

Combined Convection in Horizontal Circular Sector Ducts  
of Various Cross-Sectional Orientations

by

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COMBINED CONVECTION IN HORIZONTAL CIRCULAR SECTOR DUCTS  
OF VARIOUS CROSS-SECTIONAL ORIENTATIONS

BY

CHASIN CHINPORNCHAROENPONG

A Thesis submitted to the Faculty of Graduate Studies of the University of Manitoba in partial fulfillment of the requirements for the degree of

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## Abstract

For laminar mixed convection in horizontal circular sector ducts with uniform heat input axially, fully developed Nusselt numbers ( $Nu$ ) [and similarly for the product of friction factor ( $f$ ) and Reynolds number ( $Re$ )] depend on duct apex angle ( $2\phi$ ) and are functions of modified Grashof number ( $Gr^+$ ) (average heat flux level), Prandtl number ( $Pr$ ) (fluid type) and duct orientation with respect to the gravity vector (as would occur, for example, in horizontal multi-pass tubes). Regarding the latter, the steady pattern of secondary flow (buoyancy induced) varies as the duct changes its cross-sectional orientation, thus influencing heat transfer and pressure drop. A theoretical/numerical study was made of the fully developed results for the H1 thermal boundary condition.  $Nu$  and  $fRe$  were computed (via finite difference method; modified SIMPLER algorithm) for various  $\phi$ , each at various orientations and  $Pr$  over a range of  $Gr^+$  up to approximately  $5 \times 10^7$ . For a fixed  $Gr^+$ ,  $Pr$ , and orientation, it was found that Nusselt ratios (relative to pure forced convection) first increased rapidly with  $2\phi$  but tended to plateau between about  $2\phi = 120^\circ$  and  $180^\circ$ . In these plateau regions, enhancements, for example for  $Gr^+ = 10^7$  and  $Pr = 4$ , reached factors of between about 3.7 and 4.4 depending on orientation. On the other hand, under the same condition,  $fRe$  ratios fell in the range of about 1.4 to 1.6. For a given circular sector duct in a fixed orientation, it was found (as expected) that for a given  $Pr$  that  $Nu$  and  $fRe$  increased with  $Gr^+$  in an exponential fashion, i.e. buoyancy effects became progressively more influential as the heat flux level was increased. For a fixed duct and orientation, for a given  $Gr^+$ ,  $Nu$  increased with  $Pr$  whereas  $fRe$  decreased with  $Pr$ . These dependencies could be removed

by scaling  $Nu$  on the product  $Gr^+Pr$  and  $fRe$  on the parameter  $Gr^+/Pr^2$ . Concerning the effect of orientation, for a fixed duct operating at fixed  $Gr^+$  and  $Pr$ , orientation was a fairly significant factor, being largest for large  $\phi$ . For example, for  $2\phi = 120^\circ$ , these differences were about 15% at  $Gr^+ = 10^7$  with  $Pr = 4$ .

A series of experiments using a water loop were conducted on a semicircular duct oriented at  $45^\circ$ ,  $90^\circ$ ,  $135^\circ$  and  $180^\circ$  in addition to the  $0^\circ$  orientation in the previous study. The flow was fully developed at entry and both the thermal entry region and the fully developed region were explored for uniform heat input axially. The various tests at different orientations spanned from  $Re$  of 500–1500 and  $Gr$  of  $5 \times 10^6$ – $5 \times 10^7$ . The magnitudes of local Nusselt numbers were observed to be steady in the region where  $x^*$  was larger than about 0.01 consistently for all  $\alpha$  and therefore the fully developed Nusselt numbers were determined in this region. Strong dependence between  $Nu$  and  $Gr^+$  were observed. The maximum Nusselt numbers were obtained with  $\alpha = 90^\circ$  and  $135^\circ$  whose magnitudes were similar. Similar  $Nu$  magnitudes were also observed for  $\alpha = 45^\circ$  and  $180^\circ$ . Unlike the theoretical/numerical results, the correlation of  $Nu$  vs  $Gr^+Pr$  showed little improvement as compared to  $Nu$  vs  $Gr^+$ . The differences in Nusselt number magnitudes for  $\alpha = 90^\circ/135^\circ$  was about 10% compared to  $\alpha = 45^\circ/180^\circ$  and about 40% compared to  $\alpha = 0^\circ$ . The predictions agreed reasonably well with the experiments in terms of Nusselt number magnitudes. The locations of the optimum orientation obtained by the experiments also agreed well with the predictions. In general, the H1 boundary condition was observed to overestimate the Nusselt number in the high  $Gr^+Pr$  region.

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# NOMENCLATURE

$A_{fl}$	=	cross-sectional area of the fluid flow passage, $m^2$
$A_{iw}$	=	interior wall area of the heated section, $m^2$
$c_p$	=	specific heat of fluid, $kJ/kg \cdot K$
$D_h$	=	hydraulic diameter, $m$
$f$	=	friction factor (see equation 3.9)
$\mathbf{g}, g$	=	gravitational acceleration; vector and scalar forms, $m/s^2$
$Gr$	=	modified Grashof number, $\beta g \rho^2 D_h^4 Q_f / \mu^2 k A_{iw}$
$Gr^+$	=	modified Grashof number, $\beta g q' R_o^3 / \nu^2 k$
$H1$	=	thermal boundary condition, axially uniform heat flux and peripherally uniform temperature
$H2$	=	thermal boundary condition, uniform heat flux both axially and peripherally
$h$	=	average heat transfer coefficient, $kW/m^2 \cdot K$
$k$	=	thermal conductivity of fluid, $kW/m \cdot K$
$\dot{m}$	=	mass flow rate, $Kg/s$
$Nu$	=	Nusselt number, fully developed, (see equation 3.19)
$Nu_x$	=	Nusselt number, local (see equation 4.4)
$P$	=	fluid pressure, $kPa$
$P', P_x$	=	cross stream pressure deviation, axial pressure, $kPa$
$Pr$	=	Prandtl number, $\mu c_p / k$

$Q_f$	=	heat gain by the fluid, kW
$q'$	=	heat transfer rate per unit length, kW/m
$R, R_o$	=	radial coordinate, radius of the duct, m
$Re$	=	Reynolds number, $D_h \dot{m} / \mu A_{fl}$
$S$	=	duct wet perimeter, m
$t, t_m$	=	fluid temperature, fluid bulk mean temperature, °C or K
$t_w$	=	duct wall temperature, peripherally uniform, °C or K
$t_{x,\theta}$	=	duct wall temperature, local, °C or K
$U, V, W$	=	fluid velocity in $R, \theta, X$ directions, respectively, m/s
$x^*$	=	dimensionless axial length, $X/D_h Re Pr$
$\alpha$	=	oriented angle, measured from duct cross section centre line to the gravitational force, deg
$\beta$	=	thermal expansion coefficient of fluid, $K^{-1}$
$\theta$	=	angular coordinate, deg
$\phi$	=	half of the apex angle of circular sector ducts, deg
$\mu$	=	fluid dynamic viscosity, kg·s/m
$\nu$	=	kinematic viscosity, $m^2/s$
$\rho$	=	fluid density, $kg/m^3$
<i>subscript</i>		
$o$	=	for pure forced convection
$m$	=	mean condition at the averaged fluid inlet and outlet temperature

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# Chapter 1

## Introduction

The studies of heat transfer in uniformly heated horizontal ducts in the area of laminar forced convection have been well established and the solutions for many cross-sectional geometries are commonly known. However, pure forced convection analyses omit buoyancy forces and this can result in failure of the prediction in some situations. Such situations arise when the rate of heating is so high that the effect of buoyancy forces is significant. As a result, for these situations, the knowledge of combined or mixed convection is required instead. The ideas of multipassage tubes that led to the study of combined convection in circular sector ducts, along with the effect of duct cross-sectional orientations and the effect of Prandtl number, are introduced. The objectives and scope of this study are presented at the end of this chapter.

## 1.1 Introduction and Motivation

Generally speaking, laminar flow heat transfer has abundant applications in industries [1]; for example, heat exchangers designed for viscous liquids in chemical processes and food industries, compact heat exchangers for gas flows, and heat exchangers for biomedical applications. Furthermore, the use of multipassage or internally finned tubes to earn extra interface areas as compared to circular ducts, has perpetuated laminar flow conditions since their use result in smaller hydraulic diameters and consequently smaller Reynolds numbers. Regarding duct axial orientation, the horizontal orientation is the most common for industrial interest [1]. It is for these reasons that laminar flow heat transfer in horizontal ducts have gained large attention in convective heat transfer studies.

### 1.1.1 Combined Convection

In pure forced convection analysis, the motion of fluid is induced by an external forcing condition such as a pump or fan, thus any flow in the cross-sectional planes is ignored. For the hydrodynamic and thermal fully developed conditions, Nusselt numbers and friction factors are found to be constant. In free or natural convection, as we well know, the fluid can flow due to a temperature gradient. The existence of temperature gradients results in density gradients and hence buoyancy forces arise according to the differences of body forces acting on each part of the fluid in such conditions. Therefore,

forced convection flow can not be purely established in practice because temperature gradients always exist. At high heating rates (high Grashof numbers) which is a situation that creates large temperature gradient within the fluid body, many experimental results showed that the magnitudes of Nusselt numbers and friction factors significantly departed from their corresponding forced convection solutions. Research reports consistently attribute the deviations to the effect of buoyancy force or the effect of free convection. Consequently, the analyses are called combined or mixed free and forced convection when the buoyancy forces are added in the formulations. The term combined or mixed convection is also used for the sake of brevity.

There are some close relationships between combined and forced convection. First, the characteristics of combined convection are often compared to pure forced convection, e.g. in terms of a ratio of their Nusselt numbers. This is useful for indicating the buoyancy effect. Secondly, pure forced convection can be regarded as a specific case of combined convection when the heating rate is so low that the free convection flow can be neglected. For experimental studies, forced and mixed convection experiments can share the same test facility if the heating rate can be raised sufficiently to exhibit free convection effect. Finally, it is noted that combined convection is practically meaningful only when the heating rate is high.

## 1.1.2 Multipassage Tubes and Circular Sector Ducts

The effect of buoyancy force enhances Nusselt numbers and friction factors as compared to their corresponding forced convection solutions. The free convection flow is in the duct cross-section planes and the flow in the duct becomes vortical due to the combination of the cross-stream and axial velocity components and hence the temperature and axial velocity fields are modified. The enhancement of Nusselt number is much larger than the enhancement of friction coefficient especially at large Grashof number.

The enhancement of Nusselt number can be accomplished by many techniques [2]. One of those techniques is the extension of the duct wall and fluid interface area, as in, for example, internally finned tubes and twist tape inserts in tubes. In the case of an internally finned tube with full length fins, the tube becomes a circular-sector duct which is a possible component passage of a multipassage tube. Knowledge of combined convection in circular sector ducts is therefore useful for the study of such multipassage ducts.

## 1.1.3 Semicircular Ducts

When the number of passages in a circular sector multipassage tube is reduced to two, each passage becomes a semicircular shape. A semicircular passage can also be formed when the pitch of the inserted twist tape is infinitely large. The multipassage tube which allow a larger heat transfer rate can be used as a substitute for the circular

duct and thus the application at a higher heating condition is possible.

Circular sector passages can also be assembled in semicircular ducts instead of circular tubes. The outer flat wall of a semicircular duct can fit well with flat surfaces and consequently, a good interface contact can be established, e.g. in fin and tube heat exchangers. This should lead to a new design of heat exchanger components where compact and high heat flux operation are required. These ducts should find many industrial applications such as in nuclear or other power plants.

#### 1.1.4 Effect of Cross-Sectional Orientation

In circular sector multipassage ducts, each of the circular sector passages could orient differently, e.g. the apex may point upward, downward, or any other angle. A circular sector duct comprises two flat walls and a curved wall. As the passage changes its orientation, each flat wall makes an angle with a vertical which is the direction of the gravitational force. As we know, the cross-stream flow is induced by the gravitational force therefore this angle implies the angle of attack, i.e. between the vertical velocity component of the flow and the wall. This indicates that the orientation must modify the pattern of the cross-stream flow because the circular sector passage is not axisymmetric. The modification involves cross-stream velocity, axial velocity and temperature fields and, consequently, Nusselt number and friction factor. How the cross-stream flow impacts on heat transfer and axial flow on each circular sector duct are important for the analyzing the performances of this type of multipassage ducts.

### 1.1.5 Effect of Prandtl Number

Prandtl number is a fluid property that represents the ratio of momentum to thermal diffusion. In terms of boundary layer development, the velocity boundary layer will develop much faster than the thermal boundary layer if Prandtl number is very high and vice versa. The effect of Prandtl number is clear for cases where there is no cross-stream flow, e.g. axisymmetric flow in a tube where Prandtl number has an effect only within the developing region. However, for cases where cross-flows exist, the effect of Prandtl number is more complicated and is still poorly understood. This study attempts to bring about a better understanding of the role of Prandtl number on the heat transfer and flow characteristics in circular sector ducts.

### 1.1.6 Effect of Apex Angle

For a fixed radius, the sizes of circular cross sections depend directly on apex angles. Since cross-stream flow is confined by the duct cross-sectional shapes, apex angle would significantly affect the cross-stream flow patterns and hence flow and combined convection characteristics. The duct cross-sectional orientation can cause further modification on flow and heat transfer characteristics. Therefore the variations of apex angles and orientations should provide a broad view for combined convection in circular sector ducts.

## 1.2 Objectives

Theoretical study in the combined convection area is still limited due to the complication of the governing equations. However, recent solution techniques allow the possibility of a numerical study of combined convection. But even though theoretical study facilitates a broader view of studies, experimental studies are equally important. Among circular sector ducts, the semicircular duct is quite unique owing to its single-piece flat wall and hence a variety of potential applications. Therefore more attention is paid to semicircular ducts in this thesis. The purpose of this investigation is to explore the heat transfer and flow performances for laminar combined convection in horizontal ducts with detail as follow.

1. The theoretical study for

- effects of cross-sectional orientations for semicircular ducts,
- effects of cross-sectional orientations for other circular sector ducts,
- effects of Prandtl number for circular sector ducts, and
- effects of apex angle for circular sector ducts.

2. The experimental study for effects of cross-sectional orientation for a semicircular duct.

## 11.3 Scope of This Work

The theoretical study of combined convection in horizontal circular sector ducts will be confined to laminar fully developed flow with axially uniform heating (H1) conditions. These conditions are common for research in the area of forced convection. The duct orientation angle,  $\alpha$ , is varied from  $0^\circ$  to  $180^\circ$ . This covers all cases of orientation with a complete round of rotation since the circular sector shape is symmetric about its centre line. The increment of  $\alpha$  is chosen to be  $45^\circ$  for semicircular ducts and  $90^\circ$  for other circular sector ducts. The Prandtl number of 4 will be used for the numerical analyses for the comparison to experimental data. Additional numerical analyses for Prandtl numbers of 10 and 50 are provided for the study of the effect of Prandtl number. The research should achieve the results as summarized in the followings.

1. Numerical solutions for a semicircular duct for  $Pr = 4$  with the orientation angles of  $0^\circ$ ,  $45^\circ$ ,  $90^\circ$ ,  $135^\circ$  and  $180^\circ$ .
2. Experiments\* for a semicircular duct with the orientation angles of  $45^\circ$ ,  $90^\circ$ ,  $135^\circ$ , and  $180^\circ$ .
3. Numerical solutions for circular sector ducts, apex angles of  $60^\circ$ ,  $90^\circ$  and  $120^\circ$ , for  $Pr = 4$  with the orientation angles of  $0^\circ$ ,  $90^\circ$ , and  $180^\circ$ .
4. Numerical solutions for the same problems as in 1 and 3 but for  $Pr = 10$  and 50.

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\* The experimental studies also include the thermally developing region. Heating is uniform axially, but wall temperature is not exactly constant at any cross-section, i.e. the H1 condition is not attained exactly.

## Chapter 2

### Literature Survey

Although combined convection heat transfer has been studied for decades, the literature in this area is quite sparse as compared to the area of pure forced convection. This is partially due to the difficulty of the analyses that limits the range of theoretical study. Because combined convection problems involve a set of hydrodynamic governing equations, the literature dealing with solution techniques in this area is useful and is included in this survey. Early research work in the area mainly dealt with experiments and analytical solutions, while numerical solutions started to appear more frequently later. The topics in this chapter are accordingly arranged in more or less a chronological order. The first section reviews the reality of free convection flow as observed through flow visualization or other experimental methods. This is followed by a large section on the development of the combined convection study. The last section concerns numerical study of combined convection and its related areas. Basically, the review will focus on horizontal ducts which differ fundamentally from vertical ducts. The review is also generally restricted to only laminar flow and the uniform heating condition (H1). However, works dealing with the uniform temperature (T) boundary condition are

sometimes included where especially relevant.

## 2.1 Demonstration of Free Convection Flow by Experimentations

The evidence of free convection flow arising in forced-convective flow can be demonstrated by experimentation either directly or indirectly. The direct way is by flow visualisation while the indirect way is indicated by Nusselt number or the wall-to-fluid temperature difference compared to the pure forced convection case. Although this section is mainly devoted to experimental demonstrations, a few analytical studies are mentioned to illustrate the indirect approach.

### 2.1.1 Free Convection Flow Phenomena

In most fluids, free convection is induced by buoyancy forces arising from density gradients which occur due to the existence of temperature gradients. Therefore, in most physical situations, free convection appears as a by-product of forced convection flow and hence it is called secondary flow. When pumped fluid flowing in a horizontal tube is heated, the fluid particle paths are spiral, flowing upward near side walls and falling down near the centre of the tube [3]. The spiral velocity profile was first detected by Woolfenden [4].

Morton [5] presented a mathematical model for laminar combined convection in horizontal tubes which in fact established the idea that the fluid circulation caused by free convection is in the plane normal to the tube axis. The spiral path can be explained as the combination of an axial velocity component and a cross-stream velocity component. The term cross-stream flow is preferred to the term secondary flow due to a broader meaning of the latter term. Bergles and Simonds [6] observed spiral streamlines in flowing water in a nearly transparent tube using a dye injection technique. In addition, when raising the heat flux at a constant flow rate or when decreasing the flow rate at a constant heat flux, the pitch of the spiral was found to decrease. With a similar technique, Newell and Bergles [7] observed a couple of vortexes near the bottom of a small pyrex tube. The spiral streamlines were also seen. Some bibliographies of the free convection flow visualization in water, air and oil are also provided in [7]. Excellent pictures of the cross-stream flow patterns were provided by Mori and Futagami [8]. By introducing  $\text{NH}_4\text{Cl}$  smoke into a uniformly heated transparent tube of 30 mm i.d., a pair of vortices which was symmetric to the vertical plane was observed. The eyes of the vortices moved closer to the tube wall as the heating rate was increased. The above observations consolidate the idea of how fluid flows in a circular pipe in the combined convection situation, i.e. each fluid particle follows a spiral path along the axial direction whereas a pair of symmetric vortices form at each cross-section.

Yousef and Tarasuk [9] used the Mach-Zehnder interferometer to observe the air temperature field in an aluminum tube of 25 mm i.d. with uniform temperature heating. The interferogram of isotherms showed the low temperature contours near the bottom

wall. This indicated the descending cool core fluid. The crescent shapes of these contours provided evidence of the rising of hot fluid near the side walls.

### 2.1.2 The Effect of Free Convection Flow

McComas and Eckert [3] found that the local Nusselt numbers in the down stream side of the heated tube were larger than for pure forced convection and this effect increased with the ratio of Grashof number to Reynolds number. Ede [10] tested seven tubes of 1.27 to 5.08 cm diameters with air and water for Reynolds number of 300 to 100,000. The Nusselt number data for laminar flow were clearly dependent on Grashof number. The magnitudes of Nusselt numbers were enhanced by free convection effects up to six fold. Mori *et. al.* [11] found that the Nusselt numbers at  $ReRa$  about  $10^5$  were twice that for the laminar forced convection case and distorted velocity and temperature profiles were observed. For turbulent flow, the buoyancy effect showed little influence on both velocity and temperature fields. Later, the effect of free convection or buoyancy force was better recognized and well described, e.g. [1]. Based on [1], [3], [6-7], and [9-15], the main characteristics are summarized as follows.

1. The velocity and temperature profiles of the fluid as compared to the forced convection case were distorted by the symmetric vortex flows.

2. By investigating the results of Nusselt number versus the axial distance, the enhancement of Nusselt number was small near the inlet but became higher and further departed from the forced-convection case as the axial distance increased.

3. The velocity and temperature profiles and the Nusselt number were further distorted by raising the heat flux.

4. The measured Nusselt numbers appeared to depend on Grashof number or its equivalent dimensionless groups ( $ReRa$ ).

## 2.2 Development of Combined Convection Study

The studies of combined and forced convection for circular, rectangular, and circular sector (including semicircular) ducts are reviewed in the following subsections along with internally finned tubes. Bifurcation, which is so far only found in numerical analyses for combined convection, is also included in this section.

### 2.2.1 Circular Ducts

Morton [5] presented the very first analytic solution for fully developed combined convection by the perturbation method. The solution for Nusselt number contained a power series for  $ReRa$  while  $Pr$  appeared as the coefficients. Unfortunately, the applicable range of  $ReRa$  was limited to only 3000. Del Casal and Gill [17] extended Morton's work by allowing the axial pressure gradient to vary. Similar to Morton's solution, it was applicable only for very low heating rates. Mori and Futagami [8] used a boundary-layer integral method to solve the same problem. They mentioned that their solution was limited to Prandtl number about 1.

Shannon and Depew [12,13] used the relation  $Nu = K_1(GrPr)^{3/4}$  where  $K_1$  was a constant as suggested by Mikesell [16]. The experimentation was performed with a rather small tube of 9.53 mm dia. and 6.1 m long. They presumed that  $(GrPr)^{3/4}$  might account for the free convection contribution only. The experimental results [13] showed that as  $(GrPr)^{3/4}Nu_{Gz} < 2$  the local Nusselt numbers were independent of  $(GrPr)^{3/4}$ . It is worth noting that the previously mentioned range corresponds to the region very near the inlet where the magnitude of Graetz number is large. It was believed that the Nusselt number in that region depended on viscosity hence a viscosity ratio with a fraction exponent was added to the correlation in order to account for the variable viscosity effect. It seemed that their works were based on the idea that the temperature and velocity profiles were altered by the variable fluid viscosity while the natural convection flow aided energy transfer to or from the fluid.

However, Siegwarth *et. al.* [14] ignored the variable viscosity in their analysis of combined convection problem for large  $GrPr$ . Similar to Mori and Futagami [8], in this analysis the flow domain was divided into two distinct zones; one zone within the temperature boundary layer and the other, called core, exterior to that layer. Therefore, two sets of governing equations were separately simplified from the more complex original governing equations. Unlike the external flow problem, the core condition could not be known *a priori*. The main problem with this method was the necessity to find the conditions for the core and to discover how the core and boundary layer interacted. By following assumptions:  $(Pr/Re) \rightarrow \infty$ , constant viscosity, and small core velocity, it was shown that  $Nu = 0.471(GrPr)^{3/4}$ . In their subsequent work [15], the method of solution

was a finite difference method solving for the velocity and temperature profiles to compare with the experimental results.

The common ideas of [12-15] seem to be the use of  $(GrPr)^{3/4}$  and the ideas about variable viscosity and backflow are attributed by the influence of Mikesell [16]. On the other hand, Hwang and Cheng [18] obtained the results for various Prandtl numbers and  $ReRa$  up to  $10^5$  by the finite difference method. Their results agreed well with Morton's [5] at low  $ReRa$ . Cheng and his colleagues solved many combined convection problems. For thermal developing problems, they assumed infinitely large Prandtl number to simplify the problems, e.g. [19-20] for circular ducts. The product  $ReRa$  was shown by Patankar *et. al.* [21] to be proportional to modified Grashof number,  $Gr^+$  which was directly defined by the heating rate. This dimensionless variable has been widely accepted and used in the later generation of computational researches. The solutions for combined convection in circular duct was presented by Nandakumar *et. al.* [22] and the results agreed very well with Hwang and Cheng [18].

For the uniform wall temperature condition, the combined convection has a strong effect only within the developing region therefore the main interest is for that region. Similar to [19-20], a large Prandtl number assumption was also used in analyzing the problem with uniform wall temperature, e.g. Hieber and Sreenivasan [23], Yao [24] and Hieber [25]. Eventually, the solution without a large Prandtl number assumption was successfully solved by Hishida *et. al.* [26] and for the H2 condition by Chou and Hwang [27].

## 2.2.2 Rectangular Ducts

For fully developed combined convection in rectangular ducts with uniform heat flux, Cheng and Hwang [28] presented their finite difference solution for a couple of Prandtl numbers. Again, the Prandtl number was assumed infinitely large in their following works [29-30] for the entrance region. The results were believed to be good only for Prandtl number larger than 10. Later, such Prandtl number restriction was relaxed; for example, for the uniform heat flux condition [31-32], and for non-uniform heat flux condition [33-34].

## 2.2.3 Circular Sector Ducts

Circular sector passages represent a large family of passages due to the variant of the apex angle therefore the generalized study for circular-sector passages entails exploring the apex angle effects. There is no such generalized study for combined convection known to the author. However, such study for fully developed laminar forced convection was accomplished by Lei and Trupp [35] and [36] for Nusselt numbers and friction coefficients, respectively. The apex angle was varied from  $0^\circ$  to  $360^\circ$  with small increments thus rather detailed results were obtained from the closed form solutions. The Nusselt numbers were obtained by the velocity field established in [36]. The locations of maximum velocities were observed to move from the curved wall toward the apex as apex angle was increased. Sparrow and Haji-Sheikh presented a similar work in [37] but with variable thermal boundary conditions. For uniform heat flux condition, they

compared the Nusselt numbers and  $fRe$  of circular sector ducts to two types of triangular ducts, the results of circular sector ducts were higher and the differences increased as apex angle increased [38]. Trupp and Lau [39] solved Nusselt numbers for circular sector ducts with uniform temperature. The Nusselt numbers of this thermal condition were smaller than the H1 condition but higher than those of isosceles triangular ducts with the same wall condition. For a group of H2 heating conditions, Nusselt numbers of circular sector ducts were studied by Trupp and Lei [40]. The Nusselt numbers of H1 condition were larger than those of H2 conditions for all apex angles.

For the entrance region, Soliman *et. al.* [41] studied laminar flow of four circular sector ducts with the apex angle up to  $90^\circ$  by finite difference method. Semicircular ducts, a special case of the circular sector ducts, were studied by Hong and Bergles [42] for uniform heat flux condition and Zhang *et. al.* [43] for uniform temperature condition. For circular sector ducts, Lei and Trupp [44] analyzed the problem with H1 and H2 thermal boundary conditions.

The combined convection in a semicircular duct with flat wall on the top was studied by Lei [45] and Lei and Trupp [46-47] both numerically and experimentally. For the experimental study, the Nusselt number results appeared to be fully developed at the down stream side and  $Nu$  and  $fRe$  were compared to the predictions. The effects of Reynolds number, and Rayleigh number were also presented. Nadakumar *et. al.* [22] also solved the same problem but with the flat wall at bottom. This is a main reference used in [45-46] and the present study for comparisons of the numerical results.

## 2.2.4 Internally Finned Tubes

Internal finning is a practical heat transfer augmentation technique as internally finned tubes are commercially manufactured. However, the theoretical study is limited due to the complexity of the problem and the alternative of fin shapes. Although some experiments with water in the turbulent region exhibited optimum heat transfer performance for short spiral fins, but for air flow, straight fins with larger height showed the best heat transfer performance [48]. Hu and Chang [48] analyzed laminar forced convection with uniform heat flux in internally finned tubes with straight fins of zero thickness and infinite thermal conductivity. The best Nusselt number, almost 20 times that of the finless tube, appeared when the number of fins was 22 and fin height was 0.8 of the tube radius. Masliyah and Nandakumar [49] were aware of the idealistic assumptions therefore they used a triangular fin with peripherally uniform temperature on both the fins and tube surfaces. A more practical shape of fins was presented by Soliman and Feingold [50,51] in their analyses. The results from [49-51] were similar to those of [48] except that the Nusselt number decreased as the fins were thicker. Soliman *et. al.* [52] further modified the thermal boundary condition toward reality by allowing fin temperature to vary radially. A new parameter involving fin thickness and the ratio of fin and fluid thermal conductivity allowed the investigation of the Nusselt number behaviours. Furthermore, Soliman [53] proposed a technique for solving the energy equation by subdividing the flow domain into two regions and the analytical solutions were presented. For the entrance region problems, Prakash and Liu [54] studied laminar forced convection

for simultaneous development of velocity and temperature while Rustum and Soliman [55] studied the thermal developing region with fully developed velocity at entry.

Fully developed combined convection in internally finned tubes was studied by Mirza and Soliman [56]. There were only two fins with zero thicknesses located at the top and bottom of the tube. The effects of fin height and Grashof number were investigated. Combined convection experiments were conducted by Rustum and Soliman [57] with four commercial internally finned tubes and a smooth tube. The effect of free convection appeared strongly at high Rayleigh number. The Nusselt numbers approached those of forced convection when the Rayleigh numbers were low. The numerical analysis by Rustum and Soliman [58] showed that the flow and heat transfer characteristics depended on modified Grashof number, Prandtl number, number of fins and fin height. The finning was found to retard the free convection effect that resulted in less enhancement compared to the smooth tube. Finning in semicircular ducts was studied by Zhang and Ebadian [59] for the entrance region.

### 2.2.5 Bifurcation

The cross-stream flow pattern of combined convection in horizontal ducts comprise two vortexes which will be called the *two-vortex* or *two-cell* pattern. Such a pattern has been verified experimentally in circular ducts. Numerical solutions also show two-cell pattern, e.g. [22] for rectangular, circular and semicircular ducts.

At an adequate high heating rate, flow patterns of the numerical solutions may

become four vortexes, or so-called *four-cell*. The term bifurcation will be used for such transformation of the flow patterns or for the appearance of the four-cell pattern. The earliest detection of bifurcation in combined convection was found in rectangular ducts [30]. It, later, appeared more often, e.g. [21-22], [34], [45-46], for rectangular, circular and semicircular ducts.

The bifurcation can be induced not only by the buoyancy force in combined convection flow but also by the centrifugal force in curved duct flow (without heating). The bifurcation in a curved semicircular duct was found in the numerical solutions and it was confirmed by flow visualisation, see Masliyah [60]. The numerical solutions of curved tubes also showed bifurcation [61].

## 2.3 Numerical Study of Combined Convection

In solving a combined convection problem, it is necessary to solve the underlying flow field which is characterized by continuity and vector momentum equations in addition to a scalar equation for temperature. The discretization of the convective and diffusive terms in the momentum equations directly controls the accuracy of the predicted flow field and therefore the scalar variable. The discretization techniques are applicable to both the flow and scalar equations. On the other hand, the coupling between continuity and momentum equations is important to the solution procedures which usually depend on how this coupling is handled. Therefore the first two sub-sections are devoted to the

convective-diffusion term discretization and the solution techniques.

The finite difference and finite element methods are currently two major approaches for solving the partial differential equations arising from heat transfer problems [62]. Other methods such as boundary element, spectral or pseudo-spectral, and linearization methods have limited applications. This can be observed in a recent review of the development in numerical methods for heat transfer problems by Patankar [63] which particularly addressed the problems of spectral method, e.g. internal discontinuities, and nonuniform transport properties and sources.

In the finite element method, a partial differential equation is transformed by either weighted residual or Rayleigh-Ritz method. Unlike the weighted residual method, the Rayleigh-Ritz method is not suitable because the variational principles can not be developed when advection terms are present [62]. The alternative versions of weighted residual method are classified by the choice of the approximation function and weighted function. If they are chosen to be the same, the method is called Galerkin method. The finite element methods formulated by all these fashions were initially for stress analysis in irregular geometries thus these methods do not directly enforce a conservation principle on confined volumes [64]. In addition, the employment of reduced-order pressure interpolation, in order to avoid a spurious pressure field [65], degrades the accuracy of the pressure field representation [64]. The equal-order elements also suffer the same problem as the non-staggered grid version of the finite difference method. The basic differences and similarities of finite difference and finite element methods are compiled in [62] and they suggested that the choice is up to the users because both methods have

their own advantages.

### 2.3.1 Convection-Diffusion Formulations

Spalding [66] exhibited that the well-known upwind (upstream) difference scheme was more accurate than the central difference scheme when the ratio of convection to diffusion across the grid, frequently called grid Peclet or Reynolds number, was larger than 2 by comparing to the exact solution for the one-dimensional equation with zero source term. Spalding proposed a new scheme called hybrid scheme which had better accuracy than the upwind scheme based on such exact solution. Afterward, Patankar [67] proposed a power-law scheme which employed a less expensive computation while maintained a close accuracy. The power law scheme and hybrid scheme have a similar behaviour because they switch from second-order accuracy to first-order accuracy as grid Reynolds number becomes high [68].

It is interesting to note that the central difference scheme is equivalent to first-order interpolation while upwind scheme is equivalent to zeroth-order interpolation and they have second-order and first-order truncation errors, respectively. As a matter of fact, the schemes were formulated with one-dimensional models and applied to two- or three-dimensional applications therefore such conclusions were extrapolated from the characteristics observed on the one-dimensional problem. The central difference scheme is more accurate when grid Reynolds number is less than 2 but at high Reynolds number it may result in a wiggly solution (oscillation or under and overshoot) and it is unstable,

see e.g. [69-71]. Patankar [67] and Roache [72] pointed out that it possibly caused negative coefficients and the discretized equations might diverge while the upwind scheme was still stable even at high Reynolds number.

Although stability, the major advantage of the upwind scheme, is obtained by sacrificing accuracy, for some specific situations the upwind scheme can introduce additional error. To eliminate such effects, Raithby [70] introduced two new schemes, SUDS (Skew Upstream Differencing Scheme), and SUWDS (Skew Upstream Weighted Differencing Scheme). Both of them were intended to reduce the error arising from flow crossing a grid line with a small angle while the second scheme had an additional feature of low error at high Reynolds number. The skew upstream scheme was later modified by Hassan *et. al.* [71] due to its marginal stability caused by the negative coefficient.

In general, higher order of approximations provide low diffusion results. Runchal [73] proposed an alternative to the central-difference scheme which had unconditional stability. The problems of central-difference and upstream schemes were exhaustively examined by Leonard [74] and he proposed new interpolation schemes called QUICK (Quadratic Upstream Interpolation for Convective Kinematics) and QUICKEST (QUICK with Estimated Streaming Terms) for steady and unsteady flows, respectively. These schemes utilize three nodal data for the interpolation instead of two as in previous schemes. The new schemes have third-order accuracy, better accuracy than the central difference scheme, while the stability is maintained. The upstream interpolations with second-order or higher were studied by Warming *et. al.* [75], Atias *et. al.* [76], Shyy [77] and Vankar [78]. While Shyy was satisfied by his results although it was not yet free

from difficulties and inaccuracies, Vankar found that his second-order schemes could not perform well in terms of accuracy and computational stability. Later, Shyy *et. al.* [79] compared the central-difference scheme and three alternative second-order upstream schemes. The upwind-type schemes were more accurate and exhibited no oscillation. The flux-spline scheme, another high-order interpolation, was demonstrated by Karki *et. al.* [80] and it exhibited accurate solutions without excessive grid refining.

The variety of the discretization schemes and their specific-tuned applications made the comparison task very difficult. In May 1981, Smith and Hutton [81] collected results of a test problem from many participants who used various schemes with finite difference and finite element methods. The report concluded that no particular scheme was the best. Huang *et. al.* [82] conducted their own test with a few widely used schemes. They favoured the quadratic upwind over the skew upwind and power-law schemes. Recently, Zurigat and Ghajar [83] conducted a survey on a number of comparative studies and concluded that the second-order upwind, skew upwind and quadratic upwind schemes tended to exhibit better accuracy than the weighted upwind scheme. Another recent comparison, similar to [83], was conducted by Sharif and Burnaina [84]. Their general conclusion was that the schemes which produced less numerical diffusion suffered from more oscillations. Miller and Schmidt [85] also found that while QUICK scheme produced less numerical diffusion, its rates of convergence were poorer than hybrid or upwind schemes. Another series of scheme testings was conducted by Patel *et. al.* (appeared in [65]). They found that QUICK and its variants gave accuracy but failed to converge at high flow rates and coarse grids. The upwind scheme was the best choice by

their conclusion. All of these comparison results agree well with what Smith and Hutton [81] stated earlier that; "It seems that advection modelling remains the art of compromise between diffusive and oscillatory".

### 2.3.2 Solution Techniques

The solution procedure primarily depends on how the governing equations are formulated. For a two-dimensional flow problem, e.g. flow in an infinite width channel or axisymmetric flow in a tube, the governing equations comprise a continuity and two momentum equations. There are four main methods to formulate such problem: primitive variables, vorticity-velocity, vorticity-streamfunction and biharmonic formulation. The first method keeps the original governing equations unchanged and solves for the three dependent variables; pressure and two components of the velocity. The last three methods eliminate the pressure by introducing a streamfunction and a vorticity function. The vorticity-velocity method has three equations; two velocity Poisson equations and a vorticity transport equation, e.g. see [86]. For the vorticity-streamfunction method, e.g. see [87-88], the number of governing equations becomes two; a streamfunction equation and a vorticity transport equation. In the last method, the elimination of vorticity in the vorticity transport equation results in a single governing equation for the streamfunction in biharmonic form and this equation can be solved by Newton's method or likewise, e.g. see [89-90].

The methods involving vorticity or stream functions have an advantage of avoiding

the explicit appearance of pressure. There are a few drawbacks arising from the use of vorticity. First, the imposing of vorticity at the solid walls is difficult and lacks physical meaning. Secondly, except for the last method, to determine vorticity demands evaluating derivatives up to second-order in the case of the vorticity-stream function method. Another disadvantage of vorticity-streamfunction is that it is difficult to extend to three dimensions. Otherwise, the primitive variable method is the most popular method. It has the convenience of handling the variables in their original forms rather than those disadvantages of the other methods. But this method suffers two major problems; the spurious pressure field and the coupling between momentum and continuity equations. The spurious pressure can be eliminated by the staggered grid arrangement. This, in turn, is a major problem for the nonstaggered grid system. Owing to such distinct nature, the review proceeds separately for staggered and nonstaggered grid arrangements.

### 2.3.3 Staggered Grid Arrangement

Patankar and Spalding [91] presented a calculation procedure called SIMPLE (Semi-Implicit Method for Pressure Linked Equations) for heat, mass and momentum transfer. A revised version of SIMPLE, called SIMPLER was presented by Patankar [92]. The difference between the two versions is about the treatment of the pressure field due to SIMPLE is likely to diverge because of the overestimation of the pressure correction field. The details of these methods can be found in [93]. The large pressure corrections of SIMPLE resulted in a requirement of underrelaxation which in turn delayed the

convergence. There are many variants of SIMPLE which were released for overcoming that problem, for example, PULS, PUMPIN [94-95], and SIMPLEC [96]. The emerging of these modified versions indicates that the handling of pressure updating and pressure correction is very important for the SIMPLE-like methods.

The treatment of pressure-velocity coupling was continued by Issa [97] who proposed the PISO (Pressure-Implicit with Splitting of Operators) method. This method derived the pressure equation from the discretized forms of the momentum and continuity equations. The algorithm, involving the pressure and velocity operations, was split into a series of predictor-corrector steps. The PISO was similar to SIMPLER due to its predictor-corrector nature. In fact, for some it was believed to be the same as SIMPLER [65,99]. In spite of such belief, Jang *et. al.* [98] conducted a test with three problems and concluded that PISO was superior to SIMPLER and SIMPLEC for the problems where the momentum equation was not coupled to a scalar variable. When the temperature was strongly coupled to the momentum equation, SIMPLER and SIMPLEC exhibited better performance while the superiority between SIMPLER and SIMPLEC was difficult to figure out. Latimer and Pollard [99] developed the FIMOSE (Fully Implicit Method for Operator-Split Equations) as a fully implicit alternative of PISO which is semi-implicit.

So far, all of the methods mentioned previously are called segregated method because each dependent variable (e.g. pressure, velocities) are solved sequentially. The calculations involve the solution for predictor equations followed by the solution for the corrector equations of pressure and each velocity separately. In contrast to the segregated solution, the pressure and velocities can be evaluated simultaneously in order to make use

of the new values immediately in the next step. This is called block (or coupled) method. The idea is that the dynamically updated pressure and velocity data should speed up the convergence. The earliest proposal seems to be SIVA (SIMultaneous Variable Adjustments) by Caretto *et. al.* [100] and later by Vankar [101-103]. All these works were point-block method, e.g. the grid was scanned point by point. Vankar used Gauss-Seidel point relaxation and the process was called symmetrical coupled Gauss-Seidel (SCGS). The convergence rate was reported up to a tenth of that required by SIMPLE in term of computational time, see [103]. It is perhaps worth noting that in [103] the grid numbers were as large as 321x321. Hutchinson *et. al.* [104] commented that such spectacular gains must be viewed in the context that SIMPLE performance degrades drastically as grid is refined.

The line sweeping for block solution was proposed by Raithby's group [105-106] as CELS method (Coupled Equation Line Solver). This method is similar to the previous method except that it is line-by-line sweep instead of point by point. The advantage of point block over line block method should be the simplicity and convenience to fully vectorize on super computers [102]. In addition, the earlier work of Vankar [107] used plane-by-plane technique to cope with the three-dimensional flow in curved ducts.

### 2.3.4 Nonstaggered Grid Arrangement

In nonstaggered grid systems, the pressure and velocities of each cell locate at the same grid points. This is preferable in contrast to the staggered grid which is chosen for avoiding the checkerboard pressure and velocity. The staggered grids have been widely used in spite of their inconvenience for irregular geometry. One way to solve such problems is to use the general orthogonal coordinate, e.g. [108], which seemed to be too complex. Nonstaggered grid is probably a better choice for curvilinear coordinates.

Rhi and Chow [109] solved a flow problem with nonstaggered grid by using natural coordinates. The calculation procedure was similar to SIMPLE. Later, Majumdar [110] found that the results attained from the so-called momentum interpolation used by Rhi or Peric (as appears in [110]) was affected by the underrelaxation factor therefore he proposed an alternative momentum interpolation for avoiding such effect. Patankar [65] indeed recognized this drawback, not only by himself but also by some others, and mentioned that the finite-element method with equal-order interpolation suffered this problem as well. However, there was no report of such problem in very recent works [111-113] which studied or extended the above methods.

A generic perspective for nonstaggered grid study was given by Shih and Ren [114]. They demonstrated three methods to handle the continuity equation (or, in fact, the hidden pressure). The first method, the continuity equation is discretized as usual. They showed that the oscillatory pressure can simply be avoided if the nodal pressure is properly linked to the boundary. In the second method, the pressure Poisson equation is

used instead of the continuity equation. The third method used another form of pressure equation which was derived from the discretized momentum and continuity equations. Incidentally, the solution techniques for nonstaggered grid can probably be classified in accordance with the above three methods. In another words, all methods solve the same momentum equations but with different equations for evaluating the pressure. Therefore, all works mentioned above, except for those clearly stated in [114], fall into the first method. The second method, Poisson pressure equation method can be computer time and storage consuming [114-115]. However, some special techniques were employed in recent studies [116-118]. There is no research yet using the third method.

### 2.3.5 Multigrid and Other Techniques

There are some techniques that can be added to other computational methods to improve the convergence rates, or accuracy of the solutions, or both. These techniques are increasingly important when the computational time is concerned. The calculation on a fine mesh can be accelerated by the multigrid method. This method employs a number of meshes with different grid spaces. The idea is the fine mesh poorly smooths out the high frequency errors and this delays the convergence. In contrast, the coarse mesh can reduce the high frequency errors efficiently. By switching back and forth between these meshes, the errors are reduced faster and convergence should be reached sooner. A good source for multigrid method is Brandt [119]. For applications in fluid flow and heat transfer problems, multigrid method was successfully applied, e.g. [120], [88], [102], and

[106]. Other techniques that can improve the convergence is to minimize the residual. Chatwani and Turan [121] demonstrated that the underrelaxation factor for the pressure correction in SIMPLE-like methods can be determined from the global residual of the momentum equation. Lee and Dulikravick [121-122] method is quite similar to the above method except that the relaxation factor was determined for each dependent variable.

# Chapter 3

## Theoretical Formulation

The combined convection in circular sector duct analysis is different from the forced convection analysis because it accounts for the effect of the buoyancy force. This results in two additional momentum equations for describing the flow in the cross-stream plane. Only the assumptions and derivation of the governing equations are presented in this chapter.

### 3.1 Defining the Problem and Assumptions

Combined convection is affected by cross-sectional orientation with respect to gravitational vector. This effect can be studied by keeping the duct cross-section fixed while the gravitational force is allowed to rotate to a desired angle. The oriented angle,  $\alpha$ , is measured from the duct centre line (i.e. the line bisecting the apex angle which is on the  $Z$  axis in Fig. 3.1) to the gravitational force in the counter clockwise direction. The duct cross-sectional size is allowed to vary according to the magnitude of the apex

angle,  $2\phi$ . In this way, the definition of the domain under consideration is independent of the oriented angle and the duct size, e.g.  $\gamma \leq \theta \leq 2\phi$  where  $\gamma$  is always equal to  $\pi/2 - \phi$ . The maximum size of  $\phi$  is limited to  $\pi/2$  which corresponds to a semicircular cross section.

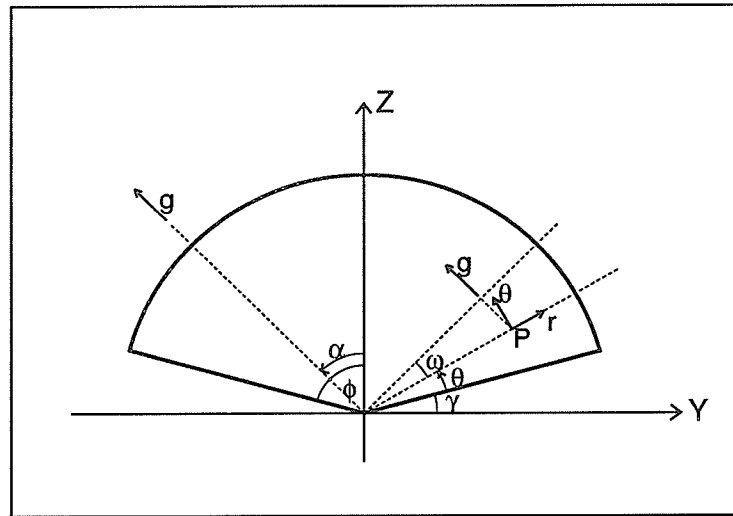


Fig. 3.1 Cross-sectional geometry of a circular sector duct.

The governing equations are formulated according to these assumptions.

1. The fluid is viscous and incompressible.
2. The flow is steady, laminar, and hydrodynamically and thermally fully developed.
3. The fluid viscous dissipation and axial diffusion are negligible.

4. All properties are constant except for the density associated with the buoyancy forces.

5. The duct is subjected to an axially uniform heat input and the wall temperature is peripherally uniform, i.e. the H1 boundary condition.

## 3.2 Derivation of the Governing Equations

With the above assumptions, the Navier-Stokes equation for a cylindrical coordinate system in vector form is

$$V \cdot \nabla V - \nu \nabla^2 V = - \frac{\nabla P}{\rho} + g. \quad (3.1)$$

Fluid pressure  $P$  at any point within the duct can be described as

$$P(R, \theta, X) = P'(R, \theta) + P_x(X) - \rho_w g h, \quad (3.2)$$

where  $\rho_w g h$  is the hydrostatic pressure on the duct wall at the same elevation as the point under consideration. Distance  $h$  is measured in the same direction as  $g$  from a certain reference level, e.g. at centre of the flat wall which can be expressed in general form as

$$h = C + r \sin(\alpha - \theta), \quad (3.3)$$

where  $C$  is a constant. Substituting  $P$  and  $h$  into the right-hand-side of the Navier-Stokes equation above, it becomes

$$-\frac{\nabla P}{\rho} + \mathbf{g} = -\frac{\nabla P'}{\rho} - \frac{\nabla P_x}{\rho} + \frac{\rho_w}{\rho} g \nabla \{r \sin(\alpha - \theta)\} + \mathbf{g}. \quad (3.4)$$

Utilizing  $\rho_w/\rho = 1 + \beta(t - t_w)$  and rearranging, leads to:

$$-\frac{\nabla P}{\rho} + \mathbf{g} = -\frac{\nabla P'}{\rho} - \frac{\nabla P_x}{\rho} - \{1 + \beta(t - t_w)\} g \nabla \{r \sin(\theta - \alpha)\} + \mathbf{g}. \quad (3.5)$$

Executing the operator  $\nabla$  and extracting the gravitational acceleration vector  $\mathbf{g}$  into each coordinate axis, yields:

*r component*

$$-\frac{\nabla P}{\rho} + \mathbf{g} = -\frac{1}{\rho} \frac{\partial P'}{\partial r} - \beta g (t - t_w) \sin(\theta - \alpha), \quad (3.6)$$

*$\theta$  component*

$$-\frac{\nabla P}{\rho} + \mathbf{g} = -\frac{1}{\rho r} \frac{\partial P'}{\partial \theta} - \beta g (t - t_w) \cos(\theta - \alpha), \quad (3.7)$$

*axial component*

$$-\frac{\nabla P}{\rho} + \mathbf{g} = -\frac{1}{\rho} \frac{\partial P_x}{\partial X}. \quad (3.8)$$

Transformation variables are now introduced as follows:

$$\begin{aligned} r &= \frac{R}{R_o}, & u^+ &= \frac{UR_o}{v}, & v^+ &= \frac{VR_o}{v}, & w &= \frac{W}{W_m}, & T^+ &= \frac{t - t_w}{q'/k}, \\ p^+ &= \frac{P'R_o^2}{\rho v^2}, & Re &= \frac{\rho W_m D_h}{\mu}, & f &= \frac{(-dP_x/dX)D_h}{2\rho W_m^2}. \end{aligned} \quad (3.9)$$

Modified Grashof number  $Gr^+$  and Prandtl number  $Pr$  are defined as

$$Gr^+ = \frac{\beta g q' R_o^3}{\nu^2 k}, \quad Pr = \frac{\mu c_p}{k}. \quad (3.10)$$

Finally, all governing equations are recast in dimensionless forms as follows:

*continuity equation*

$$\frac{1}{r} \frac{\partial}{\partial r}(ru^+) + \frac{1}{r} \frac{\partial v^+}{\partial \theta} = 0, \quad (3.11)$$

*radial momentum*

$$\begin{aligned} \frac{1}{r} \frac{\partial}{\partial r}(ru^+u^+) + \frac{1}{r} \frac{\partial}{\partial \theta}(v^+u^+) &= \frac{1}{r} \frac{\partial}{\partial r}\left(2r \frac{\partial u^+}{\partial r}\right) + \frac{1}{r} \frac{\partial}{\partial \theta}\left(\frac{1}{r} \frac{\partial u^+}{\partial \theta}\right) - \frac{\partial p^+}{\partial r} \\ &+ \frac{1}{r} \frac{\partial}{\partial \theta}\left(\frac{\partial v^+}{\partial r}\right) - \frac{3}{r^2} \frac{\partial v^+}{\partial \theta} - \frac{2u^+}{r^2} + \frac{v^{+2}}{r} + Gr^+ T^+ \sin(\theta - \alpha), \end{aligned} \quad (3.12)$$

*angular momentum*

$$\begin{aligned} \frac{1}{r} \frac{\partial}{\partial r}(ru^+v^+) + \frac{1}{r} \frac{\partial}{\partial \theta}(v^+v^+) &= \frac{1}{r} \frac{\partial}{\partial r}\left(r \frac{\partial v^+}{\partial r}\right) + \frac{1}{r} \frac{\partial}{\partial \theta}\left(\frac{2}{r} \frac{\partial v^+}{\partial \theta}\right) - \frac{1}{r} \frac{\partial p^+}{\partial \theta} \\ \frac{1}{r} \frac{\partial}{\partial r}\left(\frac{\partial u^+}{\partial \theta}\right) + \frac{3}{r^2} \frac{\partial u^+}{\partial \theta} &- \frac{v^+}{r^2} - \frac{v^+u^+}{r} - Gr^+ T^+ \cos(\theta - \alpha), \end{aligned} \quad (3.13)$$

*and axial momentum*

$$\begin{aligned} \frac{1}{r} \frac{\partial}{\partial r}(ru^+w) + \frac{1}{r} \frac{\partial}{\partial \theta}(v^+w) &= \\ \frac{1}{r} \frac{\partial}{\partial r}\left(r \frac{\partial w}{\partial r}\right) + \frac{1}{r} \frac{\partial}{\partial \theta}\left(\frac{1}{r} \frac{\partial w}{\partial \theta}\right) &+ \frac{(1+\phi)^2}{2\phi^2} fRe, \end{aligned} \quad (3.14)$$

and energy equation

$$\begin{aligned} \frac{1}{r} \frac{\partial}{\partial r}(ru^+T^+) + \frac{1}{r} \frac{\partial}{\partial \theta}(v^+T^+) = \\ \frac{1}{Pr} \left( \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial T^+}{\partial r} \right) + \frac{1}{r} \frac{\partial}{\partial \theta} \left( \frac{1}{r} \frac{\partial T^+}{\partial \theta} \right) \right) - \frac{1}{\phi Pr} w. \end{aligned} \quad (3.15)$$

Hydrodynamic and thermal boundary conditions are

$$u^+ = v^+ = w = T^+ = 0 \quad \text{on each wall.} \quad (3.16)$$

When symmetry exists, only half of the domain need to be solved by employing these additional boundary conditions at the symmetry line

$$\frac{\partial u^+}{\partial \theta} = \frac{\partial w}{\partial \theta} = \frac{\partial T^+}{\partial \theta} = v^+ = 0 \quad \text{where } \gamma \leq \theta \leq \phi, 0 \leq r \leq 1. \quad (3.17)$$

Axial pressure gradient, which is solved in term of fRe, must satisfy the conservation of mass equation in dimensionless form as

$$\int_{\gamma}^{2\phi} \int_0^1 wr dr d\theta = \phi. \quad (3.18)$$

The average Nusselt number is determined by

$$Nu = \frac{\bar{h}D_h}{k} = -\frac{\phi}{(1+\phi)^2} \frac{1}{T_m^+}, \quad (3.19)$$

where bulk mean temperature is defined as

$$T_m^+ = \frac{1}{\phi} \int_{\gamma}^{\phi} \int_0^1 w T^+ r dr d\theta. \quad (3.20)$$

In case of semicircular duct,  $\phi = \pi/2$  and  $\gamma = 0$ , thus the last three equations become:

$$\int_0^{\pi} \int_0^1 w r dr d\theta = \frac{\pi}{2}, \quad (3.21)$$

$$Nu = \frac{\bar{h} D_h}{k} = - \frac{2\pi}{(2+\pi)^2} \frac{1}{T_m^+}, \quad (3.22)$$

and

$$T_m^+ = \frac{2}{\pi} \int_0^{\pi} \int_0^1 w T^+ r dr d\theta. \quad (3.23)$$

## Chapter 4

# Experimentations and Experimental Results for a Semicircular Duct

A semicircular duct is the most important member of the circular sector duct family due to its potential application as a stand-alone duct. In addition, its broad shape provides room for the cross-stream flow to be significantly affected by the duct cross-sectional orientation which is illustrated in Chapter 5. Using water as the working fluid, the experiments were conducted for four orientations, i.e.  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ , and the results are presented in this chapter. The streamwise developments of the duct wall temperatures and local Nusselt numbers are examined corresponding to Grashof and Reynolds numbers. These developments indicated that the fully developed Nusselt numbers were obtained quite early. The lengths of the fully developed regions were larger than a half of the total length of the heated section. The fully developed Nusselt number results are presented with  $Gr^+$  and  $Gr^+Pr$  as independent variables. Finally, the friction factors in term of  $fRe$  are presented. The comparisons of  $Nu$  and  $fRe$  to the

predicted results are given in Chapter 5.

## 4.1 Experimental Facility

The major components of the experimental apparatus are shown in Fig. 4.1. This experimental facility was basically the same as the one used in [45] where a complete description is given. However, the essential details are again reported here together with the additional data due to some modifications.

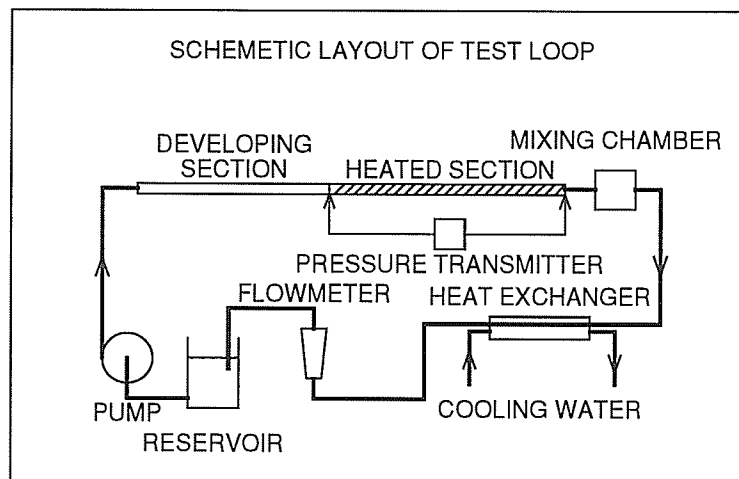


Fig. 4.1 Schematic layout of the test loop.

### 4.1.1 Test Section

The semicircular test section was constructed from type K copper tubes, inside diameter of 49.8 mm and outside diameter of 54.0 mm. The test-section consisted of two parts, a heated section of 4.7 m length and a preceding section of 3.8 m for flow development. The heat flux was generated by flat electric heater wires, (Copel® ribbon, Hoskins Manufacturing Company). After a few layers of electrical insulating varnish coating, the duct was wrapped with a layer of fibre glass insulating tape to protect the varnish surface from the heater wires. Two heaters were carefully wound in parallel and with a uniform pitch. The resistance of the twin heaters was axially uniform within about 5%. The heating wires covered with high temperature cement to ensure that they remained firmly in place in spite of the different temperature conditions. The total resistance between both ends of the heater was 6.95 ohms. With an adjustable AC power supply, it was possible to deliver a heating power up to more than 5 kW. The duct was finally covered with 5-cm-thick fibre glass thermal insulation over all the heated section. The average heat loss was about 3% (measured by a heat flux meter, HEATPROBE model HA-100, 5% accuracy) of the electrical power input for the most severe operating condition. The test section was aligned horizontally within  $\pm 1$  mm at the centre of the flat wall. The oriented angle was adjusted to an accuracy of  $\pm 1$  degree for each orientation.

## 4.1.2 Other Components

The test facility was installed in a loop whereby the outlet fluid from the test section was recycled by a centrifugal pump. The working fluid, distilled water, from a storage tank of 227 L capacity was partially fed to the test section by the pump while the main part of the fluid was fed back to the tank via a bypass line. This provided not only a stable pressure head over the test section but also fluid circulation in the tank to ensure that the fluid temperature was homogeneous and stable. The hot fluid, leaving the test section, was cooled down to near room temperature by a heat exchanger and passed to a rotameter for flow measurement before returning to the tank. The calibrated accuracy was within 3.3% for all rotameters. The pressure drop was measured across the heated section by a differential pressure transmitter, Rosemount model 1151DP, connected to an analog computer, Macsym 2, which indicated the average values of pressure. The pressure was detected by four pressure taps at each end of the heated section as shown in Fig. 4.2. The pressure transmitter was initially calibrated with water columns. The result was again confirmed with dynamic pressure reading, by running the tests without heating for various flow rates. This test also yielded the critical Reynolds number when the critical flow rate was reached.

For safe operation of the test loop, a pressure switch was installed near the outlet of the test section. The heater circuit was controlled through a magnetic relay by this switch which was normally open unless there was pressure due to fluid flow in the system. Therefore the heating could not be activated without the flowing fluid in the test section.

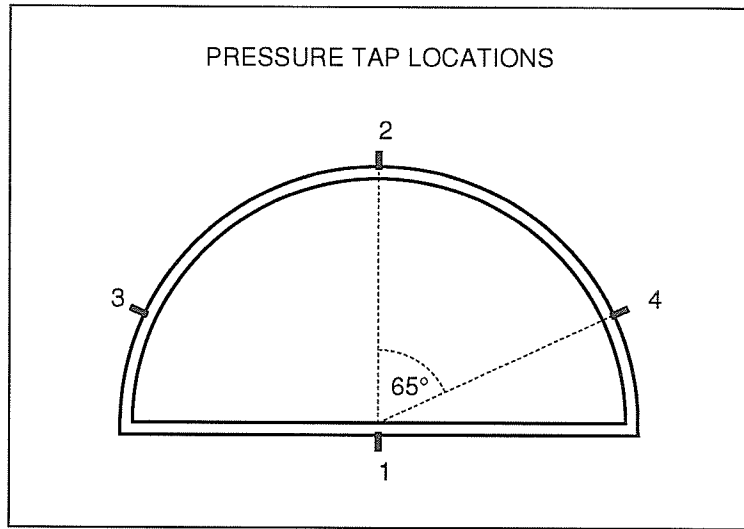


Fig. 4.2 Locations of the four pressure taps.

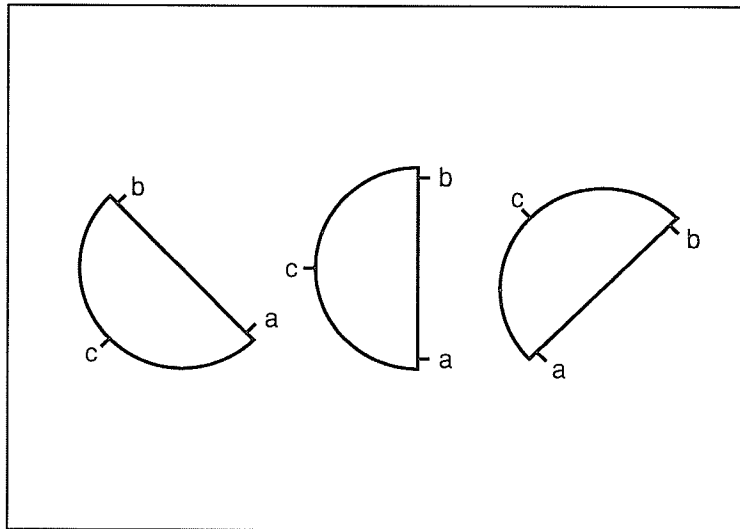


Fig. 4.3 Locations of the thermocouples, for  $\alpha = 45^\circ, 90^\circ, 135^\circ$ .

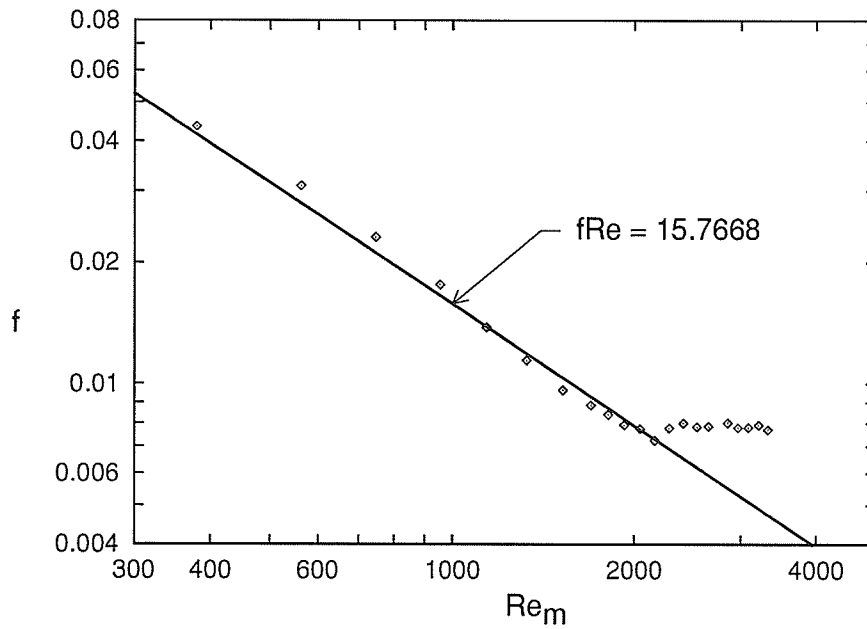
### 4.1.3 Temperature Measurements

The fluid inlet and outlet temperatures were measured by gauge 24 type T thermocouples. The outlet fluid was passed through a mixing chamber before measuring the temperature. The duct wall temperatures were measured at 19 axial locations\* within the heated section. There were 3 peripheral thermocouples at each axial location. Two thermocouples, labelled *a* and *b*, were on the flat wall, 5 mm from the corner. The third thermocouple, labelled *c*, was on the middle of the arc wall. The locations of these thermocouples are shown in Fig. 4.3 for  $\alpha = 45^\circ$ ,  $90^\circ$ , and  $135^\circ$ . For  $\alpha = 180^\circ$ , the thermocouple *a* and *b* are in the same elevation because the flat wall is horizontal for this case. The thermocouple, Type T, gauge 22, with fibre-glass covers, was directly soldered to the duct wall. These thermocouple were calibrated on-site (therefore any installation effect was eliminated). To create various wall temperature conditions, water at different temperatures was circulated around the loop at near maximum flow rate. The calibration was conducted over 18 runs covering a range of wall temperature from  $15^\circ$  to  $76^\circ$  C ( $60^\circ$  to  $170^\circ$  F). All the temperature readings agreed very well, for example, within  $0.17^\circ$  C at  $22^\circ$  C ( $0.3^\circ$  F at  $72^\circ$  F), and a maximum of within  $0.56^\circ$  C at  $76.7^\circ$  C ( $1.0^\circ$  F at  $170^\circ$  F). The calibration was done by comparing to two selected thermocouples which were already calibrated with a reference junction (Omega®, Temperature Reference Probe, model TRP-T) over a range of  $10^\circ$  to  $90^\circ$  C in a water

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\*For actual spacing between stations, see Appendix 1 (for example) which summarized the experimental data for  $\alpha = 45^\circ$ .

bath inside a vacuum flask. Generally speaking, the two thermocouple were identical, e.g. only variations within 1 least significant digit ( $0.1^{\circ}$  F) were observed. These thermocouple junctions were gently and firmly attached to the duct wall by screws and high thermal conductivity paste (Omegatherm 201) at each end of the heated section. The two reference junctions showed maximum differences only  $0.2^{\circ}$  F during the on-site calibrations.



**Fig. 4.4** The friction factor versus  $Re_m$ .

## 4.2 Experimental Procedure

The test section was tested for the critical Reynolds number by monitoring the friction factor against Reynolds number. The test were conducted for various flow rates, without heating, and the Reynolds number approximately ranging from 380 to 3300. In Fig. 4.4, the departure of friction factor from the  $fRe = 15.7668$  line[36], which indicated the transition of flow, appeared at a Reynolds number of about 2200. The friction factor agreed well with the line of  $fRe = 15.7668$  for Reynolds number up to the critical value. For  $Re_m$  between 380~2000, the averaged and maximum errors for the measured friction factors are 5.4% and 10.5% compared to the prediction from the  $fRe = 15.7668$  relation.

For combined convection, each experiment was run until steady fluid and wall temperatures were established. All the data were recorded when the steady state was established by the fluid and wall temperatures, the heating power and the fluid flow rate. For consecutive experiments, before the next experiment was started, the system was flushed by the fluid at the maximum flow rate to eliminate any possible effects remaining from the previous experiment.

The experiments were generally scheduled for  $Re_m$  ranging between 400 and 1600 so as to ensure that the flows were laminar, e.g. the local  $Re$  were well below 2100. The heat flux was varied corresponding to the  $Gr_m$  nominally ranging from  $1 \times 10^6$  to  $5 \times 10^7$  for every orientation. The range of actual operating conditions, evaluated at the

average inlet and outlet fluid temperatures, are summarized in Table 4.1. Summaries of the experimental data are given in Appendices 1 to 4 inclusive.

Table 4.1 Range of operating conditions of the experiments.

$\alpha$	$Gr_m$	$Re_m$	$Pr_m$	$\dot{m}$ (g/s)	no. of runs
45°	$1.0 \times 10^6 - 4.9 \times 10^7$	497–1529	4.2–7.7	12–53	30
90°	$1.1 \times 10^6 - 5.1 \times 10^7$	498–1549	4.6–7.6	14–51	24
135°	$1.1 \times 10^6 - 4.9 \times 10^7$	495–1505	4.2–6.1	11–43	21
180°	$1.2 \times 10^6 - 4.8 \times 10^7$	496–1500	4.2–6.2	12–43	20

### 4.3 Data Analyses

The main purpose of the experiments was to obtain the fully developed results for the duct at various orientations. In addition, the wall temperature data at the upstream side were used for demonstrating the development of Nusselt number, the effect of buoyancy force, etc. A local Nusselt number can be determined at an axial location,  $x$ , and any peripheral location,  $\theta = a, b, c, \dots$ , by

$$Nu_{x,\theta} = \frac{h_{x,\theta} D_h}{k} = \frac{Q_f D_h}{A_{iw} k (t_{x,\theta} - t_{m,x})}, \quad (4.1)$$

where  $t_{x,\theta}$  is a duct wall temperature,  $t_{m,x}$  is a fluid bulk mean temperature,  $Q_f$  is the heat gain by the fluid,  $A_{iw}$  is the inside wall area exposed to the heat flux and  $k$  is the fluid thermal conductivity evaluated at  $t_{m,x}$ . The axially local Nusselt number could be obtained from the average peripheral temperature,  $Nu_{x,t}$  or from the average peripheral heat transfer coefficients,  $Nu_{x,h}$ , as follows:

$$Nu_{x,t} = \frac{Q_f D_h}{A_{iw} k (\bar{t}_{x,\theta} - t_{m,x})} , \quad (4.2)$$

$$Nu_{x,h} = \frac{Q_f D_h}{A_{iw} k (\bar{t}_{x,\theta} - t_{m,x})} . \quad (4.3)$$

In fact, these two average Nusselt numbers,  $Nu_{x,t}$  and  $Nu_{x,h}$ , showed some differences after the experimental results were analyzed, therefore the average of these Nusselt numbers,  $Nu_x$ , is used consistently. Literally,  $Nu_x$  is a local average Nusselt number, axially local and peripherally averaged (from both methods), which is simply obtained from

$$Nu_x = \frac{Nu_{x,t} + Nu_{x,h}}{2} . \quad (4.4)$$

The modified Grashof number,  $Gr$ , used in experimental data analysis, is defined based on the total heat gain as

$$Gr = \frac{\beta g \rho^2 D_h^4 Q_f}{\mu^2 k A_{iw}} . \quad (4.5)$$

On the other hand, the other modified Grashof number,  $Gr^+$ , which emerged in the theoretical analysis, is defined based on the heat flux as

$$Gr^+ = \frac{\beta g q' R_0^3}{\nu^2 k} . \quad (4.6)$$

For a semicircular duct, it can be shown that

$$\frac{Gr}{Gr^+} = \frac{(2\pi)^4}{(\pi+2)^5} = 0.4337 . \quad (4.7)$$

The friction factor is defined as

$$f = \frac{\Delta P D_h \rho A_{fl}^2}{2 L \dot{m}^2} , \quad (4.8)$$

where  $\Delta P$  is the pressure drop across the heated section of length  $L$ , and  $A_{fl}$  is the cross sectional area of the fluid flow passage. The Reynolds number is determined from

$$Re = \frac{D_h \dot{m}}{\mu A_{fl}} . \quad (4.9)$$

The uncertainties for the computed results, e.g.  $Re$ ,  $Gr$ , were estimated from the tolerances of the measurements or the uncertainties of the calibrations, mainly based on the method in [123]. The estimated uncertainty for  $Re$  was within 4% for  $Re_m \approx$

400–1600 and within 8% for  $f$ . This resulted in about 9% uncertainty for  $fRe$ . The computation for  $Gr$  or  $Nu$  depends on the total heat gain,  $Q_f$  which had the maximum uncertainty of 4.6%. The maximum uncertainties were estimated to be about 6% for  $Gr$  and about 9% for  $Nu$ .

## 4.4 Wall Temperature Distributions

The duct wall temperature distributions for  $Gr_m$  about  $1 \times 10^7$  and  $Re_m$  about 1000, which is a typical average operating condition of these experiments, are shown in Figs. 4.5(a)–(d) for  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ . The wall temperatures are plotted along with the fluid bulk temperature so that the magnitude of the wall-to-fluid temperature difference (drop) can be seen. The temperature drop profiles are almost constant beyond  $X \approx 30D_h$  (station 6) for all four cases. In the near inlet region where  $X < 30D_h$ , wall-to-bulk fluid temperature differences are the largest and peaks in these temperature drops can be readily seen. The heights of the peak are approximately  $1.6^\circ$  to  $1.7^\circ$  C in excess of the wall temperature drops in the down stream region. The raised temperature drop indicates that the cross-stream flow is still weak and have little effect on the heat transfer. On the contrary, the decrease of the temperature drop (with axial distance) indicates that the cross-stream flow has developed and become effective in boosting the heat transfer and this results in the wall temperature decreasing as compared to the fluid bulk temperature. After  $X = 30D_h$ , the temperature drops are

uniform. The fully developed temperature drop profile implies that the effect of buoyancy force is steady.

The magnitudes of the two peripheral temperatures at the corners, i.e.  $t_a$  and  $t_b$ , can be relatively indicated by their elevations. The higher corner always showed a higher temperature. For all  $\alpha$ , the corner  $b$  was rotated to the top and therefore  $t_b$  was always higher than  $t_a$  as shown in Fig. 4.5(a), (b) and (c). The magnitudes of  $t_c$  also behaved similarly; e.g. being the lowest for  $\alpha = 45^\circ$  and the highest for  $\alpha = 135^\circ$  and  $\alpha = 180^\circ$ . For  $\alpha = 180^\circ$ ,  $t_a$  and  $t_b$  were symmetric and this is reflected by them being equal within about  $0.2^\circ$  C. It is also noted that the differences between the maximum and minimum wall temperatures were always quite uniform throughout the fully established temperature region. For the data in Fig. 4.5, the differences averaged over the last 12 stations were  $1.33^\circ$ ,  $1.19^\circ$ ,  $1.08^\circ$  and  $1.17^\circ$  C for  $\alpha = 45^\circ$ ,  $90^\circ$ ,  $135^\circ$  and  $180^\circ$  respectively. Generally, the differences in local wall temperatures (i.e. between wall thermocouples) increased with increasing  $Gr_m$ . Difference as large as  $3.0^\circ$  C ( $\alpha = 90^\circ$ ) to  $3.7^\circ$  C ( $\alpha = 45^\circ$ ) were observed at  $Gr_m = 5 \times 10^7$ . However, these effects can be illustrated, more generally, in term of Nusselt number as well, and further discussion is made in the next section.

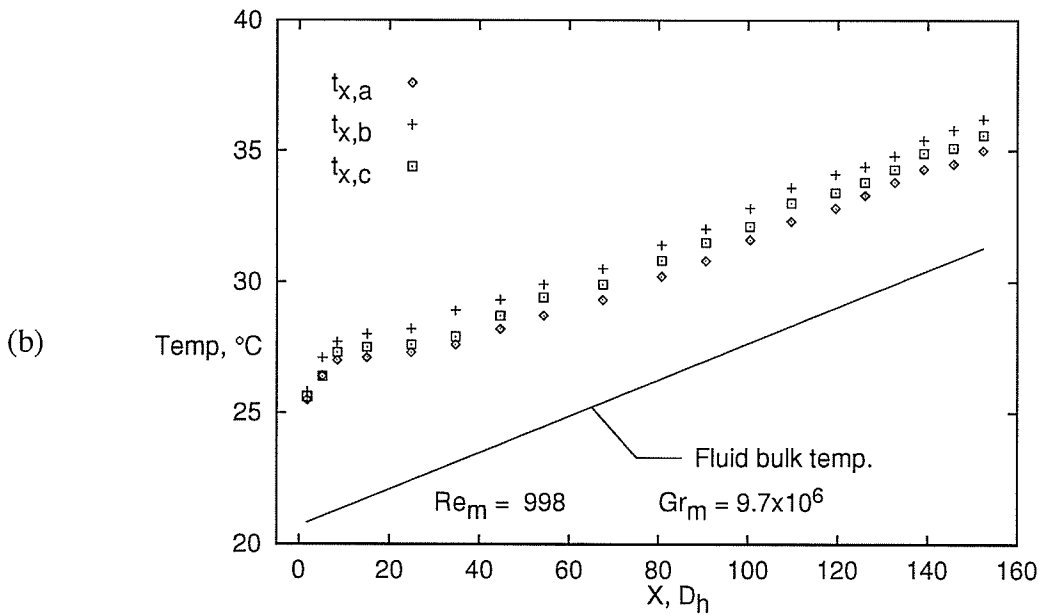
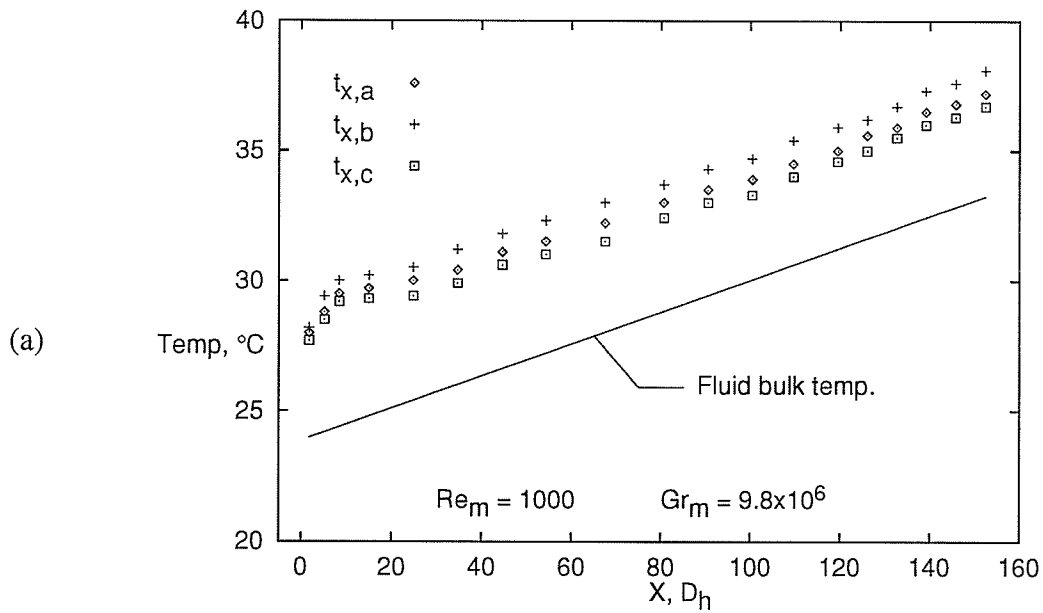
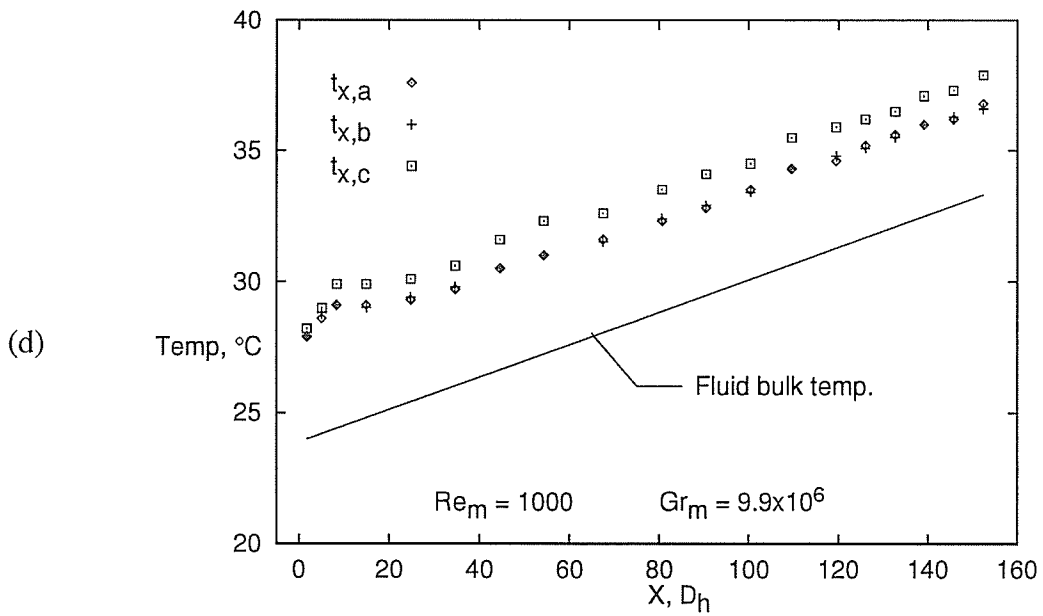
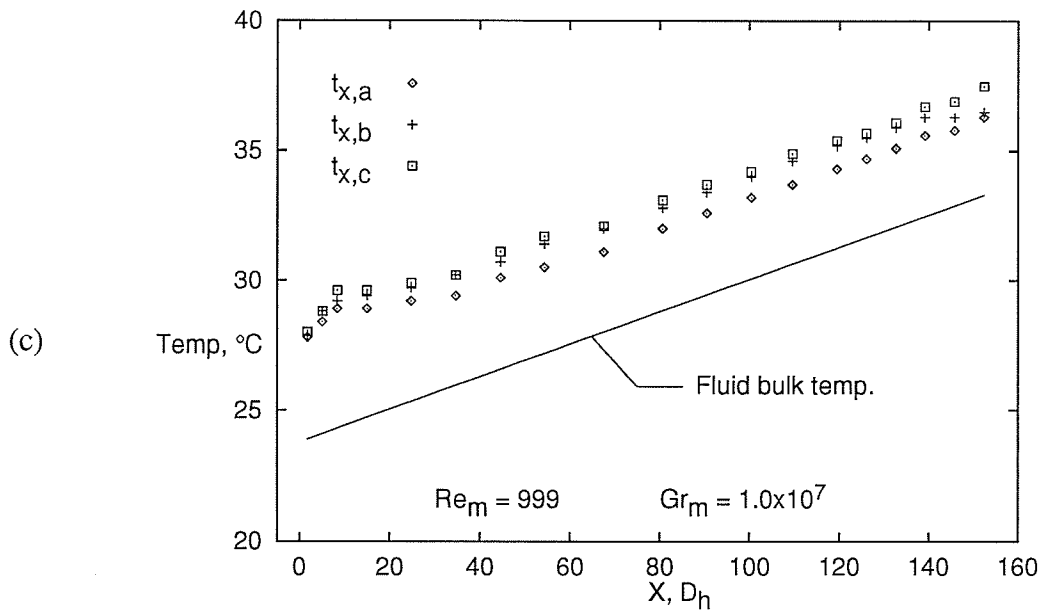


Fig. 4.5 The wall temperature distributions for  $\alpha =$  (a)  $45^\circ$ , (b)  $90^\circ$ ,



**Fig. 4.5(cont)** The wall temperature distributions for  $\alpha =$  (c)  $135^\circ$ , and (d)  $180^\circ$ .

## 4.5 Distributions of Local Nusselt Number

The local Nusselt numbers,  $Nu_{x,a}$ ,  $Nu_{x,b}$ , and  $Nu_{x,c}$ , were evaluated from  $t_{x,a}$ ,  $t_{x,b}$  and  $t_{x,c}$  via equation (4.1). These three local Nusselt numbers along with  $Nu_x$  are plotted against the axial distance,  $X$  in Fig 4.6 using the same data as for Fig. 4.5. The Nusselt number profile is approximately the inverse of the temperature drop profile, e.g. the peaks near the inlet in Fig. 4.5 become sags in Fig. 4.6, and the uniform temperature drops become the uniform Nusselt numbers. After  $X \approx 30D_h$ , the Nusselt numbers are approximately uniform, however, there is some fluctuation which seem to be normal for combined convection results, e.g. see [3], and [45] for discussion. In fact, the fluctuations are somewhat exaggerated by the Nusselt numbers, as compared to the small variations of the temperature drop profile in Fig. 4.5.

The results from Fig. 4.6 are replotted against  $x^*$  in Fig. 4.7, together with the forced convection solutions for H1 and H2 conditions [45]. The Nusselt numbers at the first station agree quite well with the forced convection solutions. This confirms that the buoyancy effect or the cross-stream flow is negligible there. From the first station, the Nusselt number decreases to the minimum near  $x^*$  viscosity  $\approx 0.001$  and increases to a steady level near  $x^* \approx 0.01$ . The uniform magnitude of Nusselt number after  $x^*$  about 0.01 implies that the buoyancy effect is fully developed. The fully developed region includes about eleven stations from the downstream end. In the fully developed region, Nusselt numbers exceed the H1 pure forced convection value (4.0880[35]) by typically a factor of about 5 to 6 as can be seen from Fig. 4.7.

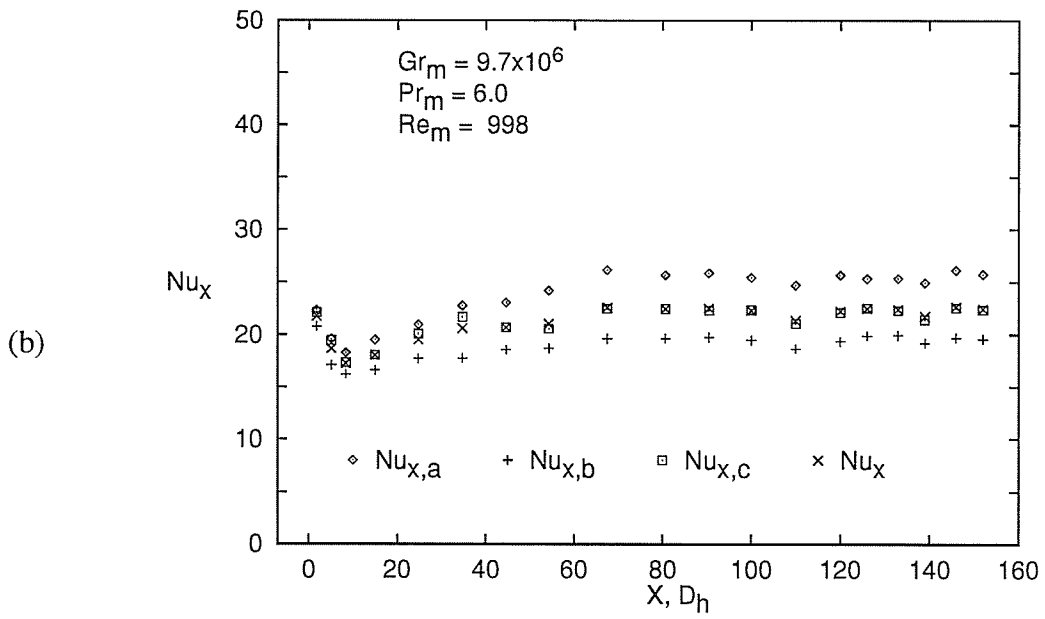
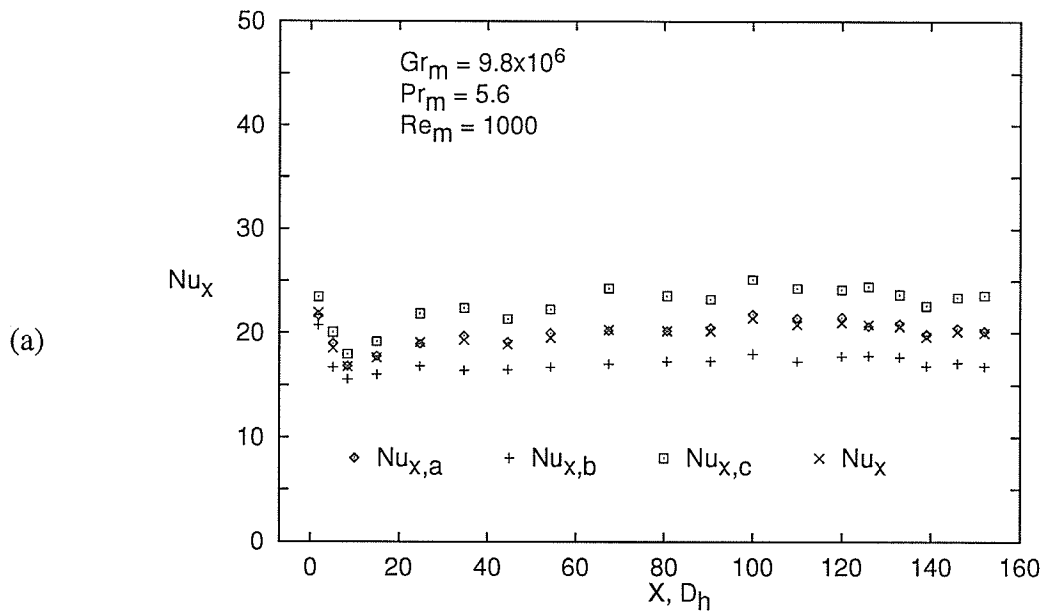


Fig. 4.6 Local Nusselt number distribution of  $X$  for  $\alpha =$  (a)  $45^\circ$ , (b)  $90^\circ$ .

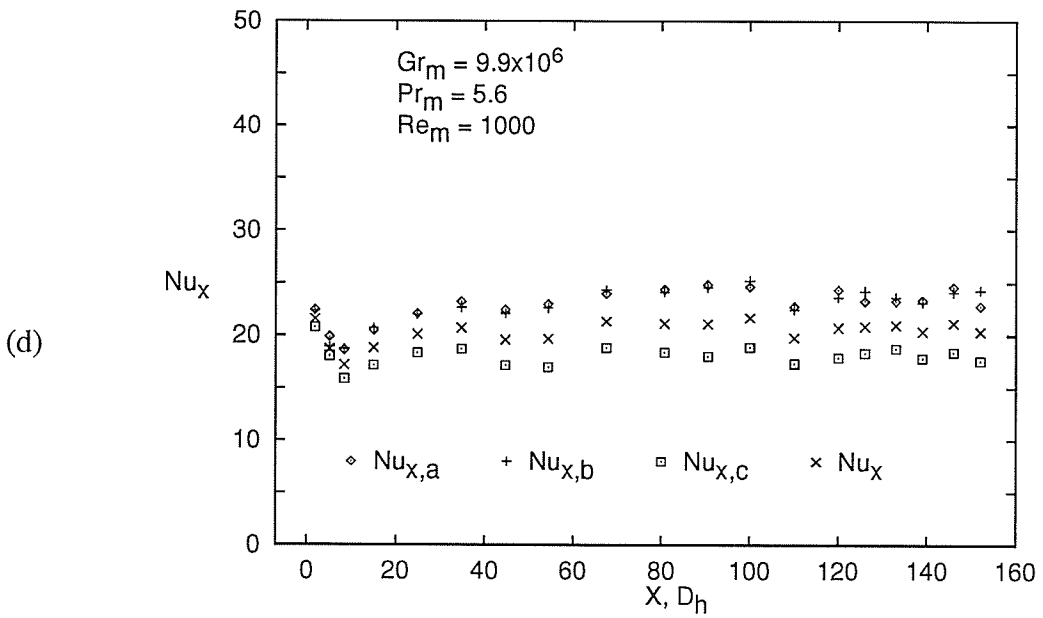
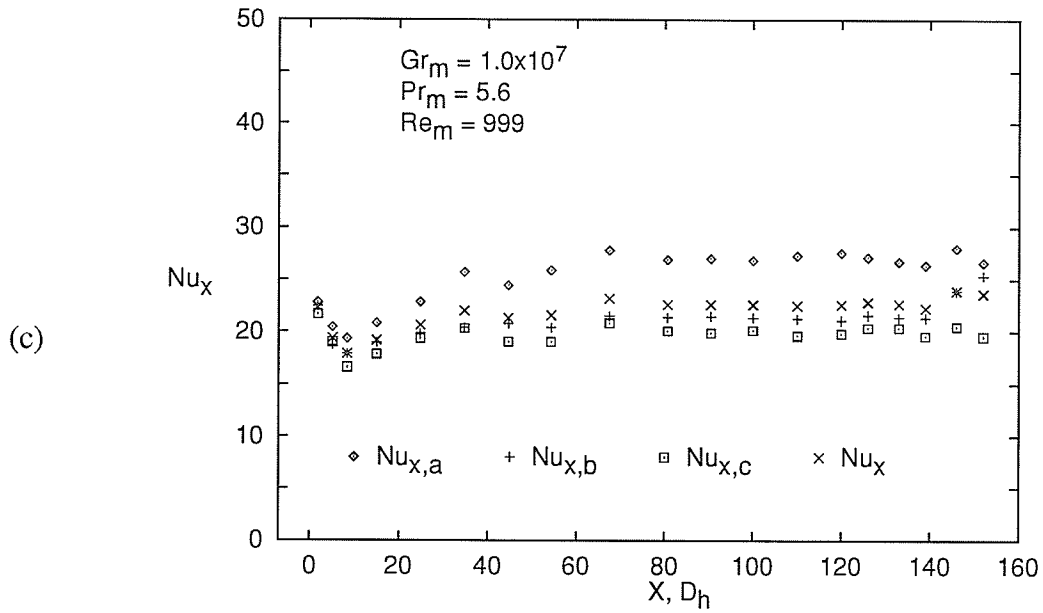
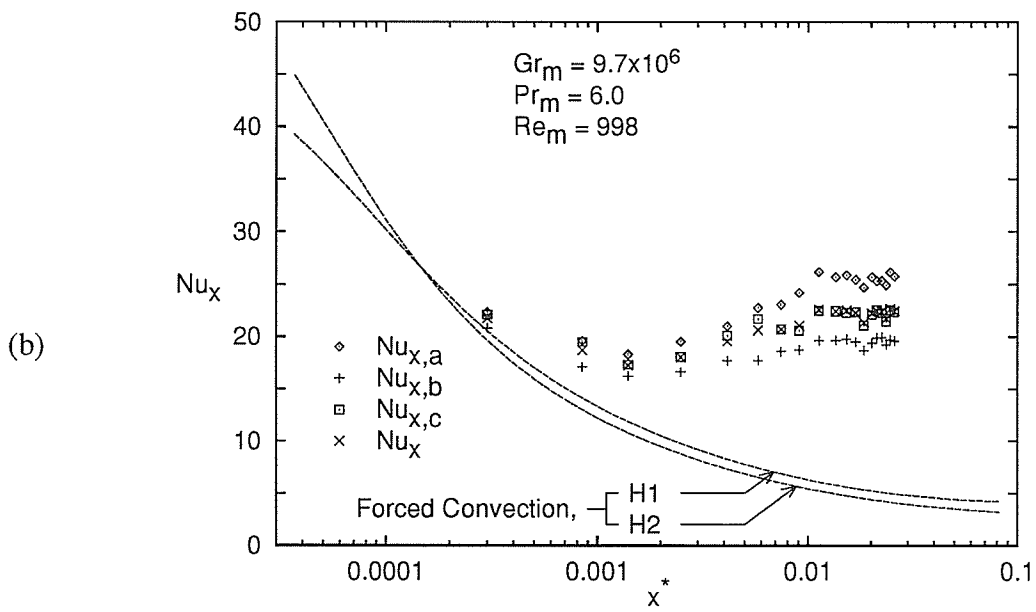
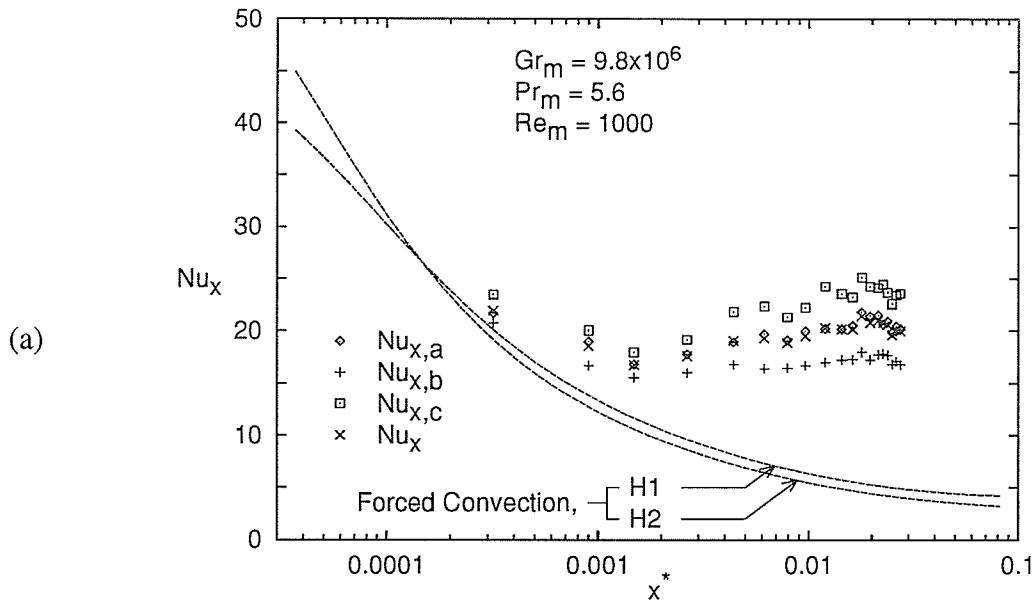
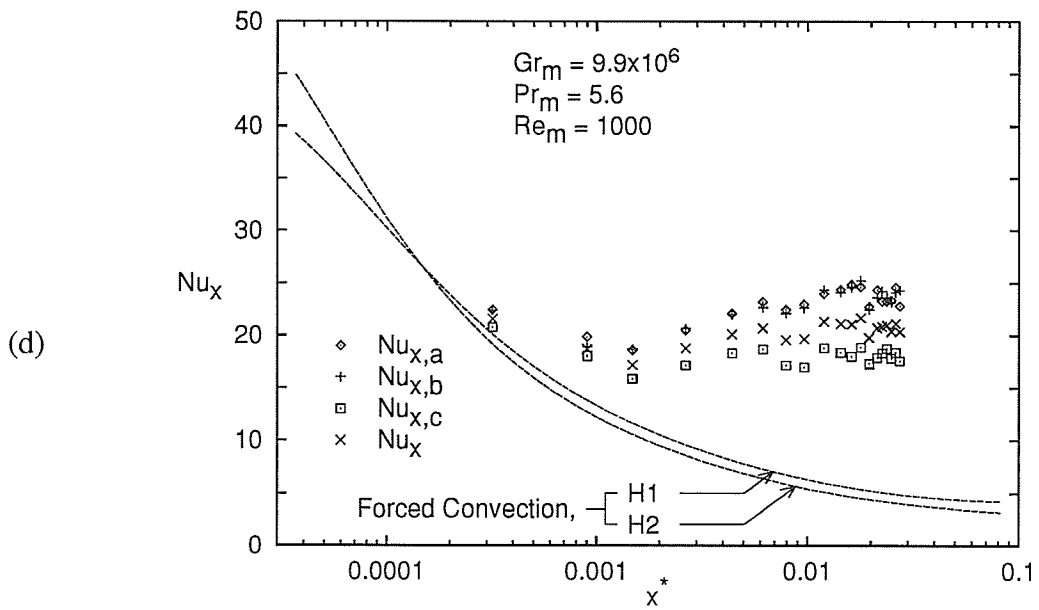
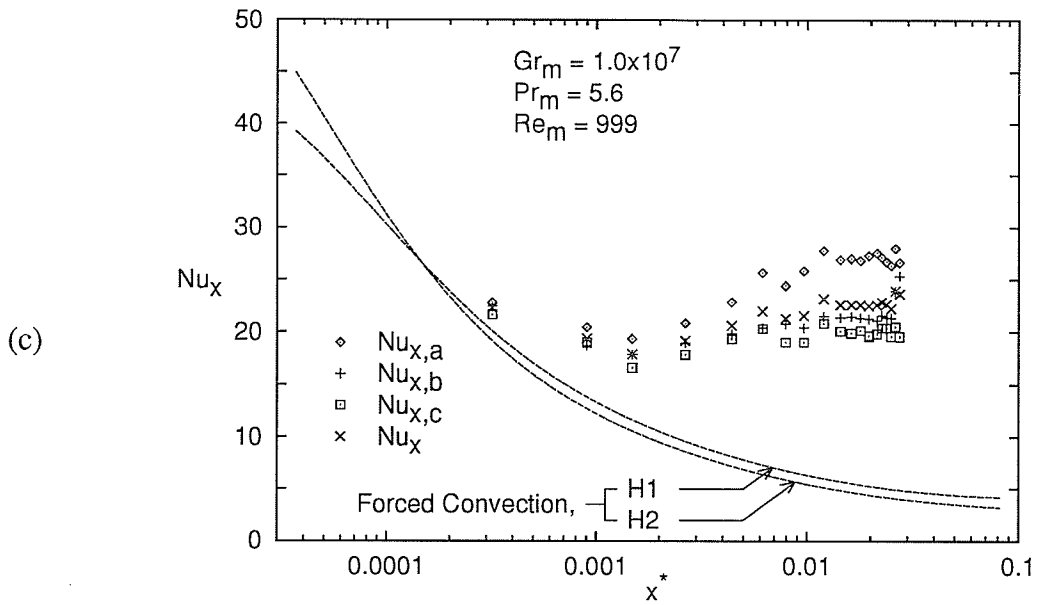


Fig. 4.6(cont) Local Nusselt number distribution of  $X$  for  $\alpha =$  (c)  $135^\circ$ , and (d)  $180^\circ$ .



**Fig. 4.7** Local Nusselt number distribution of  $x^*$  for  $\alpha =$  (a)  $45^\circ$ , (b)  $90^\circ$ ,



**Fig. 4.7(cont)** Local Nusselt number distribution of  $x^*$  for  $\alpha =$  (c)  $135^\circ$ , and (d)  $180^\circ$ .

### 4.5.1 Effect of Reynolds Number

The effect of Reynolds number on the local Nusselt number was investigated by examining the Nusselt number distributions for a set of experimental results with the same  $Gr_m$  and  $Pr_m$  but with three different  $Re_m$  of about 500, 1000 and 1500. The results for  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$  for  $Gr_m$  of  $1 \times 10^7$  are shown in Figs. 4.8(a)–(d) in term of  $Nu_x$  versus  $x^*$ .

For all cases of  $Re$  and  $\alpha$ ,  $Nu_x$  was maximum at the first station and decreased to the minimum at station 3. (According to Fig. 4.8, the locations of each station is arranged with respect to the abscissa. The experimental data given in the appendices have a few additional stations in the inlet vicinity. To locate the same physical station, station number in Fig. 4.8 must be added by 2, e.g. station 3 in Fig. 4.8 and other similar figures correspond to station no. 5 in the appendices.) From station 3,  $Nu_x$  increased to a level at station 6. The magnitudes of  $Nu_x$  was approximately uniform (with some fluctuations) from station 6 to the end of the heated section. This range agrees well with the region of the fully established temperature drops observed in the last section. Between stations 6 to 8,  $Nu_x$  decreased again and the second minimum occurred at station 7 or 8. From station 9 to the end of the heated section, the ripple magnitude was smaller due to the exclusion of station 6–8. Further decrease of the ripple was generally observed if the range was limited to between station 13 to 18, therefore the fully developed Nusselt number was determined from these six stations (13 to 18). Comparing to the magnitude of the fully developed Nusselt number, the

differences averaged over stations 9 to 19 were within 3–4% for all the data in Fig. 4.8. The maximum variations were typically about 5–7%. The averaged differences decreased to 2–3% for the range where the fully developed Nusselt numbers were evaluated.

For each  $\alpha$ , the fully developed Nusselt number magnitudes for each  $Re$  showed very small differences (about 3–5%). Therefore, the effect of  $Re$  is small for the magnitude of the fully developed Nusselt numbers. This trend was observed before in [45].

The differences in magnitudes of the local and fully developed Nusselt numbers were larger in the entrance regions. For  $\alpha = 45^\circ, 90^\circ$  and  $135^\circ$ , the minimum  $Nu_x$  at station 3 were about 20–30% less than the fully developed Nusselt number for  $Re$  of 500 and 1000 and were about 10–15% for  $Re$  of 1500. For  $\alpha = 180^\circ$ , the same trend was observed but with smaller figures of the differences. Hence the  $Re$  of 1500 showed a slight tendency of an earlier thermal fully developed due to the smaller ripple in the entrance region. Due to such tendency was not consistently observed for all cases, station 9 can be safely regarded as the beginning of the fully developed region. The magnitudes of  $x^*$  for station 9 were varied due to the magnitudes of  $Re$ . For  $Re$  of 500, 1000, and 1500,  $x^*$  values for station 9 are 0.022–0.026, 0.011–0.012 and 0.007–0.008, respectively. These values are 19–23%, 9–10%, and 6–7% of the entrance length for the pure forced convection. These figures can be considered as thermal entrance lengths for design purposes.

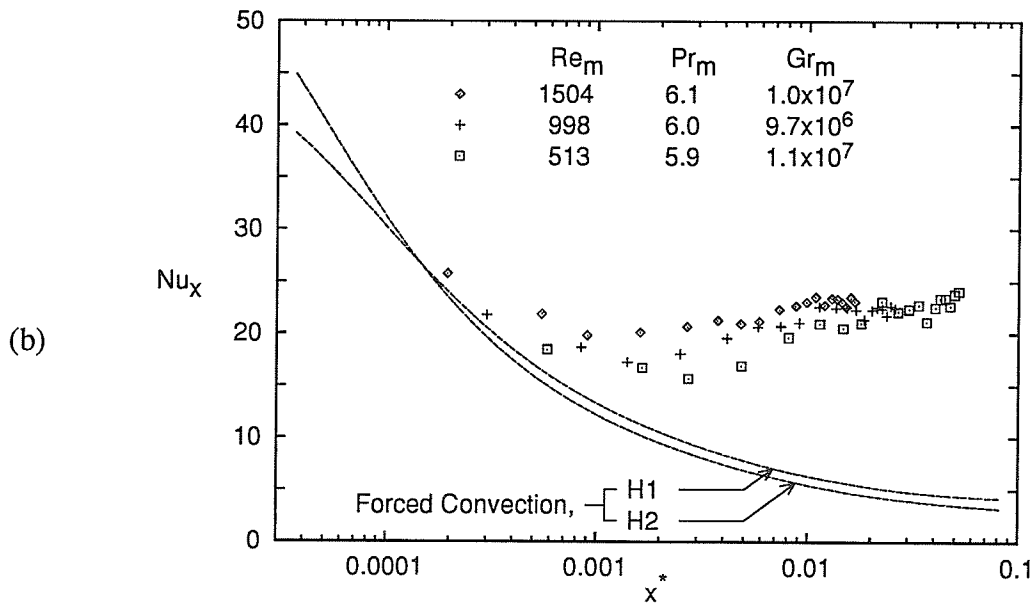
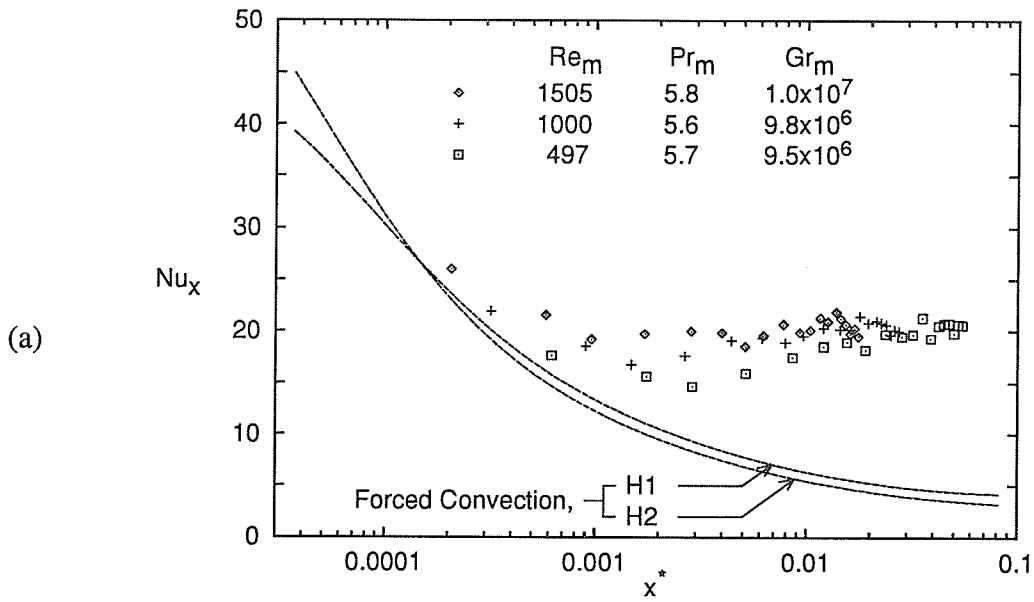


Fig. 4.8 Comparisons of the  $Nu_x$  for various  $Re_m$ ,  $\alpha =$  (a)  $45^\circ$ , (b)  $90^\circ$ .

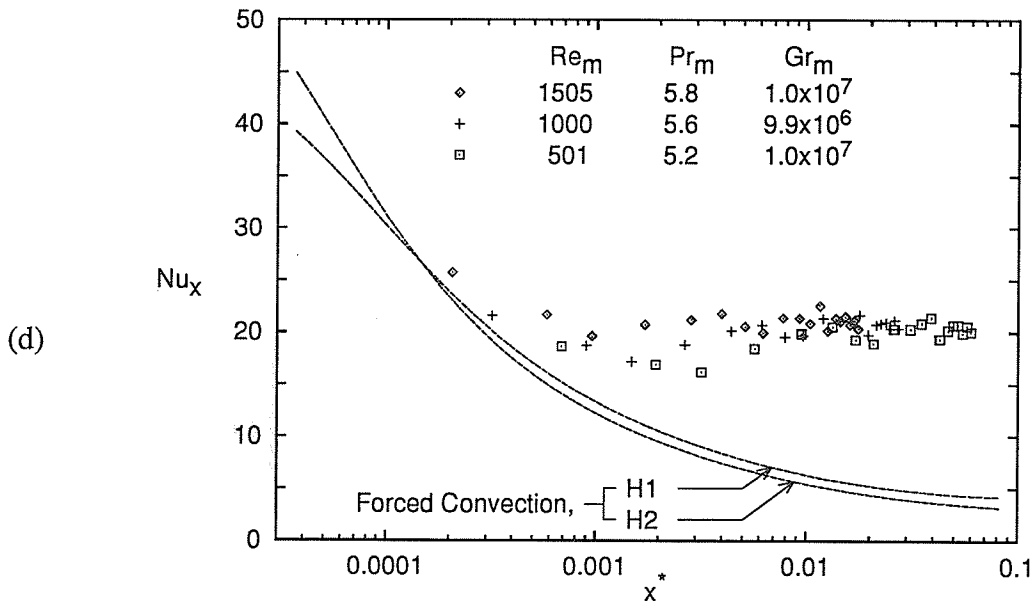
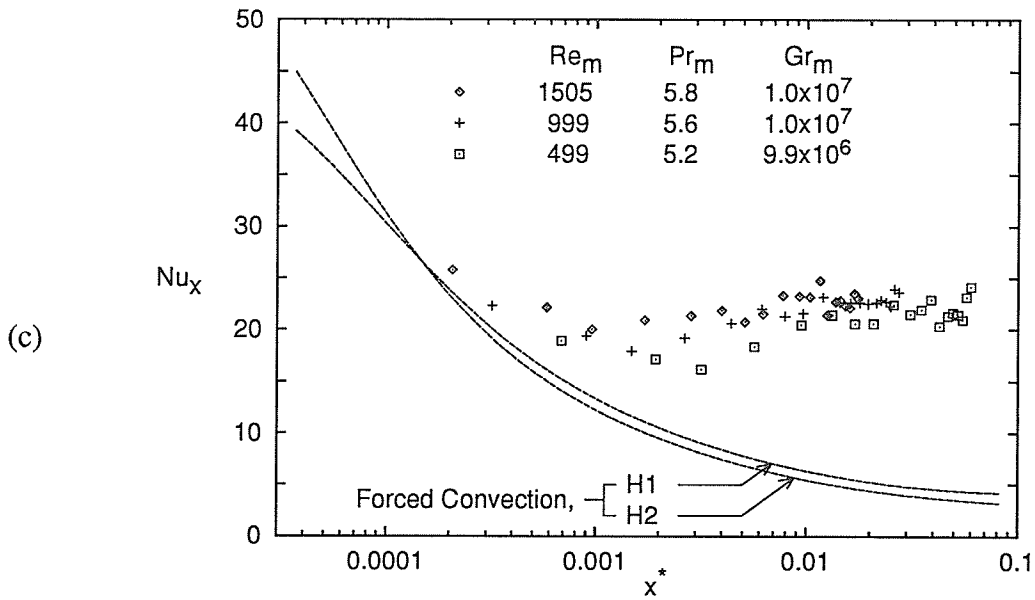


Fig. 4.8(cont) Comparisons of the  $Nu_x$  for various  $Re_m$ ,  $\alpha =$  (c)  $135^\circ$ , and (d)  $180^\circ$ .

## 4.5.2 Effect of Grashof Number

The Grashof number is the major controlling parameter for the Nusselt number. The effect of Grashof number, in term of  $Gr_m$ , is demonstrated in Figs. 4.9(a)–(d) for  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ , respectively, which contain the data for a fixed  $Re_m$  of about 1000 with three  $Gr_m$  values of approximately  $5 \times 10^6$ ,  $1 \times 10^7$  and  $5 \times 10^7$ . The Nusselt and Grashof number relationship is well known in that Nusselt number increases with increasing Grashof number. This trend is clearly shown in Fig. 4.9. At the downstream end, the Nusselt numbers for  $Gr_m = 5 \times 10^7$  are much higher than those for  $Gr_m = 1 \times 10^7$  while the differences are comparatively small between  $Gr_m$  of  $5 \times 10^6$  and  $1 \times 10^7$ . This indicates that the effect on  $Nu_x$  of Grashof number is more prominent as its magnitude increases. The plot of the fully developed Nusselt number and Grashof number in the next section will demonstrate this relationship more clearly.

The variations of  $Nu_x$  were observed similarly to the last subsection. The averaged variations of  $Nu_x$  compared to the fully developed values were typically within 2–3% for station 9 through the last station. The maximum variations are within 2–8% in general. For station 13–18, the averaged variations decreased somewhat, and the similar magnitudes as for the previous case were observed.

The magnitudes of the fully developed Nusselt number for  $Gr$  of  $5 \times 10^7$  are about 50–60% larger than those of  $Gr \approx 5 \times 10^6$ . This roughly indicates that the fluctuations of  $Nu_x$  (which are about 2–3% of the fully developed Nusselt numbers uniformly for each  $Gr$ ) in the fully developed region increase in magnitudes about 50–60% as  $Gr$

increases from  $5 \times 10^6$  to  $5 \times 10^7$ . This can be observed in Fig. 4.9 as well. The same conclusion was also obtained by Lei [45].

From the inlet, the magnitudes of  $Nu_x$  decreased to the minimum at station 3 and then increased to near the fully developed values between stations 5–8. Only small influences of the magnitudes of  $Gr$  on the beginning point of the fully developed region were observed for all cases of  $\alpha$ . Again, due to the reasonably smooth distributions of  $Nu_x$  as mentioned earlier, station 9 was regarded as the beginning point of the fully developed region. The entrance lengths in term of  $x^*$  for station 9 are about 0.010, 0.012, and 0.015 for  $Gr$  of  $5 \times 10^6$ ,  $1 \times 10^7$ , and  $5 \times 10^7$  respectively. Compared to the entrance lengths for the pure forced convection in semicircular ducts[45], the entrance lengths are about 9%, 10%, and 13% for  $Gr$  of  $5 \times 10^6$ ,  $1 \times 10^7$ , and  $5 \times 10^7$  respectively.

The entrance length was experimentally observed to slightly decrease with increasing  $Gr$  in the previous experiments for  $\alpha = 0^\circ$  by Lei [45], i.e. 7%, 9% and 15% for  $Gr$  of about  $4 \times 10^7$ ,  $6 \times 10^6$ , and  $1 \times 10^6$  respectively. Even though the figures given by Lei [45] and the present results are in the reverse order, their magnitudes are similarly about 10%. Cheng and Ou [19] compared their predictions to the experimental results for water (conducted by Petukhov and Polyakov) for circular ducts with uniform heating and concluded that the free convection effect shortens the entrance lengths, similar to Lei's observation above.

The patterns of Petukhov and Polyakov data for Rayleigh number of  $8.75 \times 10^4$ ,  $3.75 \times 10^4$ , and  $7.5 \times 10^5$  are actually very similar to Fig. 4.9 and Fig. 6.8 in [45] in the manner that local Nusselt number decreased to the minimum near the inlet and then

increased to a steady level. The Nusselt number distributions predicted by [19] also showed this pattern but only for low  $Ra$  ( $10^4$  and  $10^5$ ). The appearance of the minimum was discussed in [19] that it is because of the balance of the entrance (inlet fluid stream containing no cross-stream circulation yet) and free convection effects. If there is no minimum point, the patterns become many horizontal lines simply branching from the pure forced convection curve to the right, e.g. see Fig. 13 in [124]. It is true for such patterns that the onset of combined convection (the point on the  $x^*$  axis where local Nusselt number is 5% higher than that of pure forced convection) and the entrance length ( $x^*$ ) becomes smaller as Rayleigh number increases. This is because the curve for a higher Rayleigh number separates from the pure forced convection curve at a higher vertical level and hence at a smaller  $x^*$  due to the shape of the pure forced convection curve. On the other hand, when the patterns with large sags (e.g. for the present results, the minimum points differed from the fully developed values up to 30%) are observed, such regions must be excluded from the fully developed regions where the variations should be normally much smaller. And hence the entrance lengths were observed to be approximately the same in terms of  $X$  and slightly increased with increasing  $Gr$  in terms of  $x^*$ .

In fact, the magnitude of  $x^*$  ( $= X/D_p Re Pr$ ) depends on the magnitudes of  $Re$  and  $Pr$ . For a given value of  $X$ , a smaller  $Re$  results in a larger  $x^*$ . This can be consistently observed either in Fig. 4.8 or [45]. For Fig. 4.8, most of the data for  $Re \approx 500$  locate where  $x^* > 0.02$  whereas the data for  $Re \approx 1500$  locate in the region where  $x^* < 0.02$ . Although the entrance lengths in terms of  $X$  are slightly dependent on  $Re$ , the entrance

lengths in terms of  $x^*$  depend on  $Re$  quite strongly for the same conditions of  $Pr$  and  $Gr$ . The  $Pr$  has the similar effect as  $Re$  but with much less influence due to a smaller variations for the data obtained by using water as the working fluid. This appeared in either Fig. 4.9(b) or [45] both of which showed the same trend, e.g. as  $Pr$  value decreases from about 6 to about 4.6, the majority of the data points in the downstream region shifted significantly to the right. On the other hand, if the magnitudes of  $Pr$  were the same, all the corresponding data for each  $Gr$  should line above or below each others more evenly. This, in addition to the slight dependence between the entrance lengths in terms of  $X$  and  $Gr$ , implies that the effect of  $Gr$  on the entrance length in terms of  $x^*$  is small for given values of  $Pr$  and  $Re$ .

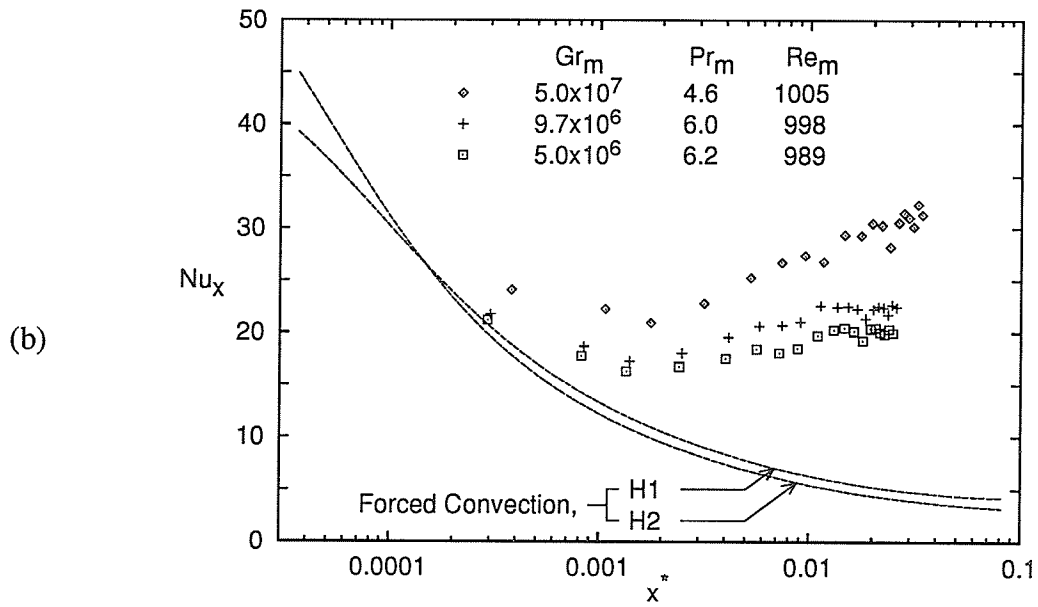
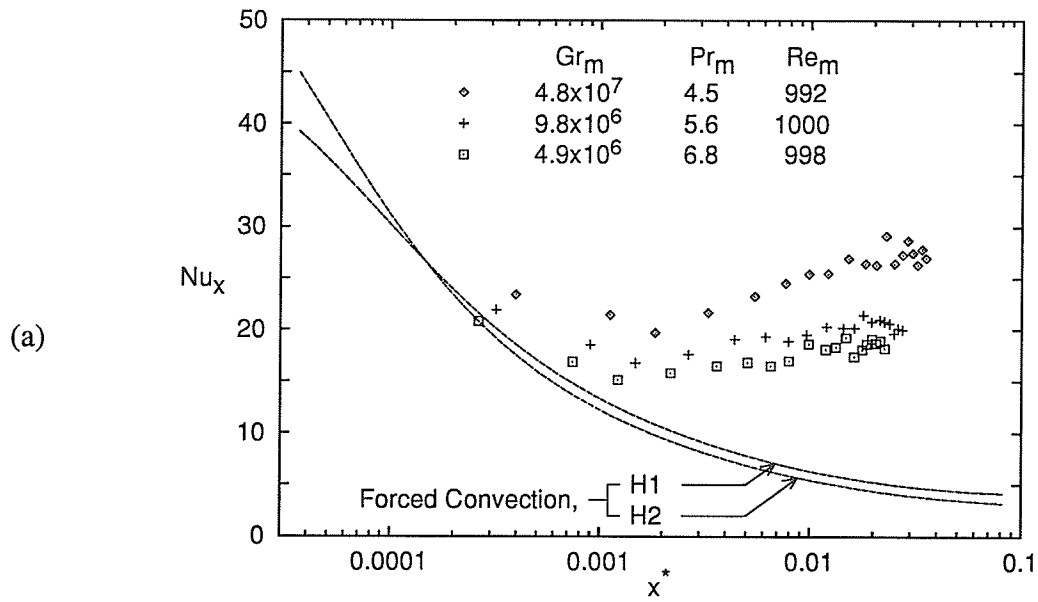


Fig. 4.9 Comparisons of the  $Nu_x$  for various  $Gr_m$ ,  $\alpha =$  (a)  $45^\circ$ , (b)  $90^\circ$ ,

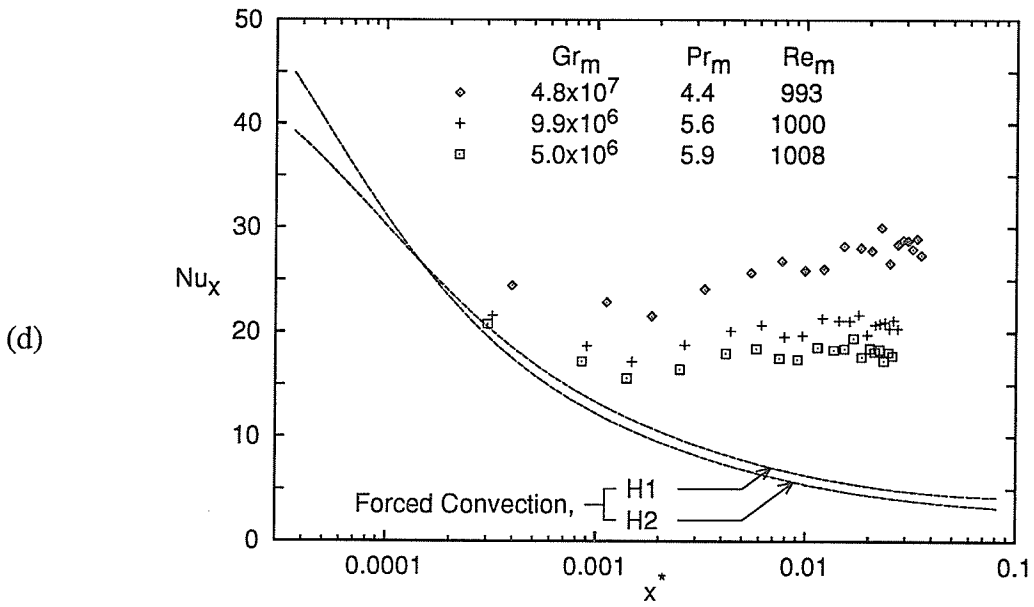
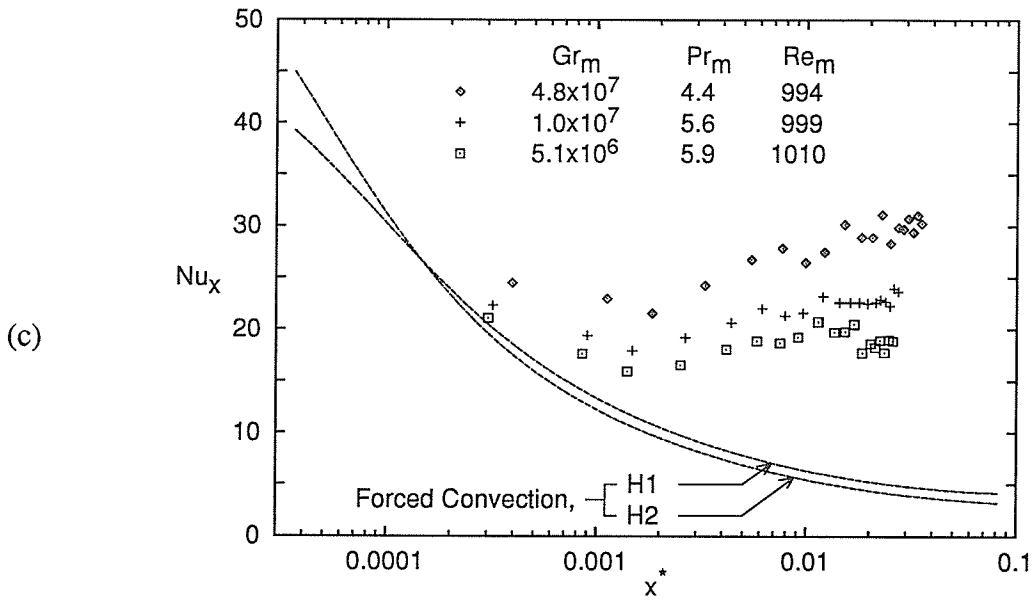


Fig. 4.9(cont) Comparisons of the  $Nu_x$  for various  $Gr_m$ ,  $\alpha =$  (c)  $135^\circ$ , and (d)  $180^\circ$ .

## 4.6 Fully Developed Nusselt Number

In the previous sections, it has been shown that the magnitude of the fully developed Nusselt number depend heavily on Grashof number. In this section, the fully developed Nusselt numbers are evaluated and investigated with the Grashof number as a dependent variable. The effect of Prandtl number, which as shown in the theoretical part of this thesis can be eliminated if the dependent variable  $Gr^+Pr$  is used instead of  $Gr^+$ , is also included in this section.

### 4.6.1 With Grashof Number Variable

The examination of  $Nu_x$  for all data showed that it was approximately uniform after  $x^* \approx 0.01$  which corresponded to more than 10 stations from the downstream end. Fully developed Nusselt number was evaluated by the length mean average of the last six  $Nu_x$ , not including the last station (to avoid any end effect even though it was not noticeable). The average thermal conditions for the fully developed region are shown in Table 4.2.

Table 4.2 The average thermal conditions of the fully developed region.

$\alpha$	$Gr^+$	$Pr$	$Gr^+Pr$	$Re$
45°	$2.7 \times 10^6 - 2.1 \times 10^8$	3.4–7.6	$2.0 \times 10^7 - 8.1 \times 10^8$	513–1727
90°	$2.5 \times 10^6 - 2.1 \times 10^8$	3.7–7.4	$1.6 \times 10^7 - 8.4 \times 10^8$	514–1787
135°	$2.7 \times 10^6 - 2.0 \times 10^8$	3.4–6.1	$1.6 \times 10^7 - 7.0 \times 10^8$	505–1730
180°	$2.9 \times 10^6 - 1.9 \times 10^8$	3.4–6.1	$1.7 \times 10^7 - 6.9 \times 10^8$	507–1729

Table 4.3 The coefficients for eqn. (4.10) and standard deviations.

$\alpha$	$a$	$b$	$\sigma$	$\sigma_{\max}$	no. of data points
45°	0.119	0.204	3.3%	6.3%	30
90°	0.107	0.217	3.3%	7.4%	24
135°	0.098	0.220	3.2%	6.6%	21
180°	0.123	0.203	2.7%	5.2%	20

The average fully developed Nusselt number,  $Nu$ , will be presented in its relative form as  $Nu/Nu_o$ , where  $Nu_o$  is a fully developed Nusselt number for the H1 forced convection solution which has a magnitude of 4.0880[35]. The relative Nusselt number are plotted against the modified Grashof number,  $Gr^+$ , which is equivalent to  $0.4337Gr$ , in Fig. 4.10 for  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ . The data are fitted using the following form

$$\frac{Nu}{Nu_o} = 1 + a(Gr^+)^b \quad . \quad (4.10)$$

The coefficients  $a$  and  $b$  for  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$  and the standard deviation are

summarized in Table 4.3. The data are represented by the equation very well for  $\alpha$  of  $45^\circ$ ,  $135^\circ$  and  $180^\circ$  while the  $90^\circ$  data show more scatter as is reflected by the slightly larger maximum value of the standard deviation.

The correlations in Table 4.3 are used in generating the curves in Fig. 4.10 whereas the Nusselt number data are shown in terms of points. Fig. 4.10 illustrates that Nusselt number is strongly dependent on Grashof number. The strong relationship of these two parameters has been commonly recognized by both the theoretical analysis (e.g. [5]) and the experiments (e.g. [10]). In addition, the theoretical solutions of the present results demonstrate that the locus of the Nusselt number on the Nusselt–Grashof number plane is always a well defined curve for a given  $\alpha$ . The shapes of the curves in Fig. 4.10 show that Nusselt number exponentially increases with increasing Grashof number. This suggests that cross-stream flow strengthens progressively with increasing heating rate which results in continual decreasing of the wall-to-fluid temperature drop.

To demonstrate the effect of the duct orientation, the curves in Fig. 4.10 are replotted in Fig. 4.11. The Nusselt number curves for  $\alpha = 90^\circ$  is a little higher than that for  $135^\circ$  but are very close for the entire range of  $Gr^+$ . For  $\alpha = 45^\circ$  and  $180^\circ$ , the Nusselt number curves are also very close and the curve for  $\alpha = 180^\circ$  is a little higher. Considering the two pairs, the differences in Nusselt number magnitudes are the highest at high  $Gr^+$ . Between  $\alpha = 45^\circ$  and  $90^\circ$ , the difference is about 10% at  $Gr^+ \approx 2 \times 10^8$ . The difference is a little smaller between  $\alpha = 90^\circ$  and  $180^\circ$ . On the contrary, at the lower end of the  $Gr^+$  range, the differences for all curves are almost negligible. This indicates that the orientation significantly affects the heat transfer performance only if

the heating rate is high and the effect decreases with a decrease in heating rate. As we can extrapolate from the results in Fig. 4.11, the four curves must collapse and approach the pure forced convection result if  $Gr^+$  keep decreasing toward zero.

In Fig. 4.11, the Nusselt number ratios are about 7–8 at  $Gr^+ = 2 \times 10^8$  for all  $\alpha$ . The rate of increases for each curve suggests that a very high Nusselt number is likely if  $Gr^+$  is raised much beyond  $2 \times 10^8$ .

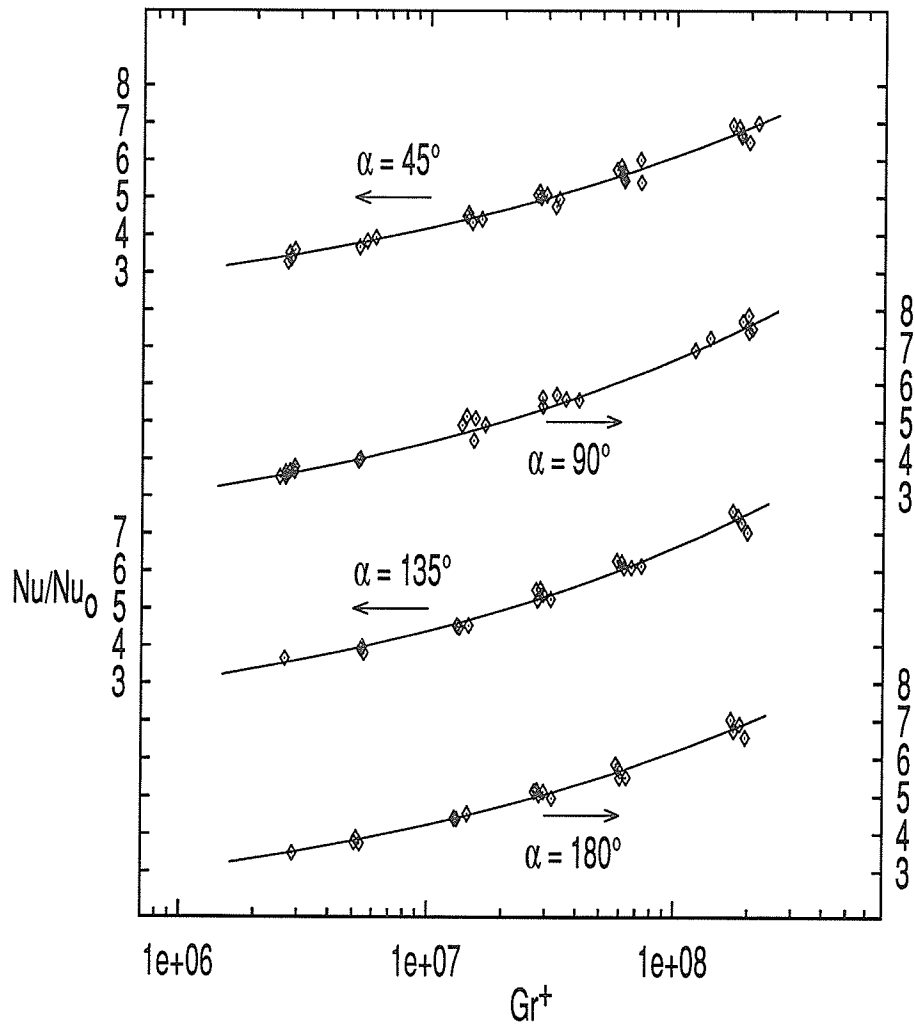


Fig. 4.10 Fully developed Nusselt number versus  $Gr^+$  for  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ .

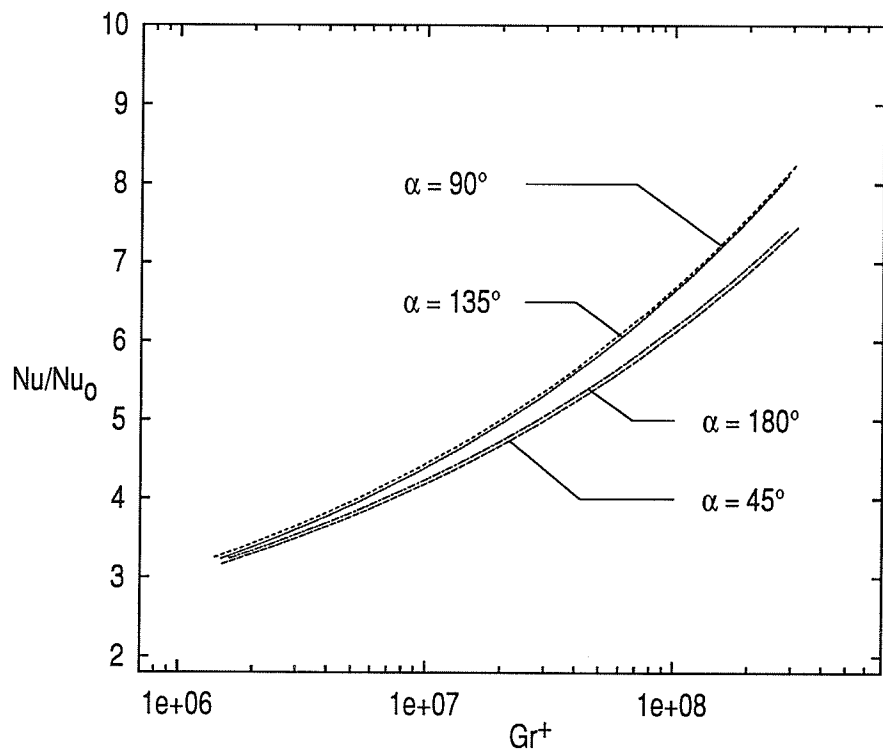


Fig. 4.11 Comparison of Nusselt number for  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ .

## 4.6.2 With $Gr^+Pr$ Variable

The same data as previously presented are now modified due to the change of the independent variable from  $Gr^+$  to  $Gr^+Pr$  and the results are shown in Fig. 4.12. The  $Nu/Nu_o$  data are correlated to  $Gr^+Pr$  using the form similar to eqn. (4.10) and the results are summarized in Table 4.4.

**Table 4.4** The coefficients and standard deviations for  $Nu/Nu_o = 1 + a(Gr^+Pr)^b$  relationship.

$\alpha$	$a$	$b$	$\sigma$	$\sigma_{\max}$
45°	0.0389	0.246	2.7%	4.9%
90°	0.0414	0.247	2.2%	2.8%
135°	0.0378	0.252	2.5%	5.6%
180°	0.0515	0.232	2.0%	3.5%

A slight improvement of the relation of  $Nu/Nu_o-Gr^+Pr$  over  $Nu/Nu_o-Gr^+$  can be observed by the reduction of  $\sigma$  in Table 4.4 and also in Fig. 4.12. The points in Fig. 4.12 seem to contract as compared to Fig. 4.10. These improvements are so small that it is not appropriate to draw any conclusion, however, the trends can be seen. This small variation is due to a limited range of Prandtl number data which were from a single fluid type.

The comparison of these curves according to the orientation angles are shown in Fig. 4.13 including the data for  $\alpha = 0^\circ$  from the previous experiment [45]. Again the curve for  $\alpha = 90^\circ$  and  $135^\circ$  are very close but the  $135^\circ$  curve is now slightly higher at

high  $Gr^+$ . The gap between the  $90^\circ$  or  $135^\circ$  and the  $45^\circ$  curves are wider in Fig. 4.13 compared to Fig. 4.11. At  $Gr^+Pr \approx 10^9$ , Nusselt number for  $\alpha = 135^\circ$  is about 10% larger than those of  $\alpha = 45^\circ$  and  $180^\circ$  and about 40% larger than that of  $\alpha = 0^\circ$ . At the low end of the  $Gr^+Pr$  range, the Nusselt number for  $\alpha = 90^\circ$  is somewhat larger than that of  $\alpha = 135^\circ$ . At  $Gr^+Pr \approx 2 \times 10^7$ , the Nusselt number for  $\alpha = 90^\circ$  is about 2%, 6%, and 28% larger than those of  $\alpha = 180^\circ$ ,  $45^\circ$  and  $0^\circ$  respectively. At low  $Gr^+$ , the curves for  $\alpha = 90^\circ$ ,  $135^\circ$  and  $180^\circ$  agree well while that of  $45^\circ$  is somewhat lower. These results suggest that the effect of  $\alpha$  can enhance Nusselt number as large as 10–40% at high heating rate.

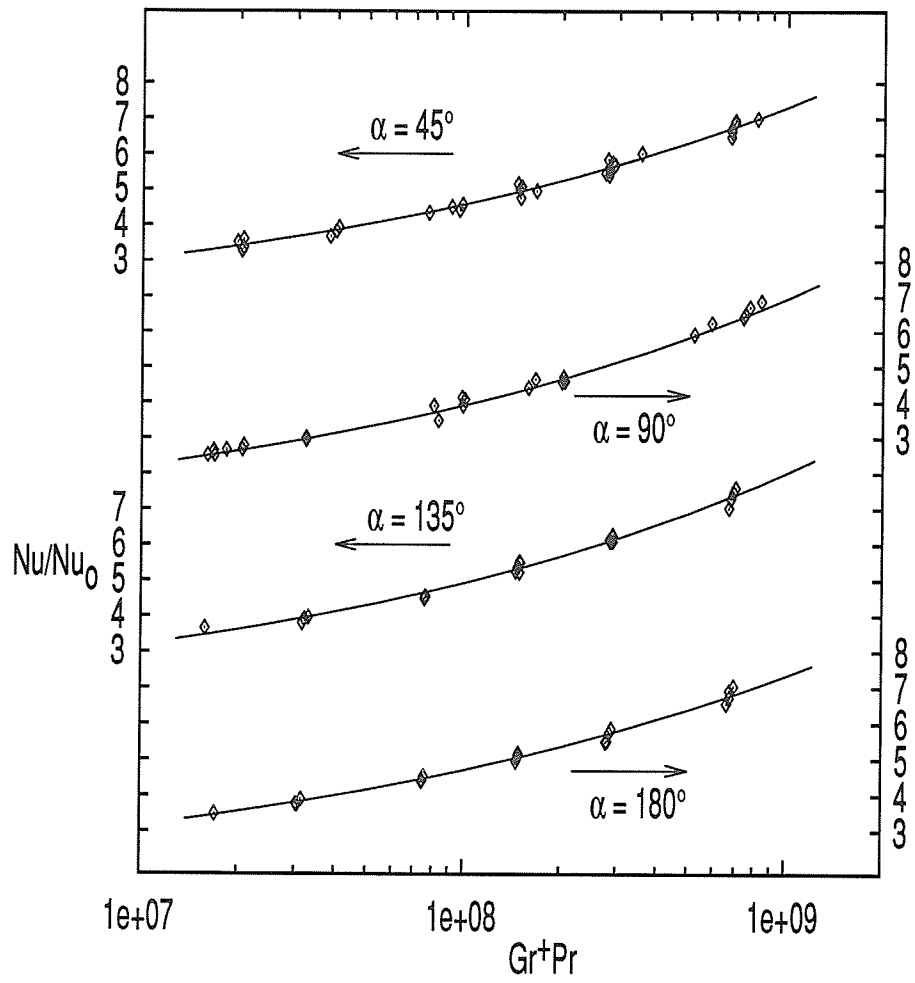


Fig. 4.12 Fully developed Nusselt number versus  $Gr+Pr$ ,  
for  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ .

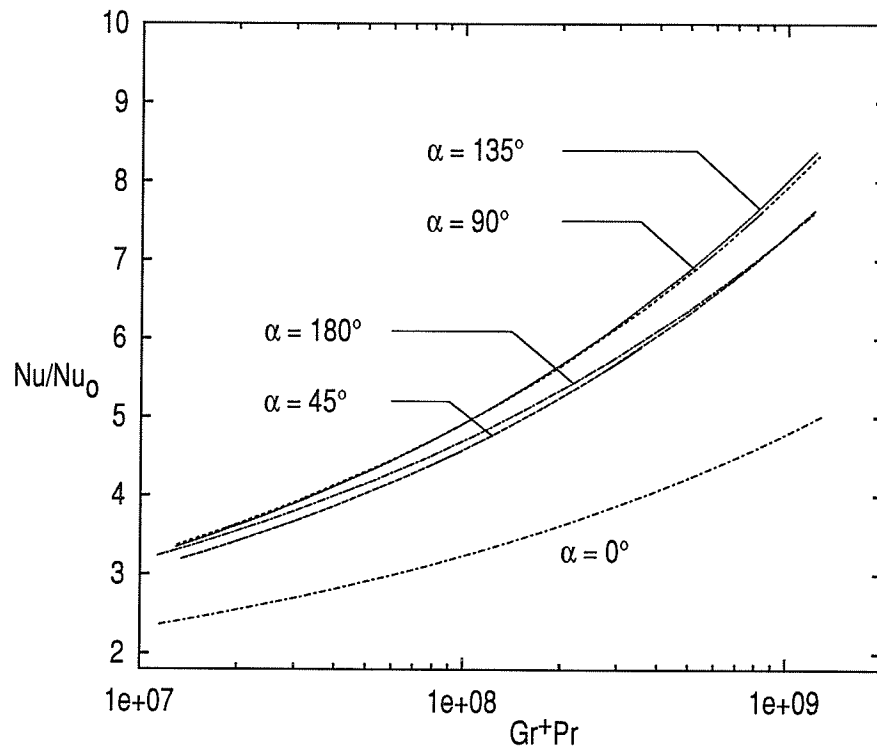


Fig. 4.13 Comparison of  $Nu$  for  $\alpha = 0^\circ$ [45],  $45^\circ$ ,  $90^\circ$ ,  $135^\circ$  and  $180^\circ$ .

## 4.7 Friction Factor

Friction factor were determined using eqn. (4.8) which involved measured mass flow rates and pressure drops. Due to the fact that the pressure drops were measured across the whole heated section which included both the thermal developing and fully developed regions, it is first emphasized that they reflect the average pressure drops for both regions. The Reynolds numbers were consequently evaluated at the average of the fluid inlet and outlet temperatures, which should be called  $Re_m$ . However, the product of the friction factor and Reynolds number will be referred to simply as  $fRe$ . Also it is noted that the symbol  $fRe$  is used in the theoretical analyses when dealing with the fully developed condition only.

The  $fRe$  is normalized by the forced convection solution  $(fRe)_o = 15.7668$ [36] and correlated with the variable  $Gr^+$  similar to eqn. (4.10). The coefficients and the standard deviations are listed in Table 4.5. The  $fRe/(fRe)_o$  ratios are plotted in Fig. 4.14 against  $Gr^+$  for  $\alpha = 0^\circ$ [45],  $45^\circ$ ,  $90^\circ$ ,  $135^\circ$  and  $180^\circ$ . Buoyancy force has a little effect on  $fRe$  for  $Gr^+$  less than about  $1 \times 10^7$  which can be seen from the stable magnitudes of  $fRe/(fRe)_o$  in that region. The  $fRe$  increases gradually as  $Gr^+$  exceeds about  $2 \times 10^7$ . The effect of duct orientation is shown in Fig. 4.15. The  $fRe$  product is the highest for  $\alpha = 90^\circ$ , followed by those for  $\alpha = 135^\circ$ ,  $45^\circ$ ,  $180^\circ$  and  $0^\circ$ . The  $fRe$  magnitude for  $\alpha = 45^\circ$  is somewhat higher than that of  $180^\circ$  but the latter becomes higher at the high end of the  $Gr^+$  range. For  $\alpha = 0^\circ$  and  $45^\circ$ , the  $fRe$  ratios are about 1.3 and 1.5 at  $Gr^+ \approx 10^8$ .

For all  $\alpha$ , the increases between  $Gr^+$  of  $2 \times 10^7$  to  $10^8$  are quite rapid and the ratios approach the value of 2. However, the increase of  $fRe$  ratio due to the cross-stream flow is small compared to corresponding increase in Nusselt ratio.

Table 4.5 The coefficients and standard deviations  $fRe/(fRe)_o = 1 + a(Gr^+)^b$ .

$\alpha$	$a$	$b$	$\sigma$	$\sigma_{\max}$
45°	$1.0483610 \times 10^{-6}$	0.7094759	4.2%	10.2%
90°	$4.8994200 \times 10^{-6}$	0.6469435	4.3%	8.4%
135°	$5.5703270 \times 10^{-6}$	0.6281309	3.5%	6.8%
180°	$6.4307020 \times 10^{-8}$	0.8597980	3.6%	9.7%

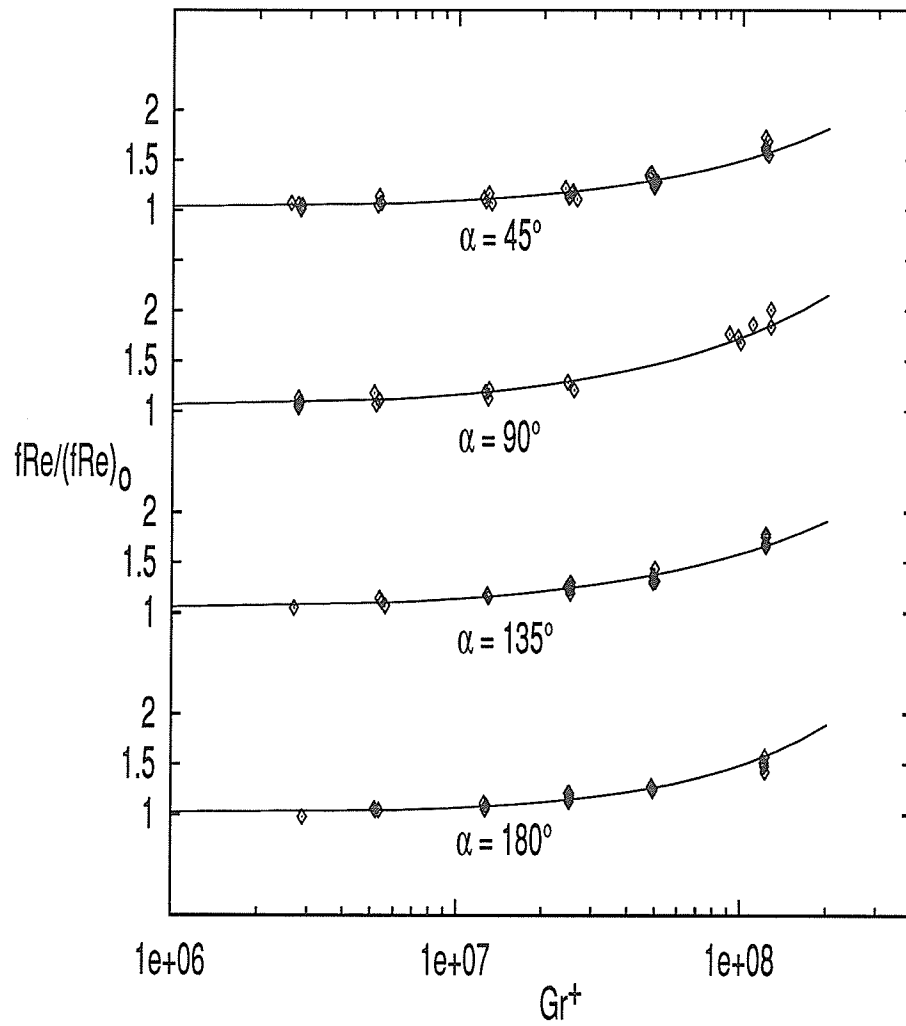


Fig. 4.14 The  $fRe/(fRe)_0$  ratios for  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ .

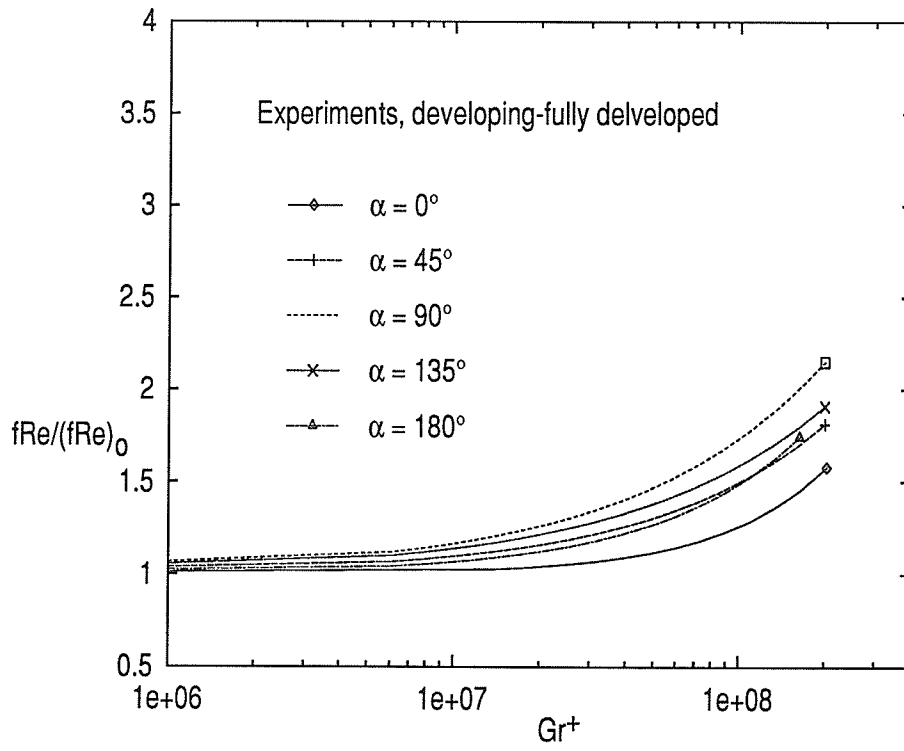


Fig. 4.15 Comparison of  $fRe/(fRe)_0$  for  $\alpha = 0^\circ$ [45],  $45^\circ$ ,  $90^\circ$ ,  $135^\circ$  and  $180^\circ$ .

## Chapter 5

# Numerical Results for the Semicircular Duct

The numerical results were obtained by solving the governing equations in Chapter 3 by the finite difference method with the aid of a perturbation approach. This chapter is started with the method of solution and accuracy of the solutions. For  $Gr^+ = 0$ , the macroscopic results from the grid refining exercises are observed by direct comparison to the exact solutions. For cases of  $Gr^+ > 0$ , the present predicted results were compared to the only\* existing published data which correspond to a case of  $\alpha = 180^\circ$ . The final solutions for  $Pr = 4$  were obtained only after good accuracy was established through mesh refining for  $\alpha = 0^\circ, 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ . The Nusselt number and friction factor ( $fRe$ ) results are presented and discussed according to the effect of  $\alpha$ . The solutions for additional Prandtl numbers of 10 and 50 are also presented

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\* The present predictions are based on a modified version of the numerical procedure developed by Lei [45]. Thus comparison to Lei's results for  $\alpha = 0^\circ$  is inappropriate.

to illustrate  $Pr$  effect. The merging of the results for various  $Pr$  leads to an introduction of a new key parameter  $Gr^+Pr$  which is a better replacement for  $Gr^+$  due to the existence of a unique relationship between  $Nu$  and  $Gr^+Pr$ . Again the present results are compared to the published data in term of  $Gr^+Pr$  with a good agreement and then the effect of Prandtl number is meaningfully demonstrated. Finally, the numerical results are compared to the experimental results from the previous chapter.

## 5.1 Solution Algorithm

There are five dimensionless governing partial differential equations comprising continuity, three momentum, and an energy equation. These correspond to five dependent variables: three velocity components, cross-stream pressure deviation, and temperature. In addition,  $fRe$  which is proportional to the axial pressure drop is virtually the sixth dependent variable of the system and it must satisfy the total mass conservation equation. The governing equations were discretized by using the control volume integration finite difference method. The coefficients related to the convective-diffusion terms in the discretized equations were formulated by employing the power-law scheme. A modified SIMPLER algorithm (the same version as used in [45]) was employed in solving these equations. The modification was made by solving the pressure and pressure correction fields over the whole domain simultaneously by a band storage linear system solver instead of the iteration technique as suggested in the SIMPLER algorithm.

The rest of the independent variables, which were velocity and temperature fields, were traditionally solved by the line iteration technique. It is interesting to remark that the way of handling the pressure and velocity coupling is very important and this has attracted much attention, e.g. see [100–106]. The simultaneous solution of the pressure field is basically a plane-by-plane technique. The accurate pressure and, as the consequent, velocity solutions obtained by such technique secure the convergence and improve the convergent rate. The convergence were judged by the absolute differences between the current and the previous iterative values of each primitive variables at each node being smaller than  $10^{-5}$ . The mass source terms which indicated the error of local continuity of each control volumes and the  $fRe$  correction which indicated overall axial mass balance were also required to be smaller than  $10^{-5}$ . The computer program was developed from the one used in [45]. The modification was that the restriction for the semicircular duct was released in order that the circular sector ducts with any apex angle less than  $180^\circ$  was able to be automatically accepted by the program. Another main change was the frequently used subroutines, e.g. in evaluating the coefficients of the variables, which were observed taking a large portion of the run time, were simply modified for speed. Reduction of about 30% of the computing time was observed. The staggered grid arrangement used by the computer program is shown in Fig. 5.1.

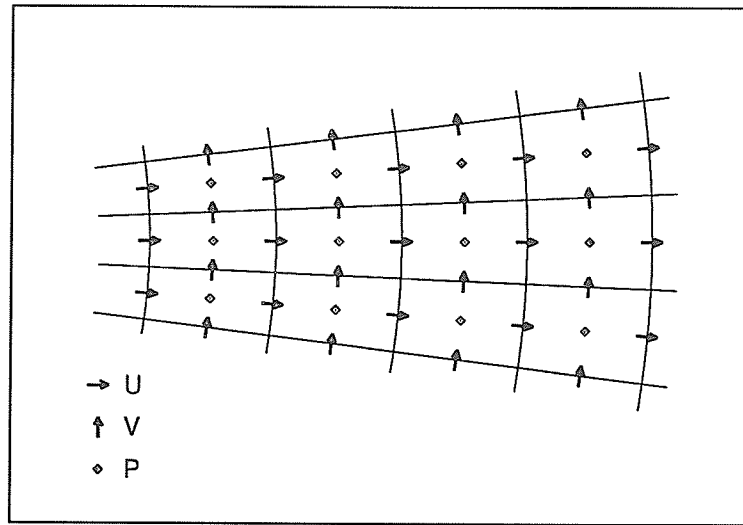


Fig. 5.1 The mesh and location of variables.

## 5.2 Solution Method

According to the mathematical model of this problem, the parameters that control the flow and heat transfer characteristics are  $Gr^+$ ,  $Pr$ ,  $\phi$  and  $\alpha$ . The  $\phi$  and  $\alpha$  are duct half apex angle and oriented angle, respectively. The  $Pr$  nominally represent fluid type although for a given fluid, its  $Pr$  value varies according to the fluid temperature. If the fluid properties are assumed constant then a fluid will have a single  $Pr$  value and the  $Gr^+$  is proportional to the applied heat flux. The representations of these parameters are

useful in the discussion that will appear later. The general idea in solving the problem is by specifying the four parameters to obtain the solutions consisting of velocity and temperature fields and  $fRe$ . Then the average Nusselt number is evaluated from the axial velocity and temperature distributions. Either a heating or cooling problem can be addressed by simply setting  $Gr^+$  to be positive or negative respectively.

For a certain problem, the values of  $Pr$ ,  $\phi$  and  $\alpha$  are fixed. In practice, the solutions for a given  $Gr^+$  can be obtained directly only if the  $Gr^+$  is small, e.g.  $5 \times 10^3$ . The solution takes more time and is prone to divergence as  $Gr^+$  increases. Increasing  $Pr$  also exhibits a similar effect but not as strongly as  $Gr^+$ . This seems to be a nature in solving the combined convection problem which in fact is a set of non-linear partial differential equations, and hence the perturbation approach must be employed. The idea of perturbation was suggested by Morton [5] who stated that one way to solve the combined convection problems ( $Gr^+ > 0$ ) is to perturb the forced convection ( $Gr^+ = 0$ ) solutions, i.e. using the forced convection velocity and temperature fields as the first approximation for the combined convection solution. The idea was probably originated for the perturbation method in Morton's analysis. However, this idea has appeared to work very well in many, if not all, finite difference analyses which reached high  $Gr^+$ . The techniques obviously stated in [21-22], and [28], whether inspired by [5] or not, appeared to be such perturbation manners, i.e. the solution for a  $Gr^+$  can be obtained by using (perturbing) the converged solutions of another, often slightly lower,  $Gr^+$  as the initial approximation. The perturbation approach is not limited to the perturbing of lower  $Gr^+$ , however, that is the usual way to start solving a problem.

### 5.3 Accuracy of the Solutions

The accuracy of the solutions for  $Gr^+ = 0$  was simply gauged by comparing to the exact solutions. This is in contrast to the case of  $Gr^+ > 0$  where there is no exact solution. The pure forced convection solution is independent of either the oriented angle or Prandtl number due to the absence of the buoyancy forces and hence the cross-stream flow. The problem can be solved over the entire domain of interest (called *full region solution*, for convenience), or within a half of the domain (called *half region solution*), by making use of the symmetry. The use of the half region solution offers some benefit other than saving the computational time for the case that  $Gr^+ > 0$  which is cited in subsection 5.3.2.

**Table 5.1** Comparisons of  $Nu$  and  $fRe$  for  $Gr^+ = 0$ , half region solutions, to exact solutions [35] and [36].

Mesh $rx\theta$		20x21	30x24	40x60
$Nu$ error%	4.0880[35]	4.03027 [1.41%]	4.04644 [1.02%]	4.07746 [0.26%]
$fRe$ error%	15.7668[36]	15.64701 [0.76%]	15.66643 [0.64%]	15.71600 [0.32%]

**Table 5.2** Comparisons of  $Nu$  and  $fRe$  for  $Gr^+ = 0$ , full region solutions, to exact solutions [35] and [36].

Mesh $rx\theta$		16x40	20x40	24x40	30x48
$Nu$ error%	4.0880[35]	4.06194 [0.64%]	4.07520 [0.31%]	4.08217 [0.14%]	4.08490 [0.08%]
$fRe$ error%	15.7668[36]	15.73902 [0.18%]	15.73931 [0.17%]	15.73942 [0.17%]	15.74803 [0.12%]

### 5.3.1 For $Gr^+ = 0$

The  $Nu$  and  $fRe$  results for  $Gr^+ = 0$ , solved with various mesh sizes, are compared to the exact results from Ref. [35, 36] in Table 5.1 and 5.2 for the half and full region solutions. The errors for the full region solutions are less than 0.7% for all meshes and about 0.1% for the 30x48 mesh. For the half region solution, the errors are about 0.3% for the 40x60 mesh and larger as the mesh becomes coarser. These meshes arose due to the investigation of the mesh size effect during the computation of the combined convection problem.

### 5.3.2 For $Gr^+ > 0$

Owing to the lack of exact solutions for the case of  $Gr^+ > 0$ , the accuracy of numerical results was observed by mesh refining. This process led to the selection of an adequate mesh for the main course of the computation. Initially, the results for  $Pr = 4$  were required for comparison to the corresponding experimental study that used water as the working fluid. In addition to the mesh refining, the results were compared to Ref. [22], the only available source of data. Their semicircular duct solutions were for  $Pr = 5$  and corresponded to  $\alpha = 180^\circ$  in the present study. Therefore the effect of mesh size on the Nusselt number was observed at such conditions to allow a direct comparison to [22], e.g. at the same  $Pr$  and  $Gr^+$ . It is worth noting here that in the study of Prandtl number effect in Section 5.8 the Nusselt number results for  $Pr = 4, 10$  and 50 are compared to [22] again.

To compare to [22], the problem with  $\alpha = 180^\circ$  was solved for  $Pr = 5$  and  $Gr^+ = 6.28 \times 10^5$ ,  $9.42 \times 10^5$  and  $1.26 \times 10^6$  which corresponded to the high end of the range in [22]. It was supposed to be the worst cases as the maximum errors between [22] and the present results were expected at such high  $Gr^+$  range. During the computation, it was found that the full region solution suffered the convergent difficulty and could not reach that  $Gr^+$  range thus the half region solution was used instead. Because the results in [22] were in graphical forms and were presented in terms of the ratios relative to the forced-convection solutions, the  $fRe$  result was skipped due to its narrow variation (about 1.0 to 1.3). The  $Nu$  results from the  $30 \times 24$ ,  $30 \times 48$  and  $40 \times 60$  meshes are compared to [22] in Table 5.3 and the agreements are within 3.4%, 1.9% and 2.3%, respectively for the two-cell pattern and within 6.3%, 4.9% and 4.0%, respectively for the four-cell pattern. The discrepancies for the two-cell solutions are quite satisfactory while those of the four-cell solution are larger but still acceptable. The comparison among the present results for the coarser meshes to the  $40 \times 60$  mesh are given in Table 5.4. The agreements of the  $30 \times 24$ ,  $40 \times 30$ , and  $30 \times 48$  to the  $40 \times 60$  meshes are within 1.9%, 0.6% and 0.61%, respectively for the two-cell pattern results and are within 2.24%, 0.88% and 1.1% respectively for the four-cell results. Since the maximum discrepancies between the coarsest and finest mesh is within about 2.5%, one can conclude that all results are essentially mesh independent, i.e. even the  $30 \times 24$  mesh is adequate.

**Table 5.3** Comparisons of  $Nu$  to [22],  $Pr = 5$ .

Mesh $r \times \theta$	$Gr^+$					
	two-cell			four-cell		
	$6.28 \times 10^5$	$9.42 \times 10^5$	$1.26 \times 10^6$	$6.28 \times 10^5$	$9.42 \times 10^5$	$1.26 \times 10^6$
Ref. [22]*	8.79	9.41	10.14	9.57	10.14	10.67
30x24	9.048 2.9%	9.725 3.4%	10.254 1.1%	10.059 5.1%	10.781 6.3%	11.336 6.2%
30x48	8.940 1.7%	9.588 1.9%	10.091 0.5%	9.939 3.9%	10.642 4.9%	11.181 4.8%
40x60	8.995 2.3%	9.541 1.4%	10.147 0.1%	9.869 3.11%	10.544 4.0%	11.059 3.6%

\* approximate values (as extracted from graphical results).

**Table 5.4** Comparisons of  $Nu$  to the 40x60,  $Pr = 5$ .

Mesh $r \times \theta$	$Gr^+$					
	two-cell			four-cell		
	$6.28 \times 10^5$	$9.42 \times 10^5$	$1.26 \times 10^6$	$6.28 \times 10^5$	$9.42 \times 10^5$	$1.26 \times 10^6$
30x24	9.048 0.59%	9.725 1.93%	10.254 1.05%	10.059 1.92%	10.781 2.24%	11.336 2.50%
40x30	8.941 0.60%	9.592 0.53%	10.099 0.48%	9.942 0.74%	10.631 0.82%	11.157 0.88%
30x48	8.940 0.61%	9.588 0.49%	10.091 0.55%	9.939 0.70%	10.642 0.92%	11.181 1.10%
40x60	8.995	9.541	10.147	9.869	10.544	11.059

The further investigation of the effect of mesh size indicates that if the 40x30 mesh was chosen for the half region solution then its equivalent mesh for the full region solution, which was the only choice for  $\alpha = 45^\circ, 90^\circ$  and  $135^\circ$ , should be a 40x60 mesh. Unfortunately, the 40x60 mesh for full region solution showed severe problem of convergence even at moderate  $Gr^+$  in addition to the drastic increase in computing time. As a result, the mesh 30x24 was chosen as a compromise between accuracy and cost of the computation.

While the half region solution was preferred for the symmetric cross-stream flow as it can reach higher  $Gr^+$ , the full region solution was necessary for the cases of unsymmetric cross-stream flow. Therefore, a combination of the half and full region solution was required to cover all cases of the orientations. To verify the compatibility of both solutions, the problems with  $\alpha = 0^\circ$  and  $180^\circ$  were solved by the half and full region solutions on the 30x24 and 30x48 meshes, respectively. Very good agreements of better than 1% for  $Nu$  and  $fRe$  were achieved for all ranges of the  $Gr^+$ . The results for are shown in Table 5.5.

Table 5.5  $Nu$  and  $fRe$  for the half and full region solutions,  $Pr = 4$ .

$Gr^+$	$\alpha = 0^\circ$						$\alpha = 180^\circ$					
	Half			Full			Half			Full		
	$fRe$	$Nu$		$fRe$	$Nu$		$fRe$	$Nu$		$fRe$	$Nu$	
0	15.666	4.046		15.748	4.085		15.666	4.046		15.748	4.085	
$5 \times 10^3$	15.693	4.261		15.781	4.292		15.709	4.295		15.780	4.306	
$1 \times 10^4$	15.745	4.540		15.835	4.562		15.768	4.589		15.834	4.590	
$5 \times 10^4$	16.080	5.621		16.173	5.635		16.106	5.732		16.160	5.724	
$1 \times 10^5$	16.370	6.270		16.463	6.283		16.373	6.396		16.423	6.386	
$2 \times 10^5$	16.787	7.048		16.881	7.058		16.736	7.160		16.781	7.149	
$5 \times 10^5$	17.588	8.298		17.680	8.306		17.399	8.340		-	-	
$1 \times 10^6$	18.421	9.444		18.510	9.450		18.070	9.402		-	-	
$5 \times 10^6$	21.287	12.983		21.359	12.979		-	-		-	-	
$1 \times 10^7$	22.998	15.016		23.084	15.024		-	-		-	-	

## 5.4 Results for $Pr = 4$

The problem was solved for  $Pr = 4.0$  for flow and temperature fields,  $fRe$  and  $Nu$  for the oriented angles  $\alpha$  of  $0^\circ$ ,  $45^\circ$ ,  $90^\circ$ ,  $135^\circ$  and  $180^\circ$  which represents a complete round of duct cross-sectional rotation with  $45^\circ$  increment. All the results were obtained using the perturbation approach by keeping  $Pr$  and  $\alpha$  constant in each series of solution. It is interesting to note that in the perturbation approach the  $Gr^+$  marching is somewhat similar to the temporal marching in transient problems or the spacial marching in entrance region in duct flow problems except that the marching in the later cases is advanced by the time or length according to their derivatives. The perturbation approach yielded consistent solutions, e.g. the same results for a given  $Gr^+$  were always obtained (if the convergences were possible) by perturbing the converged solutions of the different  $Gr^+$ , irregardless as to whether it was higher or lower than the given  $Gr^+$ . During the course of computations, it appeared that to obtain the convergences of the velocities and temperatures by the absolute difference of  $10^{-5}$ , the residuals of the local and total mass balances usually dropped to less than  $10^{-10}$ . This indicated the accuracy of the mass conservation, i.e. the continuity equation was well satisfied both locally and globally. Other than the more common two-cell flow patterns, four-cell patterns also emerged as another solution in some orientations. The results for these flow patterns are presented in the following subsections.

### 5.4.1 Two-cell Pattern Results

The two-cell solutions were the only common solutions among all of the oriented angles and these are summarized in Table 5.6. The effect of oriented angle on  $Nu$  and  $fRe$  is demonstrated by plotting  $Nu$  and  $fRe$  against  $Gr^+$  for each  $\alpha$  in Figs. 5.2(a) and (b) where these magnitudes are normalized by the forced convection solutions;  $Nu_o = 4.0880$  and  $(fRe)_o = 15.7668$ . The magnitudes of  $Nu$  and  $fRe$  for  $\alpha = 90^\circ$  and  $\alpha = 135^\circ$  are very much the same, except that  $fRe$  for  $\alpha = 135^\circ$  is a little lower at  $Gr^+ < 10^6$  and a little higher at  $Gr^+ > 10^6$ , and they are obviously the highest of the group. Similarly, the magnitudes of  $Nu$  and  $fRe$  for  $\alpha = 0^\circ$  and  $\alpha = 180^\circ$  are almost the same except that  $fRe$  for  $\alpha = 180^\circ$  is a little lower near  $Gr^+ = 10^6$ . For  $\alpha = 45^\circ$ ,  $Nu$  and  $fRe$  curves are in between the highest and lowest curves, however the  $fRe$  curve is rather close to the  $0^\circ$  curve. The Nusselt numbers of  $90^\circ$  and  $135^\circ$  are about 20% higher than that for  $\alpha = 0^\circ$  at  $Gr^+$  near  $10^7$  while the differences of their  $fRe$  are less than 10%. It is worth emphasizing that the cases of  $\alpha = 0^\circ$  and  $180^\circ$  which correspond to symmetric cross-stream flows provide the lowest  $Nu$  and  $fRe$  of this group.

It is clear in Figs. 5.2 that, for a given  $\alpha$ ,  $Nu$  and  $fRe$  increase monotonically with increasing  $Gr^+$ . In addition, the data on both  $Nu-Gr^+$  and  $fRe-Gr^+$  domains form well defined curves. Compared to the pure forced convection data, i.e.  $Nu_o = 4.0880$  and  $(fRe)_o = 15.7668$ , at  $Gr^+ = 2 \times 10^7$ , the  $Nu/Nu_o$  ratios are over 4 for  $\alpha = 0^\circ$  and over 5 for  $\alpha = 90^\circ$  or  $135^\circ$  while  $fRe/(fRe)_o$  ratios are only 1.6 for  $\alpha = 0^\circ$  and about 1.7 for  $\alpha = 90^\circ$  or  $135^\circ$ . This indicates that the effect of buoyancy force on Nusselt number is much more significant.

Table 5.6  $Nu$  and  $fRe$  for all orientations,  $Pr = 4$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 45^\circ$		$\alpha = 90^\circ$		$\alpha = 115^\circ$		$\alpha = 135^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	15.666	4.046	15.748	4.085	15.748	4.087	15.748	4.085	15.748	4.085	15.666	4.046
$5 \times 10^3$	15.693	4.261	15.779	4.286	15.774	4.261	15.773	4.259	15.775	4.274	15.709	4.295
$1 \times 10^4$	15.745	4.540	15.836	4.577	15.828	4.553	15.822	4.530	15.824	4.541	15.767	4.589
$5 \times 10^4$	16.080	5.621	16.238	5.809	16.284	5.947	16.246	5.888	16.196	5.794	16.106	5.732
$1 \times 10^5$	16.370	6.270	16.590	6.554	16.708	6.791	16.664	6.747	16.570	6.625	16.373	6.396
$5 \times 10^5$	17.588	8.298	17.975	8.855	18.362	9.375	18.363	9.386	18.193	9.225	17.399	8.340
$1 \times 10^6$	18.421	9.444	18.848	10.158	19.394	10.846	19.450	10.893	19.266	10.708	18.070	9.402
$5 \times 10^6$	21.287	12.983	21.735	14.281	22.758	15.411	23.034	15.585	22.835	15.338	-	-
$1 \times 10^7$	22.998	15.016	23.464	16.675	24.709	17.984	25.098	18.215	24.895	17.949	-	-
$2 \times 10^7$	25.036	17.454	25.548	19.525	27.005	20.990	27.506	21.285	27.298	21.008	-	-
$5 \times 10^7$	-	-	-	-	30.603	25.696	31.272	26.102	31.087	25.850	-	-

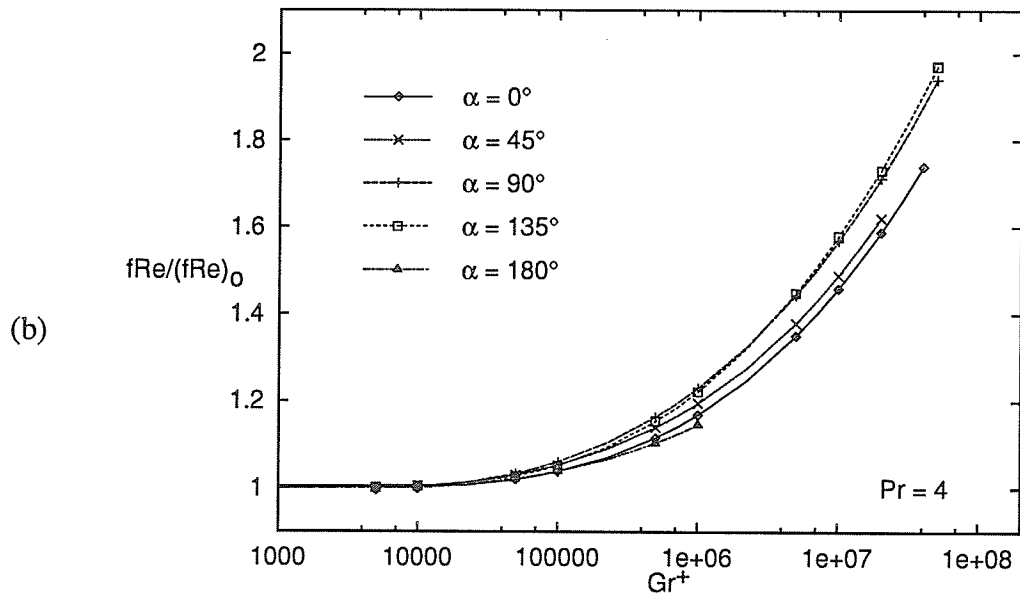
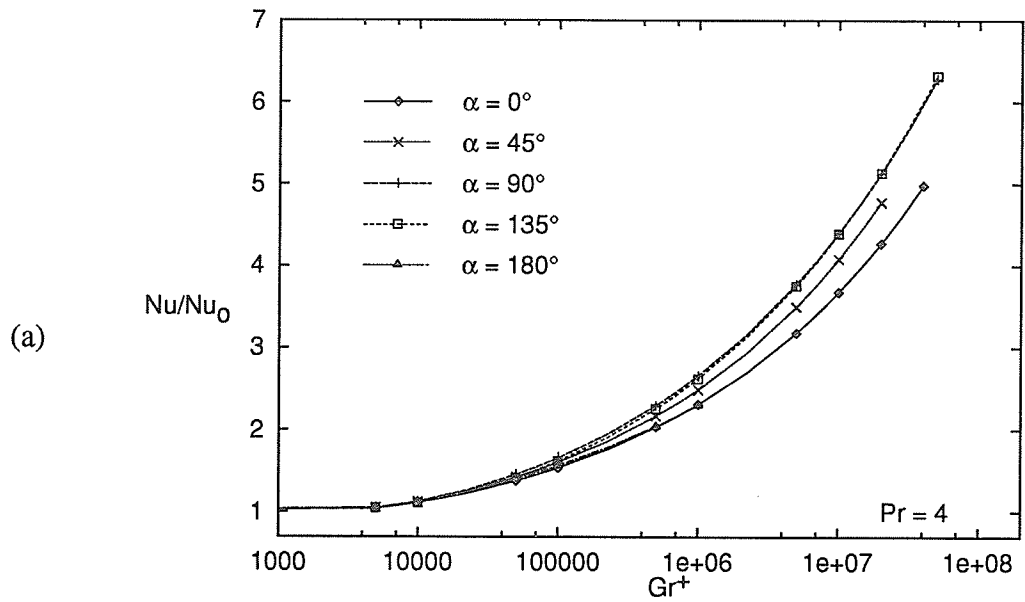


Fig. 5.2 (a)  $Nu/Nu_0$  (b)  $fRe/(fRe)_0$  for  $Pr = 4$ .

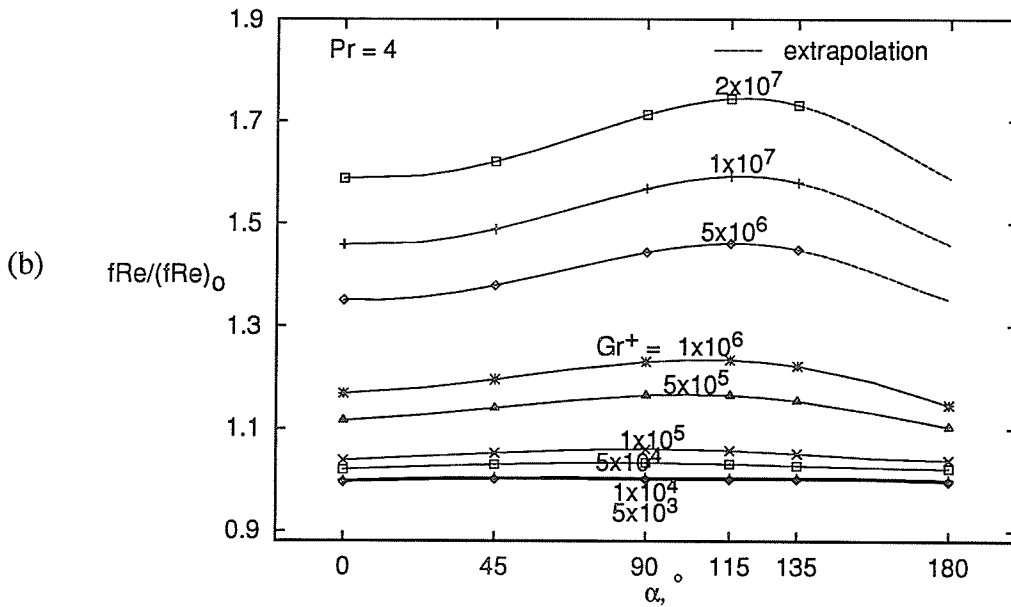
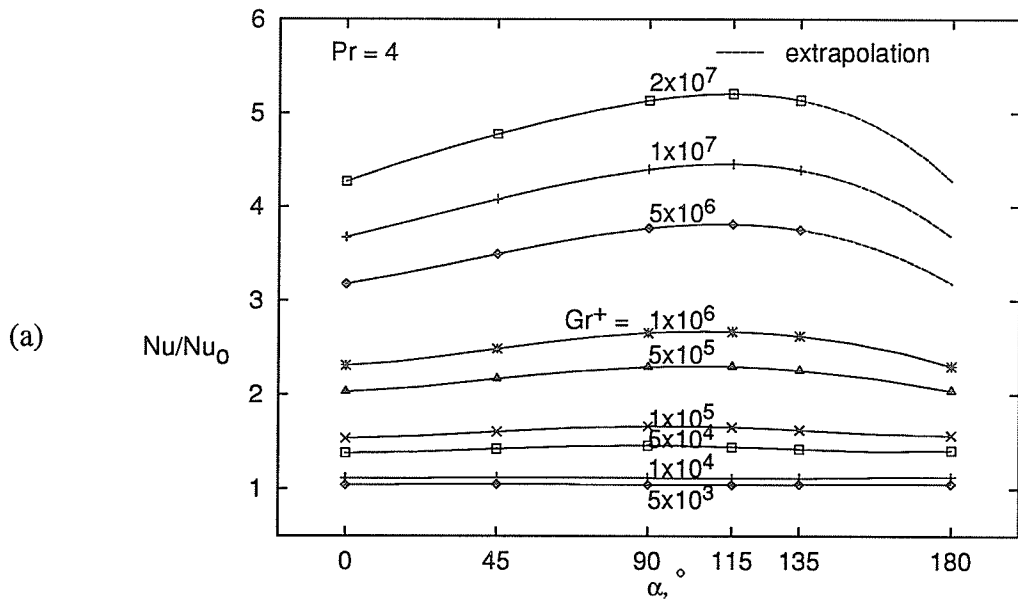


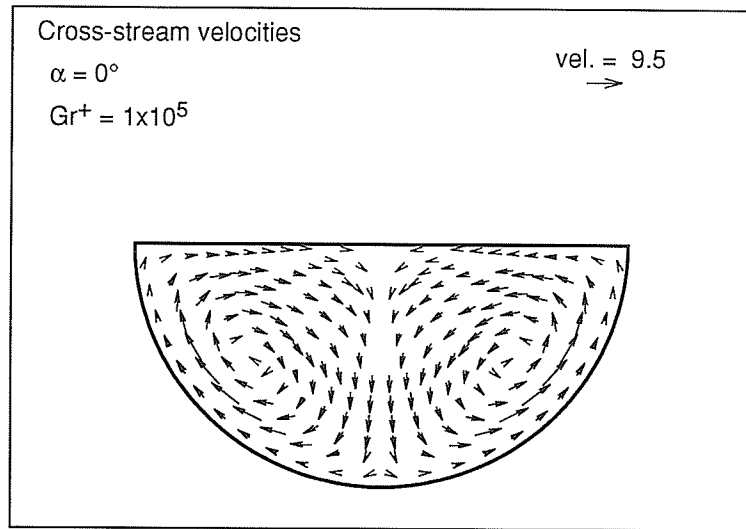
Fig. 5.3 (a)  $Nu/Nu_0$  (b)  $fRe/(fRe)_0$  against  $\alpha$ ,  $Pr = 4$ .

The oriented angle was scanned, to search for maximum Nusselt numbers, using a 20x40 mesh with 5° increments from  $\alpha = 75^\circ$  to  $120^\circ$ . It was found that the values of Nusselt numbers were nearly insensitive to  $\alpha$  where  $\alpha$  was between  $90^\circ$  and  $120^\circ$ . However, the maximum  $Nu$  at high  $Gr^+$  was found at  $\alpha = 115^\circ$ . The Nusselt number and  $fRe$  results were generated again with the 30x48 mesh and are added to Table 5.6. In Figs. 5.3,  $Nu/Nu_o$  and  $fRe/(fRe)_o$  are plotted against  $\alpha$  at constant  $Gr^+$ . The maximum  $Nu$  are at  $\alpha \approx 45^\circ-90^\circ$  for  $Gr^+ = 10^4$ , at  $\alpha \approx 90^\circ$  for  $Gr^+ = 10^5$  and at  $\alpha \approx 115^\circ$  for  $Gr^+ \geq 10^6$ . The similar pattern can be observed in Fig. 5.3(b) for  $fRe$ . For  $Gr^+ = 5 \times 10^6$  to  $2 \times 10^7$ , the curves in Figs. 5.3(a) and (b) were extrapolated from  $\alpha = 135^\circ$  to  $180^\circ$  by presuming the same values at  $\alpha = 0^\circ$  and  $180^\circ$ . Even though it is clear that the maximum moves to a higher  $\alpha$  as  $Gr^+$  increases, the differences are very small for  $\alpha$  between  $90^\circ$  to  $135^\circ$ .

The effect of buoyancy force on Nusselt number can be investigated from the patterns of cross-stream flows. Generally, a larger Nusselt number results from the wall-to-bulk temperature difference dropping due to the cooler part of fluid being pushed to absorb heat energy from the duct wall by the cross-stream current. The resultant cross-stream velocities are computed in term of  $\sqrt{(u^+)^2 + (v^+)^2}$  and are plotted in Figs. 5.4(a)–(e) for  $\alpha = 0^\circ, 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ , respectively for a fixed  $Gr^+$  of  $10^5$ . For  $\alpha = 45^\circ, 90^\circ, 135^\circ$ , the flows are non-symmetric and each cell is different in size. The flow dominated by one major cell provides a rather thorough mixing of hot and cool parts of the fluid therefore the higher Nusselt number is achieved. The quality of this

mixing can be noticed by the reduction of the area enclosed by a low temperature contour, e.g. contour line number 9 (without label and next to number 8) in Fig. 5.5. For  $\alpha = 90^\circ$  and  $135^\circ$  in Figs. 5.5(c) and (d), it is obvious that the areas are approximately the same and this perhaps explains why their Nusselt numbers are comparatively equal. For  $\alpha = 0^\circ$  and  $180^\circ$  in Figs. 5.5(a) and (e), the areas are similar in shape and they are the largest and this corresponds to the lowest Nusselt numbers in this group. Differences in Nusselt number appear to be linked to the shape and orientation of the semicircular cross-section. For  $\alpha = 90^\circ$  or  $135^\circ$ , the sharp corner of the cross section is turned to the bottom so that the cooler fluid can not spread widely as in the case of  $\alpha = 0^\circ$  or  $180^\circ$ , as a result, the cooler fluid in such a confined space is forced to blend with the, warmer, downward return stream. This results in a smaller size of the cool core fluid.

(a)



(b)

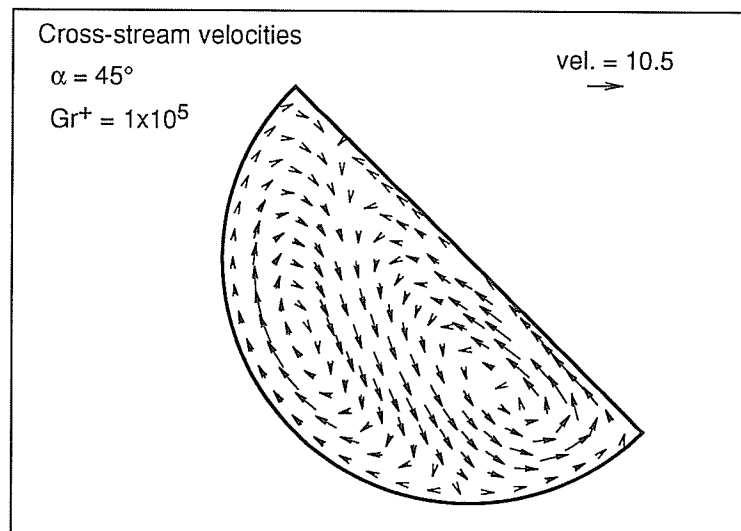
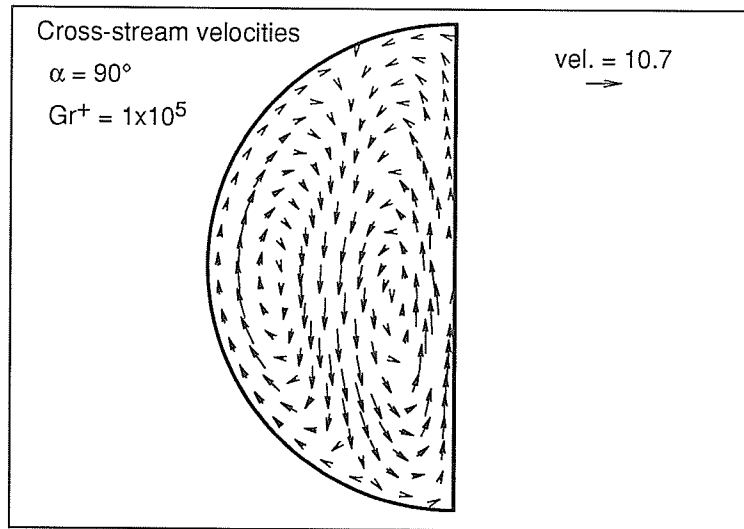


Fig. 5.4 Cross-stream flow pattern, two-cell,  $\alpha =$  (a)  $0^\circ$ , (b)  $45^\circ$ .

(c)



(d)

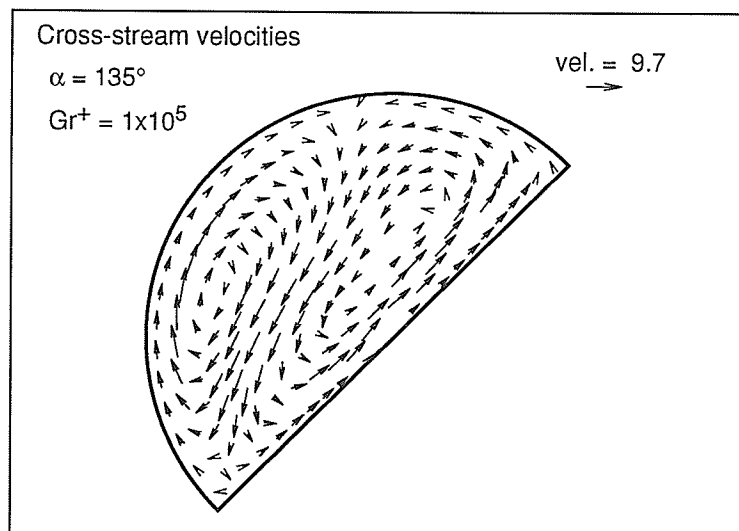


Fig. 5.4(cont) Cross-stream flow pattern, two-cell,  $\alpha =$  (c)  $90^\circ$ , (d)  $135^\circ$ .

(e)

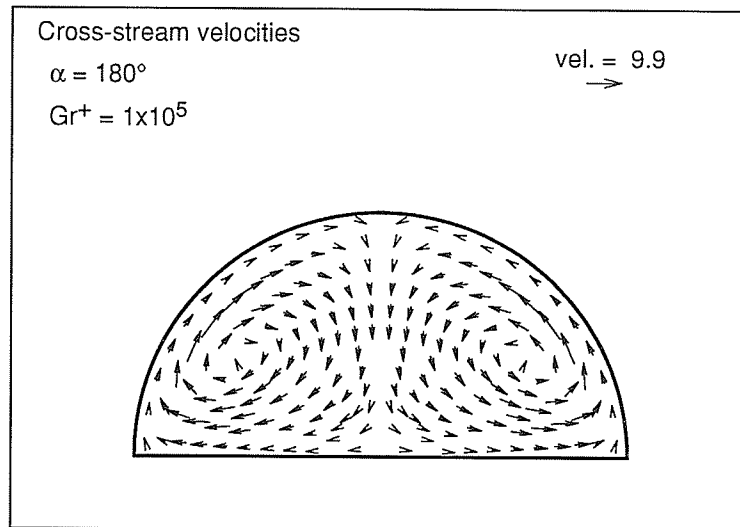


Fig. 5.4(cont) Cross-stream flow pattern, two-cell,  $\alpha =$  (e)  $180^\circ$ .

(a)

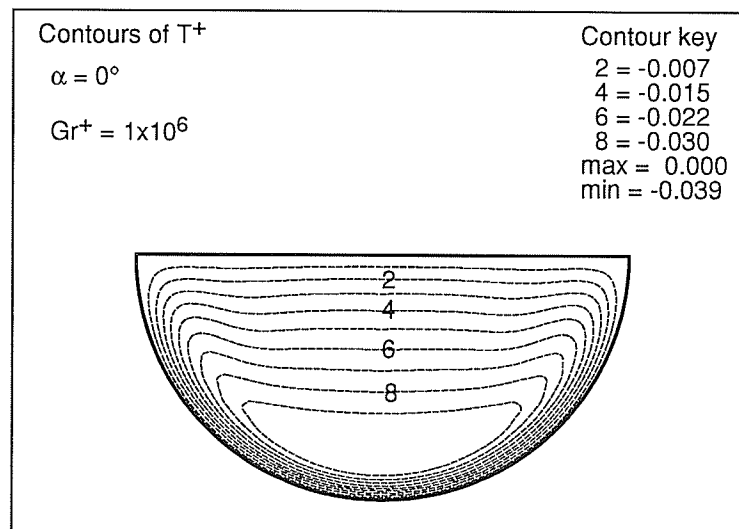
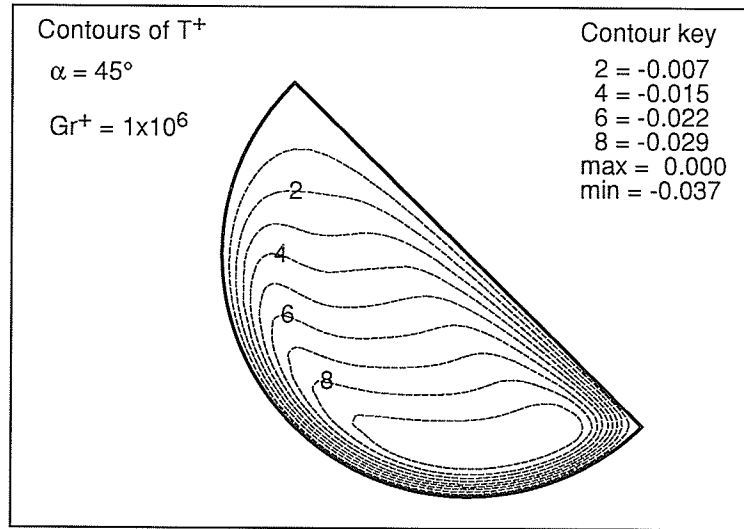


Fig. 5.5 Contour of  $T^+$ , two-cell,  $\alpha =$  (a)  $0^\circ$ .

(b)



(c)

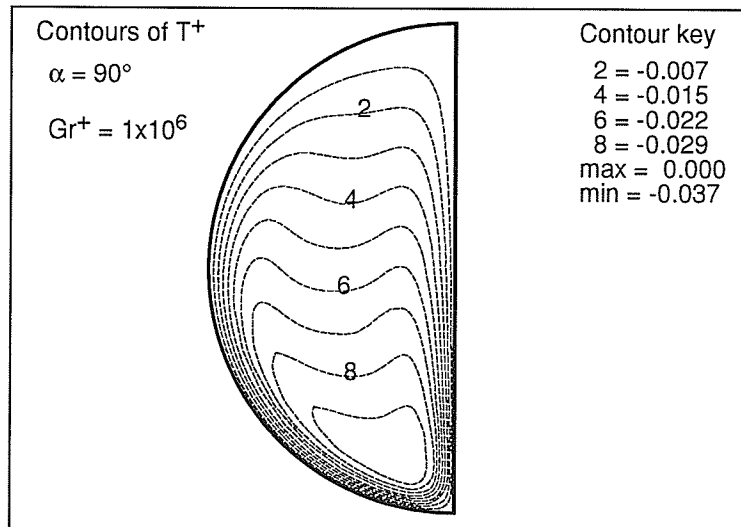
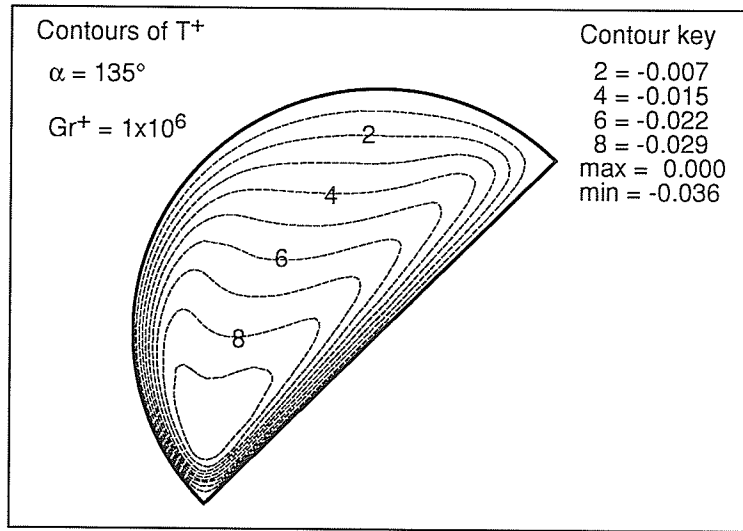


Fig. 5.5(cont) Contour of  $T^+$ , two-cell,  $\alpha =$  (b)  $45^\circ$ , (c)  $90^\circ$ .

(d)



(e)

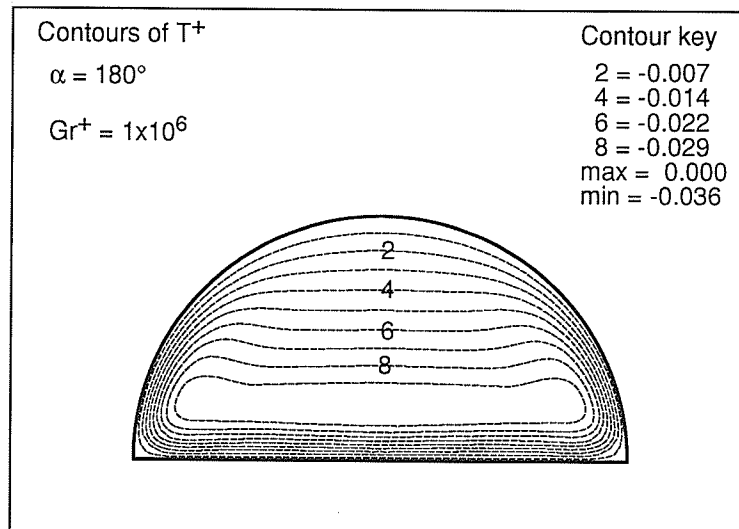


Fig. 5.5(cont) Contour of  $T^+$ , two-cell,  $\alpha =$  (e)  $180^\circ$ .

Table 5.7  $Nu$  and  $fRe$  of two- and four-cell patterns for  $Pr = 4$ .

$Gr^+$	$\alpha = 0^\circ$						$\alpha = 180^\circ$					
	two-cell		four-cell		two-cell		four-cell		two-cell		four-cell	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	15.666	4.04644	-	-	-	-	15.666	4.046	-	-	-	-
$5 \times 10^3$	15.693	4.26122	-	-	-	-	15.709	4.295	-	-	-	-
$1 \times 10^4$	15.745	4.540	-	-	-	-	15.768	4.589	-	-	-	-
$5 \times 10^4$	16.080	5.621	-	-	-	-	16.106	5.732	-	-	-	-
$1 \times 10^5$	16.370	6.270	-	-	-	-	16.373	6.396	-	-	-	-
$2 \times 10^5$	16.787	7.048	17.077	7.467	16.736	7.160	17.028	7.805	17.399	8.340	18.215	9.297
$5 \times 10^5$	17.588	8.298	18.328	8.885	18.070	9.402	19.227	10.450	19.462	10.059	10.59	13.556
$1 \times 10^6$	18.421	9.444	19.462	10.059	22.815	13.556	-	-	22.815	13.556	-	-
$5 \times 10^6$	21.287	12.983	22.815	13.556	24.679	15.576	-	-	24.679	15.576	-	-
$1 \times 10^7$	22.998	15.016	24.679	15.576	26.881	18.028	-	-	26.881	18.028	-	-
$2 \times 10^7$	25.036	17.454	26.881	18.028	28.339	19.679	-	-	28.339	19.679	-	-
$3 \times 10^7$	26.391	19.098	28.339	19.679	29.982	21.288	-	-	29.982	21.288	-	-
$4 \times 10^7$	27.431	20.372	29.982	21.288	31.017	22.377	-	-	31.017	22.377	-	-
$5 \times 10^7$	-	-	31.017	22.377	-	-	-	-	-	-	-	-

## 5.4.2 Four-cell Pattern Results

The nature of the perturbation technique employed in solving this problem is one whereby new solutions are obtained by gradually increased  $Gr^+$ . In the cases of non-symmetric cross-stream flow, the solution successfully continued to  $Gr^+$  as high as  $2 \times 10^7$  without extra difficulty. For symmetric cases,  $\alpha = 0^\circ$  and  $180^\circ$ , the full region solution was attempted in the first place but difficulty in getting converged solutions was encountered for  $Gr^+ > 2 \times 10^5$  for  $\alpha = 180^\circ$  and for  $Gr^+ > 10^7$  for  $\alpha = 0^\circ$  hence half region solutions were used to enable proceeding to higher  $Gr^+$ . When  $Gr^+$  reached  $5 \times 10^7$ , for the case of  $\alpha = 0^\circ$ , it appeared that the solution was only a four-cell pattern instead of the usual two-cell pattern. This four-cell pattern was the only solution for  $Gr^+ \geq 5 \times 10^7$ . Conversely, when  $Gr^+$  was gradually decreased, the solutions were still four-cell pattern until  $Gr^+$  dropped to  $10^5$  where the four-cell pattern was no longer a solution. Therefore, there were two possible solutions for  $2 \times 10^5 \leq Gr^+ \leq 4 \times 10^7$ . The  $Nu$  and  $fRe$  results are shown in Table 5.7 and Fig. 5.6. The bifurcation phenomena were tested by feeding the four-cell results to a  $Gr^+$  less than  $2 \times 10^5$ . It appeared that the converged results were always two-cell patterns which were the same ones obtained by gradually raising  $Gr^+$  from zero in the first place. The four-cell solutions were also fed to the unsymmetric orientations at  $Gr^+ > 2 \times 10^5$ . The outcome was similar to the previous test due to four-cell result was not established. The coexistence of the two- and four-cell solutions, called dual solution, were found before in rectangular, circular and semicircular ducts, e.g. [21-22], [34], and [46], and extensive discussions are available in [22] and [34].

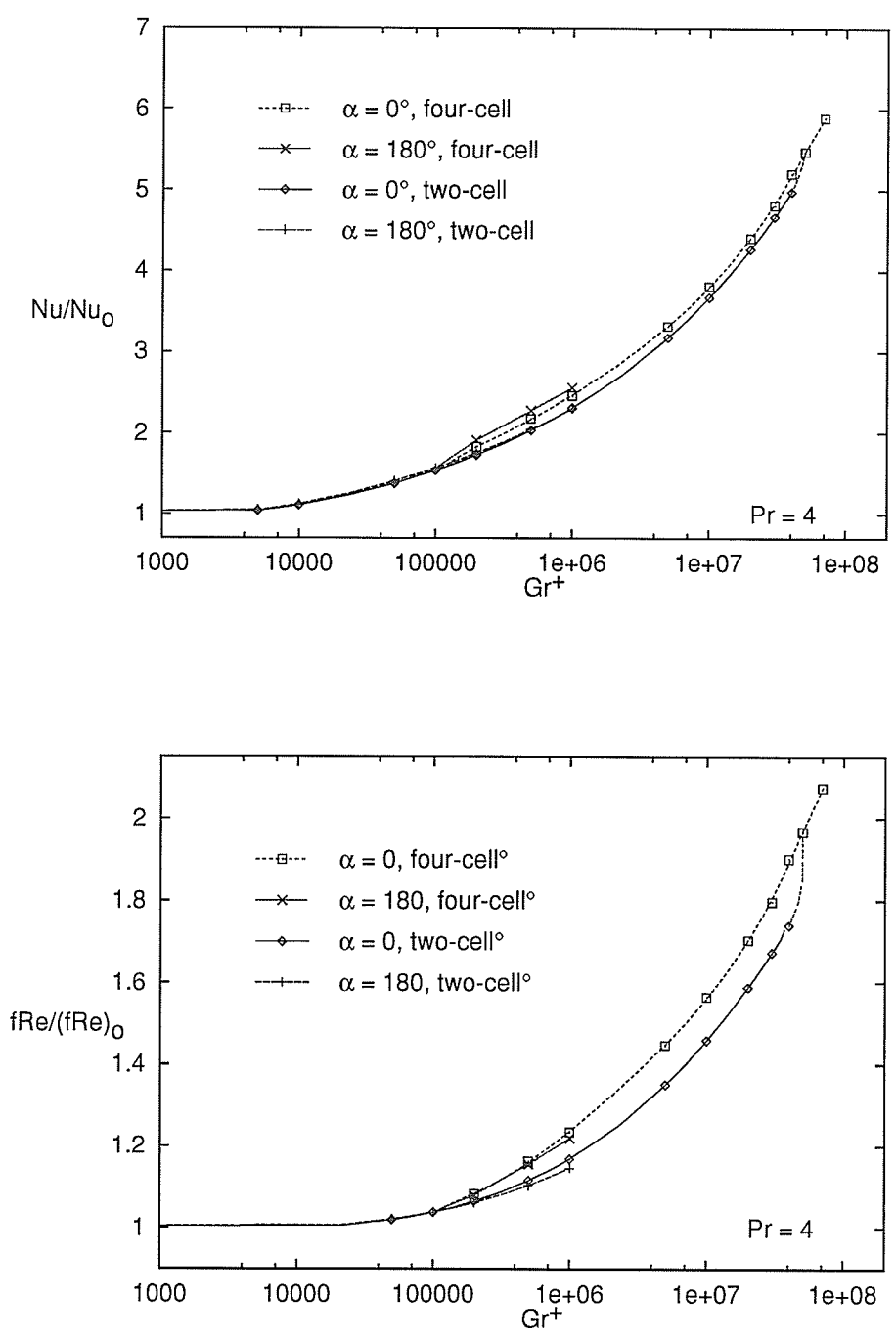


Fig. 5.6 The four- and two-cell results for  $\alpha = 0^\circ$  and  $180^\circ$ .

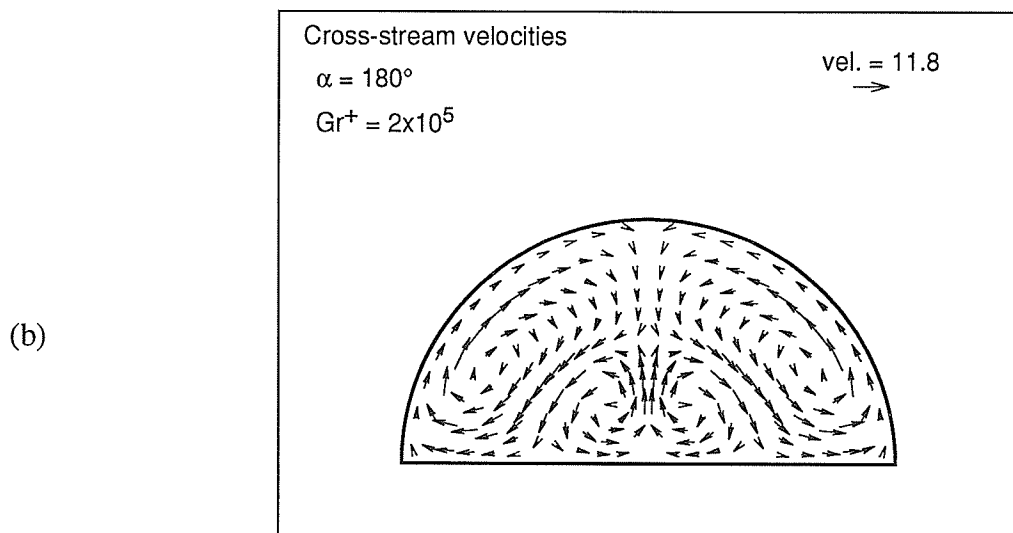
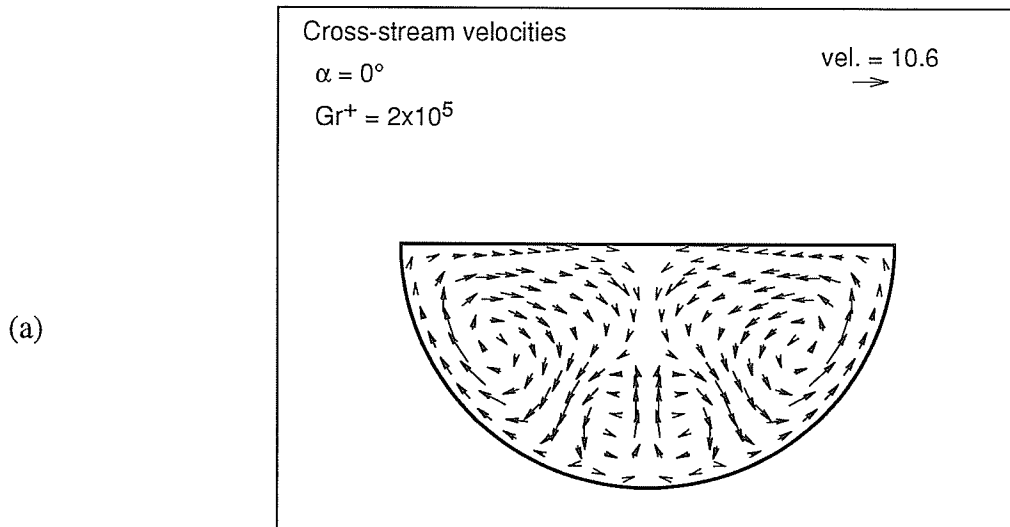
For  $\alpha = 180^\circ$ , the four-cell solution could not be obtained by the previously described way so that the initial four-cell pattern was prepared from the case of  $\alpha = 0^\circ$ . After the first four-cell solution was established with  $\alpha = 180^\circ$ ,  $Gr^+$  was varied to get other four-cell solutions. Such transfer process was repeated by taking different solutions from the case  $\alpha = 0^\circ$  but the same outcome was observed. The dual solution occurred at the same critical  $Gr^+$  of  $2 \times 10^5$  but the second critical Grashof number could not be reached due to convergence difficulties at high Grashof number.

Fig. 5.6 demonstrates that Nusselt number and  $fRe$  of four-cell pattern were always higher than those of the two-cell patterns. The differences are about 9 to 11% for the case of  $\alpha = 180^\circ$  and about 3 to 7% for  $\alpha = 0^\circ$ . The four-cell flow patterns for  $\alpha = 0^\circ$  and  $180^\circ$  are shown in Figs. 5.7(a) and (b). In both cases, there is an extra pair of cells near the bottom wall which provides additional circulation to the cross-stream flow so that some improvement in Nusselt number was obtained relative to the two-cell pattern.

The formation of the four-cell pattern can be explained as a bifurcation from the two-cell pattern. The two-cell pattern is the original pattern which emerges when the cross-stream flow is slight. This pattern is built up by the rising of hot fluid along both side walls. These two streams turn and run toward each other due to the constraint of the top wall. The downward flow is formed after the collision and combining of these two streams. As a result, two symmetric cross-stream cells are formed. If the heat flux is high enough to encourage the rising of the hot fluid near the middle of bottom wall to encounter the combined downward stream, another collision will occur near the centre

of the duct cross-section. This causes a pair of minor cells to be created near the bottom wall. This happens only at a sufficiently high Grashof number such that the upward rising current from the bottom wall is strong enough to maintain the circulation of these minor cells. In Fig. 5.7, each pair of minor cells can be viewed near the bottom walls.

It is reasonable to believe that the four-cell pattern is maintained by the symmetry of cross-stream flow. In this study, no four-cell patterns were found for orientation of  $\alpha = 45^\circ$ ,  $90^\circ$  and  $135^\circ$  for which the cross-stream flow is not symmetric. In these cases, the direction of the main downward return stream is not normal to the bottom wall. In addition, the sizes of each cells are different. The downward return current is dominated by the larger cell and the flow approaches the bottom wall with an angle other than a right angle. This suggests that any upward rising current from the bottom wall can be easily swept away by the main downward stream which attacks it at an angle. It is for this reason that there are no four-cell patterns for  $\alpha = 45^\circ$ ,  $90^\circ$  and  $135^\circ$ . The symmetry of the cross-stream flow is necessary for bifurcation. This was also observed in [22] and [34] but it was referred to as "a flat wall geometry facing the body force" which corresponds to the case that  $\alpha = 0^\circ$  or  $180^\circ$  in the present work.



**Fig. 5.7** The four-cell pattern for  $\alpha =$  (a)  $0^\circ$  and (b)  $180^\circ$ ,  $Gr^+ = 2 \times 10^5$ .

## 5.5 Results for $Pr = 10$ and $50$

The results for various Prandtl numbers were useful for exploring the influence of Prandtl number on the flow and heat transfer characteristics. The  $Nu$  and  $fRe$  results for  $Pr = 10$  and  $50$  are presented in this section where the influence of Prandtl number is illustrated in addition to the effect of  $\alpha$ . The further study of the effect of Prandtl number arising from the merging of the data for  $Pr = 4, 10$  and  $50$  is presented in Section 5.6 and 5.7.

The problem was solved similar to the previous section but for  $Pr = 10$  and  $50$  and the numerical results are in Table 5.8 and 5.9. The  $Nu$  and  $fRe$  for  $\alpha = 0^\circ, 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$  are plotted against  $Gr^+$  in Fig. 5.8 for  $Pr = 10$  and Fig. 5.9 for  $Pr = 50$ . Similar to the case of  $Pr = 4$ ,  $Nu$  and  $fRe$  for  $Pr = 10$  and  $50$  are maximum when  $\alpha = 90^\circ$  or  $135^\circ$  and minimum when  $\alpha = 0^\circ$  or  $180^\circ$ . The magnitudes for each pair of  $\alpha$  are very close for all range of  $Gr^+$ . The case of  $\alpha = 45^\circ$  again shows the magnitudes in the middle of the range between maximum and minimum. The differences of  $Nu$  for  $\alpha$  of  $0^\circ$  and  $90^\circ$  are about 21% for  $Pr = 10$  at  $Gr^+ = 1 \times 10^7$  and 21.5% for  $Pr = 50$  at  $Gr^+ = 2 \times 10^6$ . On the  $fRe$  side, the differences at the same  $Gr^+$  conditions are about 7% for  $Pr = 10$  and 1.8% for  $Pr = 50$ .

Table 5.8  $Nu$  and  $fRe$  for all orientations,  $Pr = 10$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 45^\circ$		$\alpha = 90^\circ$		$\alpha = 115^\circ$		$\alpha = 135^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	15.666	4.046	15.748	4.085	15.748	4.085	15.748	4.085	15.748	4.085	15.666	4.046
$1 \times 10^3$	15.669	4.116	15.750	4.147	15.749	4.136	15.749	4.138	15.749	4.145	15.670	4.133
$5 \times 10^3$	15.680	4.659	15.768	4.705	15.767	4.681	15.765	4.648	15.765	4.655	15.691	4.706
$1 \times 10^4$	15.699	5.094	15.794	5.198	15.795	5.233	15.790	5.170	15.787	5.133	15.714	5.165
$5 \times 10^4$	15.815	6.526	15.953	6.852	15.992	7.103	15.979	7.048	15.949	6.909	15.833	6.658
$1 \times 10^5$	15.919	7.354	16.092	7.798	16.171	8.151	16.158	8.117	16.109	7.958	15.929	7.487
$5 \times 10^5$	16.387	9.894	16.670	10.725	16.903	11.405	16.926	11.440	16.833	11.228	16.333	9.963
$1 \times 10^6$	16.733	11.235	17.068	12.417	17.400	13.268	17.461	13.351	17.351	13.112	16.618	11.360
$2 \times 10^6$	17.198	13.023	17.587	14.446	18.040	15.481	18.152	15.614	18.026	15.354	-	-
$5 \times 10^6$	18.036	15.794	18.514	17.761	19.141	19.030	19.332	19.223	19.189	18.948	-	-
$1 \times 10^7$	18.867	18.391	19.437	20.824	20.191	22.256	20.444	22.497	20.289	22.223	-	-
$2 \times 10^7$	19.892	21.504	20.588	24.398	21.442	25.994	21.766	26.302	21.610	26.050	-	-

Table 5.9 Nusselt numbers and  $fRe$  for all orientations,  $Pr = 50$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 45^\circ$		$\alpha = 90^\circ$		$\alpha = 115^\circ$		$\alpha = 135^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	15.666	4.046	15.748	4.085	15.758	4.085	15.748	4.085	15.748	4.085	15.666	4.046
$1 \times 10^3$	15.666	4.666	15.749	4.709	15.749	4.680	15.749	4.646	15.749	4.657	15.668	4.706
$2 \times 10^3$	15.666	5.106	15.750	5.208	15.750	5.237	15.750	5.170	15.750	5.137	15.669	5.110
$5 \times 10^3$	15.667	5.853	15.753	6.077	15.754	6.225	15.753	6.146	15.752	6.037	15.672	5.958
$1 \times 10^4$	15.669	6.556	15.757	6.891	15.759	7.136	15.758	7.068	15.757	6.919	15.676	6.687
$5 \times 10^4$	15.682	8.740	15.779	9.410	15.789	9.912	15.788	9.893	15.783	9.685	15.690	8.883
$1 \times 10^5$	15.693	9.970	15.797	10.856	15.814	11.502	15.815	11.513	15.807	11.278	15.701	10.114
$5 \times 10^5$	15.745	13.769	15.877	15.442	15.926	16.466	15.937	16.577	15.919	16.293	15.752	13.975
$1 \times 10^6$	15.785	15.963	15.940	18.096	16.013	19.287	16.032	19.428	16.008	19.143	15.790	16.199
$2 \times 10^6$	15.842	18.610	16.030	21.250	16.134	22.608	16.165	22.771	16.134	22.497	-	-
$5 \times 10^6$	-	-	-	-	16.365	27.856	16.418	28.047	16.378	27.806	-	-
$7 \times 10^6$	-	-	-	-	16.475	30.031	16.539	30.235	16.497	30.001	-	-
$1 \times 10^7$	-	-	-	-	16.607	32.469	16.688	32.689	16.644	32.451	-	-

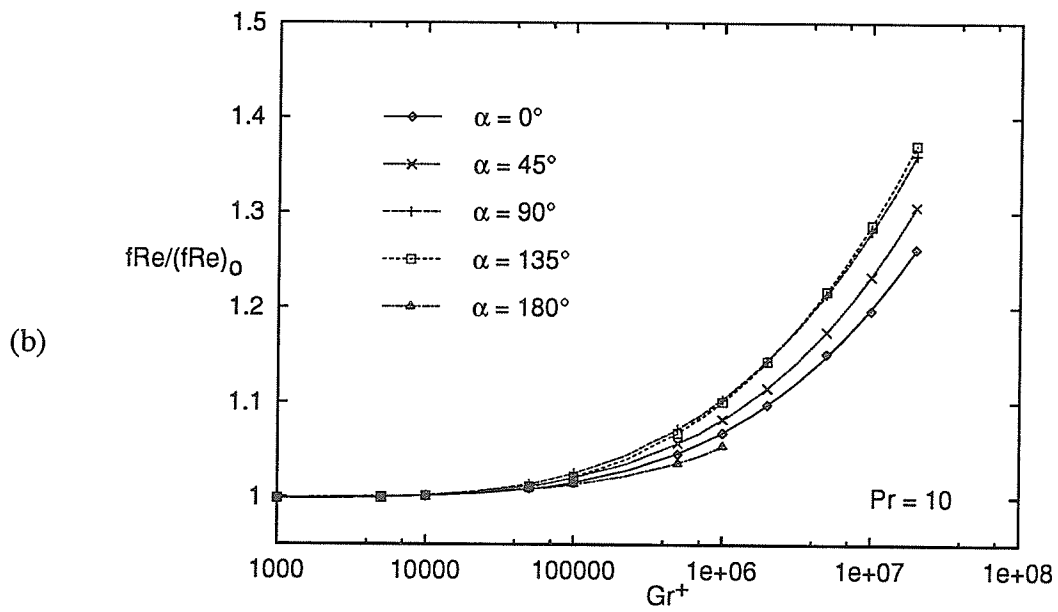
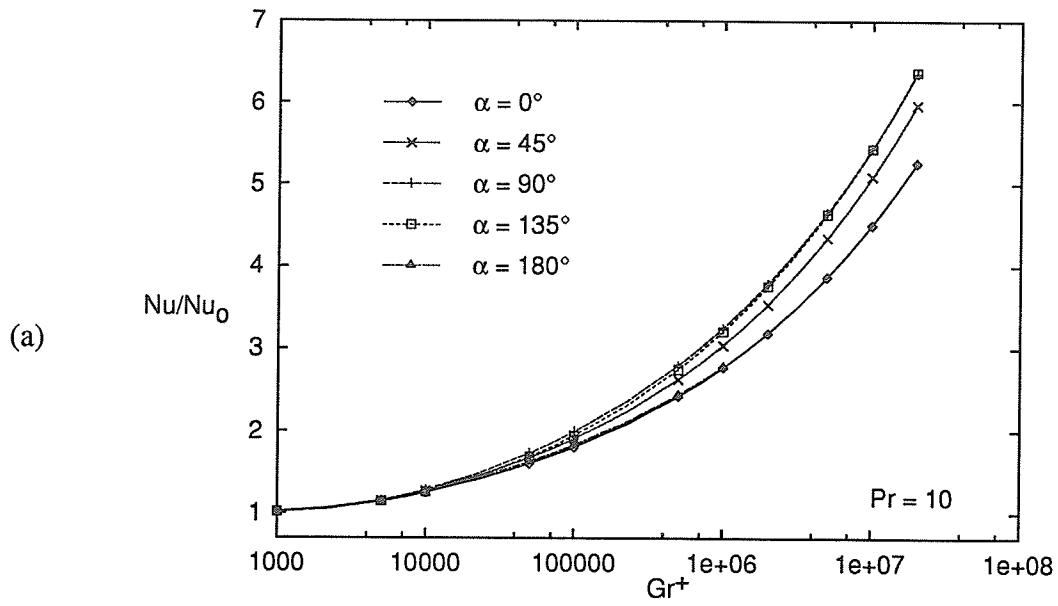


Fig. 5.8 (a)  $Nu/Nu_0$  and (b)  $fRe/(fRe)_0$  for  $Pr = 10$ .

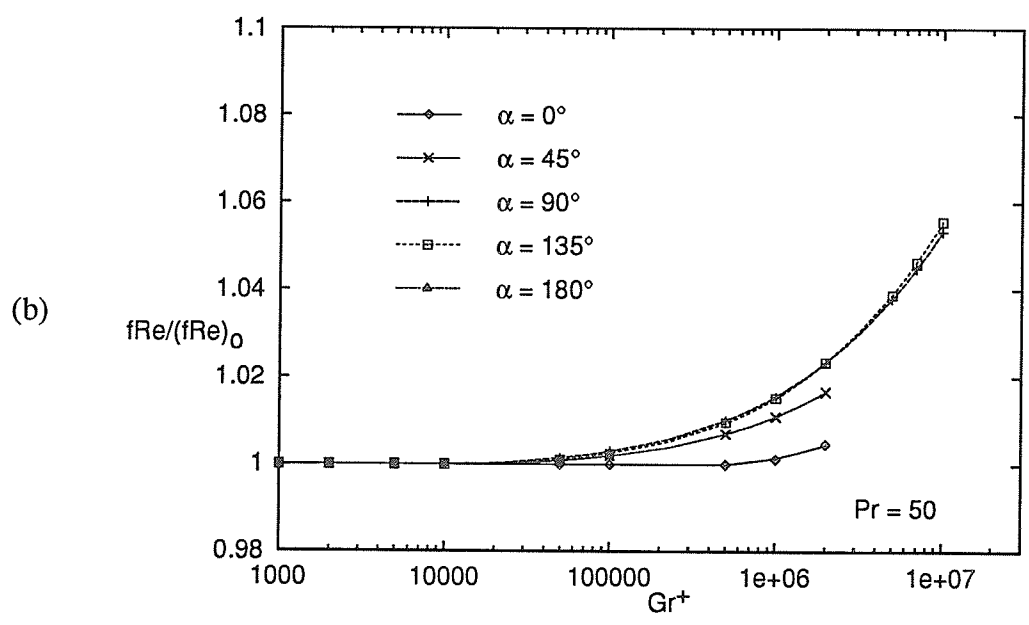
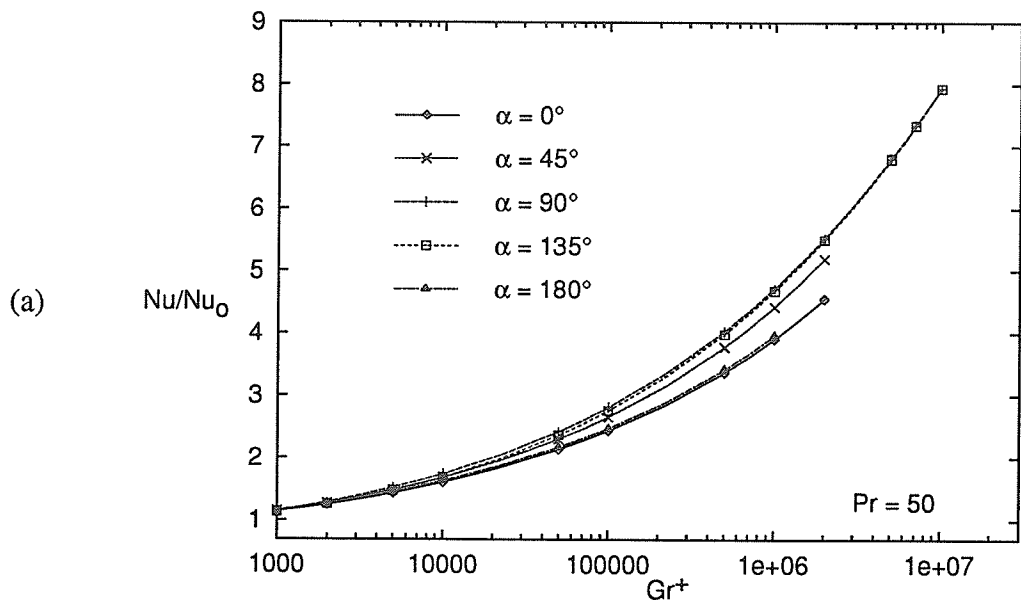
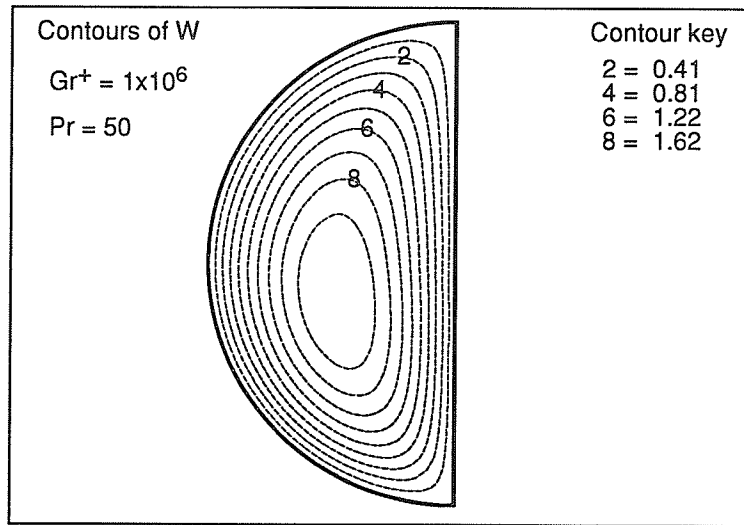


Fig. 5.9 (a)  $Nu/Nu_0$  and (b)  $fRe/(fRe)_0$  for  $Pr = 50$ .

(a)



(b)

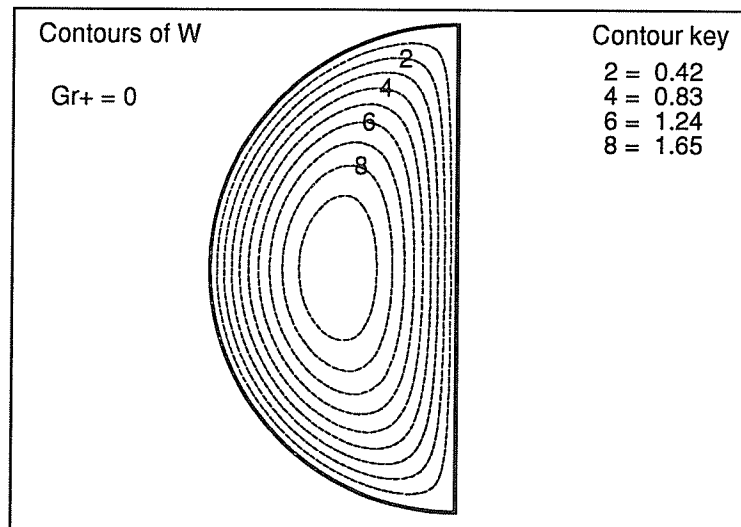


Fig. 5.10 Axial velocity contours for (a)  $Pr = 50$ ,  $Gr^+ = 10^6$ , (b)  $Gr^+ = 0$ .

(c)

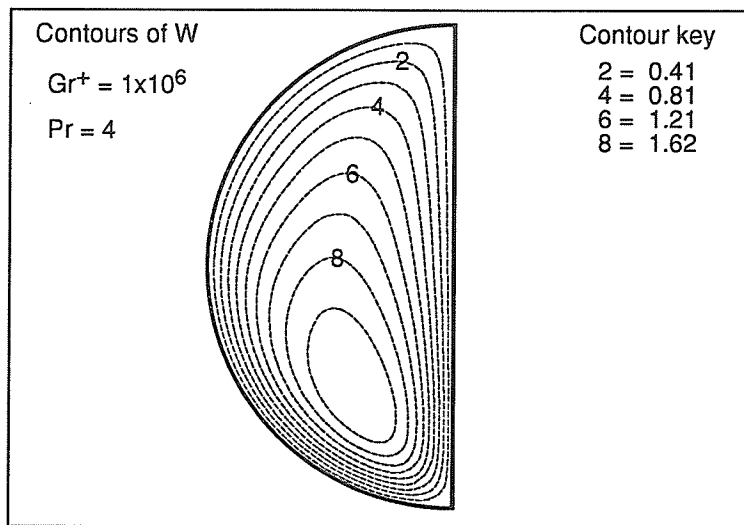


Fig. 5.10(cont) Axial velocity contours for (c)  $Pr = 4$ ,  $Gr^+ = 10^6$ .

It is interesting to note that the increase of  $fRe$  due to the buoyancy effect was so small, e.g. 1.053 fold at  $Gr^+ = 10^7$  for  $\alpha = 90^\circ$  at which the maximum buoyancy effect occurred. This indicates that distortion of the axial velocity fields caused by the buoyancy flow is small due to the very viscous fluid. The contours of the axial velocity for  $Pr = 50$  and  $Gr^+ = 10^6$  are shown in Fig. 5.10(a) whereas Fig. 5.10(b) is for  $Gr^+ = 0$ . The similarity of the two contour plots supports the above observation that distortions decrease with increasing  $Pr$ . A further illustration is provided by Fig. 5.10(c) which is for  $Gr^+ = 10^6$  and  $Pr = 4$ . Here a large amount of distortion can be observed compared to Fig. 5.10(a) and (b).

The  $Nu$  and  $fRe$  for  $Pr = 10$  and  $50$  were also computed at  $\alpha = 115^\circ$  which is the location where Nusselt number was found maximum at high  $Gr^+$  for the case of  $Pr = 4$ . The  $Nu/Nu_0$  versus  $\alpha$  are illustrated in Figs. 5.11(a) and (b) for  $Pr = 10$  and  $50$ . For  $\alpha = 180^\circ$ , where the data are not available, the curves are extrapolated by assuming the same values as at  $\alpha = 0^\circ$ . The maximum again appears at  $\alpha = 115^\circ$  in both cases of  $Pr$ . At the highest  $Gr^+$ , the  $Nu$  for  $\alpha = 115^\circ$  is only 1.2% higher from its neighbours. This is true for both  $Pr$  and a more precise location of  $\alpha$  was not searched for these new  $Pr$ . The maximums shift from  $\alpha = 45^\circ$  at the lowest  $Gr^+$  to  $\alpha = 90^\circ$  and  $135^\circ$  as  $Gr^+$  increases. This trend agrees well with the case of  $Pr = 4$ . The  $fRe$  results are shown only for  $Pr = 10$  in Fig. 5.12. The  $fRe$  results for  $Pr = 50$  is skipped due to its tiny variations (less than 2%) for all orientations. The locus of the maximums for  $fRe$  is basically similar to the case of  $Nu$ .

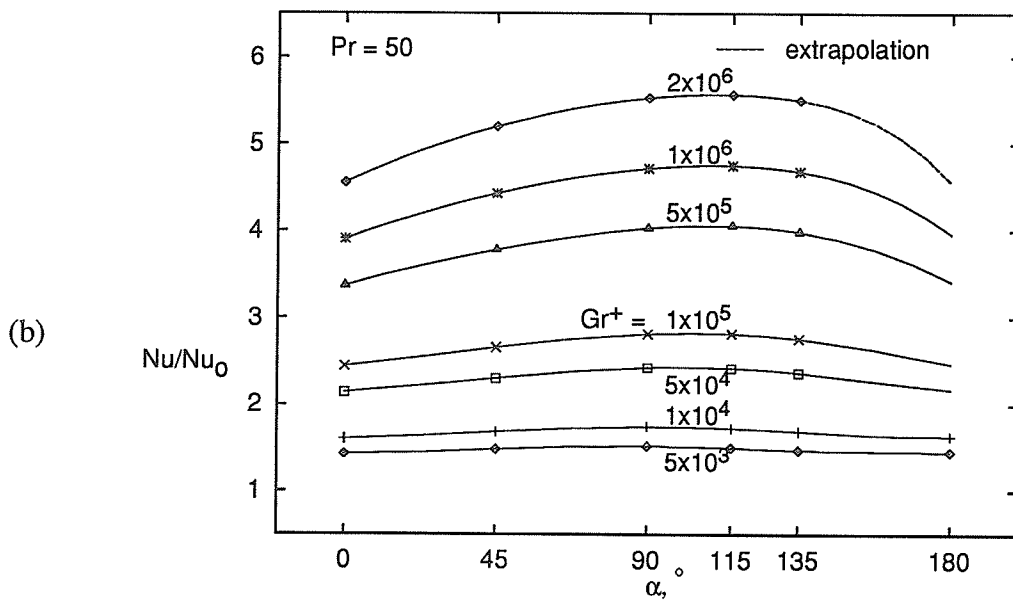
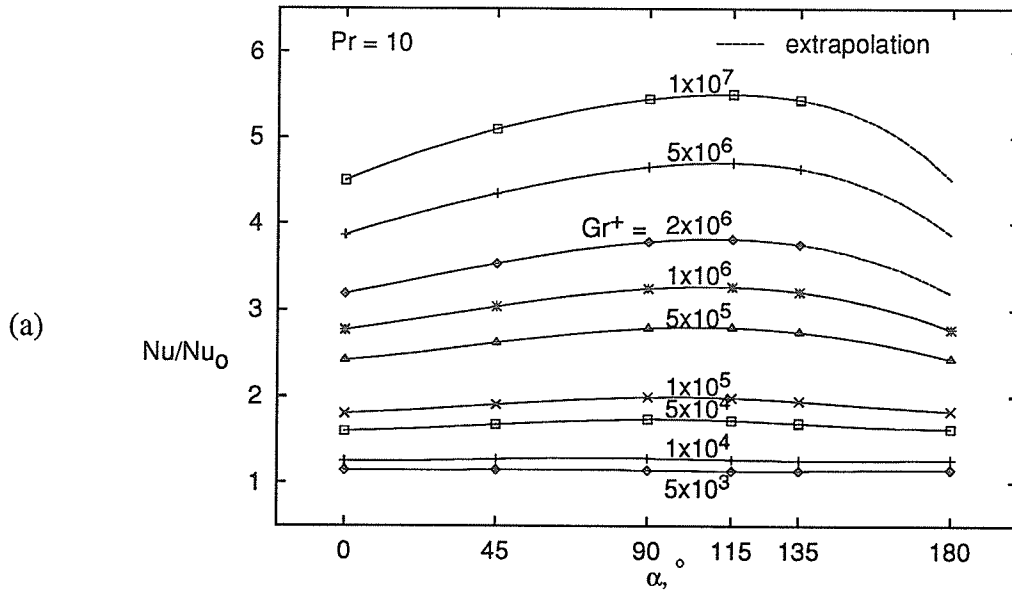


Fig. 5.11  $Nu/Nu_0$  versus  $\alpha$  for  $Pr =$  (a) 10 and (b) 50.

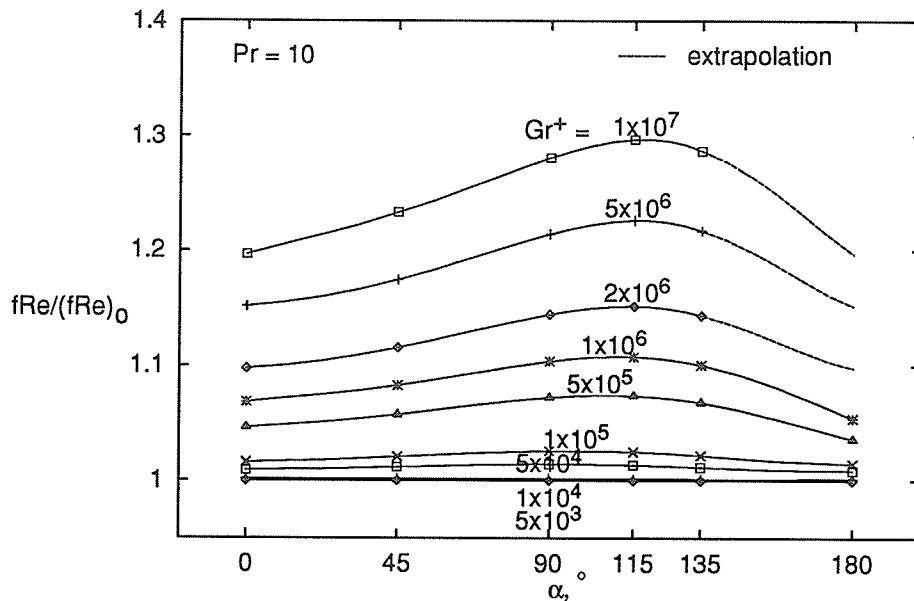


Fig. 5.12  $fRe/(fRe)_0$  versus  $\alpha$  for  $Pr = 10$ .

## 5.6 Prandtl Number Effect on Nusselt Number

The Nusselt number results for  $Pr = 4, 10$  and  $50$  from the previous sections are analyzed in this section. It has been shown in the previous sections that the  $Nu$  data form a well defined curve on the  $Nu-Gr^+$  domain for a given  $Pr$ . The primary influence of Prandtl number is that the Nusselt number increases with increasing Prandtl number at a given  $Gr^+$ . For simplicity, the data for  $\alpha = 0^\circ$  and  $180^\circ$  were merged due to their similarities in magnitude and these are plotted in Fig. 5.13(a) which clearly shows, for fixed  $Gr^+$ , that  $Nu$  increase with increasing  $Pr$ .

The analyses of combined convection problem have shown that the fully developed Nusselt number depends on only three dimensionless quantities in term of  $ReRa$  and  $Pr$ , e.g. see Morton [5], Mori *et. al.* [8]. Later, Patanka *et. al.* [21] presented a use of  $Gr^+$  as a replacement for  $ReRa$  by showing that they are equivalent. Thus the variables reduced to  $Gr^+$  and  $Pr$ . For an ideal fluid,  $Gr^+$  and  $Pr$  represent the heat input and fluid type while  $Nu$  represents the inverse of wall-to-fluid temperature drop. In most works, e.g. [18–22], the solution of Nusselt number was done by keeping  $Pr$  fixed and varying  $Gr^+$  (or  $ReRa$ ). Those results were similar to Fig. 5.14 that a  $Nu-Gr^+$  curve is for a  $Pr$  value and the curve for a larger  $Pr$  will lie above that of a lower  $Pr$ , e.g. see [8], [18], [21], [29–32], etc. This coupled effect of  $Pr$  can be eliminated if the product  $Gr^+Pr$  is used as a sole independent variable for  $Nu$ . This was originally demonstrated by Lei [45] and Lei and Trupp [46]. When the same data as in Fig. 5.13(a) are plotted against  $Gr^+Pr$  in Fig. 5.13(b), the three curves collapse and all the data can be well represented by a single curve. To verify this evidence, the Nusselt number results of the semicircular duct with  $\alpha = 180^\circ$  and  $Pr = 0.7$  and  $5.0$  from [22] were carefully duplicated and are plotted in Fig. 5.14(a) in term of  $Gr^+$  and in Fig. 5.14(b) in term of  $Gr^+Pr$ . The data for the multiple values of  $Pr$  again form a single and well defined curve on the  $Nu-Gr^+Pr$  domain. To compare the results from [22] and the present results, data from Figs. 5.13(b) and 5.14(b) are shown in Fig. 5.14(c). Excellent agreement of both data confirms not only the comparable accuracy of both data but also the single curve on the  $Nu-Gr^+Pr$  domain. Further confirmations were done on other ducts; a circular duct [22] and a rectangular duct [28] with uniform heat flux, and a

circular duct with non-uniform heat flux [34] (which included the data from [21] already). These data are plotted against  $Gr^+Pr$  in Figs. 5.15(a)–(c). The outcome is excellent for the circular duct as can be seen in Fig. 5.15(a). For the square duct, there is some small separation of points in Fig. 5.15(b), but this is somewhat due to the very narrow range of the data so that the plot exaggerates small differences. For the third case, the twisting shape of the original curves in [34] indicates a complicated heat transfer and flow characteristics under the non-uniform heating condition. In spite of such facts, the outcomes of Fig. 5.15(c) confirms  $Gr^+Pr$  scaling.

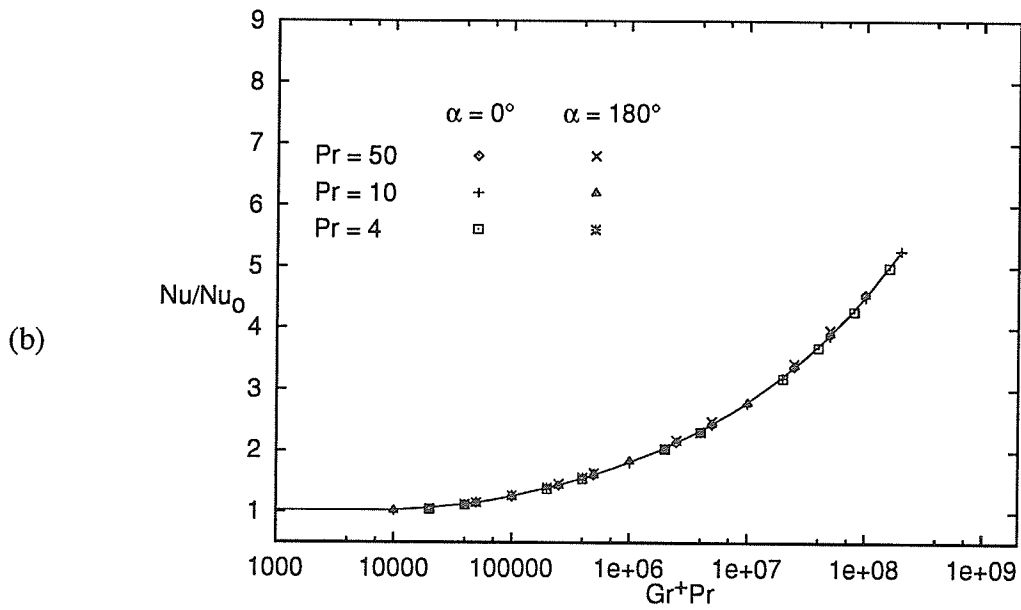
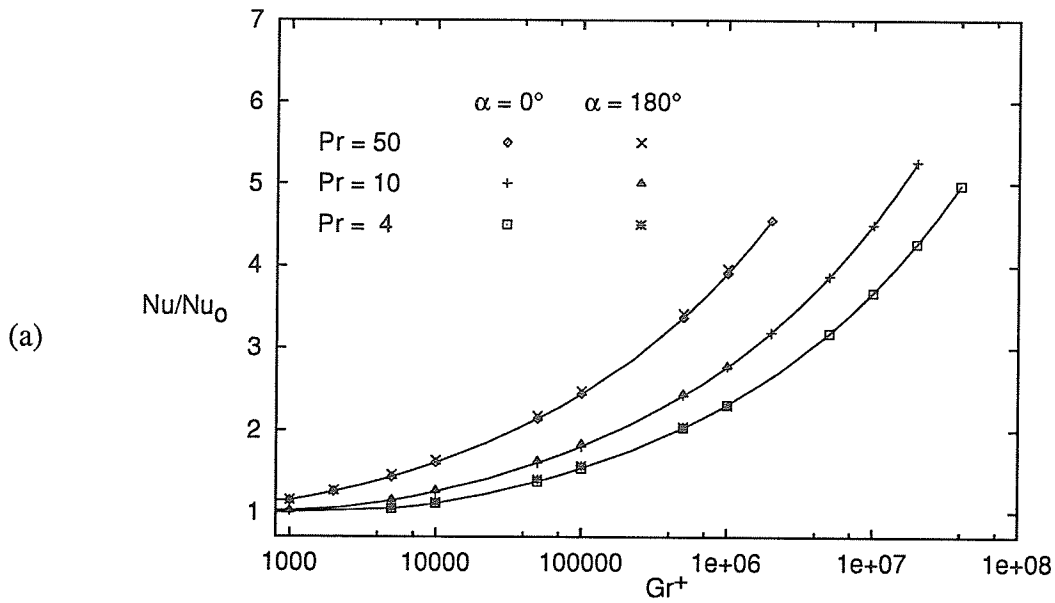


Fig. 5.13  $Nu/Nu_0$  versus (a)  $Gr^+$  and (b)  $Gr^+Pr$ , for all  $Pr$ .

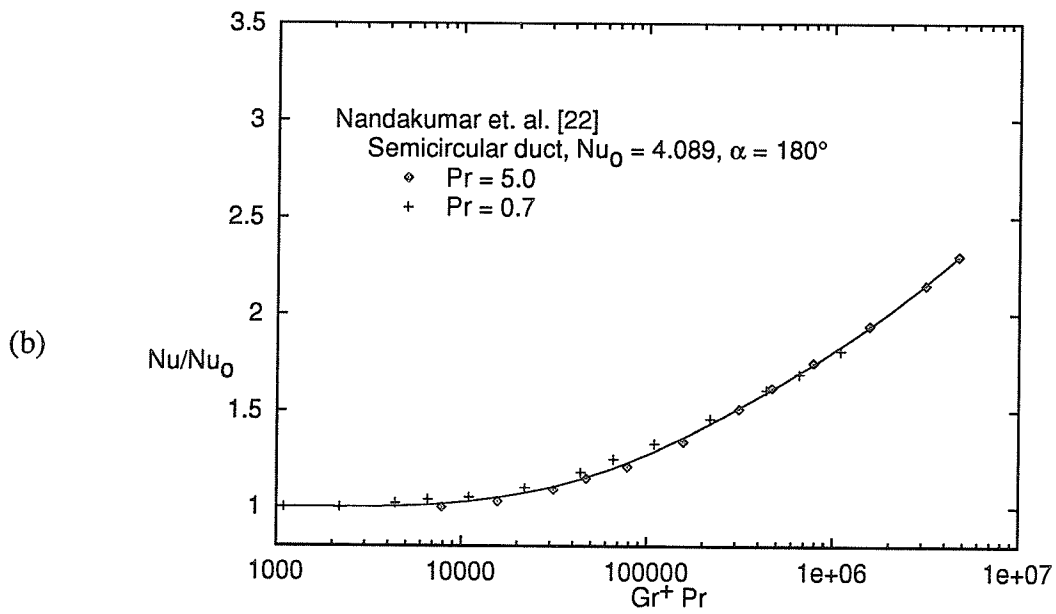
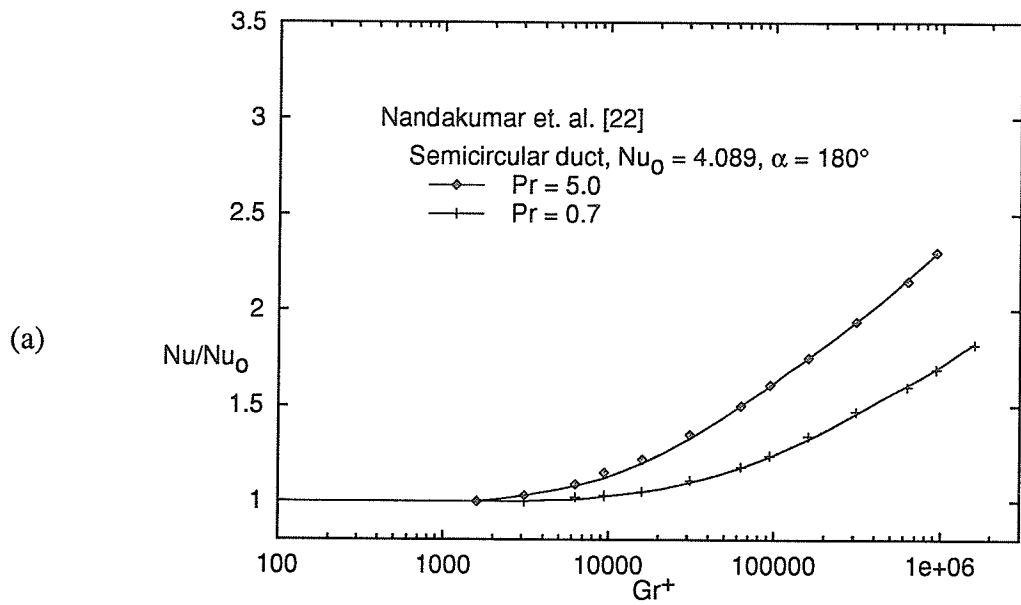


Fig. 5.14  $Nu/Nu_0$  from [22] versus (a)  $Gr^+$ , (b)  $Gr^+ Pr$ .

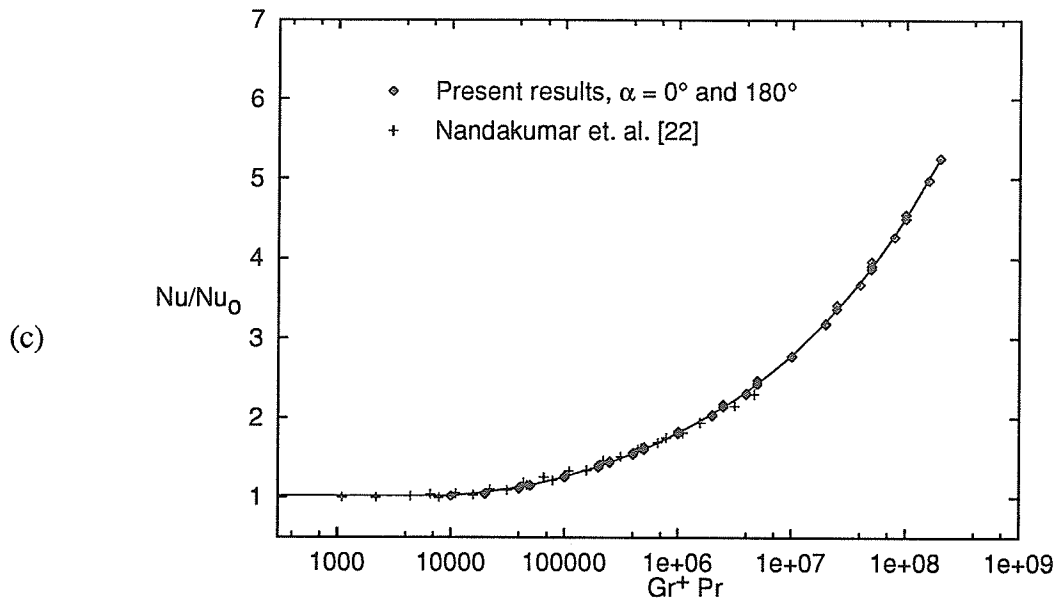


Fig. 5.14(cont)  $Nu/Nu_0$  from [22] (c) comparison to the present results.

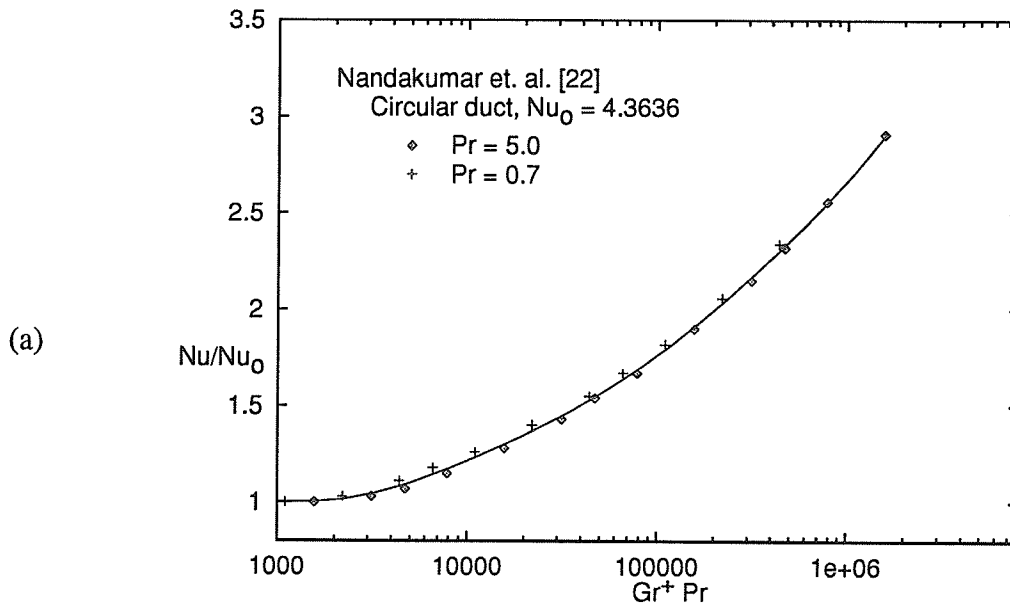


Fig. 5.15  $Nu/Nu_0$  of (a) circular duct [22].

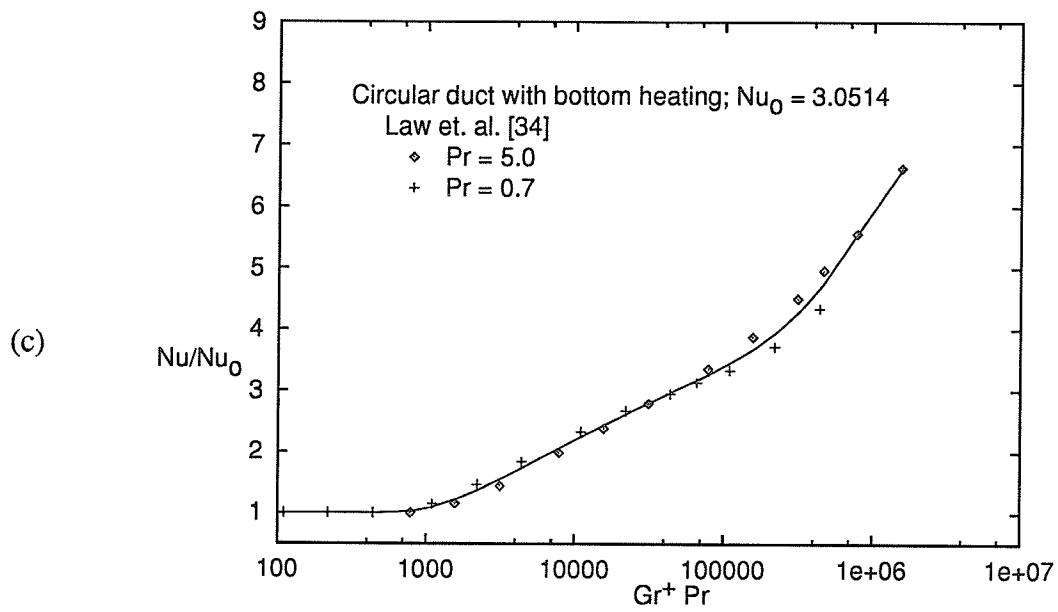
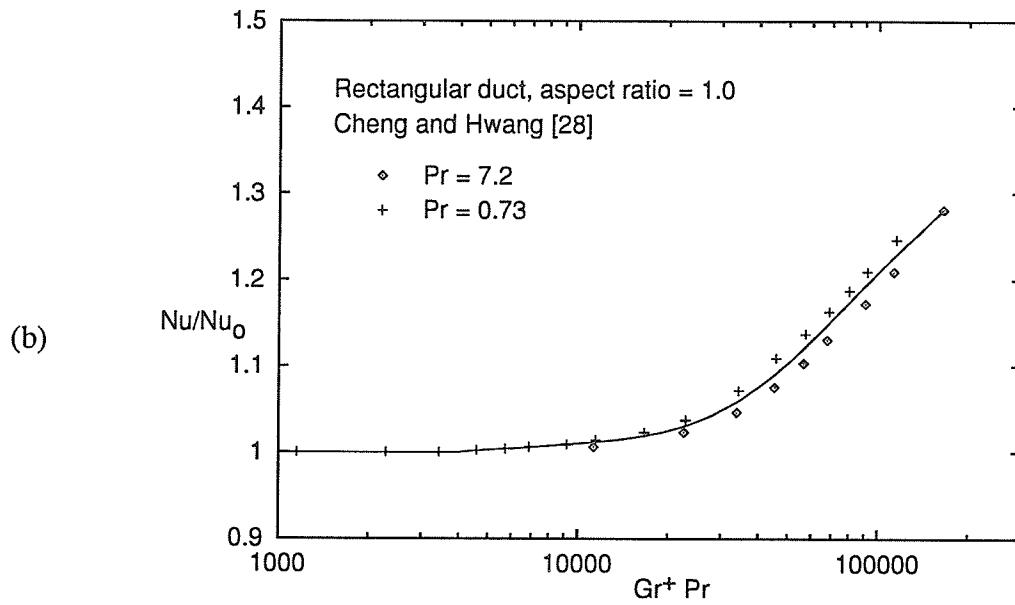


Fig. 5.15(cont)  $Nu/Nu_0$  of (b) rectangular duct [28], and (c) circular duct with non-uniform heat flux [34].

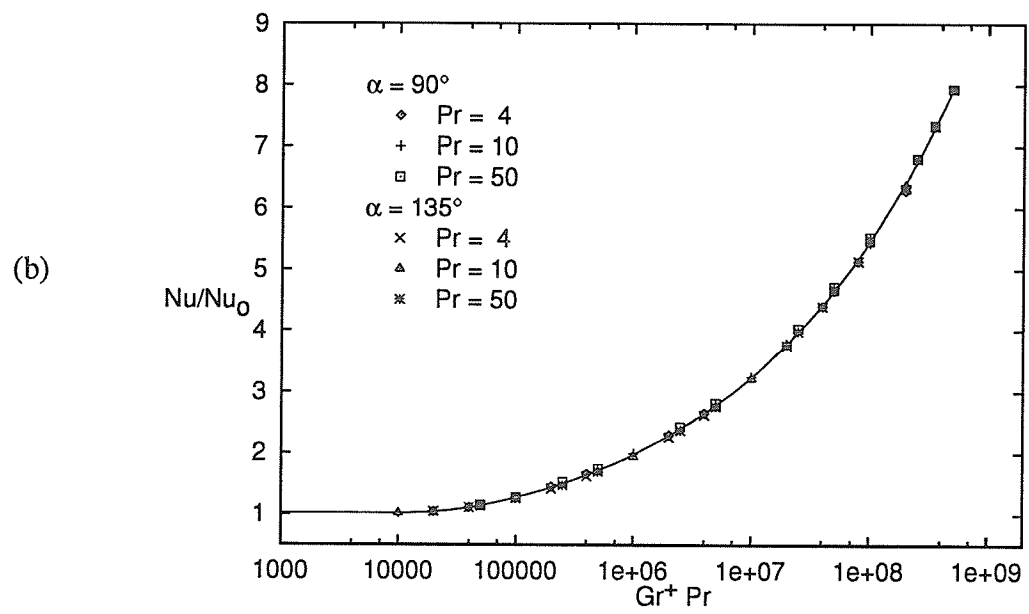
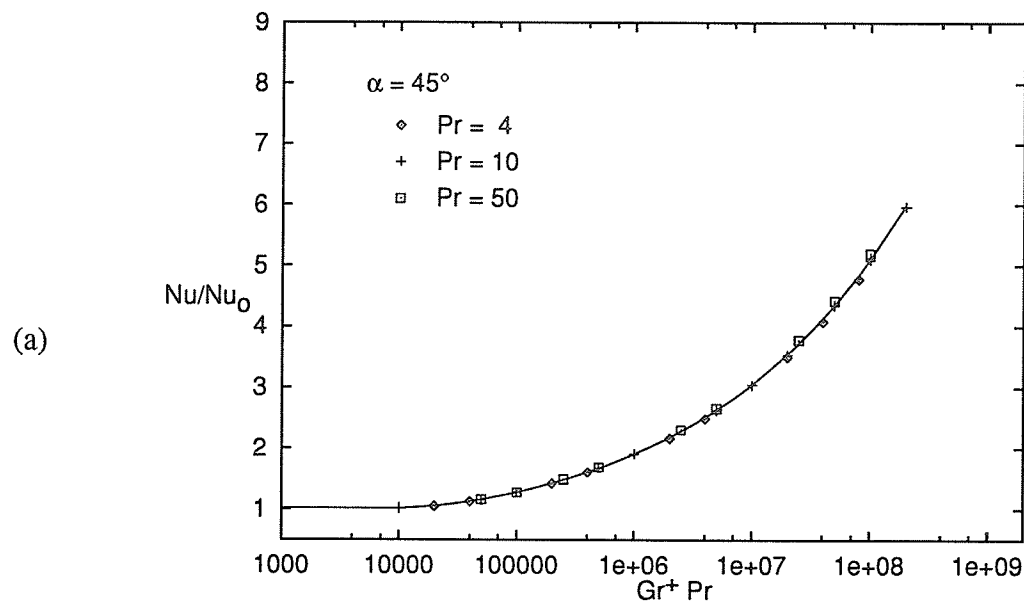


Fig. 5.16  $Nu/Nu_0$  versus  $Gr^+Pr$  for  $\alpha =$  (a)  $45^\circ$ , (b)  $90^\circ/135^\circ$ .

(c)

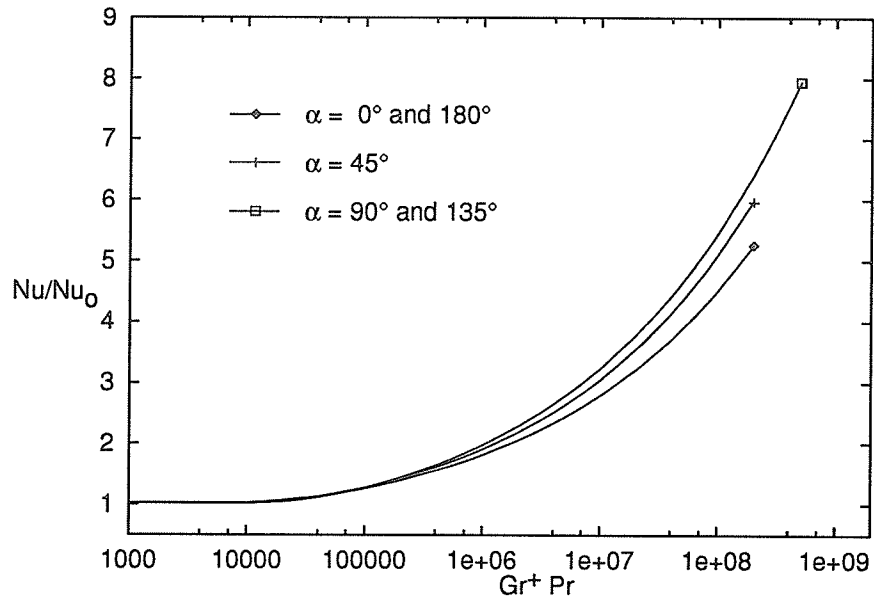


Fig. 5.16(cont)  $Nu/Nu_0$  versus  $Gr^+Pr$  for (c) all  $\alpha$ .

For presentation purposes, the data for  $\alpha = 90^\circ$  and  $135^\circ$  were combined due to their almost identical magnitudes. The results for  $\alpha = 45^\circ$  and  $90^\circ/135^\circ$  are shown in Figs. 5.16(a) and (b) respectively, and again excellent  $Gr^+Pr$  scaling is illustrated. A comparison among the three cases is shown in Fig. 5.16(c) in which the Nusselt number curves arrange in the same sequence as they did on the  $Nu-Gr^+$  domain.

The uniqueness of the  $Nu-Gr^+Pr$  relation is very useful if a precise prediction for a specific Prandtl number is required. The lengthy perturbation process can be avoided by computing the Nusselt number from the  $Nu-Gr^+Pr$  relation. The  $Nu$  data are modelled in a polynomial form as

$$\frac{Nu_i}{Nu_o} = C_0 + C_1x_i + C_2x_i^2 + \dots + C_nx_i^n \quad (5.1)$$

where  $i = 1, 2, \dots, m$  which  $m$  is the total number of data, and

$$x_i = \log_{10}(Gr^+Pr) \quad (5.2)$$

The residual norm and standard deviation are determined by

$$R = \sum_{i=1}^m \left( C_0 + C_1x_i + C_2x_i^2 + \dots + C_nx_i^n - \frac{Nu_i}{Nu_o} \right)^2 \quad (5.3)$$

$$\sigma = \sqrt{\frac{R}{m-1}} \quad (5.4)$$

The order of the polynomials was varied from 3 to 9, for all data. It was found that  $n$

$n = 4$  was the best choice in terms of computing cost and accuracy because the residuals only decreased slightly as  $n$  was increased from 4 to 9. The coefficients for  $n = 4$  and  $\alpha = 0^\circ, 45^\circ, 90^\circ, 135^\circ, 180^\circ, 0^\circ/180^\circ,$  and  $90^\circ/135^\circ$  are compiled in Table 5.10. It should be noted here that the data were also tested with the exponential model similar to Chap. 4 but the fit was not as good since each set of data already form a well defined curve which the polynomial modelling was a much better choice.

It is interesting to note that the  $Gr^+Pr$  parameter did not surface theoretically in the mathematical model in Chap. 3, but rather it was empirically formulated from the numerical Nusselt number results. Since there is no obvious theoretical basis, it may be too early to generalize that the Nusselt number depends on a sole parameter for a fixed orientation horizontal duct. In addition, the result in [45] and [46] showed a small amount of separation from the unique curve for  $Pr = 0.7$ . This separation is also noticeable in Fig. 5.15(b) which contains the  $Pr \approx 0.7$  data. On the other hand, the result in Fig. 5.14(b) which also contains the data of this Prandtl number did not show this sign. It is possible that the unique  $Nu-Gr^+Pr$  relation could somewhat deteriorate as  $Pr$  is less than 1. Nevertheless, it is clear that for  $4 \leq Pr \leq 50$ , the  $Gr^+Pr$  works well for the semicircular duct.

**Table 5.10** Coefficients of the polynomials in equation 5.1 for the  $Nu/Nu_0$ .

$\alpha$	Coefficients							$\sigma_{\max}$
	$C_0$	$C_1$	$C_2$	$C_3$	$C_4$	$\sigma$		
0°	0.95150982E+1	-0.62369480E+1	0.16698433E+1	-0.19990874E+0	0.98561374E-2	0.00269777	0.04544893	
45°	0.98366232E+1	-0.65221121E+1	0.17634421E+1	-0.21453426E+0	0.10845645E-1	0.00636744	0.09586126	
90°	0.14531336E+2	-0.96409864E+1	0.25125910E+1	-0.29242278E+0	0.13907628E-1	0.00619665	0.12110808	
135°	0.12096346E+2	-0.78315638E+1	0.20317778E+1	-0.23848557E+0	0.11737342E-1	0.00438557	0.08410378	
180°	0.14866670E+2	-0.10350876E+2	0.28333789E+1	-0.34329864E+0	0.16369740E-1	0.00351722	0.03922880	
0° and 180°	0.83017243E+1	-0.54070747E+1	0.14595934E+1	-0.17638511E+0	0.88781484E-2	0.00312848	0.07435869	
90° and 135°	0.13313841E+2	-0.87362750E+1	0.22721843E+1	-0.26545417E+0	0.12822485E-1	0.00417036	0.12150593	

## 5.7 Prandtl number effect on $fRe$

The Prandtl number affects  $fRe$  quite differently from its effect on Nusselt number. For a given  $Gr^+$ ,  $fRe$  decreases with increasing  $Pr$  as demonstrated in section 5.6 and in Fig. 5.17(a) where  $fRe/(fRe)_o$  vs  $Gr^+$  is shown for  $\alpha = 90^\circ$  as a sample. The higher viscosity of the fluid due to its higher Prandtl number retards the buoyancy flow thus resulting in a less distorted axial velocity field. Therefore  $fRe$  is not much different from its original forced-convection value. Similar to Nusselt number,  $fRe$  is governed by  $Pr$  and  $Gr^+$ . The coupled effect of  $Pr$  was overcome by using  $Gr^+/Pr^{1.8}$  as initially illustrated in [45] and [46]. The present results were test by  $Gr^+/Pr^n$  which  $n$  was varied from 2.0 to 1.2. The outcome of  $n = 2.0$  was the best even though it was only slightly different from that of  $n = 1.8$ . The data of Fig. 5.17(a) are plotted against  $Gr^+/Pr^2$  in Fig. 5.17(b) where a single curve is formed. Successful scaling is further illustrated in Figs. 5.18(a), (b) and (c) where results are presented for  $\alpha = 0^\circ$  and  $180^\circ$ ,  $45^\circ$ , and  $90^\circ$  and  $135^\circ$ . The single curves are formed well for each case.

The  $fRe/(fRe)_o$  is formulated with an equation similar to eqn. (5.1) where  $x_i = \log_{10}(Gr_i^+/Pr^2)$ . The coefficients are shown in Table 5.11. The  $\sigma$  and  $\sigma_{\max}$  for  $fRe/(fRe)_o$  are generally smaller than those for  $Nu/Nu_o$  for each  $\alpha$ . This is due to the smaller magnitudes of  $fRe$  ratios rather than other reasons. In fact, both the  $Nu$  and  $fRe$  data are represented very well by these polynomial models in terms of the very small standard variations (less than 1% for both cases). The polynomial models were actually

used for generating the curves in all of the related figures, e.g. see Figs. 5.18, 5.16(a), and (b).

It is worth noting that even though the data for  $Pr = 50$  are combined with the rest, they share only a small portion in the  $fRe/(fRe)_o - Gr_i^+/Pr^2$  relations. This is due to two reasons. First, the data for  $Pr = 50$  were shifted much toward the left when the abscissa was  $Gr^+/Pr^2$  instead of  $Gr^+$  (as compared to the data for smaller  $Pr$ ). Secondly, the  $fRe/(fRe)_o$  magnitudes for  $Pr = 50$  are very close to 1. Therefore the  $Pr = 50$  data locate only within the flat parts of the curves in Fig. 5.18. The curves in  $Gr^+/Pr^2 > 10^4$  rely on the data for  $Pr = 4$  and 10 only. Finally, the results from Fig. 5.18 are shown together in Fig. 5.19 to allow comparison. At  $Gr^+/Pr^2 \approx 10^6$ , the  $fRe/(fRe)_o$  ratio is about 1.7 and the difference between the maximum and minimum curves is roughly 8%.

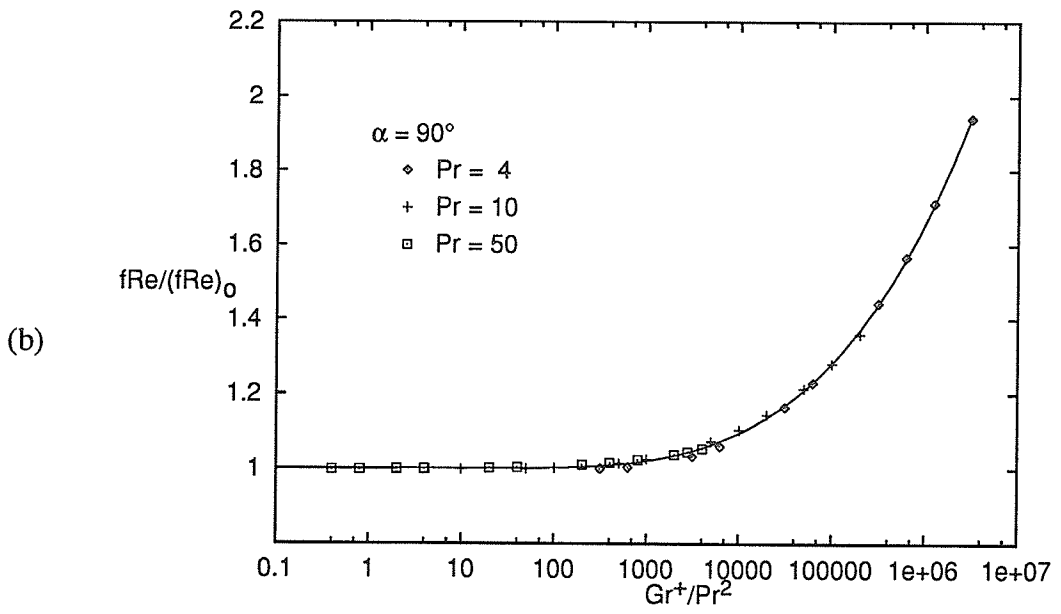
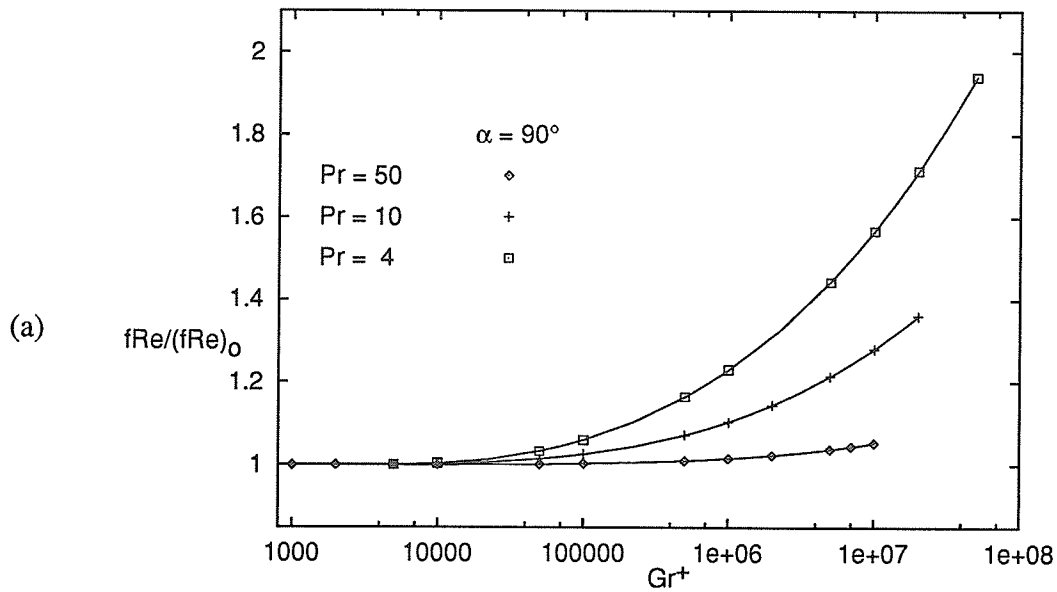


Fig. 5.17  $fRe/(fRe)_0$  versus (a)  $Gr^+$  and (b)  $Gr^+/Pr^2$ , for  $\alpha = 90^\circ$ .

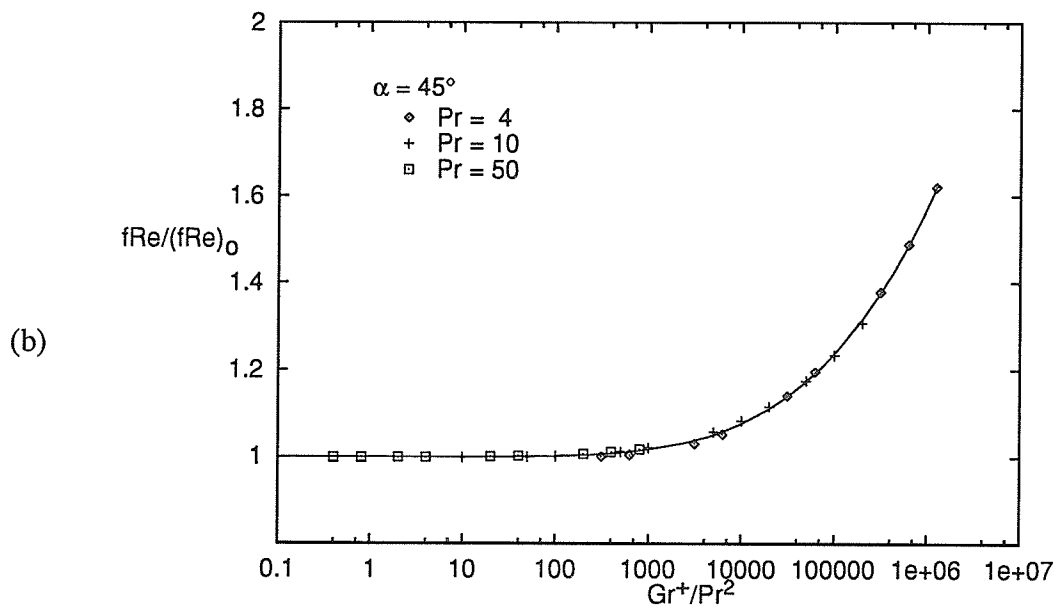
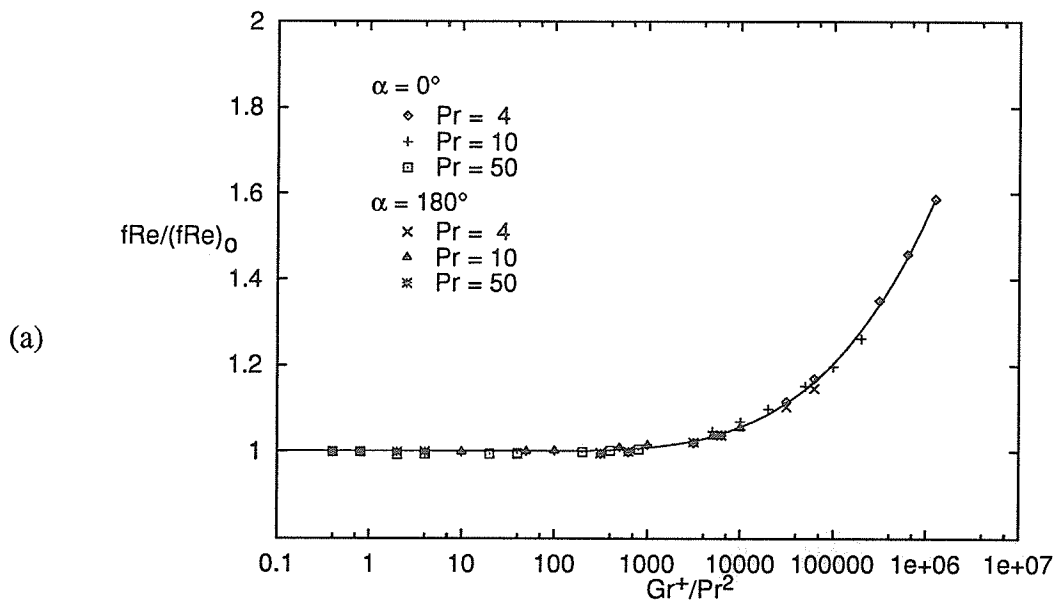


Fig. 5.18  $fRe/(fRe)_0$  versus  $Gr^+/Pr^2$  for  $\alpha =$  (a)  $0^\circ/180^\circ$ , (b)  $45^\circ$ .

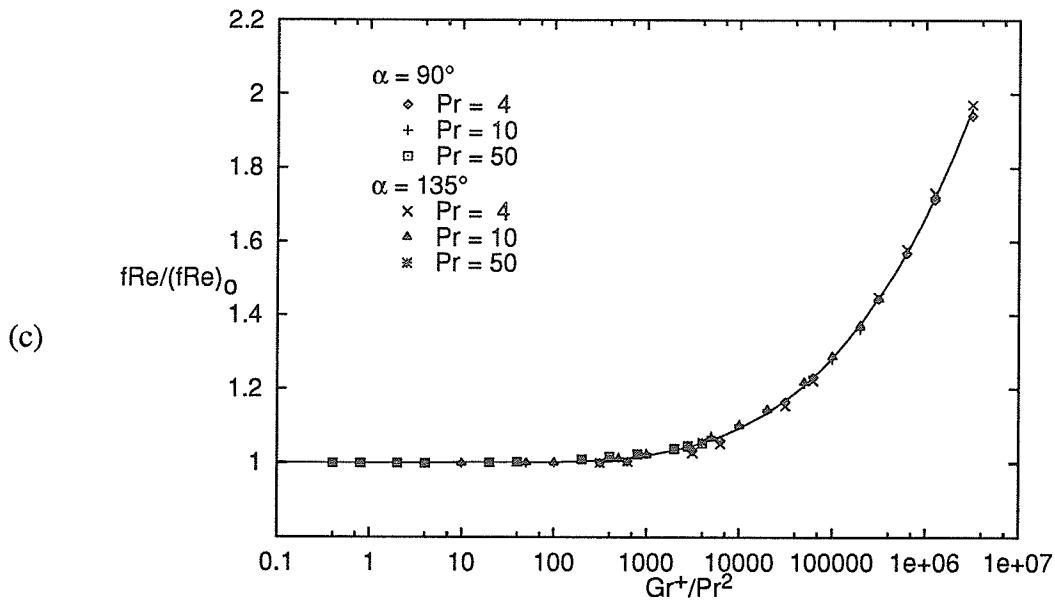


Fig. 5.18(cont)  $fRe/(fRe)_0$  versus  $Gr^+/Pr^2$  for  $\alpha =$  (c)  $90^\circ/135^\circ$ .

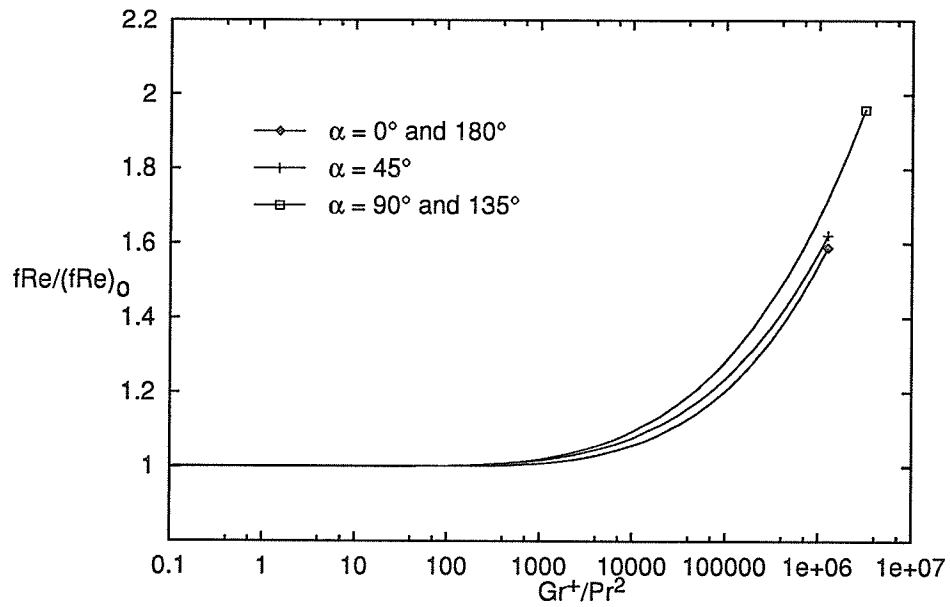


Fig. 5.19  $fRe/(fRe)_0$  versus  $Gr^+/Pr^2$  for all  $\alpha$ .

**Table 5.11** Coefficients of the polynomials for  $fRe(fRe)_\alpha$ .

$\alpha$	Coefficients						
	$C_0$	$C_1$	$C_2$	$C_3$	$C_4$	$\sigma$	$\sigma_{\max}$
0°	0.99533134E+0	-0.89212149E-2	0.14030731E-1	-0.67310238E-2	0.11944874E-2	0.00123837	0.02127804
45°	0.99824711E+0	-0.11880349E-2	0.54225957E-2	-0.37609285E-2	0.92597196E-3	0.00076026	0.00949152
90°	0.99913959E+0	0.33129348E-2	-0.50706989E-3	-0.18385361E-2	0.81410879E-3	0.00106207	0.01474402
135°	0.99889139E+0	0.33843436E-2	0.46165310E-3	-0.25336721E-2	0.91567897E-3	0.00126902	0.01891866
0° and 180°	0.99626326E+0	-0.64666543E-2	0.12547567E-1	-0.66112268E-2	0.12039536E-2	0.00094157	0.01886392
90° and 135°	0.99901549E+0	0.33486392E-2	-0.22708395E-4	-0.21861041E-2	0.86489388E-3	0.00092552	0.02039447

## 5.8 Comparisons to the Experimental Results

The numerical Nusselt number results for  $Pr = 4$  and  $\alpha = 45^\circ, 90^\circ, 135^\circ$  are compared to the experimental results from section 4.6. For  $\alpha = 180^\circ$ , the numerical results for  $Pr = 4$  were obtained for  $Gr^+$  up to only  $10^6$  whereas the experiments were conducted for  $Gr^+ > 2 \times 10^6$  therefore the comparison is not available. However, the numerical results obtained from all  $Pr$  values are fairly high compared to the range of the experimental results therefore the comparison for  $\alpha = 180^\circ$  is available in term of  $Gr^+ Pr$  variable.

The comparisons in terms of  $Gr^+$  are shown in Fig. 5.20(a) where the solid curves for the experimental results are generated by the correlations in Table 4.3 and the dash curves for the numerical results are reproduced from the data in Table 5.10. The numerical and the experimental results agreed very well near  $Gr^+ = 10^7$  and the difference between both data increases as  $Gr^+$  increases or decreases from  $10^7$ . In the low  $Gr^+$  region, the predictions are underestimations. At  $Gr^+ = 2.7 \times 10^6$ , the differences are 11%, 8% and 8% for  $\alpha = 45^\circ, 90^\circ$ , and  $135^\circ$ , respectively. In the high  $Gr^+$  region, the numerical results are higher especially at the highest range of the experimental results. However, due to the limits of the numerical results, the comparisons can be done only at  $Gr^+ = 4 \times 10^7$  where the differences for  $\alpha = 90^\circ$  and  $135^\circ$  are 6% and 7% based on the experimental results.

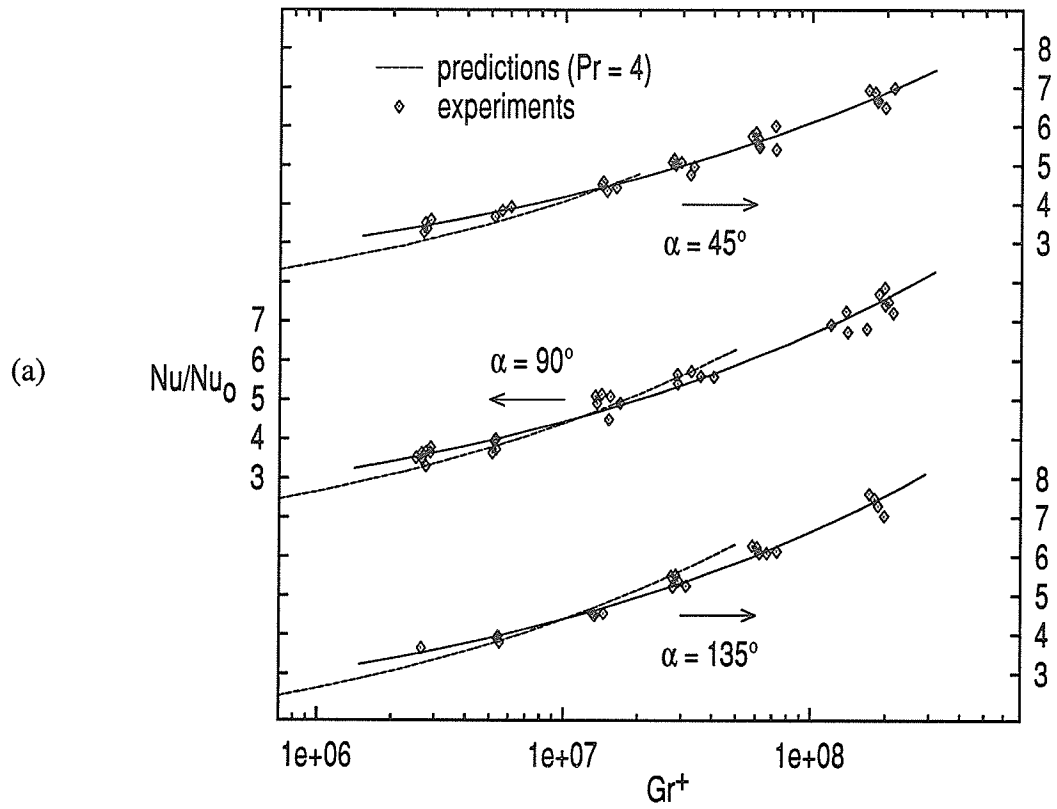


Fig. 5.20 Comparisons to the experimental results in terms of (a)  $Gr^+$

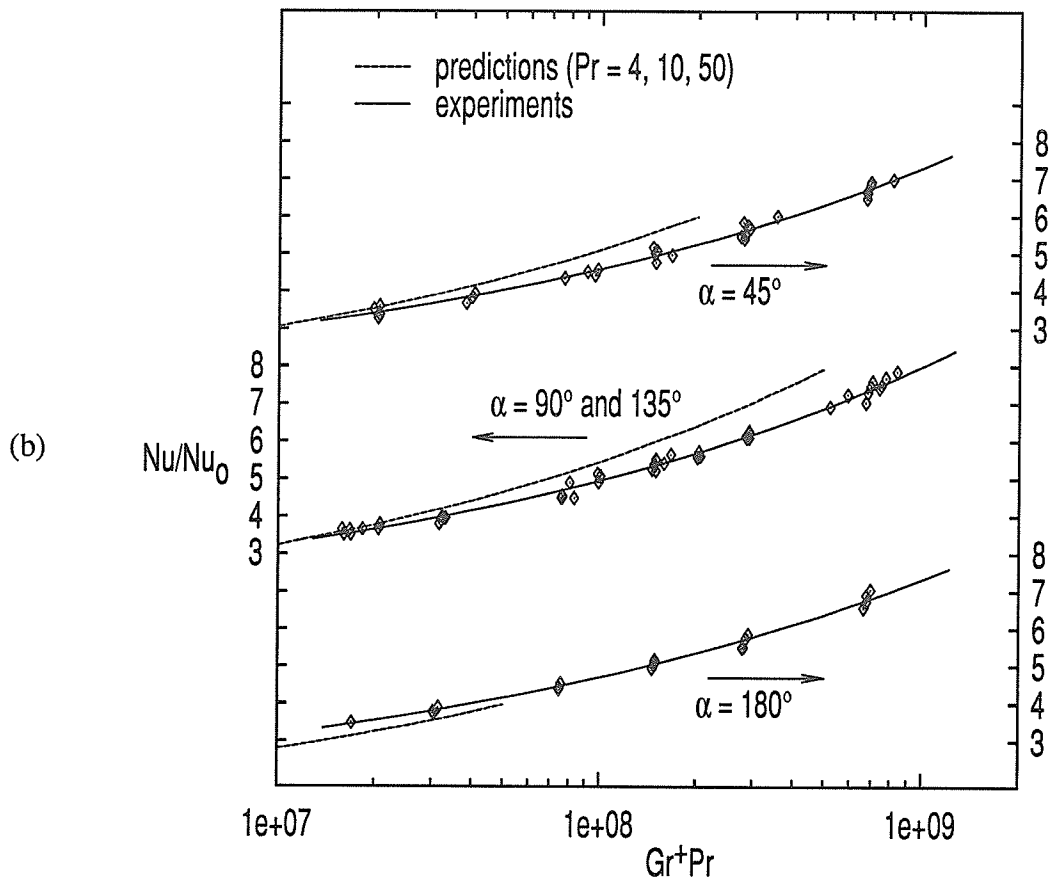


Fig. 5.20(cont) Comparisons to the experimental results in terms of (b)  $Gr^+Pr$ .

The numerical results in term of  $Gr^+Pr$  which include the results for  $Pr = 4, 10,$  and  $50,$  are compared to the experimental results in Fig. 5.20(b) for  $\alpha = 45^\circ, 90^\circ$  and  $135^\circ,$  and  $180^\circ.$  This comparison is more general than the previous one in Fig. 5.20(a) due to the variation of Prandtl number within the experimental results are absorbed in the  $Gr^+Pr$  product. All results show good agreements at the low end of the  $Gr^+Pr.$  In general, as  $Gr^+Pr$  increases, the curves for the numerical results show a steeper slope and therefore the overestimations are clearly expectable at high  $Gr^+Pr$  for all  $\alpha.$  The intersections of the numerical and experimental data are near  $2 \times 10^7$  for  $\alpha = 45^\circ,$  and  $90^\circ$  and  $135^\circ,$  and near  $5 \times 10^7$  for  $\alpha 180^\circ.$  At near the intersections, the predictions are about 5% lower for  $\alpha = 180^\circ$  and about 3.5% higher for both  $\alpha = 45^\circ,$  and  $90^\circ$  and  $135^\circ.$  The differences increase as  $Gr^+Pr$  move further from the intersections. The differences estimated at  $Gr^+Pr = 2 \times 10^8$  are 14% and 13% overestimations for  $\alpha = 45^\circ,$  and  $90^\circ$  and  $135^\circ,$  and about 9% underestimation at  $Gr^+Pr = 2 \times 10^7$  for  $\alpha = 180^\circ.$

Overall, the predictions clearly show the trends of overestimation which appear at the lowest  $Gr^+Pr$  value except for the case of  $\alpha = 180^\circ$  where the overestimation did not actually appear but it is expectable if  $Gr^+Pr$  is further increased. The overestimation is due to the H1 thermal boundary condition assumed in the mathematical model. The H1 condition assumes a uniform peripheral temperature while the experimental results showed the non-uniform temperature which increases with the magnitude of  $Gr^+.$  On the other hand, the H2 condition assumes a uniform peripheral heat flux. Comparing to the H2 condition, the H1 condition has a higher heat transfer coefficient according to a larger heat transfer potential resulted from the cooler fluid near the bottom wall and

the uniform wall temperature. As a result, the Nusselt number for H1 condition should be higher as compared to the H2 condition. The experimental results indicated when the heating rate was small, the wall temperature condition was indeed close to the H1 condition, e.g. about  $0.2^\circ$  at  $Gr^+Pr \approx 2 \times 10^7$ , and the predictions are very close to the experimental results, as appearing in Fig. 5.20(b). The duct wall temperature variation was as large as  $3.0\text{-}3.7^\circ$  as indicated in section 4.4 at the largest  $Gr^+$ . This forced the experimental condition significantly moving away from the H1 toward the H2 condition. This can be confirmed by the increasing amount of the overestimation by the predictions with H1 condition which appears in both Figs. 5.20(a) and (b). It is probably appropriate to conclude that the wall temperature condition for these experiments was H1 at low Grashof number and the prediction with this condition is acceptable up to  $Gr^+Pr$  of  $10^8$ . The further numerical study can be for the H2 condition which should be useful if the application at a higher heating rate is concerned.

In regard to the effect of duct orientation, the experimental results in Chap. 4 show that the duct orientation significantly affected the magnitude of Nusselt number. At  $Gr^+Pr \approx 10^9$ , the Nusselt numbers for  $\alpha = 90^\circ$  and  $135^\circ$  are about 10% higher compared to  $\alpha = 45^\circ$  and  $180^\circ$  and about 40% higher compared to  $\alpha = 0^\circ$ . At  $Gr^+Pr \approx 2 \times 10^8$  (maximum  $Gr^+Pr$  values for the predictions), the predictions (i.e. Fig. 5.16) indicated that Nusselt numbers for  $\alpha = 90^\circ$  and  $135^\circ$  are about 7% larger in comparison to  $\alpha = 45^\circ$  and about 20% larger in comparison to  $\alpha = 0^\circ$  and  $180^\circ$ . The locations of the maximum Nusselt numbers which appeared at  $\alpha = 90^\circ$  and  $135^\circ$  for which the Nusselt number magnitudes are very similar agreed very well between the numerical and

experimental results. For  $\alpha = 45^\circ$ , the predicted value are about 7% lower than the maximum at  $Gr^+Pr \approx 2 \times 10^8$ . The 7% difference agrees very well with the experiments which shows the same difference between  $\alpha = 90^\circ$  and  $135^\circ$  and  $45^\circ$  at the same  $Gr^+Pr$ .

Although the predictions have shown the similar Nusselt number magnitudes for  $\alpha = 0^\circ$  and  $180^\circ$ , the measured Nusselt numbers for  $\alpha = 180^\circ$  are much higher. Actually, the Nusselt numbers for  $\alpha = 180^\circ$  were comparatively the same as those for  $\alpha = 45^\circ$ . This indicates that the  $\alpha = 180^\circ$  position is better than the  $0^\circ$  position for Nusselt number enhancement in practice. Considering the cross-stream flow pattern, the upward curved wall in the  $180^\circ$  orientation allows a smoother path for the rising of the hot fluid near the side wall. Furthermore, the corners which inhibit continuous circulation and the fluid velocity should be low in the corner vicinity. In case of  $\alpha = 0^\circ$ , some part of the hot fluid is virtually blocked in the corner. This is in contrast to  $\alpha = 180^\circ$  which the corners at the bottom that accommodate cooler fluid instead of hot fluid as in the case of  $\alpha = 0^\circ$ . Therefore the wall temperature differences of about  $4^\circ$ – $5^\circ$  C at  $Gr \approx 5 \times 10^7$  were measured for  $\alpha = 0^\circ$  as compared to about  $3^\circ$  for  $\alpha = 180^\circ$ . The mathematical model with uniform wall temperature as H1 boundary condition is certainly not able to imitate this difference and hence the nearly same Nusselt numbers were obtained both orientations.

As mentioned earlier in Chap. 4 for  $fRe$ , the experimental results should be considered as the average between the thermal developing and fully developed region while the numerical  $fRe$  results are for the fully developed condition. These results are compared in Fig. 5.21 for  $\alpha = 45^\circ$ ,  $90^\circ$  and  $135^\circ$ . For  $\alpha = 45^\circ$  and  $90^\circ$ , the differences

are within about 40% and somewhat larger for  $\alpha = 135^\circ$ . The experimental results show that the  $fRe$  were affected by the buoyancy force at  $Gr^+ > 2 \times 10^7$  while the numerical results show this effect since the  $Gr^+$  is much smaller. These results indicate that the  $fRe$  increases when the cross-stream flow becomes stronger similar to the Nusselt number as discussed earlier in this chapter. In addition, the overestimation of  $fRe$  is also consistent with the Nusselt number predictions when the amount of overestimation increases with a further departing from H1 condition of the peripheral wall temperature.

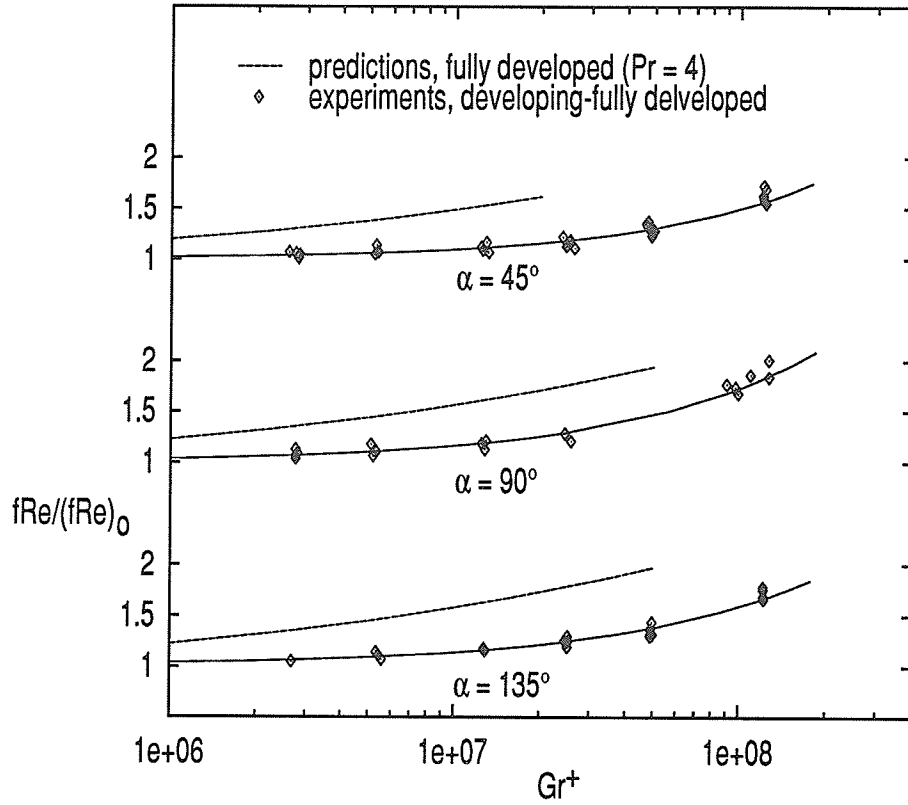


Fig. 5.21 Comparison of the two  $fRe/(fRe)_0$  data.

## Chapter 6

### Circular Sector Ducts

Circular sector ducts are a family of ducts consisting of a countless number of circular sector cross-sectional shapes according to various possible apex angles. The apex angle can be used as the identification for each duct. The flow and heat transfer characteristics of circular sector ducts can vary widely according over the wide range of apex angles. As an element in a multipassage tube, the apex angle is limited to  $180^\circ$ , therefore larger apex angles are omitted. The analyses in this chapter are basically for apex angles of  $60^\circ$ ,  $90^\circ$  and  $120^\circ$  in addition to the  $180^\circ$  (semicircular duct) presented in the last chapter. These apex angles correspond to internally finned tubes with equispaced, full fin, with 6, 4, 3 and 2 passages. The combined convection characteristics, H1 thermal boundary condition, under the influence of the gravitational orientation for the  $60^\circ$ ,  $90^\circ$  and  $120^\circ$  ducts are presented with the data for  $Pr = 4$ . For a given orientation with respect to the gravity vector, the  $Nu$  and  $fRe$  differ due to the shapes of the duct cross section which depends on apex angles. The comparative results in term of the apex angle ( $2\phi$ ) for various orientations ( $\alpha$ ) are presented to demonstrate such

results. The effect of Prandtl number is investigated by the association of the data for  $Pr = 10$  and  $50$ . Lastly, the unique correlations for  $Nu$  and  $fRe$  in term of the  $Gr^+Pr$  and  $Gr^+/Pr^2$  similar to the last chapter are presented.

## 6.1 Results for $Pr = 4$

The circular sector duct problems were solved by the same method as described in chapter 5. The accuracy of the solutions were tested similarly to the case of the semicircular duct except that all the problems were solved by the full region solution which experienced no difficulty of convergence. The solutions for  $Gr^+ = 0$  were compared to the exact results [35, 36] and the agreements were within about 0.1% for the Nusselt number for all ducts. For  $fRe$  the agreements were 0.26% for the  $60^\circ$  duct and 0.15% and 0.16% for the  $90^\circ$  and  $120^\circ$  ducts. Meshes of  $30 \times 30$ ,  $30 \times 40$  and  $30 \times 40$  were chosen for the  $60^\circ$ ,  $90^\circ$  and  $120^\circ$  ducts respectively according to the two reasons; first, the radial grid of 30 was consistent for all ducts including the semicircular duct, secondly, both  $Nu$  and  $fRe$  were independent of grid sizes with about 3% discrepancies at  $Gr^+ = 10^7$  as the meshes reached  $40 \times 50$ ,  $40 \times 60$ , and  $40 \times 60$  for the  $60^\circ$ ,  $90^\circ$  and  $120^\circ$  ducts respectively.

**Table 6.1**  $Nu$  and  $fRe$  for the  $60^\circ$  duct with  $Pr = 4$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 90^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	14.134	3.476	14.134	3.476	14.134	3.476
$5 \times 10^3$	14.137	3.494	14.137	3.494	14.136	3.494
$1 \times 10^4$	14.146	3.547	14.144	3.544	14.143	3.539
$5 \times 10^4$	14.281	4.140	14.262	4.093	14.243	4.039
$1 \times 10^5$	14.445	4.610	14.407	4.545	14.375	4.481
$5 \times 10^5$	15.175	6.034	15.119	6.034	15.091	6.018
$1 \times 10^6$	15.668	6.842	15.668	6.937	15.647	6.931
$5 \times 10^6$	17.489	9.476	17.772	9.859	17.692	9.786
$1 \times 10^7$	18.623	11.035	19.105	11.547	18.940	11.409
$2 \times 10^7$	20.020	12.909	20.728	13.515	20.432	13.338
$5 \times 10^7$	22.383	15.914	23.375	16.516	22.812	16.462

**Table 6.2**  $Nu$  and  $fRe$  for the  $90^\circ$  duct with  $Pr = 4$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 90^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	14.747	3.739	14.747	3.739	14.747	3.739
$5 \times 10^3$	14.758	3.813	14.756	3.802	14.756	3.808
$1 \times 10^4$	14.785	3.969	14.779	3.940	14.779	3.945
$5 \times 10^4$	15.034	4.875	15.022	4.857	14.994	4.786
$1 \times 10^5$	15.263	5.439	15.277	5.477	15.211	5.354
$5 \times 10^5$	16.194	7.183	16.446	7.506	16.202	7.222
$1 \times 10^6$	16.839	8.200	17.266	8.700	16.912	8.310
$5 \times 10^6$	19.113	11.464	20.111	12.464	19.384	11.688
$1 \times 10^7$	20.493	13.377	21.805	14.606	20.842	13.616
$2 \times 10^7$	22.177	15.683	23.800	17.103	22.562	15.915
$5 \times 10^7$	25.023	19.415	26.959	20.971	25.302	19.630

**Table 6.3**  $Nu$  and  $fRe$  for the  $120^\circ$  duct with  $Pr = 4$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 90^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	15.177	3.903	15.177	3.903	15.177	3.903
$5 \times 10^3$	15.198	4.041	15.193	4.014	15.196	4.036
$1 \times 10^4$	15.240	4.268	15.229	4.221	15.234	4.251
$5 \times 10^4$	15.542	5.293	15.570	5.367	15.520	5.260
$1 \times 10^5$	15.799	5.910	15.916	6.116	15.777	5.894
$5 \times 10^5$	16.849	7.859	17.390	8.482	16.899	7.955
$1 \times 10^6$	17.568	8.997	18.360	9.851	17.685	9.153
$5 \times 10^6$	20.031	12.606	21.610	14.129	20.402	12.922
$1 \times 10^7$	21.495	14.705	23.914	16.974	22.013	15.107
$2 \times 10^7$	23.305	17.257	26.058	19.713	23.942	17.719
$5 \times 10^7$	–	–	29.452	24.030	27.111	21.908

The  $Pr = 4$  is for water and therefore the solutions for this Prandtl number should find frequent applications. In addition, the solutions for  $Pr = 4$  for the  $60^\circ$ ,  $90^\circ$  and  $120^\circ$  ducts were required to complete the data for the study of the whole set of circular sector ducts for which the data for  $Pr = 4$  for the semicircular ducts were available already at that time. The  $Nu$  and  $fRe$  results for all ducts and  $\alpha = 0^\circ$ ,  $90^\circ$  and  $180^\circ$  are compiled in Table 6.1–6.3. In Fig. 6.1(a), (b) and (c)  $Nu/Nu_o$  is plotted versus  $Gr^+$ .

For the  $60^\circ$  duct, the Nusselt numbers for  $\alpha = 90^\circ$  and  $180^\circ$  are almost identical and are larger than those of  $\alpha = 0^\circ$ . Where  $Gr^+ = 2 \times 10^7$ , the difference is 4.7%. In contrast to the  $60^\circ$  duct, the  $90^\circ$  and  $120^\circ$  ducts have the nearly identical Nusselt numbers when  $\alpha = 0^\circ$  and  $180^\circ$ , similarly to the semicircular duct. For the  $90^\circ$  and  $120^\circ$  ducts, the  $90^\circ$  orientation shows the maximum Nusselt numbers while the  $0^\circ$  orientation shows the minimum Nusselt numbers for all range of  $Gr^+$ . The differences between them are 9% and 14% for the  $90^\circ$  and  $120^\circ$  ducts respectively.

The  $fRe/(fRe)_o$  are shown in Fig. 6.2. Similar patterns as for the case of Nusselt number can be observed except that  $fRe/(fRe)_o$  for  $180^\circ$  orientation of the  $60^\circ$  duct is somewhat lower than that of  $\alpha = 90^\circ$ . The differences between the maximum and minimum are 4%, 7%, and 12% at  $Gr^+ = 2 \times 10^7$ . For all ducts, the increases of  $fRe$  are less than 2 while it is about 5–6 for  $Nu$ .

The cross-stream flow patterns are shown for the  $90^\circ$  duct for  $\alpha = 0^\circ$ ,  $90^\circ$ , and  $180^\circ$  for  $Gr^+ = 10^5$  are shown in Fig. 6.3. The cross-stream flow comprises two cells similar to the flows in the semicircular duct in Chapter 5. The flow is induced by the

rising of the warm fluid near both side walls. The downward stream in the center of the cross section is due to the continuity and the descending of the cooler fluid. All the solutions conform this flow pattern consistently. The isotherms of the fluid are shown in Fig. 6.4 for  $Gr^+ = 10^6$  for the same duct with all  $\alpha$ . Enclosing the cooler fluid inside, the isotherms emphasize the cool core fluid region surrounded by the warmer fluid. The isotherms also illustrate the descending of the cooler fluid by the dwelling of the low temperature contours in the bottom part of the cross section. The crescent shapes of the top part of the temperature contours reflect the rising of the warm fluid near the side walls and the descending of the cool fluid in the middle.

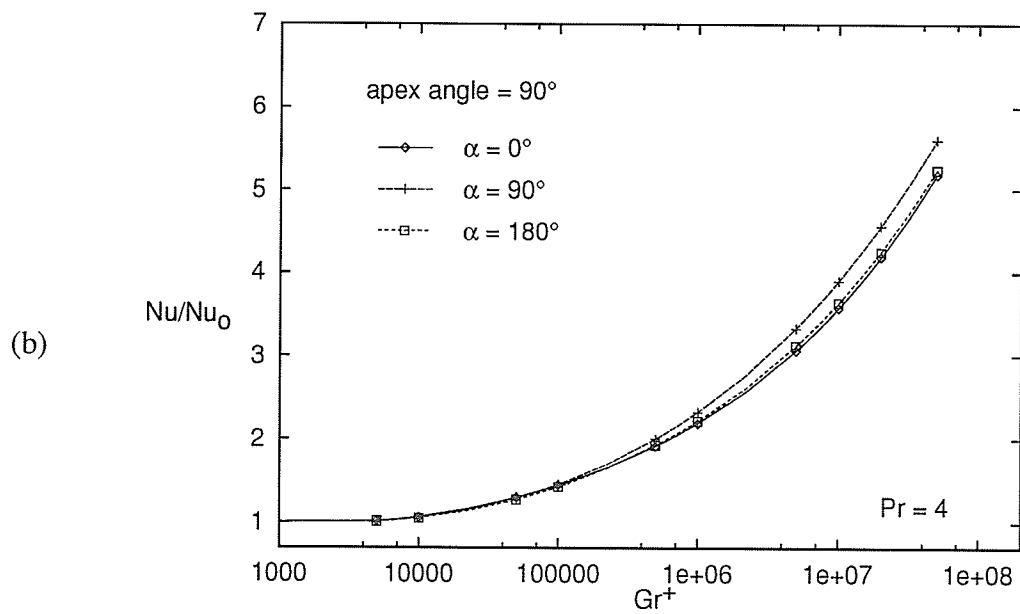
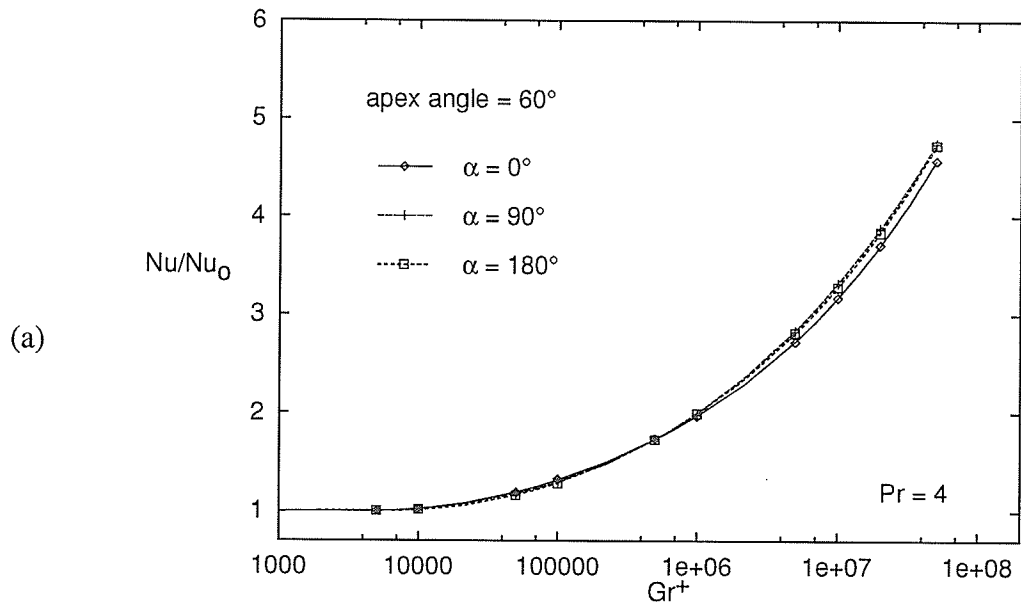


Fig. 6.1 Nusselt number for  $Pr = 4$  for the (a)  $60^\circ$ , (b)  $90^\circ$  ducts.

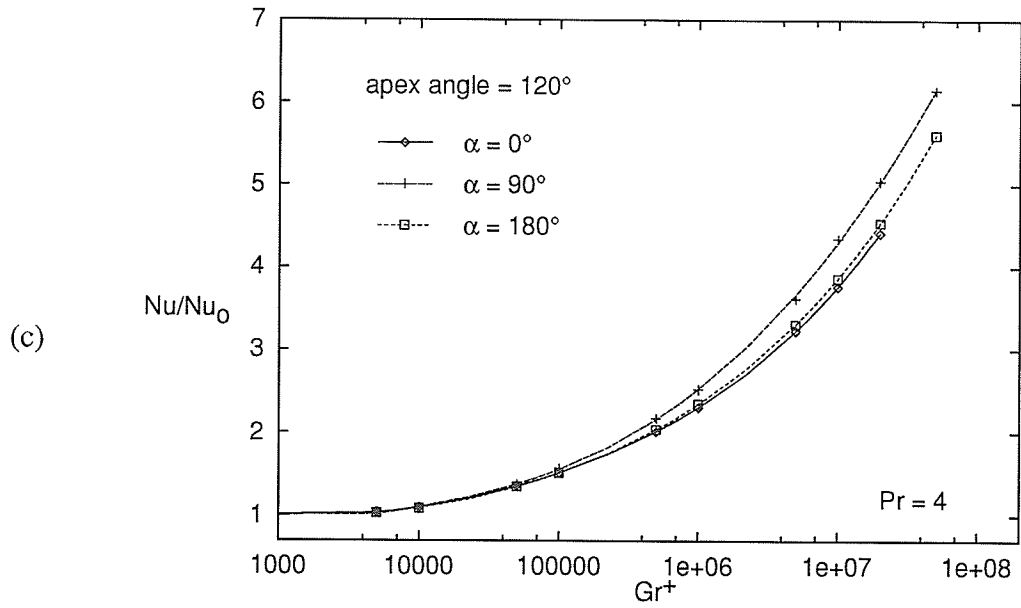


Fig. 6.1(cont) Nusselt number for  $Pr = 4$  for the (c)  $120^\circ$  ducts.

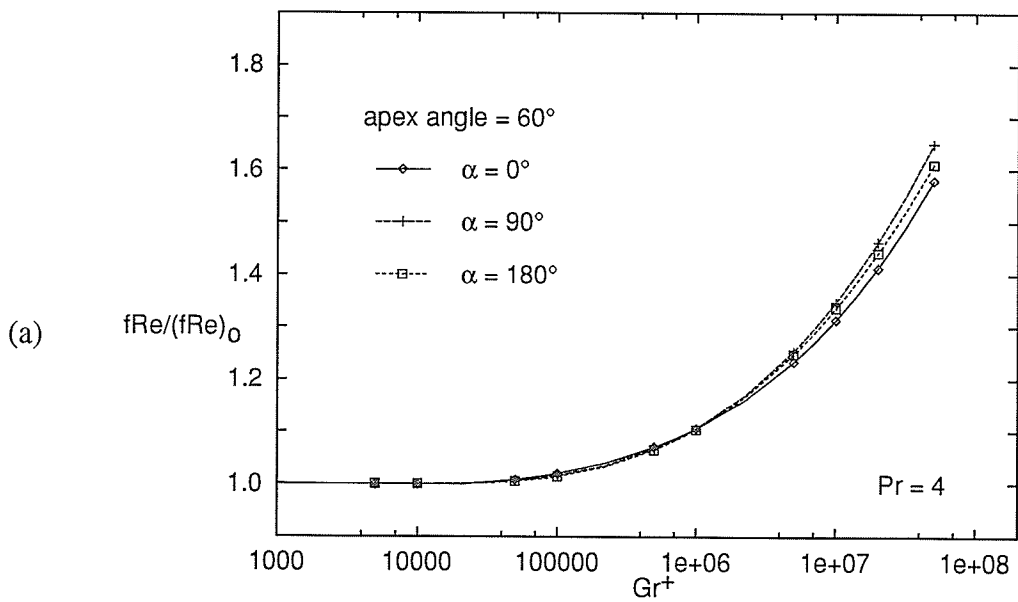


Fig. 6.2 The  $fRe$  for  $Pr = 4$  for the (a)  $60^\circ$  ducts.

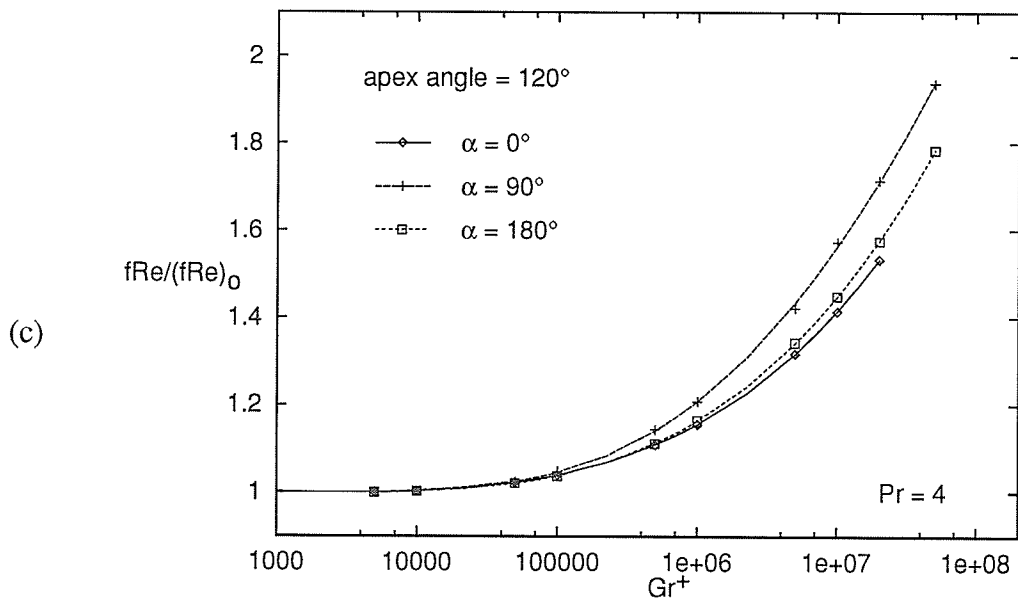
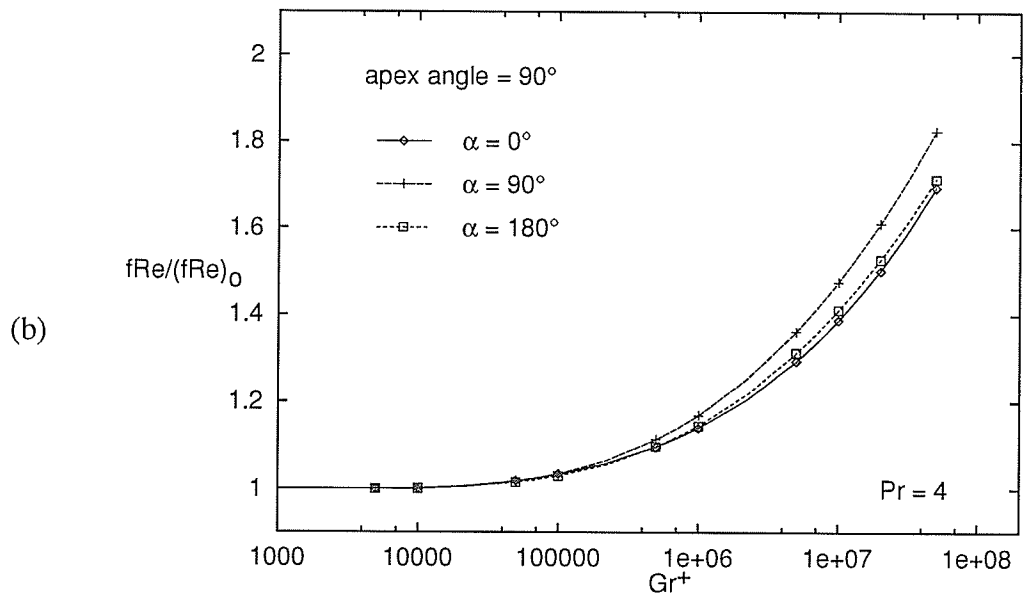
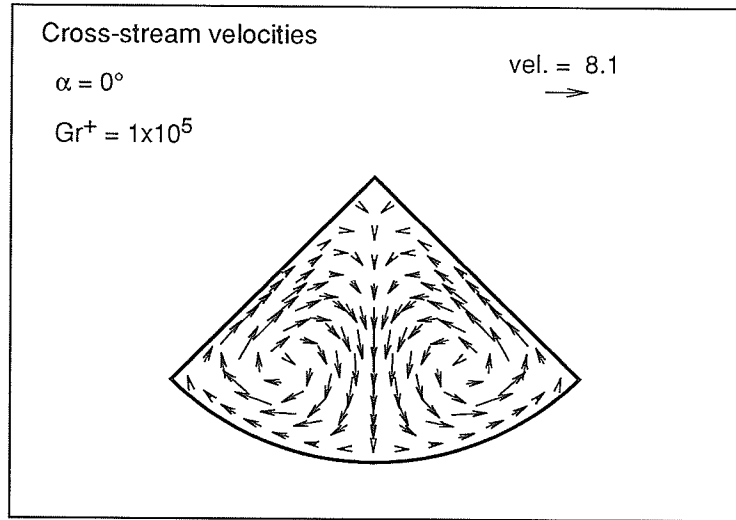
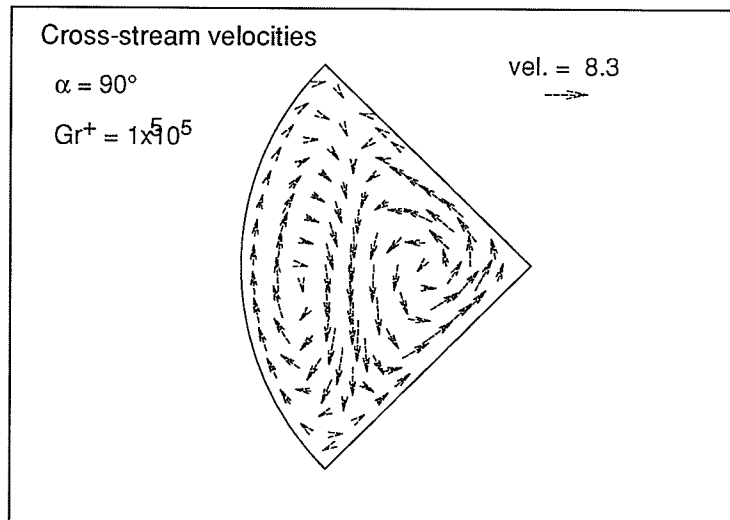


Fig. 6.2(cont) The  $fRe$  for  $Pr = 4$  for the (b)  $90^\circ$ , and (c)  $120^\circ$  ducts.

(a)



(b)



**Fig. 6.3** Cross-stream flow patterns for the  $90^\circ$  duct for  $\alpha =$  (a)  $0^\circ$ , (b)  $90^\circ$ .

(c)

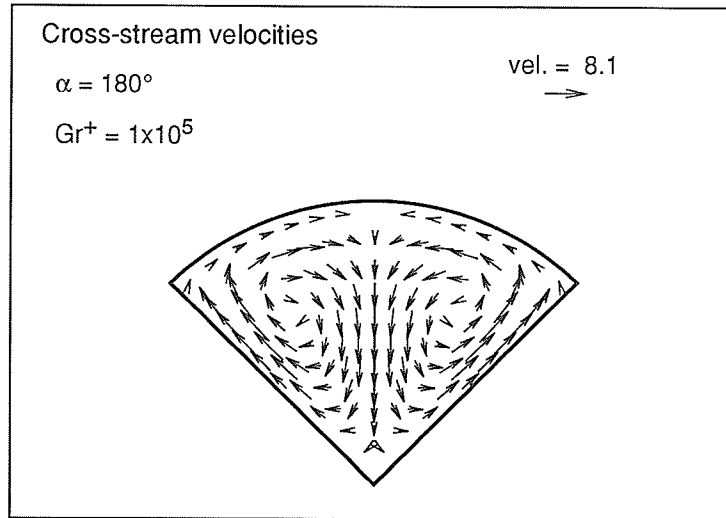


Fig. 6.3(cont) Cross-stream flow patterns for the  $90^\circ$  duct for  $\alpha =$  (c)  $180^\circ$ .

(a)

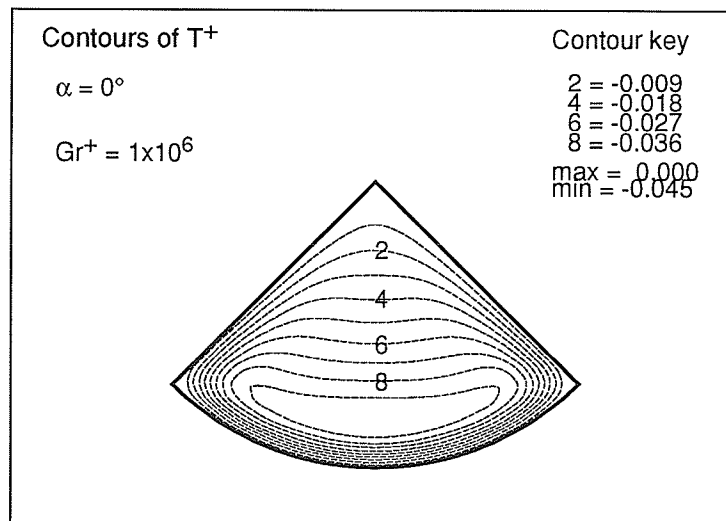
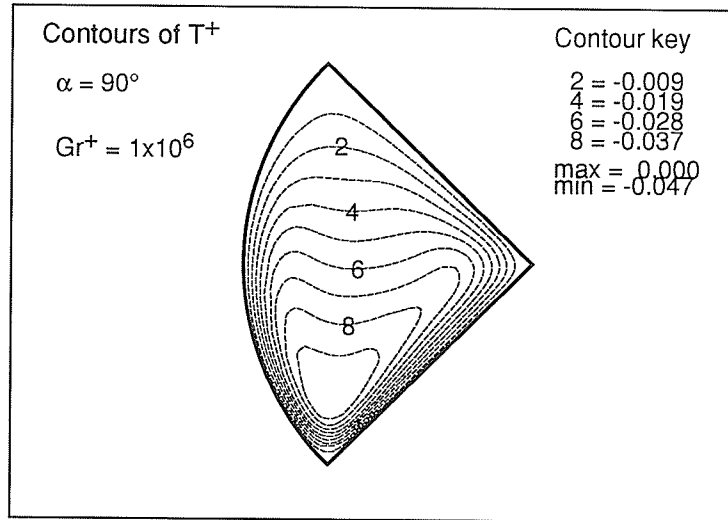


Fig. 6.4 The temperature contours for the  $90^\circ$  duct for  $\alpha =$  (a)  $0^\circ$ .

(b)



(c)

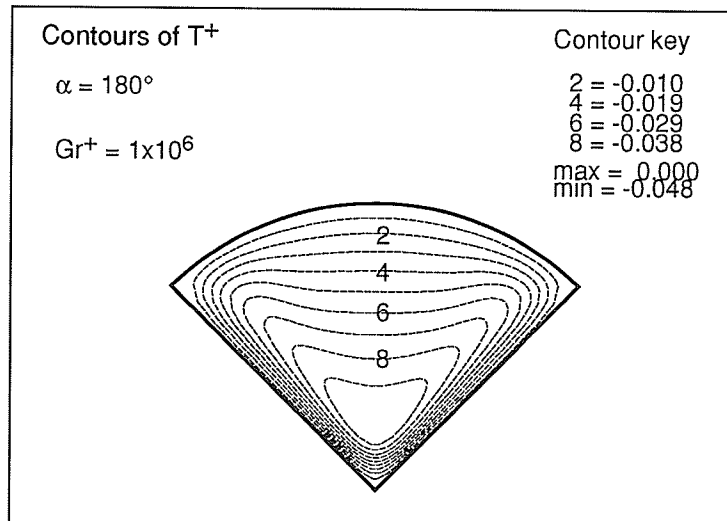


Fig. 6.4(cont) The temperature contours for the  $90^\circ$  duct for  $\alpha =$  (a)  $90^\circ$ , (b)  $180^\circ$ .

Buoyancy and orientation effects are of interest in this study. The buoyancy effects are reflected by the magnitudes of the  $Nu$  and  $fRe$  ratios, whereas the orientation effects can be observed by comparing these ratios for each  $\alpha$ . In other words, the orientation effect exhibits how much difference the buoyancy flow affects the magnitude of  $Nu$  for each  $\alpha$ . Considering the buoyancy effect, the maximum Nusselt number ratios are about 6.3, 6.2, 5.6, and 4.8 at  $Gr^+ = 5 \times 10^7$  for the  $180^\circ$ ,  $120^\circ$ ,  $90^\circ$ , and  $60^\circ$  ducts respectively whereas the corresponding  $fRe$  ratios are about 1.8–1.9 for all ducts. The decrease in Nusselt number ratio with decreasing apex angle suggests that the buoyancy effect is more prominent in broader cross sections.

The differences of Nusselt number ratios due to  $\alpha$  gradually decreases from about 20% for the semicircular duct to 14%, 9%, and 4.7% for the  $120^\circ$ ,  $90^\circ$ , and  $60^\circ$  ducts respectively. The monotonical decrease of these differences in Nusselt numbers also indicates that the combined convection is more effectively affected by the orientation in the broader cross sections. This suggests that the orientation effect should be negligibly small for ducts with apex angle smaller than  $60^\circ$ . In general, cross-stream circulation comprises two cells therefore the duct cross section should be adequately broad to allow the two cells to circulate effectively. A test was conducted using a very narrow duct with apex angle of  $10^\circ$  and the  $Nu/Nu_o$  ratio of 1.4 was observed at  $Gr^+ = 2 \times 10^7$  for  $Pr = 4$ . The Nusselt number ratios are almost the same for all orientations. Therefore the results confirm not only small orientation effect but also small buoyancy effect in very narrow ducts.

Further investigation was conducted using a  $30^\circ$  duct to ensure the above result.

The results for  $Pr = 4$  for the  $30^\circ$  duct using a  $30 \times 30$  mesh are shown in Table 6.4 and Fig. 6.3 for  $\alpha = 0^\circ, 90^\circ,$  and  $180^\circ$ . The maximum Nusselt number ratio is about 3.4 for  $\alpha = 180^\circ$  at  $Gr^+ = 5 \times 10^7$  compared to 6.3, 6.2, 5.6, and 4.8 for the  $180^\circ$  to  $60^\circ$  ducts respectively. Therefore the decrease of buoyancy effect with decreasing apex angle is again confirmed. Concerning the orientation effect, the maximum  $Nu$  was obtained from  $\alpha = 180^\circ$  which is about 10% higher than the others at  $Gr^+ = 2 \times 10^7$ . The Nusselt numbers for  $\alpha = 0^\circ$  and  $90^\circ$  are similar in magnitudes over the entire range of  $Gr^+$ . The orientation effect slightly deviates from the previous observed trend because the effect does not monotonically decrease as the apex angle decrease to smaller than  $60^\circ$ . This will be explained by the so much difference in shapes of the duct cross sections later.

Considering Fig. 6.3, the Nusselt numbers for all  $\alpha$  show only little differences until  $Gr^+$  reaches about  $5 \times 10^6$  that the  $\alpha = 180^\circ$  surpasses the others for Nusselt number magnitudes. The shifting from the minimum  $Nu$  to the maximum  $Nu$  of the  $180^\circ$  orientation actually appears earlier in the  $60^\circ$  duct and the behaviour becomes clear as apex angle decreases to  $30^\circ$ .

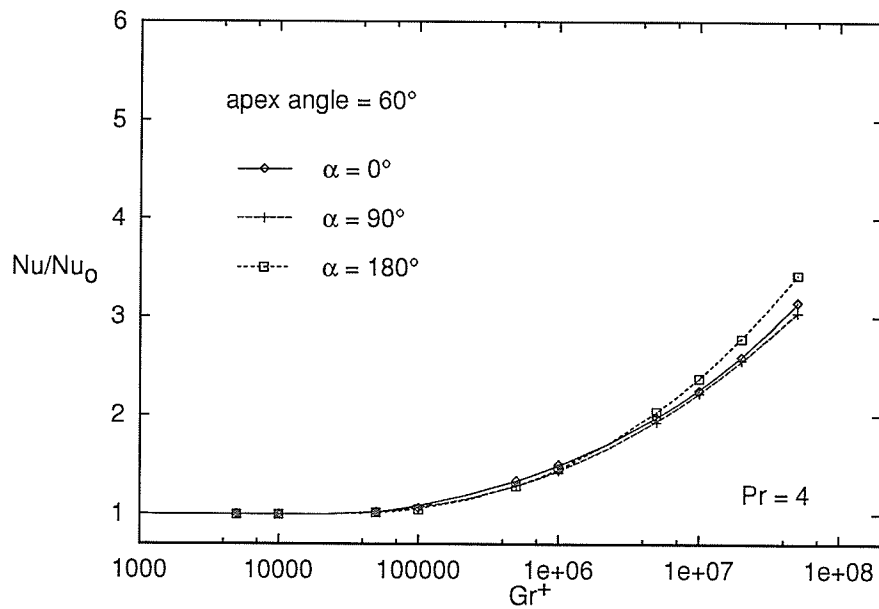
The cross-sectional shapes can be roughly classified as broad and narrow shapes for the apex angles larger and smaller than  $60^\circ$  respectively. The narrow cross sections are more like wedge shape and have rather short chord lengths compared to the radius. For the broad cross section ducts, the maximum Nusselt number is always achieved from the  $90^\circ$  orientation where the arc section becomes a side wall. The arc side wall accommodates one of the vortex cells with a rather smooth path for the rising of the warm fluid. The two flat walls on the other side of the duct cross sections accommodate

the other cell. This cell is formed by the rising of warm fluid from the lower flat wall to the higher flat wall. The size of the apex angle therefore directly controls the flow pattern of this cell.

For the narrow cross section, observed from the case of apex angle of  $30^\circ$ , the  $\alpha$  of  $90^\circ$  is the worst orientation because both the flat walls are rather horizontal and hence there is practically no side wall in the apex vicinity. Flowing along side walls allows the fluid to continuously absorb heat energy and keep rising. For  $\alpha = 180^\circ$ , the arc top wall provides space for the warm fluid to return to the bottom. The room on the top is important because the flows induced by the side walls need spaces for making turn. The apex at the bottom which does not provide much room for the cool fluid is an additional advantage as discussed earlier in Chapter 5. Since the  $0^\circ$  orientation is the reverse of the  $180^\circ$  orientation, both the advantages for  $\alpha = 180^\circ$  become the disadvantages for  $\alpha = 0^\circ$ . The shapes of the flow domains for the  $30^\circ$  duct drastically change according to the orientation therefore the Nusselt number varied in a larger amount compared to the  $60^\circ$  duct.

Generally, in a sharp apex vicinity, both the axial and cross-sectional fluid velocities are low therefore there is a limit area that allows continuous cross-stream circulation. The awkward increase in Nusselt number magnitude with the increasing  $Gr^+$  indicates the limit buoyancy effect in the narrower duct. For example, the Nusselt number ratios of about 1.3 is reached at  $Gr^+$  about  $5 \times 10^5$ ,  $1 \times 10^5$ ,  $5 \times 10^4$ , and less than  $5 \times 10^4$  for the  $30^\circ$  to  $180^\circ$  respectively. Furthermore, the orientation effect is typically smaller in the smaller duct. This can be clearly observed for the  $60^\circ$ – $120^\circ$  in Fig. 6.1.

The orientation effect is large for the  $30^\circ$  duct only at very high  $Gr^+$ . At an intermediate  $Gr^+$  value of  $5 \times 10^6$ , the maximum differences of Nusselt numbers for each orientation are about 5%, 4%, 8%, 11%, and 16% for the  $30^\circ$  to  $180^\circ$  respectively. The  $180^\circ$  orientation for the  $30^\circ$  duct begins to behave differently from other orientations as  $Gr^+$  reaches  $5 \times 10^6$ . The limit orientation effect in the  $30^\circ$  duct is overcome because the cross-stream current is adequately strong at high heating rates in addition to the exceptional difference in the shapes of the flow areas between  $\alpha = 180^\circ$  and the others.



**Fig. 6.5** Nusselt number for  $Pr = 4$  for the  $30^\circ$  ducts, for  $\alpha = 0^\circ, 90^\circ$ , and  $180^\circ$ .

**Table 6.4**  $Nu$  and  $fRe$  for the  $30^\circ$  duct for  $Pr = 4$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 90^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	13.276	2.997	13.276	2.997	13.276	2.997
$5 \times 10^3$	13.276	2.998	13.276	2.998	13.276	3.000
$1 \times 10^4$	13.277	2.999	13.276	3.000	13.276	3.000
$5 \times 10^4$	13.287	3.059	13.287	3.066	13.282	3.050
$1 \times 10^5$	13.315	3.198	13.310	3.193	13.298	3.153
$5 \times 10^5$	13.599	4.022	13.505	3.874	13.475	3.853
$1 \times 10^6$	13.870	4.512	13.681	4.316	13.677	4.374
$5 \times 10^6$	14.837	5.939	14.463	5.803	14.671	6.124
$1 \times 10^7$	15.435	6.767	15.054	6.681	15.385	7.143
$2 \times 10^7$	16.230	7.794	15.850	7.686	16.299	8.351
$5 \times 10^7$	17.528	9.449	17.225	9.135	17.844	10.289

## 6.2 Comparisons Among the Circular Sector Ducts

Nusselt numbers for the  $60^\circ$ ,  $90^\circ$ ,  $120^\circ$  and  $180^\circ$  ducts are shown in Fig. 6.3(a) and (b) for  $\alpha = 0^\circ$  and  $180^\circ$ . For a given  $Gr^+$  in both Figs.,  $Nu$  increases as the apex angle increases. Considering the difference of  $Nu$  for a duct compared to the duct of next higher  $\phi$ , the difference is largest for  $2\phi$  of  $30^\circ$  and  $60^\circ$ . In high  $Gr^+$  region, the difference decreases as the apex angle increases. Small difference for the  $120^\circ$  and  $180^\circ$  ducts, especially at large  $Gr^+$ , can be seen for both orientations.

The results show that the narrower duct cross-section yields a smaller Nusselt number. This trend is also true for mixed convection in rectangular ducts and natural convection in rectangular cavities. For mixed convection in rectangular ducts [28], Nusselt number is maximum when the duct aspect ratio of the cross section is 1 and decreases when duct cross-section becomes narrower. The minimum Nusselt number is reached when aspect ratio is zero or infinity.

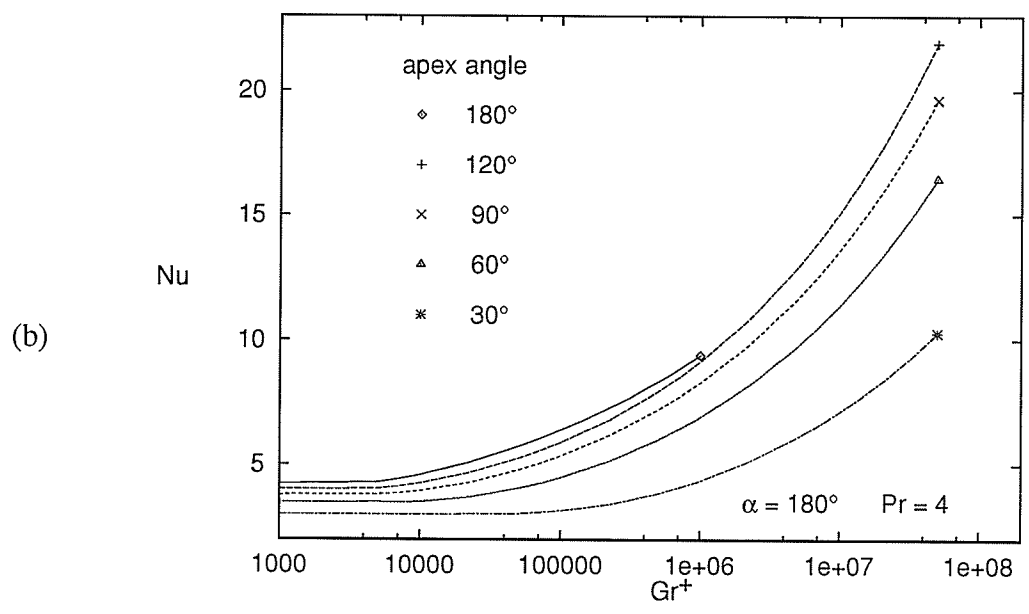
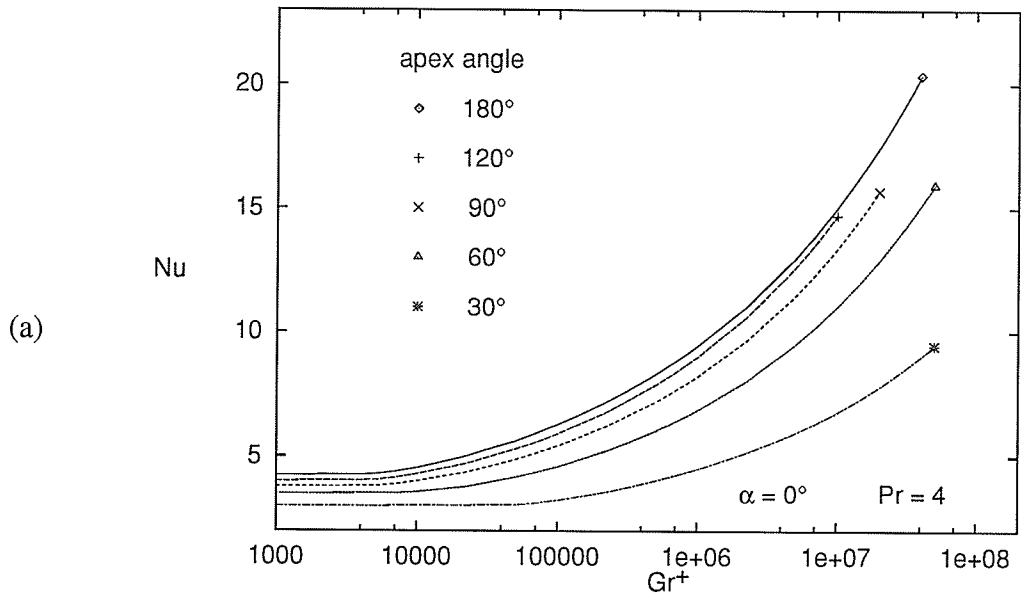
Since the  $Nu$  magnitudes for the  $120^\circ$  and  $180^\circ$  ducts are comparatively the same while the sizes of their cross sections are actually much different, the  $Nu$  figure seems to fail in communicating the difference in cross-section in this case. In practice, a better heat transfer performance should lower duct wall temperature toward fluid bulk temperature. The comparison among the ducts with different sizes as in this case can be done by comparing the duct wall temperature drop rather than Nusselt number because it is defined based on the hydraulic diameter which varies according to duct sizes. The

Nusselt number is independent of duct sizes if the eqn. (3.19) is modified as

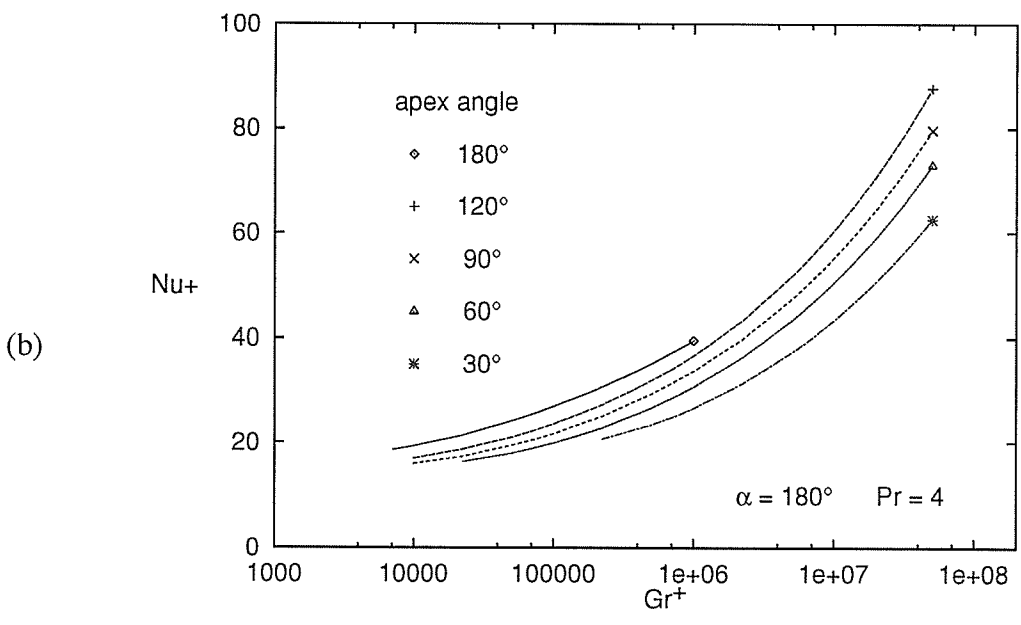
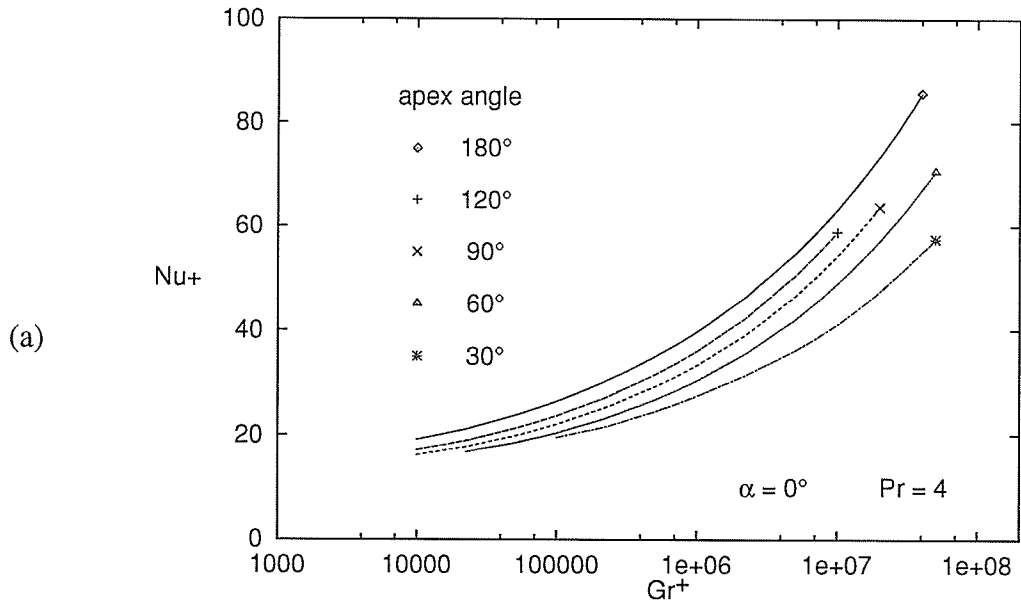
$$Nu^+ = -\frac{1}{T_m^+} = \frac{S}{D_h} Nu \quad (6.1)$$

where  $Nu^+$  is a modified Nusselt number and  $S$  and  $D_h$  are duct wet perimeter and hydraulic diameter. The  $Nu$  data are transformed to  $Nu^+$  by the ratios  $D_h/S$  which are 0.1644, 0.2256, 0.2464, 0.2499 and 0.2377 for the  $30^\circ$  to  $180^\circ$  ducts respectively. For apex angle less than about  $110^\circ$ , the ratio  $D_h/S$  decreases monotonically with decreasing apex angle. This makes the ratio  $D_h/S$  increase with the decrease of apex angle. The ratio  $D_h/S$  approaches zero as the apex angle reaches zero. Since the Nusselt numbers for pure forced convection in circular sector ducts with small apex angles have magnitudes roughly of 2, the magnitudes of  $Nu^+$  become larger than the larger ducts. For the limiting case of zero apex angle,  $Nu^+$  becomes infinity and hence the duct wall temperature is equal to fluid temperature. This is theoretically true because the temperature gradient within the fluid body disappears as the fluid has infinitesimal thickness. For pure forced convection solutions, the magnitudes of  $Nu^+$  decreases as apex angle increases from zero and becomes approximately stable when apex angle is larger than about  $60^\circ$ . The  $Nu^+$  behaviour for small apex angles with pure forced convection condition in which the buoyancy flow is ignored seems to be ambiguous. Therefore the  $Nu^+$  is plotted versus  $Gr^+$  only when the buoyancy effect is significantly large. The  $Nu^+$  is plotted in Fig. 6.6 with the Nusselt number data that are about 10% larger than the corresponding pure forced convection solutions.

In Figs. 6.6(a) and (b), larger differences between the curves for the 120° and 180° ducts can be seen while the spaces between the adjacent curves become uniform. The  $Nu^+$  of the 180° duct compared to the 60° duct is about 28% higher for  $\alpha = 0^\circ$  at  $Gr^+ = 10^7$ , and about 29% higher for  $\alpha = 180^\circ$  at  $Gr^+ = 10^6$ . Since  $Gr^+$  is defined using the duct radius which is the same for all the ducts, it represents the heat input regardless the duct size. Therefore those figures suggest that for a given heat input and the same fluid conditions, the wall temperature of the 180° duct will be about 30% lower than the 60° duct. Based on the 30° duct, at  $Gr^+ = 10^7$ , the differences of  $Nu^+$  are about 19%, 32%, 43%, and 54% for the 60° to 180° respectively.



**Fig. 6.6**  $Nu$  for circular sector ducts, apex angle 30°–180°,  $\alpha =$  (a) 0°, and (b) 180°.



**Fig. 6.7**  $Nu^+$  for circular sector ducts, apex angle 30°–180°,  $\alpha =$  (a) 0°, and (b) 180°.

**Table 6.5**  $Nu$  and  $fRe$  for the  $60^\circ$  duct with  $Pr = 10$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 90^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	14.134	3.476	14.134	3.476	14.134	3.476
$5 \times 10^3$	14.137	3.581	14.137	3.575	14.136	3.567
$1 \times 10^4$	14.144	3.782	14.142	3.759	14.141	3.731
$5 \times 10^4$	14.205	4.784	14.195	4.713	14.187	4.643
$1 \times 10^5$	14.266	5.372	14.250	5.308	14.240	5.252
$5 \times 10^5$	14.542	7.192	14.534	7.261	14.530	7.233
$1 \times 10^6$	14.755	8.267	14.770	8.428	14.768	8.381
$2 \times 10^6$	15.055	9.583	15.108	9.840	15.099	9.748
$5 \times 10^6$	15.625	11.766	15.757	12.135	15.709	11.962
$1 \times 10^7$	16.232	13.804	16.437	14.204	16.315	14.019
$2 \times 10^7$	17.048	16.181	17.317	16.531	17.059	16.476
$5 \times 10^7$	18.546	19.780	18.877	19.890	–	–

**Table 6.6**  $Nu$  and  $fRe$  for the  $90^\circ$  duct with  $Pr = 10$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 90^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	14.747	3.739	14.747	3.739	14.747	3.739
$5 \times 10^3$	14.756	4.049	14.754	4.012	14.754	4.015
$1 \times 10^4$	14.770	4.403	14.767	4.359	14.766	4.335
$5 \times 10^4$	14.864	5.654	14.871	5.705	14.850	5.563
$1 \times 10^5$	14.945	6.369	14.974	6.514	14.931	6.311
$5 \times 10^5$	15.306	8.644	15.468	9.121	15.318	8.697
$1 \times 10^6$	15.576	9.982	15.840	10.639	15.612	10.067
$2 \times 10^6$	15.943	11.598	16.345	12.454	16.008	11.700
$5 \times 10^6$	16.621	14.271	17.249	15.380	16.715	14.360
$1 \times 10^7$	17.325	16.774	18.129	18.022	17.322	16.838
$2 \times 10^7$	18.260	19.724	19.207	21.049	18.260	19.788

**Table 6.7**  $Nu$  and  $fRe$  for the  $120^\circ$  duct with  $Pr = 10$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 90^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	15.177	3.903	15.177	3.903	15.177	3.903
$5 \times 10^3$	15.191	4.371	15.189	4.320	15.189	4.348
$1 \times 10^4$	15.210	4.781	15.208	4.760	15.206	4.746
$5 \times 10^4$	15.318	6.149	15.357	6.384	15.311	6.134
$1 \times 10^5$	15.409	6.943	15.501	7.336	15.405	6.966
$5 \times 10^5$	15.808	9.475	16.143	10.338	15.838	9.624
$1 \times 10^6$	16.095	10.949	16.603	12.072	16.162	11.166
$2 \times 10^6$	16.476	12.722	17.209	14.134	16.598	13.024
$5 \times 10^6$	17.155	15.640	18.259	17.436	17.388	16.076
$7 \times 10^6$	17.473	16.917	18.721	18.832	17.751	17.393
$1 \times 10^7$	17.854	18.392	19.259	20.430	18.188	18.911
$2 \times 10^7$	–	–	20.462	23.891	19.223	22.233

**Table 6.8**  $Nu$  and  $fRe$  for the  $60^\circ$  duct with  $Pr = 50$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 90^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	14.134	3.476	14.134	3.476	14.134	3.476
$5 \times 10^3$	14.136	4.286	14.135	4.229	14.135	4.163
$1 \times 10^4$	14.137	4.797	14.137	4.723	14.136	4.647
$5 \times 10^4$	14.147	6.357	14.145	6.329	14.144	6.285
$1 \times 10^5$	14.155	7.258	14.153	7.290	14.152	7.250
$5 \times 10^5$	14.193	10.211	14.193	10.397	14.192	10.285
$1 \times 10^6$	14.224	11.971	14.227	12.198	14.224	12.035
$2 \times 10^6$	14.273	14.082	14.277	14.297	14.271	14.137
$5 \times 10^6$	14.383	17.406	14.386	17.472	14.361	17.576
$1 \times 10^7$	14.526	20.241	14.518	20.078	14.459	20.752
$2 \times 10^7$	14.754	23.191	–	–	–	–

**Table 6.9**  $Nu$  and  $fRe$  for the  $90^\circ$  duct with  $Pr = 50$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 90^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	14.747	3.739	14.747	3.739	14.747	3.739
$5 \times 10^3$	14.749	5.063	14.749	5.051	14.749	4.962
$1 \times 10^4$	14.752	5.675	14.752	5.722	14.751	5.576
$5 \times 10^4$	14.764	7.595	14.768	7.890	14.762	7.573
$1 \times 10^5$	14.774	8.724	14.783	9.163	14.773	8.734
$5 \times 10^5$	14.821	12.362	14.853	13.190	14.821	12.387
$1 \times 10^6$	14.858	14.511	14.910	15.492	14.858	14.510
$2 \times 10^6$	14.913	17.100	14.990	18.180	14.910	17.071
$5 \times 10^6$	15.034	21.216	15.149	22.349	15.010	21.234
$7 \times 10^6$	15.099	22.892	15.228	24.046	15.060	22.999
$1 \times 10^7$	–	–	15.329	25.920	–	–

**Table 6.10**  $Nu$  and  $fRe$  for the  $120^\circ$  duct with  $Pr = 50$ .

$Gr^+$	$\alpha = 0^\circ$		$\alpha = 90^\circ$		$\alpha = 180^\circ$	
	$fRe$	$Nu$	$fRe$	$Nu$	$fRe$	$Nu$
0	15.177	3.903	15.177	3.903	15.177	3.903
$5 \times 10^3$	15.180	5.502	15.180	5.599	15.179	5.465
$1 \times 10^4$	15.183	6.172	15.185	6.404	15.182	6.153
$5 \times 10^4$	15.196	8.305	15.208	8.931	15.195	8.383
$1 \times 10^5$	15.207	9.552	15.228	10.395	15.207	9.687
$5 \times 10^5$	15.254	13.532	15.324	14.986	15.260	13.863
$1 \times 10^6$	15.289	15.871	15.398	17.589	15.302	16.317
$2 \times 10^6$	15.339	18.711	15.502	20.643	15.364	19.250
$5 \times 10^6$	–	–	15.702	25.438	15.495	23.882
$7 \times 10^6$	–	–	15.799	27.404	15.565	25.782
$1 \times 10^7$	–	–	15.921	29.586	15.660	27.892

## 6.3 Results for $Pr = 10$ and $50$

The  $Nu$  and  $fRe$  results according to oriented angles of  $0^\circ$ ,  $90^\circ$  and  $180^\circ$  are in Table 6.5–6.7 for  $Pr = 10$  and Table 6.8–6.10 for  $Pr = 50$ , for the  $60^\circ$ ,  $90^\circ$  and  $120^\circ$  ducts. The results for all the ducts are also shown in Fig. 6.8 for  $Pr = 10$  and Fig. 6.9 for  $Pr = 50$ . For each duct, the differences between the maximum and minimum Nusselt numbers decreases with the increase of  $Pr$  as can be seen by the closer spaces of the curves in Fig. 6.8 comparing to Fig 6.1 (same vertical scales, except for the  $60^\circ$  duct). This agrees with the discussion in the second last paragraph of Section 6.1; the more viscous fluid reduces the flexibility for the modification of the cross-stream flow by the duct orientation. Therefore the magnitude of the Nusselt number swings within a narrower range. The differences between the maximum and minimum Nusselt numbers for  $Pr = 50$  are 2%, 6.5% and 10% at  $Gr^+ = 2 \times 10^6$  for the  $60^\circ$ ,  $90^\circ$  and  $120^\circ$  ducts. For  $Pr = 10$ , the results are 3%, 7% and 11% for the same conditions.

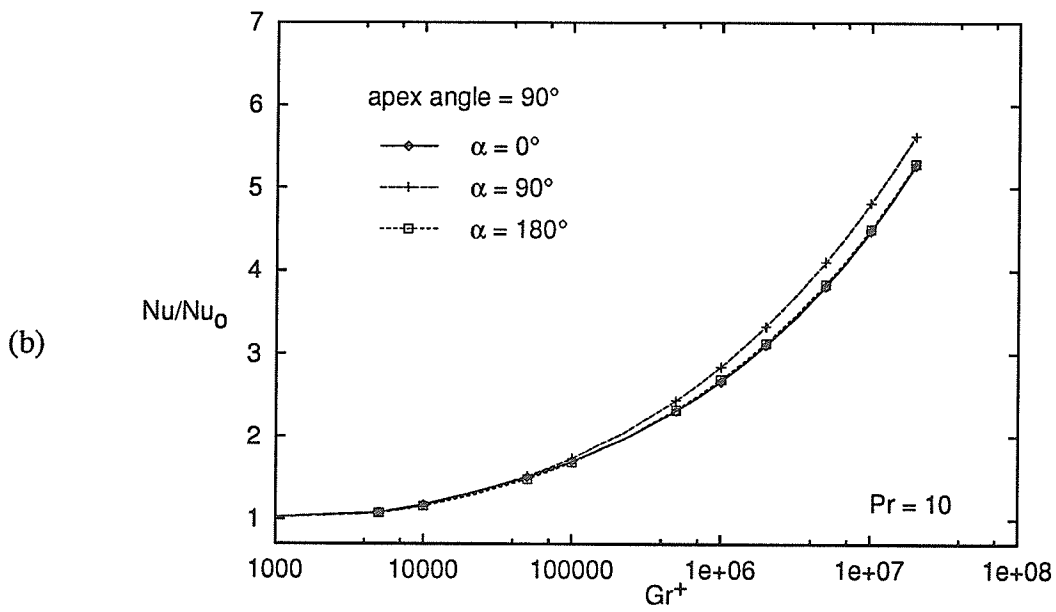
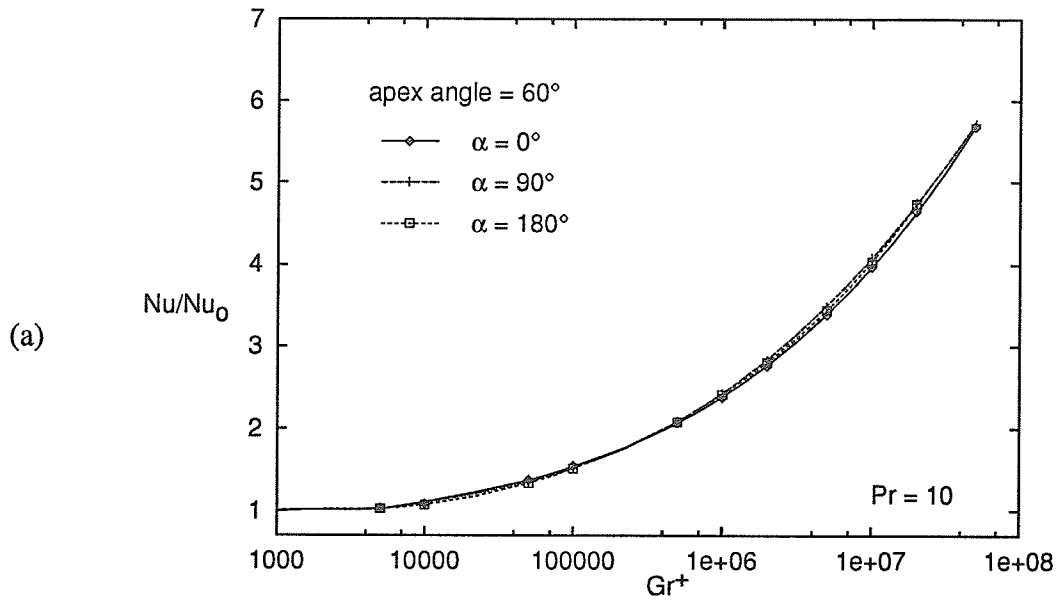


Fig. 6.8  $Nu$  ratios for  $Pr = 10$ , apex angle of (a)  $60^\circ$ , (b)  $90^\circ$ .

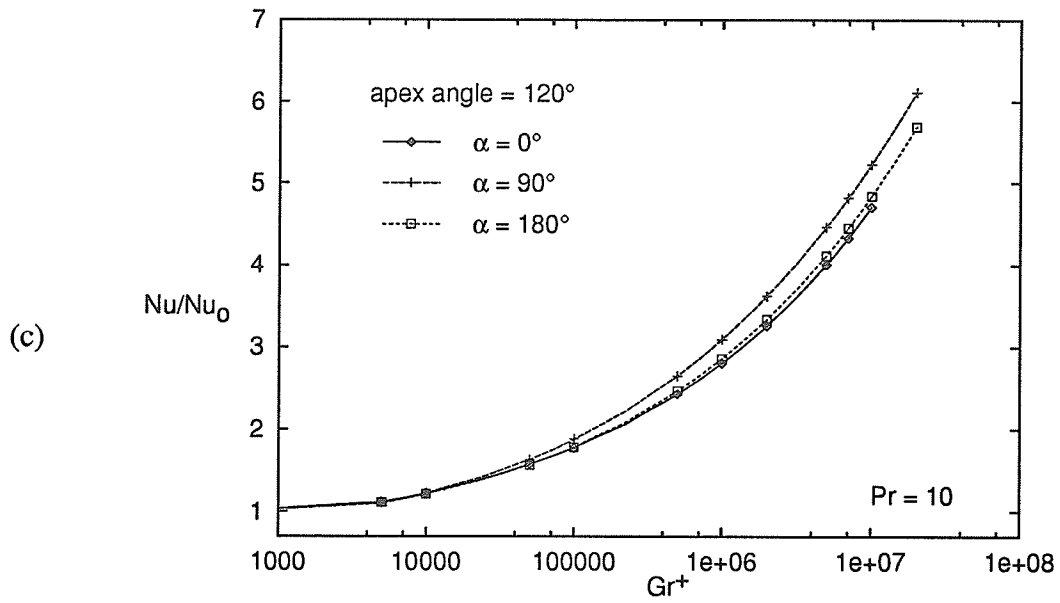


Fig. 6.8(cont)  $Nu$  ratios for  $Pr = 10$ , apex angle of (c)  $120^\circ$ .

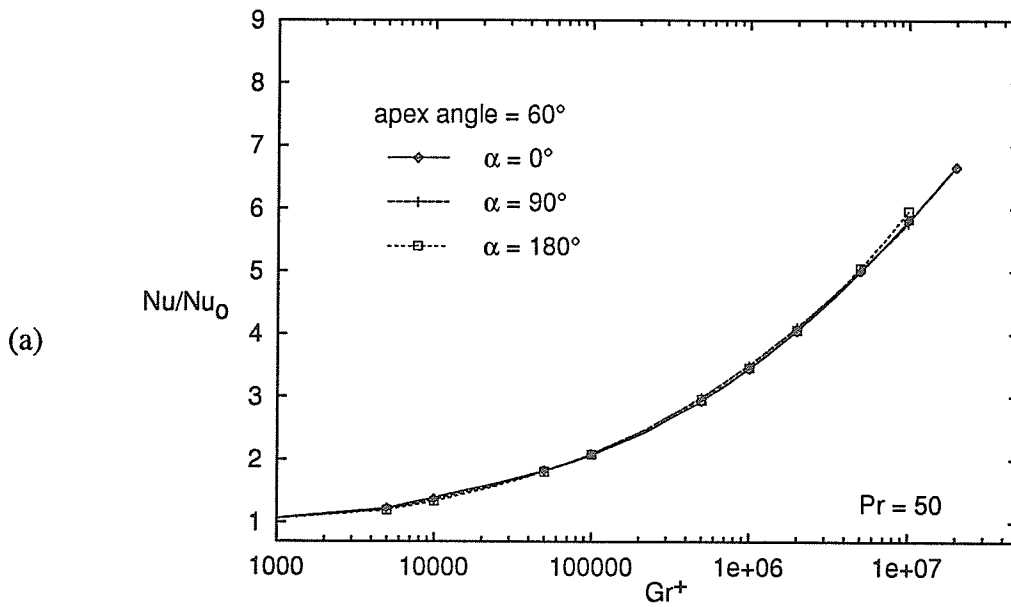


Fig. 6.9  $Nu$  ratios for  $Pr = 50$ , apex angle of (a)  $60^\circ$ .

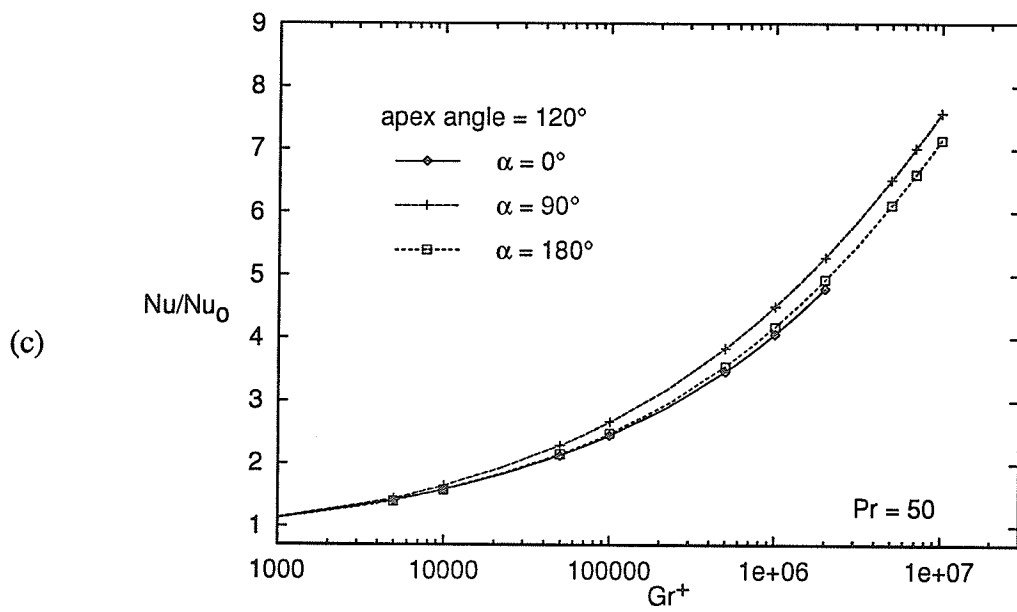
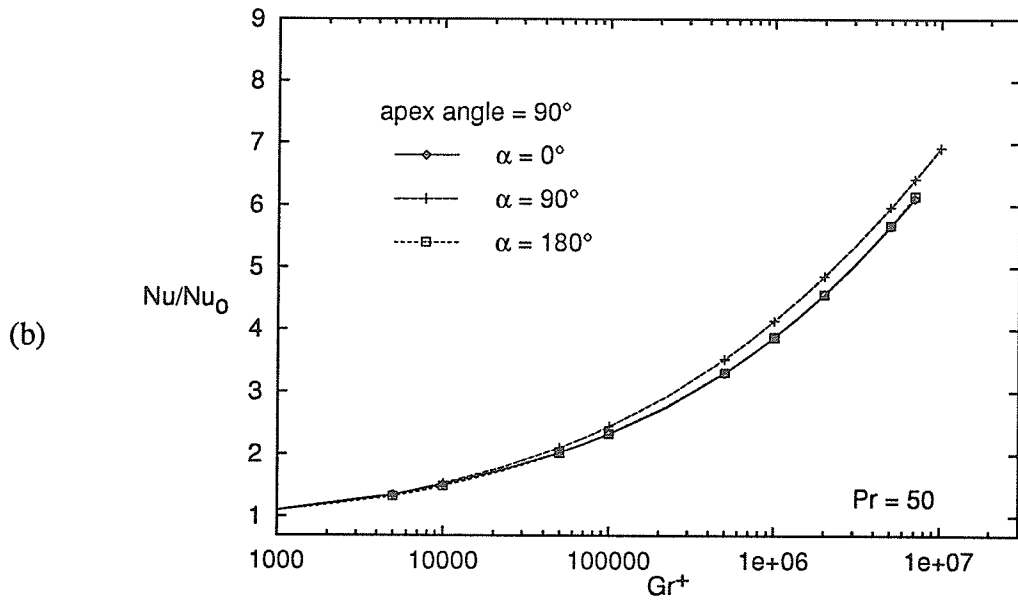


Fig. 6.9(cont)  $Nu$  ratios for  $Pr = 50$ , apex angle of (b)  $90^\circ$ , and (c)  $120^\circ$ .

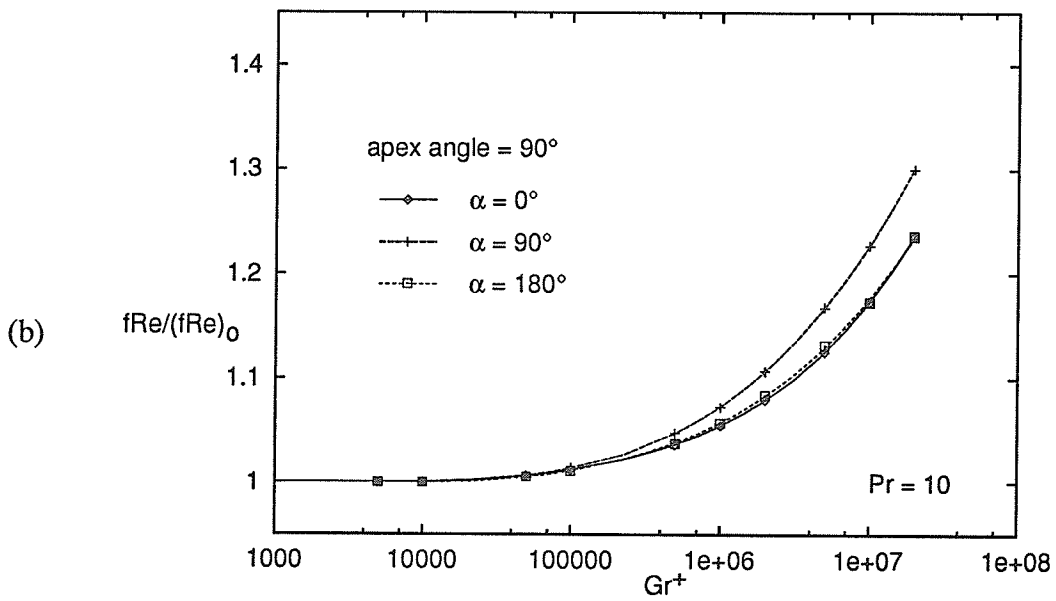
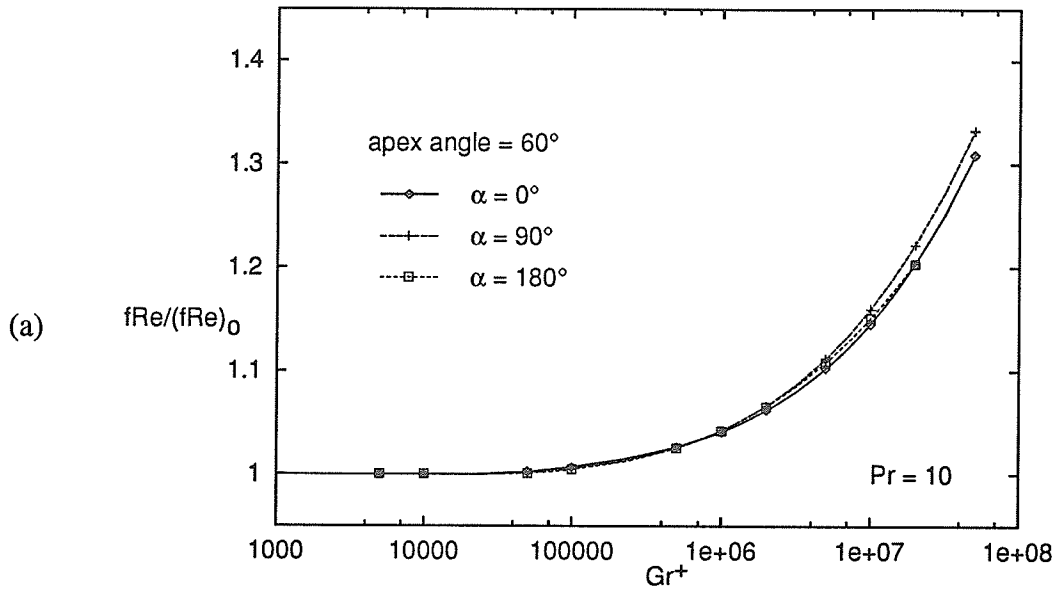


Fig. 6.10  $fRe$  ratios for  $Pr = 10$ , apex angle of (a)  $60^\circ$ , (b)  $90^\circ$ .

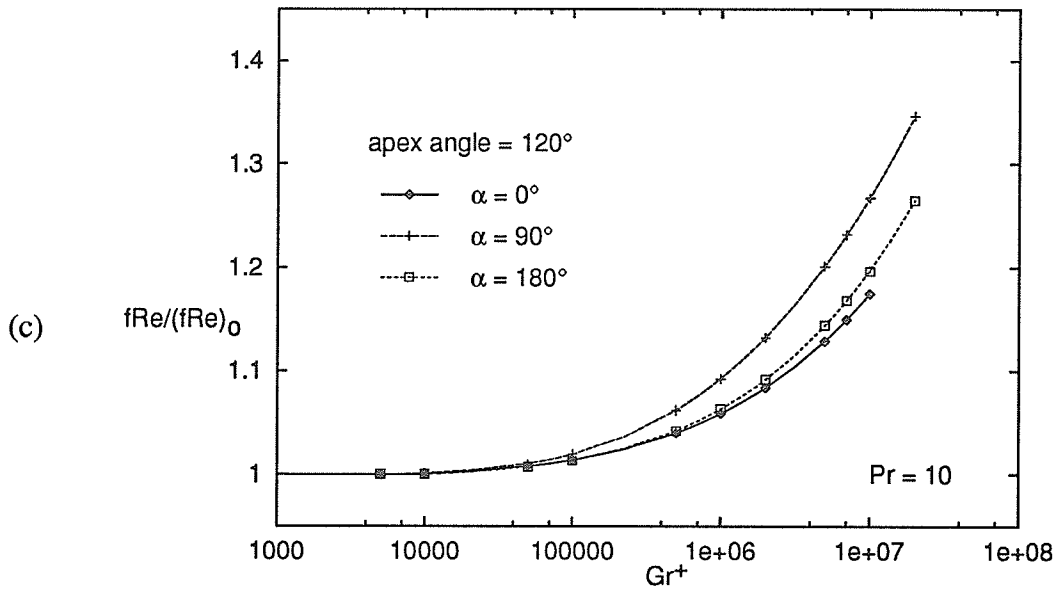


Fig. 6.10  $fRe$ (cont) ratios for  $Pr = 10$ , apex angle of (c)  $120^\circ$ .

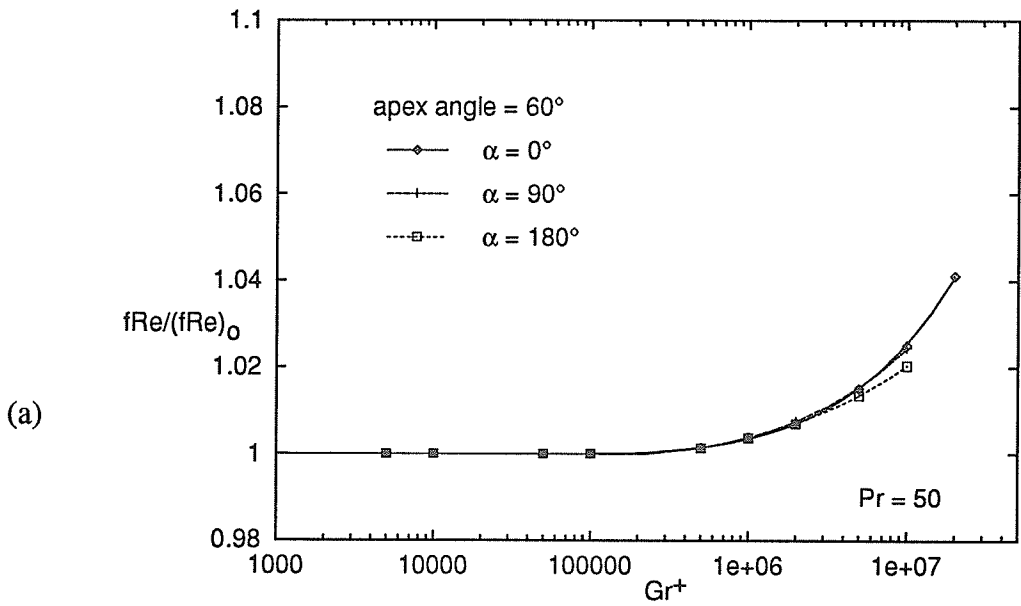


Fig. 6.11  $fRe$  ratios for  $Pr = 50$ , apex angle of (a)  $60^\circ$ .

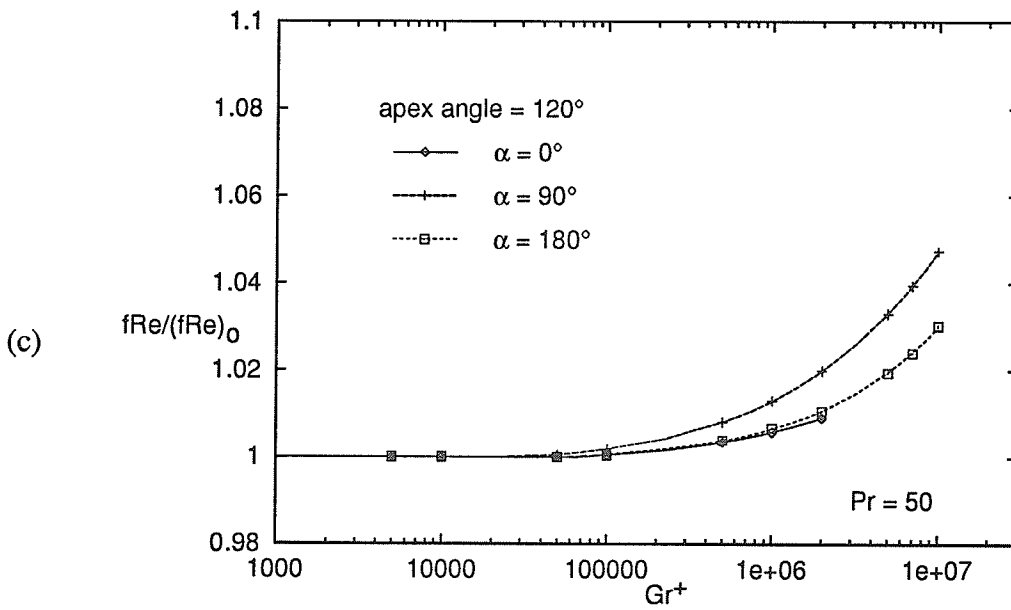
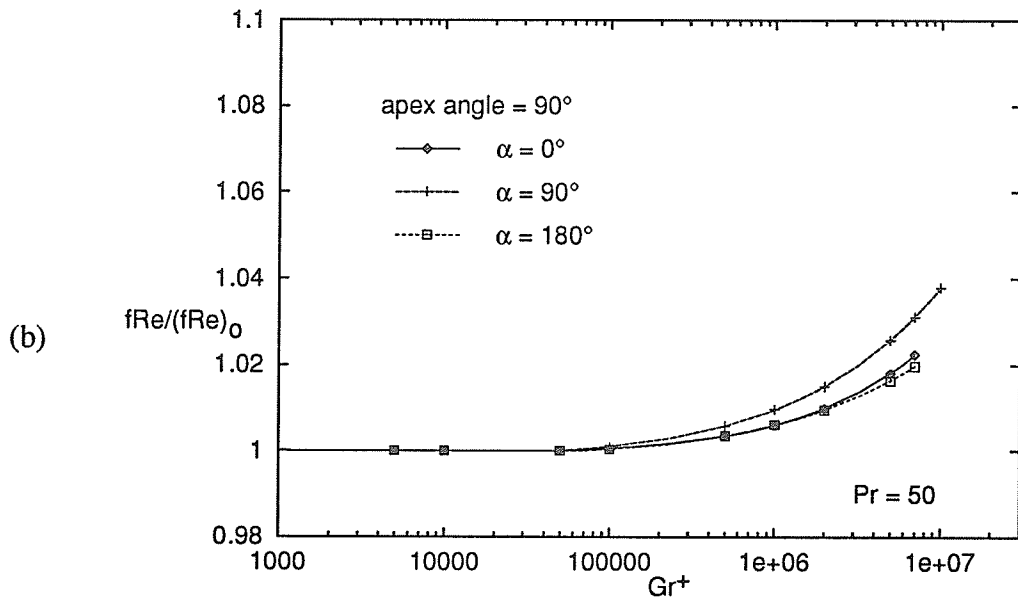


Fig. 6.11  $fRe(\text{cont})$  ratios for  $Pr = 50$ , apex angle of (b)  $90^\circ$ , and (c)  $120^\circ$ .

The  $fRe/(fRe)_o$  results for  $Pr = 10$  are shown in Fig. 6.10. The increases of  $fRe$  are less than the case of  $Pr = 4$ . For each duct, the maximum and minimum  $fRe$  magnitudes are similar to the case of Nusselt number. The maximum difference about 10% can be expected for  $120^\circ$ , however, the  $fRe$  of each orientation does not depart much from the  $(fRe)_o$ . The  $fRe$  ratios are about 1.3 at the highest  $Gr^+$ . The results for  $Pr = 50$  are shown in Fig. 6.11 where the ratios drop to very close to 1 for all ducts. This is similar to the case of semicircular duct which the discussion is in Chapter 5.

## 6.4 Effect of Prandtl number

The effect of Prandtl number was studied by analyzing the data of  $Pr = 4, 10$  and  $50$ . The results for each  $Pr$  form a curve on the  $Nu-Gr^+$  plane in the manner that the  $Nu$  increases with increasing  $Pr$ , e.g. see Fig. 6.12 which is for the  $90^\circ$  duct with  $\alpha = 90^\circ$ . It has been shown in Section 5.7 that the effect of  $Pr$  disappears because the relation  $Nu-Gr^+Pr$  is unique. Similar results are observed for the circular sector ducts.

For the  $60^\circ$  duct, the results for all  $Pr$  agree very well on the  $Nu/Nu_o-Gr^+Pr$  domain. In fact, since  $Nu$  varies little with orientation, the results for all orientations and all  $Pr$  are plotted in Fig. 6.13 where good agreement can be observed. The  $Nu/Nu_o-Gr^+Pr$  relationships in the polynomial form in eqn. 5.1 are presented in Table 6.11 for each orientation separately.

Table 6.11 Coefficients for  $Nu/Nu_o$ , eqn. 5.1.

$\alpha$	Coefficients for 60° duct						$\sigma$	$\sigma_{max}$
	$C_0$	$C_1$	$C_2$	$C_3$	$C_4$	$C_5$		
0°	0.37236269E+1	-0.20716423E+1	0.58588767E+0	-0.79943322E-1	0.49456048E-2	0.00747884	0.10591053	
90°	-0.27697589E+1	0.26881159E+1	-0.68895915E+0	0.67425445E-1	-0.12404672E-2	0.00292518	0.04325832	
180°	0.17992351E+2	-0.11241168E+2	0.27579254E+1	-0.30567945E+0	0.13667410E-1	0.00248089	0.03503270	
For 90° duct								
0°	0.15289681E+2	-0.10175859E+2	0.26737537E+1	-0.31444993E+0	0.14765026E-1	0.00734008	0.11647754	
90°	0.59300236E+1	-0.33966246E+1	0.86639363E+0	-0.10562562E+0	0.60286309E-2	0.00286737	0.04114399	
180°	0.18895444E+2	-0.12286956E+2	0.31168500E+1	-0.35406167E+0	0.16043163E-1	0.00484728	0.08386671	
For 120° duct								
0°	0.14214511E+2	-0.95934023E+1	0.25671693E+1	-0.30715689E+0	0.14703312E-1	0.00518323	0.06363539	
90°	0.60423861E+1	-0.36042099E+1	0.94463333E+0	-0.11661205E+0	0.66659136E-2	0.00437286	0.07939668	
180°	0.13118928E+2	-0.86365364E+1	0.22744789E+1	-0.26997421E+0	0.13056594E-1	0.00669726	0.11606544	

The results for the  $90^\circ$  and  $120^\circ$  ducts are shown in Figs. 6.14 and 6.15. In Figs. 6.13(a) and 6.14(a), the plots are for both  $\alpha = 0^\circ$  and  $180^\circ$  due to their similarities in magnitudes and for  $\alpha = 90^\circ$  in Figs. 6.13(b) and 6.14(b). The three curves in Fig. 6.12 collapse and only a single curve represents the  $Nu-Gr^+Pr$  relation for all  $Pr$  data in Fig. 6.14(b). In general, very good agreements between the data for each Prandtl number can be seen in these Figures. The coefficients for  $Nu/Nu_0-Gr^+Pr$  relations are also compiled in Table 6.11 for each orientation. The results again confirm the unique Nusselt number distribution on the  $Gr^+Pr$  abscissa which is a very useful achievement for combined convection study. In addition, the Nusselt numbers for  $\alpha = 90^\circ$  are higher than the case of  $\alpha = 0^\circ$  and  $180^\circ$  for both the ducts. The figures of these differences are not so much different from those mentioned in Section 6.1 and 6.3.

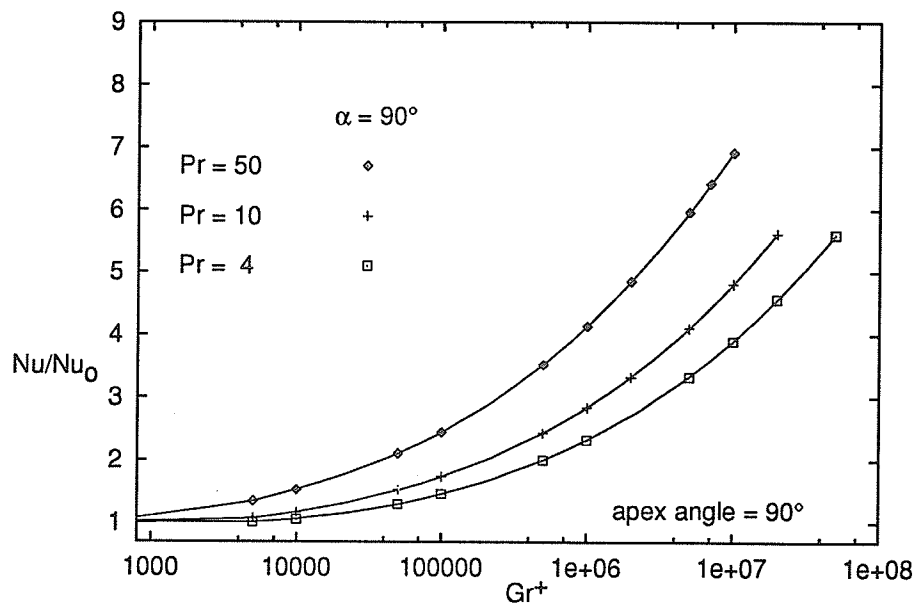


Fig. 6.12  $Nu$  ratios for the  $90^\circ$  duct,  $\alpha = 90^\circ$ , and all  $Pr$ .

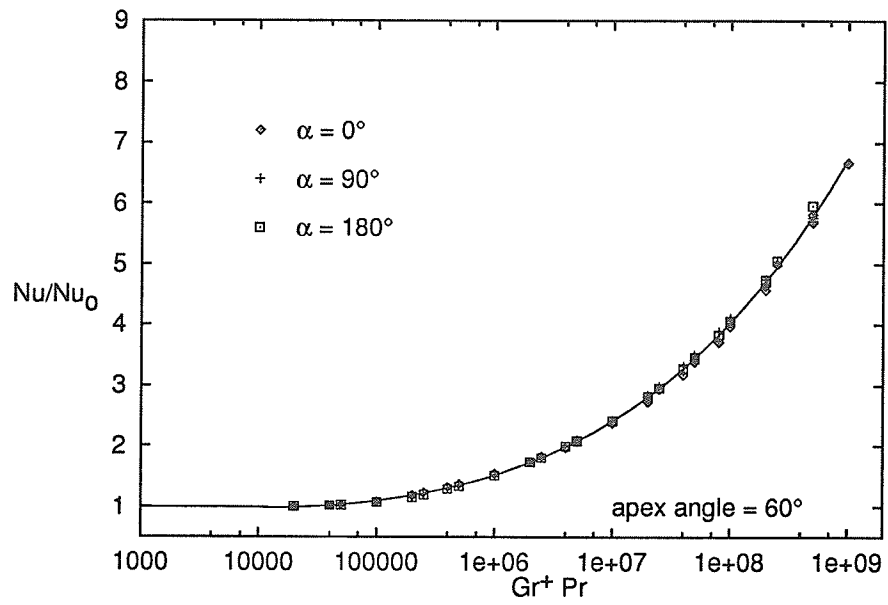


Fig. 6.13  $Nu$  ratios, all  $Pr$  and all  $\alpha$ , for the 60° duct.

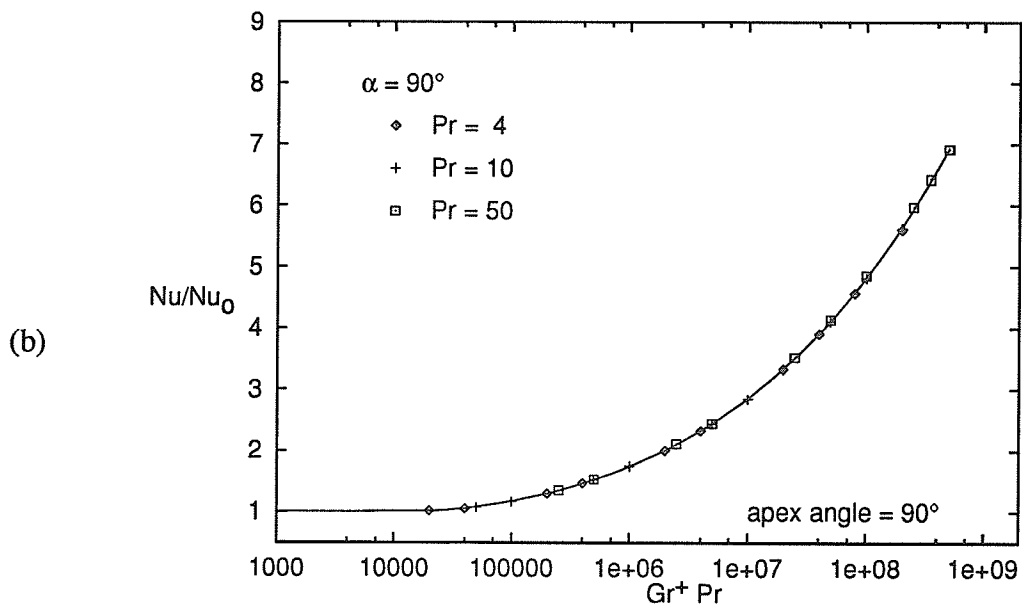
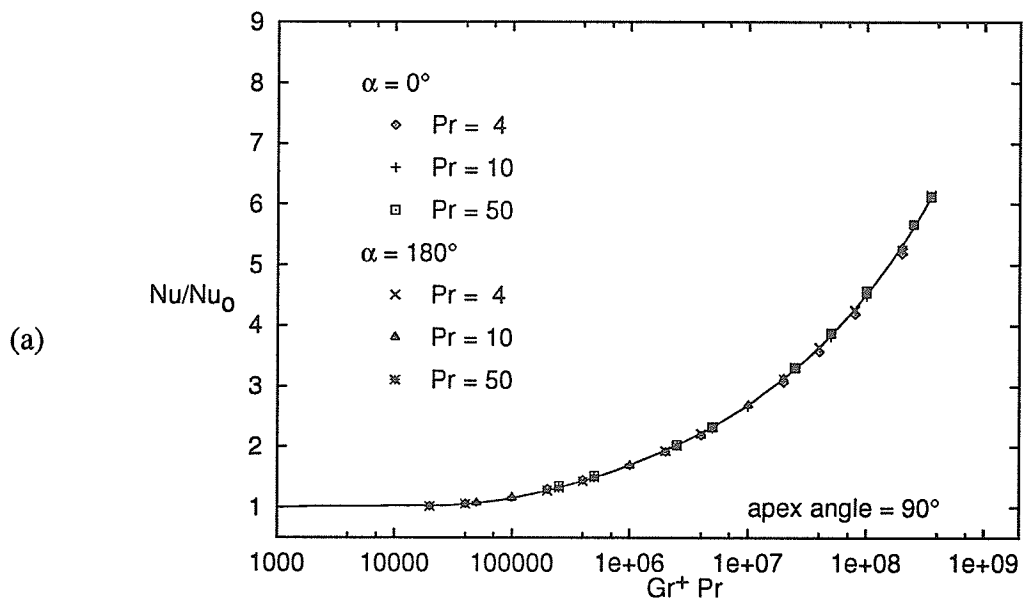


Fig. 6.14  $Nu$  ratios for all  $Pr$  and  $\alpha =$  (a)  $0^\circ$  and  $180^\circ$ , and (b)  $90^\circ$ , for the  $90^\circ$  duct.

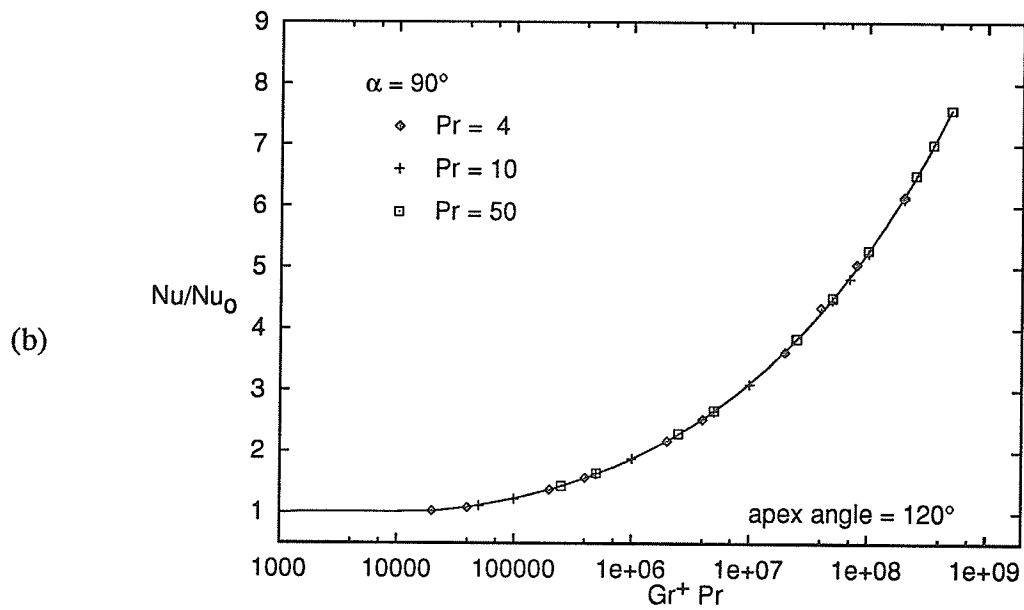
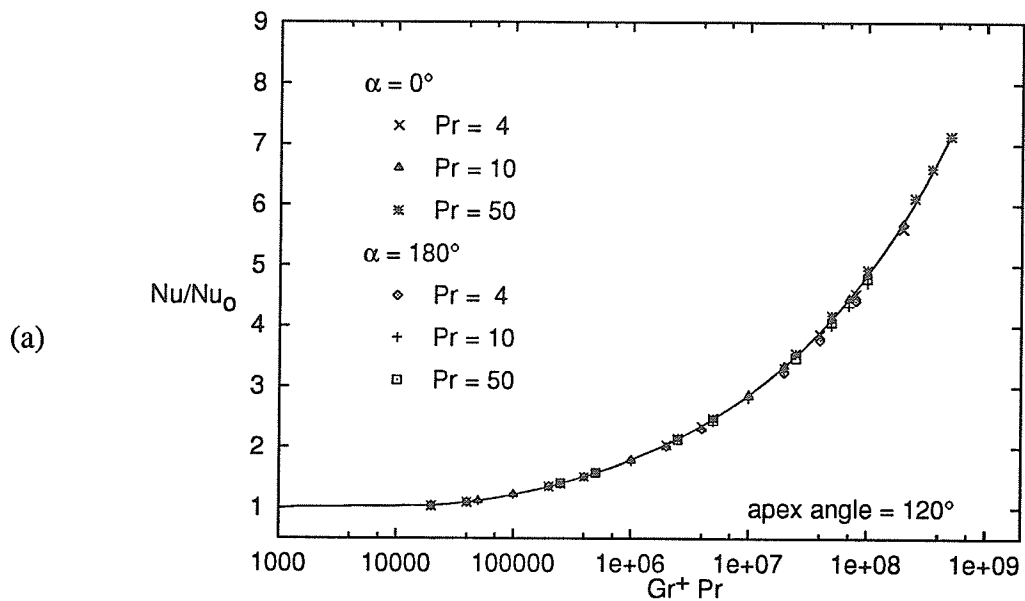


Fig. 6.15  $Nu$  ratios for all  $Pr$  and  $\alpha =$  (a)  $0^\circ$  and  $180^\circ$ , and (b)  $90^\circ$ , for the  $120^\circ$  duct.

The effect of Prandtl number on  $fRe$  for the  $60^\circ$ ,  $90^\circ$  and  $120^\circ$  ducts are studied similarly to the case of the semicircular duct in Chapter 5 that the Prandtl number effect can be eliminated if  $fRe$  is plotted against  $Gr^+/Pr^2$ . The  $fRe/(fRe)_o$  for the  $60^\circ$  duct is shown in Fig. 6.16 where the data for all  $\alpha$  and all  $Pr$  are plotted against  $Gr^+/Pr^2$ . The separations of the data for each orientation are observable, however, with small differences.

For the  $90^\circ$  duct, the  $fRe/(fRe)_o$  is shown in Fig. 6.17(a) for  $\alpha = 0^\circ$  and  $180^\circ$  and Fig. 6.17(b) for  $\alpha = 90^\circ$ . In both Figures, the curves where  $Gr^+/Pr^2 > 10^4$  comprise the data for  $Pr = 4$  and  $10$  only because  $fRe$  ratios for  $Pr = 50$  are very close to 1. This is easier to observe in Fig. 6.17(b). The agreements among each  $Pr$  data are very good.

For the  $120^\circ$  duct, the results are in Fig. 6.18 in similar fashion as the case of the  $90^\circ$  duct. The outcomes for the  $120^\circ$  duct are also similar to the case of  $90^\circ$  duct except that the  $fRe$  ratios are slightly larger. For both ducts, the  $fRe$  for  $\alpha = 90^\circ$  is the highest as compared to the other orientations. These results verify the unique  $fRe-Gr^+/Pr^2$  relation in addition to the case of semicircular duct in Chapter 5. All the  $fRe/(fRe)_o$  data are correlated to the  $Gr^+/Pr^2$  with the eqn. similar to eqn. 5.1 and the results are in Table 6.12.

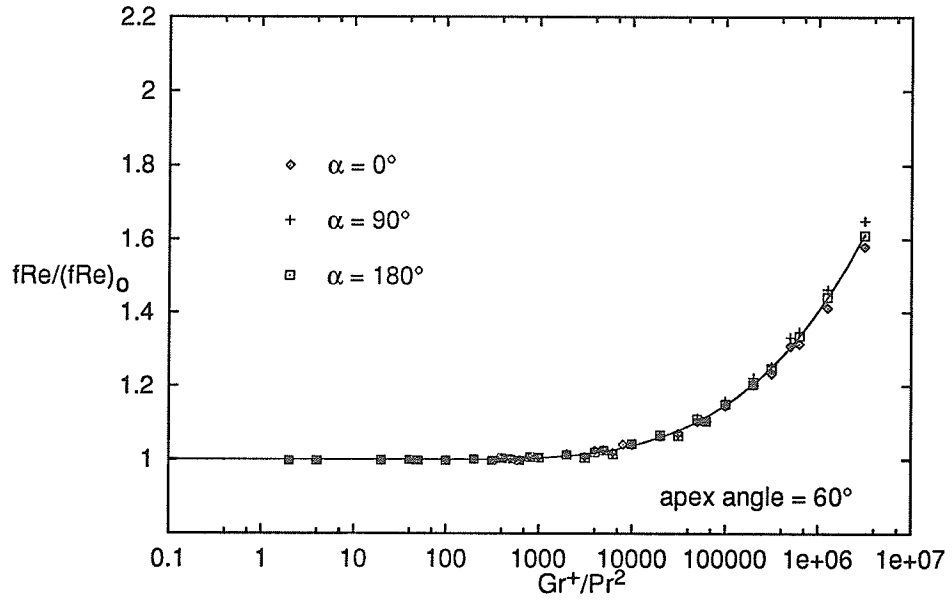


Fig. 6.16  $fRe$  ratios for all  $Pr$  and all  $\alpha$ , for the  $60^\circ$  duct.

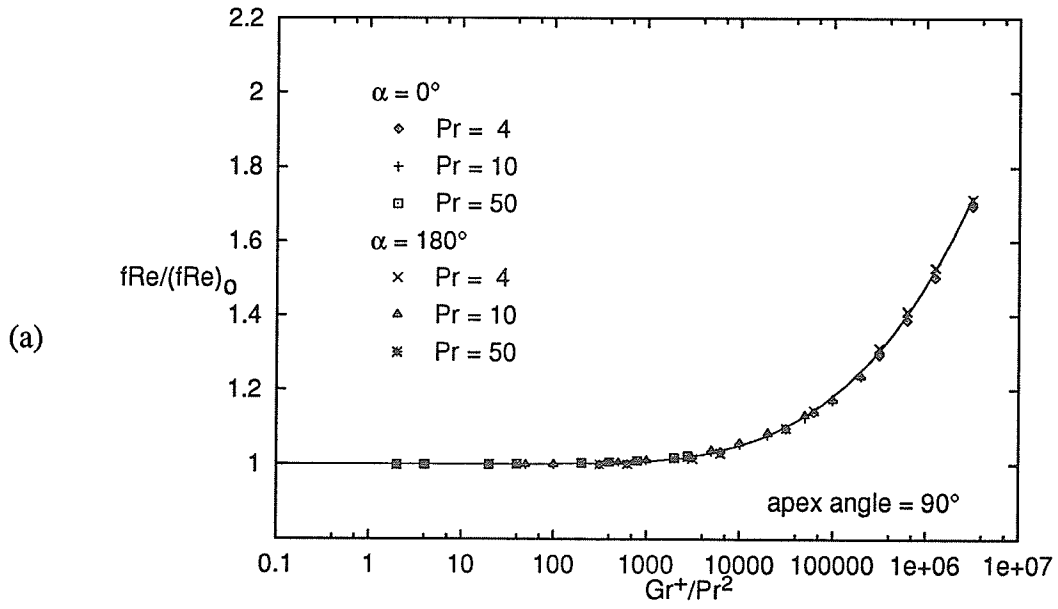


Fig. 6.17  $fRe$  ratios for all  $Pr$  and  $\alpha =$  (a)  $0^\circ$  and  $180^\circ$ , for the  $90^\circ$  duct.

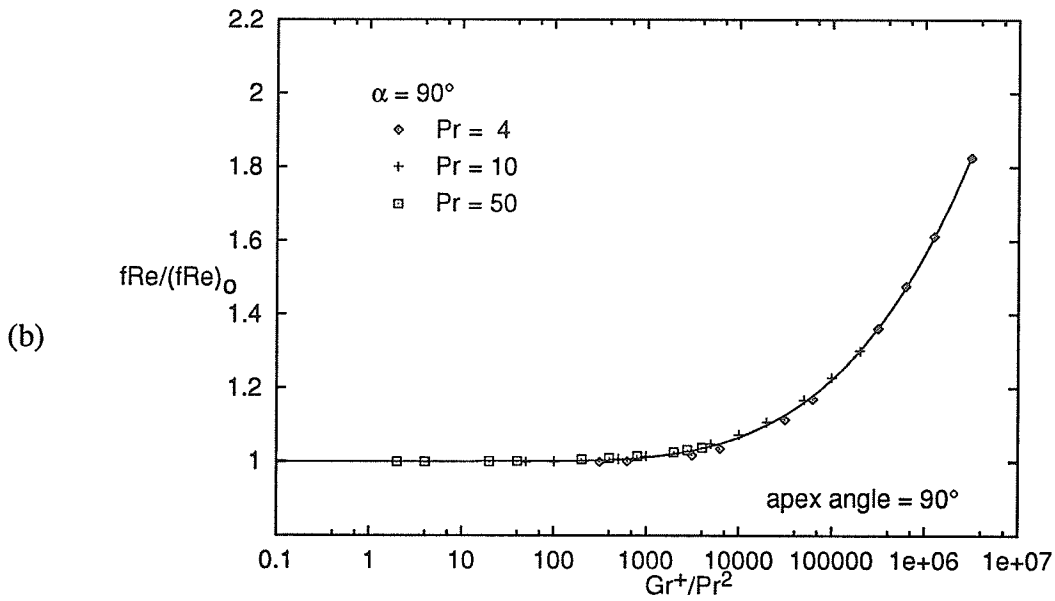


Fig. 6.17(cont)  $fRe$  ratios for all  $Pr$  and  $\alpha =$  (b)  $90^\circ$ , the  $90^\circ$  duct.

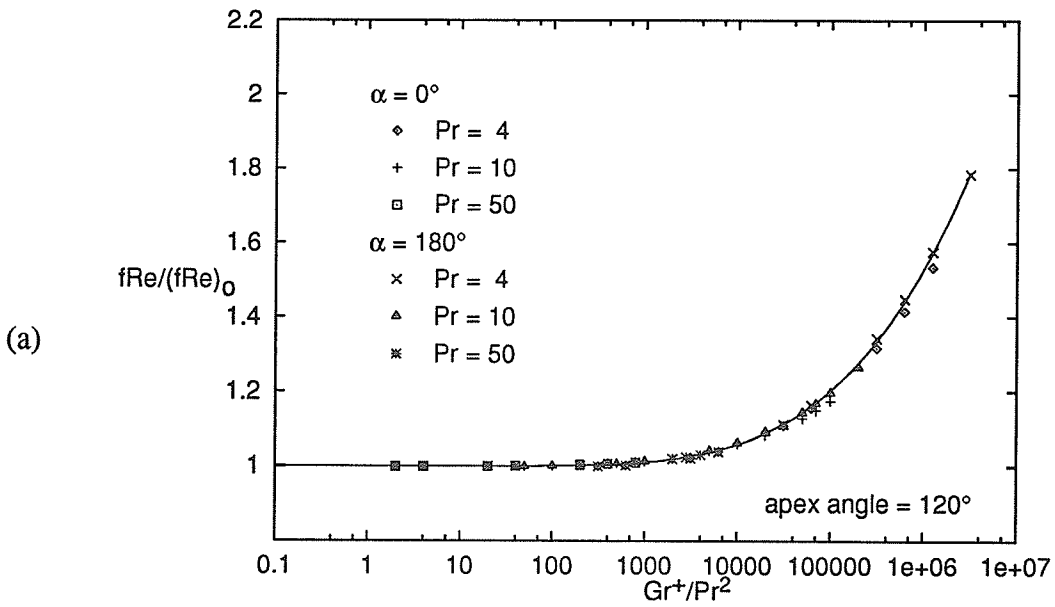


Fig. 6.18  $fRe$  ratios for all  $Pr$  and  $\alpha =$  (a)  $0^\circ$  and  $180^\circ$ , the  $120^\circ$  duct.

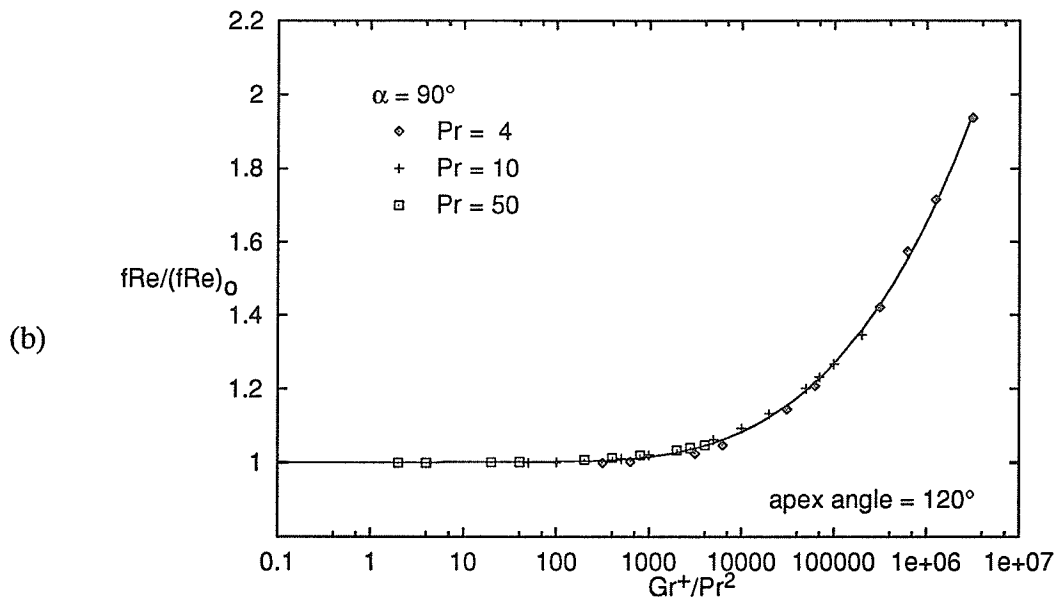


Fig. 6.18(cont)  $fRe$  ratios for all  $Pr$  and  $\alpha =$  (b)  $90^\circ$ , the  $120^\circ$  duct.

Table 6.12 Coefficients for  $fRe/(Re)_p$  similar to eqn. 5.1.

$\alpha$	Coefficients for 60° duct						$\sigma$	$\sigma_{max}$
	$C_0$	$C_1$	$C_2$	$C_3$	$C_4$	$C_5$		
0°	0.99904241E+0	-0.75210706E-2	0.10195034E-1	-0.49544964E-2	0.87430386E-3	0.00096126	0.01619538	
90°	0.99882346E+0	-0.85012493E-2	0.12796952E-1	-0.63306612E-2	0.10682752E-2	0.00114125	0.01360955	
180°	0.99745627E+0	-0.40394075E-2	0.89648714E-2	-0.51324150E-2	0.93784386E-3	0.00097588	0.01232653	
For 90° duct								
0°	0.10023471E+1	-0.13812856E-1	0.15151148E-1	-0.64804688E-2	0.10777997E-2	0.00049128	0.00531799	
90°	0.99592353E+0	0.54514809E-2	0.16518428E-2	-0.33748034E-2	0.92819257E-3	0.00112141	0.01413986	
180°	0.99770647E+0	-0.84874606E-3	0.66255275E-2	-0.46348601E-2	0.96498352E-3	0.00103086	0.01457469	
For 120° duct								
0°	0.10039440E+1	-0.19096670E-1	0.19437540E-1	-0.77441066E-2	0.12164718E-2	0.00087966	0.01208072	
90°	0.99202664E+0	0.17431339E-1	-0.69345821E-2	-0.12554769E-2	0.83096566E-3	0.00150423	0.01983553	
180°	0.99928886E+0	-0.55749857E-2	0.10140762E-1	-0.56221079E-2	0.10883432E-2	0.00075322	0.01203267	

# Chapter 7

## Concluding Remarks

The studies of combined convection in this thesis consists of two main topics. The first part is a major part dealing with a semicircular duct both theoretically and experimentally. The second part deals with the rest of the set of circular sector ducts by theoretical study. In both parts, the effect of duct cross-sectional orientation was exhaustively studied in addition to the study of the effect of Prandtl number. The second part also includes a generic view of the circular sector ducts according to the size of the apex angle associating with the buoyancy force effect. The results of these studies are summarized in this chapter along with the remarks and suggestion for further study.

### 7.1 Summaries

The first three subsections are the summaries according to the results in Chapter 4, 5, and 6. Due to the similarity of the effect of Prandtl number for all the ducts, the summaries for these results are combined in the last subsection.

### 7.1.1 Experimental Results for the Semicircular Duct

The experiments for combined convection in the semicircular duct were conducted for four orientations,  $\alpha = 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ , for investigating the effect of the duct cross-sectional orientation on the fully developed Nusselt number. The average experimental ranges were within 490–1550 for  $Re_m$ ,  $1 \times 10^6$ – $5 \times 10^7$  for  $Gr_m$ , and 4.2–7.6 for  $Pr_m$ . Examination of the streamwise wall temperatures and local Nusselt number showed that the fully developed conditions were obtained for  $X > 30D_h$  or  $x^* > 0.01$ .

The wall temperature drops (the difference between duct wall temperature and fluid bulk temperature) showed a peak at the inlet of the heated section. The peak was followed by a uniform temperature drop which occupied a large portion of the duct length. The peak temperature drop suggested the developing of the combined convection near the inlet while the uniform temperature drop indicated that the temperature drop was fully developed. The fully developed region also implied that the heat transfer mechanism reached a constant level which was not changed as the fluid proceeded to the outlet. The local Nusselt number distribution was approximately the reversion of the wall temperature drop and the peak temperature drop became a sag in Nusselt number profile.

In the developing region, the local Nusselt number decreased to the minimum over a first few stations. The effects of Reynolds number were observed for a fixed  $Gr$  of about  $10^7$  and three Reynolds numbers of about 500, 1000, and 1500. The effects of

Grashof number were observed for a fixed  $Re$  of about 1000 and three Grashof numbers of  $5 \times 10^6$ ,  $1 \times 10^7$ , and  $5 \times 10^7$ . The effects of both Reynolds and Grashof numbers were negligibly small for the dimensional entrance lengths ( $X$ ). The effects were somewhat larger for the dimensionless entrance lengths ( $x^*$ ). For a fixed  $Gr$  of about  $10^7$ , decreasing  $Re$  was observed to increase the entrance length in terms of  $x^*$ . The effect of  $\alpha$  on the entrance lengths was not noticeable.

The fully developed Nusselt number was obtained by the length mean average of six local Nusselt numbers at the downstream end. For all  $\alpha$ , fully developed regions were observed to occupy about 11 station in the downstream side where local Nusselt number varied within the average of 2–4% compared to the fully developed values. For each  $\alpha$ ,  $Nu$  depended on  $Gr^+$  in an exponential fashion. Increases in Nusselt number ratios up to eight folds were observed at  $Gr^+ \approx 2 \times 10^8$  while it was about three folds at  $Gr^+ \approx 10^6$ . The Nusselt numbers for  $\alpha = 90^\circ$  and  $135^\circ$  were very close in magnitude while that for  $\alpha = 45^\circ$  was lower, about 10% lower at  $Gr^+ \approx 2 \times 10^8$ . For  $\alpha = 45^\circ$  and  $180^\circ$ , Nusselt numbers were quite similar in magnitudes.

The Nusselt number was also plotted against the variable  $Gr^+Pr$  and an improved correlation of the data in the  $Nu-Gr^+Pr$  mode was observed. Similar to the case of  $Gr^+$  variable, at  $Gr^+Pr$  about  $10^9$ , Nusselt numbers for  $\alpha$  of  $90^\circ$  and  $135^\circ$  are about 10% higher than those for  $\alpha = 45^\circ$  and  $180^\circ$  and about 40% higher than that for  $\alpha = 0^\circ$ .

Regarding friction factor, it was found that  $fRe$  was maximum for  $\alpha = 90^\circ$ , while those of  $45^\circ$  and  $180^\circ$  were the lowest. The  $135^\circ$  orientation showed a moderate  $fRe$

which was closer to the case of  $\alpha = 45^\circ$  rather than  $90^\circ$ . The lowest  $fRe$  was observed for  $\alpha = 0^\circ$ . For all  $\alpha$ , the maximum  $fRe$  compared to the pure forced convection were less than two folds. The  $fRe$  was obtained from the pressure drop across the duct inlet and outlet therefore it included both the developing and fully developed regions.

### 7.1.2 Numerical Results for the Semicircular Duct

Numerical results were obtained for  $Pr = 4$  and  $\alpha = 0^\circ, 45^\circ, 90^\circ, 135^\circ$  and  $180^\circ$ . For  $\alpha = 0^\circ$  and  $180^\circ$ , the cross-stream flow was symmetric and could be two- or four-cell pattern while for  $\alpha = 45^\circ, 90^\circ$ , and  $135^\circ$ , it was not symmetric and the pattern was only two-cell. The two-cell pattern was the only common pattern and therefore it was used to allow comparisons for  $Nu$  and  $fRe$  among all orientations. The Nusselt numbers were the highest for  $\alpha = 90^\circ$  and  $135^\circ$  which had very small differences in magnitudes. Similarly for  $\alpha = 0^\circ$  and  $180^\circ$ , the Nusselt number magnitudes were almost the same but they were the lowest, up to 20% compared to the case of  $\alpha = 90^\circ$  or  $135^\circ$ . The Nusselt numbers for  $\alpha = 45^\circ$  were approximately in the middle between the highest and the lowest. The fluid temperature cross-stream isotherms at the lowest level were observed to be smaller for  $\alpha = 90^\circ$  and  $135^\circ$  compared to similar isotherms for  $\alpha = 0^\circ$  and  $180^\circ$ . The smaller area bounded by the isotherm indicated that the cooler part of the fluid was efficiently used and the fluid temperature became more homogenous. This appeared to relate to the shape of the duct as well. For  $\alpha = 45^\circ, 90^\circ$  and  $135^\circ$ , the sharp corner of the duct was turned to the bottom which contained the cooler fluid. This

confined space seemed to force the hotter fluid downward to mix with the cool core fluid more effectively. This would be a reason to explain why all the non-symmetric orientations had higher Nusselt numbers. For a given  $Gr^+$ , the maximum Nusselt number appeared between  $\alpha \approx 90^\circ$  and  $135^\circ$ . The maximum was at  $\alpha = 115^\circ$  for high  $Gr^+$ , however, it was only little higher than that of  $\alpha = 90^\circ$  or  $135^\circ$ .

The four-cell pattern was an additional solution for symmetric orientations when  $Gr^+$  was sufficiently high. The four-cell pattern started to appear where  $Gr^+ \geq 2 \times 10^5$ . For  $\alpha = 0^\circ$ , the two-cell solution disappeared where  $Gr^+ \geq 5 \times 10^7$ . Therefore, there were two possible solutions within the range  $2 \times 10^5 \leq Gr^+ \leq 4 \times 10^7$ . The second critical  $Gr^+$  could not be obtained for  $\alpha = 180^\circ$  due to convergence difficulty. The four-cell solutions were higher, about 9–11% for  $\alpha = 0^\circ$  and 3–7% for  $\alpha = 180^\circ$ . The four-cell pattern occurred only where  $Gr^+$  was sufficiently high, unlike the two-cell pattern that appear naturally at low  $Gr^+$ .

In general, the  $fRe$  product behaved similar to  $Nu$  in regard to the influence of duct orientation. The enhancement of Nusselt number is due to the buoyancy induced flow which actually distorts the axial velocity field as well. The  $fRe$  product increases as the axial velocity field is further distorted. Therefore the conditions producing a higher Nusselt number is always likely to generate a higher  $fRe$ . The Nusselt number enhancement is obtained by paying a higher price for the axial pressure drop, however, the gain in Nusselt number is much higher. For all cases, the increase in  $fRe$  was less than two folds compared to factors of 5–6 for Nusselt number.

### 7.1.3 Circular Sector Ducts

Circular sector ducts are identified by their apex angles. The results for  $Pr = 4$  were obtained for  $60^\circ$ ,  $90^\circ$  and  $120^\circ$  ducts in addition to the previous results for the semicircular duct. The results showed that cross-sectional orientation affected Nusselt number more for the larger ducts. The effect was almost negligible for the  $60^\circ$  duct. For the  $90^\circ$  and  $120^\circ$  ducts, the  $90^\circ$  orientation gave the maximum Nusselt numbers while those for  $0^\circ$  or  $180^\circ$  were minimum. The differences between the maximum and minimum were 4.7%, 9%, and 11.5% for the  $60^\circ$ ,  $90^\circ$  and  $120^\circ$  duct, respectively. Since the differences were not large, solutions were determined only for three orientations. The decrease of the orientation effect with the decreasing apex angle suggests that the cross-stream flow requires a broad cross section in order to become effective. The confined cross-sectional flow domain in the  $60^\circ$  duct limited the effect of the buoyancy flow therefore the Nusselt number was observed to be fairly insensitive to orientation.

A comparison of Nusselt number among the  $60^\circ$ ,  $90^\circ$ ,  $120^\circ$  and  $180^\circ$  ducts with a fixed orientation showed that Nusselt number increased with increasing apex angle. A modified Nusselt number which was inversely proportional to the duct wall mean temperature drop was introduced. The result of the comparison was similar to the case of the original Nusselt number except that the difference of the modified Nusselt number between the  $180^\circ$  and  $120^\circ$  duct was larger. This comparison implies that the smaller duct wall is hotter if the ducts are operated with the same heat input.

### 7.1.4 Effect of Prandtl Number

For all four ducts, the results for  $Pr = 10$ , and  $50$  showed that  $Nu$  increased and  $fRe$  decreased with increasing  $Pr$  at a given  $Gr^+$ . In general, all the results for  $Pr = 10$  and  $50$  had the same trend as the  $Pr = 4$  results regarding the effect of duct orientation. The  $90^\circ$  or  $135^\circ$  orientation always yielded the highest  $Nu$  and  $fRe$  while the  $0^\circ$  and  $180^\circ$  orientations were always the lowest. For the  $180^\circ$  duct, the differences between the maximum and minimum were about 20% for all cases of  $Pr$ . The increase of  $fRe$  was small for the higher Prandtl number fluid. The  $fRe$  ratio was about 1.3 for  $Pr = 10$  and 1.05 for  $Pr = 50$ . The axial velocity contours for  $Pr = 50$  and  $Gr^+ = 1 \times 10^6$  showed almost no difference from the case of  $Gr^+ = 0$ . This revealed why the  $fRe$  ratio for  $Pr = 50$  was very close to 1.

On the  $Nu$  or  $Nu/Nu_o$  and  $Gr^+$  domain, the solutions for each  $Pr$  formed a curve separately. Therefore, there were three curves for  $Pr = 4, 10, \text{ and } 50$ . When the same Nusselt number data were plotted on the  $Nu-Gr^+Pr$  domain, the data for all  $Pr$  formed a unique curve. This was true for all ducts. Hence there is a unique relationship between  $Nu$  and  $Gr^+Pr$  for a given  $\alpha$ . The validity of this relation was also true for the rectangular and circular ducts which the data were published by other authors. Similarly, the unique relationship existed for  $fRe$  when it was plotted with the  $Gr^+/Pr^2$  variable.

## 7.2 Remarks and Suggestions for Further Studies

There are some interesting points that arose during the course of the experimental and theoretical studies in this thesis. The followings should be worth further studies:

1. Regarding comparison of the experimental and numerical results in Chapter 5, predictions with the H1 thermal boundary condition compared reasonably well in the lower range of  $Gr^+$  where the actual duct wall temperature was almost uniform. However, the amount of overestimation was large in the high  $Gr^+$  range where the actual wall temperature variation in the experiment was as large as  $3^{\circ}$ – $4^{\circ}$  C. This was due to the H1 condition supplying more heat flux at the bottom than at the top of the cross section according to the constant wall temperature. This results in a larger heat transfer coefficient as compared to the H2 condition which has a lower wall temperature at the bottom. The Nusselt number predicted with H2 condition should be smaller than that predicted with the H1 condition and therefore it should be closer to the experimental results in the high  $Gr^+$  region.

2. The results in Chapter 5 and 6 suggest that the orientation effect is most remarkable in semicircular ducts. Considering the numerical and experimental results, a semicircular duct orienting at  $\alpha = 90^{\circ}$  possibly yielded the extra gain in Nusselt number magnitude up to 20–40% compared to other orientations. A circular tube with two semicircular passages seating with the angle  $\alpha$  of  $90^{\circ}$  seems capable to make use

of the buoyancy and orientation effects effectively.

3. Partitioning in a semicircular cross section has two advantages over the partitioning in circular cross sections. First, the flat wall can be conveniently attached to sheet materials to achieve a good thermal contact. Secondly, the number of interior walls is smaller for semicircular ducts compared to circular ducts partitioned with the same apex angle. The heat transfer performances of the passage possessing interior wall should deteriorate due to the indirect exposing to the applied heat flux. The  $60^\circ$  and  $90^\circ$  circular sector cross sections are two potential component passages that has considerably large buoyancy effect. Two alternates for partitioning in a semicircular cross section can be two  $90^\circ$  passages or three  $60^\circ$  passages. The latter case has two drawbacks; more interior walls, and smaller orientation effect.

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## Appendix 1

### Experimental Data for $\alpha = 45^\circ$

XP045-01

INPUT ELECTRIC POWER = 330.0 W HEAT RATE GAINED BY WATER = 334.7 W HEAT BALANCE ERROR = -1.41%  
 MASS FLOW RATE = 53.2320 G/S PRESSURE DROP = 1.3725 MM H2O FRICTION FACTOR = 0.013897 FREM = 21.1350  
 REM = 1520.8 GRM = 0.10965E+07 UPSTREAM BULK TEMPERATURE = 15.60 DEG C DOWN STREAM BULK TEMPERATURE = 17.11 DEG C  
 PRM = 7.684 RAM = 0.84255E+07 INLET BULK TEMPERATURE = 15.60 DEG C OUTLET BULK TEMPERATURE = 17.10 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVER- AGE						A	B	C	T	H	T+H
0	0.1	16.31	16.28	16.28	16.29	15.60	1491.1	7.87	0.787E+07	0.00000	40.42	41.98	42.14	41.63	41.67	41.65
1	1.5	16.45	16.42	16.42	16.43	15.61	1491.3	7.87	0.788E+07	0.00004	33.59	35.13	34.89	34.59	34.62	34.61
2	2.5	16.56	16.51	16.52	16.53	15.61	1491.4	7.87	0.788E+07	0.00007	30.09	31.57	31.19	30.98	31.01	30.99
3	5.5	16.86	16.79	16.82	16.83	15.62	1491.8	7.87	0.788E+07	0.00015	22.92	24.22	23.66	23.59	23.60	23.59
4	15.5	17.18	17.11	17.06	17.12	15.65	1493.0	7.86	0.791E+07	0.00043	18.58	19.50	20.19	19.40	19.42	19.41
5	25.5	17.30	17.35	17.30	17.32	15.68	1494.3	7.85	0.793E+07	0.00071	17.60	17.07	17.60	17.42	17.43	17.42
6	45.5	17.42	17.41	17.27	17.36	15.75	1495.8	7.84	0.798E+07	0.00128	17.05	17.16	18.73	17.61	17.64	17.63
7	75.5	17.66	17.71	17.43	17.60	15.84	1500.5	7.81	0.805E+07	0.00212	15.70	15.23	17.96	16.21	16.30	16.25
8	105.5	17.84	18.08	17.67	17.87	15.94	1504.3	7.79	0.812E+07	0.00296	14.93	13.27	16.41	14.76	14.87	14.81
9	135.5	18.07	18.19	18.03	18.10	16.04	1508.1	7.76	0.819E+07	0.00381	13.96	13.21	14.27	13.80	13.81	13.80
10	165.2	18.32	18.40	18.23	18.32	16.13	1511.9	7.74	0.826E+07	0.00464	12.97	12.53	13.52	12.99	13.01	13.01
11	205.2	18.31	18.59	18.21	18.37	16.26	1517.0	7.71	0.836E+07	0.00577	13.89	12.17	14.60	13.47	13.55	13.51
12	245.2	18.52	18.63	18.29	18.48	16.39	1522.2	7.68	0.845E+07	0.00690	13.34	12.68	14.94	13.59	13.65	13.62
13	275.2	18.52	18.74	18.35	18.54	16.48	1526.1	7.65	0.852E+07	0.00775	13.97	12.56	15.21	13.83	13.91	13.87
14	305.2	18.47	18.66	18.35	18.50	16.58	1530.0	7.63	0.860E+07	0.00860	15.00	13.50	16.04	14.77	14.84	14.81
15	333.3	18.82	19.02	18.68	18.84	16.67	1533.7	7.61	0.866E+07	0.00940	13.22	12.07	14.10	13.07	13.13	13.10
16	363.3	18.86	19.09	18.72	18.89	16.77	1537.6	7.58	0.874E+07	0.01025	13.58	12.22	14.55	13.38	13.45	13.41
17	383.3	18.87	19.08	18.77	18.91	16.83	1540.3	7.57	0.879E+07	0.01082	13.89	12.62	14.65	13.67	13.72	13.69
18	403.3	18.98	19.19	18.81	19.00	16.89	1542.6	7.55	0.883E+07	0.01139	13.58	12.33	14.80	13.49	13.57	13.53
19	423.3	19.12	19.31	18.85	19.09	16.96	1544.8	7.54	0.888E+07	0.01195	13.14	12.05	14.99	13.29	13.39	13.34
20	443.3	19.07	19.36	18.98	19.14	17.02	1547.1	7.53	0.892E+07	0.01252	13.83	12.12	14.46	13.40	13.47	13.43
21	463.3	19.19	19.44	19.00	19.21	17.09	1549.3	7.52	0.896E+07	0.01309	13.45	12.02	14.82	13.33	13.43	13.38
AVERAGE VALUES THROUGH STATIONS 15 TO 20:		391.6	18.95	19.18	18.80	16.98	1541.0	7.56	0.880E+07	0.01106	13.54	12.24	14.59	13.38	13.45	13.42

XP045-02

INPUT ELECTRIC POWER = 285.0 W HEAT RATE GAINED BY WATER = 283.1 W HEAT BALANCE ERROR = 0.66%  
 MASS FLOW RATE = 16.9880 G/S PRESSURE DROP = 0.4535 MM H2O FRICTION FACTOR = 0.045079 FREM = 22.4064  
 REM = 497.0 GRM = 0.10335E+07 UPSTREAM BULK TEMPERATURE = 15.32 DEG C DOWN STREAM BULK TEMPERATURE = 19.32 DEG C  
 PRM = 7.472 RAM = 0.77216E+07 INLET BULK TEMPERATURE = 15.33 DEG C OUTLET BULK TEMPERATURE = 19.31 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVER- AGE						A	B	C	T	H	T+H
0	0.1	16.35	16.37	16.29	16.33	15.33	472.5	7.94	0.649E+07	0.00000	23.60	23.20	25.14	24.14	24.27	24.20
1	1.5	16.52	16.54	16.46	16.50	15.34	472.7	7.94	0.650E+07	0.00013	20.52	20.10	21.55	20.81	20.93	20.87
2	2.5	16.63	16.66	16.58	16.62	15.35	472.8	7.94	0.650E+07	0.00022	18.82	18.40	19.62	19.00	19.12	19.06
3	5.5	16.97	17.02	16.93	16.97	15.38	473.1	7.93	0.652E+07	0.00048	15.09	14.68	15.46	15.07	15.08	15.07
4	15.5	17.46	17.51	17.34	17.44	15.46	474.1	7.91	0.657E+07	0.00136	12.04	11.78	12.82	12.20	12.21	12.20
5	25.5	17.63	17.85	17.68	17.72	15.54	475.2	7.89	0.662E+07	0.00224	11.53	10.45	11.26	11.06	11.08	11.07
6	45.5	17.86	17.96	17.72	17.85	15.71	477.3	7.84	0.673E+07	0.00400	11.22	10.72	12.03	11.29	11.32	11.31
7	75.5	17.88	18.10	17.65	17.88	15.97	480.5	7.78	0.689E+07	0.00664	12.61	11.29	14.32	12.62	12.74	12.68
8	105.5	18.07	18.19	17.84	18.03	16.22	483.7	7.72	0.705E+07	0.00930	13.05	12.21	14.89	13.29	13.38	13.34
9	135.5	18.24	18.41	18.14	18.26	16.48	487.0	7.65	0.721E+07	0.01196	13.66	12.44	14.48	13.47	13.52	13.50
10	165.2	18.38	18.62	18.34	18.45	16.73	490.3	7.59	0.737E+07	0.01460	14.59	12.70	14.89	13.99	14.06	14.02
11	205.2	18.69	18.93	18.59	18.74	17.07	494.3	7.52	0.758E+07	0.01816	14.78	12.92	15.75	14.39	14.49	14.44
12	245.2	19.07	19.24	18.84	19.05	17.41	498.1	7.45	0.778E+07	0.02122	14.43	13.12	16.73	14.61	14.76	14.69
13	275.2	19.18	19.36	19.02	19.19	17.67	501.0	7.41	0.793E+07	0.02440	15.79	14.18	17.74	15.77	15.90	15.84
14	305.2	19.36	19.57	19.25	19.41	17.92	503.9	7.36	0.808E+07	0.02707	16.63	14.50	17.43	16.09	16.19	16.14
15	333.3	19.82	20.03	19.69	19.85	18.16	506.7	7.31	0.823E+07	0.02959	14.44	12.77	15.65	14.19	14.29	14.24
16	363.3	20.07	20.31	19.89	20.09	18.42	509.7	7.27	0.838E+07	0.03227	14.43	12.61	16.22	14.27	14.42	14.34
17	383.3	20.21	20.42	20.04	20.22	18.59	511.7	7.23	0.849E+07	0.03406	14.74	13.04	16.37	14.59	14.72	14.66
18	403.3	20.43	20.64	20.26	20.44	18.76	513.7	7.20	0.860E+07	0.03586	14.28	12.64	15.92	14.15	14.28	14.21
19	423.3	20.61	20.81	20.47	20.63	18.93	515.8	7.17	0.870E+07	0.03765	14.16	12.66	15.48	14.00	14.10	14.05
20	443.3	20.69	20.93	20.54	20.72	19.10	517.9	7.14	0.881E+07	0.03945	14.97	13.02	16.52	14.70	14.84	14.77
21	463.3	20.87	21.07	20.67	20.87	19.27	519.9	7.11	0.892E+07	0.04125	14.89	13.24	17.01	14.89	15.05	14.97
AVERAGE VALUES THROUGH STATIONS 15 TO 20:		391.6	20.30	20.52	20.15	20.33	512.6	7.22	0.853E+07	0.03481	14.50	12.79	16.03	14.32	14.44	14.38

XP045-03

INPUT ELECTRIC POWER = 320.0 W HEAT RATE GAINED BY WATER = 321.8 W HEAT BALANCE ERROR = -0.56%  
 MASS FLOW RATE = 35.0380 G/S PRESSURE DROP = 0.9292 MM H2O FRICTION FACTOR = 0.021715 FREM = 21.5682  
 REM = 1011.6 GRM = 0.11061E+07 UPSTREAM BULK TEMPERATURE = 15.65 DEG C DOWN STREAM BULK TEMPERATURE = 17.85 DEG C  
 PRM = 7.585 RAM = 0.83894E+07 INLET BULK TEMPERATURE = 15.65 DEG C OUTLET BULK TEMPERATURE = 17.85 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVER- AGE						A	B	C	T	H	T+H
0	0.1	16.53	16.55	16.47	16.51	15.65	982.8	7.86	0.761E+07	0.00000	31.26	30.65	33.66	32.10	32.31	32.20
1	1.5	16.68	16.70	16.62	16.66	15.66	982.9	7.86	0.761E+07	0.00006	26.93	26.29	28.50	27.36	27.55	27.46
2	2.5	16.78	16.81	16.73	16.77	15.66	983.1	7.86	0.761E+07	0.00011	24.58	23.94	25.78	24.84	25.02	24.93
3	5.5	17.08	17.13	17.04	17.09	15.68	983.4	7.85	0.762E+07	0.00023	19.48	18.88	20.03	19.45	19.46	19.46
4	15.5	17.52	17.56	17.39	17.49	15.73	984.6	7.84	0.766E+07	0.00066	15.27	14.90	16.39	15.50	15.52	15.51
5	25.5	17.81	17.96	17.85	17.87	15.77	985.8	7.83	0.769E+07	0.00109	13.45	12.50	13.17	13.03	13.04	13.03
6	45.5	18.03	18.13	17.94	18.03	15.87	988.2	7.81	0.775E+07	0.00194	12.65	12.09	13.19	12.63	12.64	12.64
7	75.5	18.16	18.27	17.87	18.10	16.01	991.9	7.77	0.785E+07	0.00322	12.72	12.09	14.65	13.07	13.16	13.11
8	105.5	18.18	18.36	17.96	18.16	16.15	995.6	7.74	0.795E+07	0.00451	13.46	12.34	15.11	13.54	13.64	13.59
9	135.5	18.29	18.52	18.14	18.32	16.29	999.3	7.70	0.805E+07	0.00579	13.61	12.22	14.75	13.45	13.53	13.49
10	165.2	18.43	18.62	18.40	18.49	16.43	1002.9	7.67	0.815E+07	0.00707	13.61	12.44	13.84	13.26	13.29	13.28
11	205.2	18.58	18.82	18.43	18.61	16.61	1008.0	7.62	0.829E+07	0.00879	13.86	12.39	15.04	13.68	13.76	13.72
12	245.2	18.79	18.96	18.57	18.77	16.80	1013.0	7.57	0.843E+07	0.01051	13.68	12.63	15.45	13.83	13.92	13.87
13	275.2	18.79	19.08	18.68	18.85	16.94	1016.4	7.54	0.852E+07	0.01181	14.72	12.76	15.65	14.27	14.38	14.33
14	305.2	18.81	19.13	18.74	18.89	17.08	1019.7	7.52	0.862E+07							

XP045-04

INPUT ELECTRIC POWER = 285.0 W HEAT RATE GAINED BY WATER = 285.1 W HEAT BALANCE ERROR = -0.04%  
 MASS FLOW RATE = 16.9880 G/S PRESSURE DROP = 0.4450 MM H2O FRICTION FACTOR = 0.044231 FREM = 22.1981  
 REM = 501.9 GRM = 0.10866E+07 UPSTREAM BULK TEMPERATURE = 15.73 DEG C DOWN STREAM BULK TEMPERATURE = 19.76 DEG C  
 PRM = 7.392 RAM = 0.80315E+07 INLET BULK TEMPERATURE = 15.74 DEG C OUTLET BULK TEMPERATURE = 19.75 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVER-AGE						A	B	C	T	H	T+H
0	0.1	16.72	16.76	16.65	16.70	15.74	477.6	7.84	0.679E+07	0.00000	24.80	23.79	26.49	25.18	25.39	25.29
1	1.5	16.87	16.93	16.82	16.87	15.75	477.7	7.83	0.680E+07	0.00013	21.61	20.54	22.75	21.71	21.91	21.81
2	2.5	16.98	17.05	16.93	16.98	15.76	477.8	7.83	0.680E+07	0.00022	19.85	18.77	20.73	19.82	20.02	19.92
3	5.5	17.31	17.41	17.27	17.33	15.79	478.1	7.83	0.682E+07	0.00048	15.95	14.92	16.37	15.72	15.75	15.73
4	15.5	17.79	17.90	17.73	17.81	15.87	479.2	7.80	0.687E+07	0.00136	12.60	11.95	13.06	12.52	12.53	12.53
5	25.5	18.08	18.29	18.07	18.15	15.96	480.3	7.78	0.693E+07	0.00224	11.39	10.37	11.45	11.05	11.07	11.06
6	45.5	18.25	18.41	18.11	18.25	16.13	482.5	7.74	0.704E+07	0.00401	11.41	10.63	12.25	11.39	11.43	11.41
7	75.5	18.32	18.49	18.09	18.30	16.39	485.7	7.68	0.720E+07	0.00666	12.49	11.50	14.16	12.63	12.72	12.67
8	105.5	18.40	18.64	18.23	18.42	16.64	489.1	7.61	0.736E+07	0.00932	13.75	12.11	15.19	13.57	13.69	13.63
9	135.5	18.63	18.86	18.63	18.71	16.90	492.3	7.55	0.753E+07	0.01199	13.97	12.35	13.93	13.37	13.42	13.40
10	165.2	18.98	19.18	18.85	19.00	17.15	495.2	7.50	0.768E+07	0.01463	13.19	11.93	14.23	13.05	13.12	13.08
11	205.2	19.19	19.37	19.09	19.22	17.49	499.0	7.44	0.788E+07	0.01818	14.20	12.85	15.08	13.98	14.04	14.01
12	245.2	19.35	19.52	19.12	19.33	17.84	502.9	7.37	0.809E+07	0.02175	15.94	14.35	18.76	16.15	16.35	16.25
13	275.2	19.52	19.74	19.46	19.57	18.09	505.9	7.33	0.824E+07	0.02442	16.93	14.60	17.62	16.28	16.38	16.33
14	305.2	19.75	19.97	19.57	19.76	18.35	508.9	7.28	0.840E+07	0.02711	17.20	14.90	19.71	17.05	17.27	17.16
15	333.3	20.15	20.42	20.02	20.20	18.59	511.8	7.23	0.855E+07	0.02962	15.43	13.14	16.81	14.97	15.13	15.05
16	363.3	20.41	20.64	20.33	20.46	18.85	514.9	7.18	0.871E+07	0.03231	15.43	13.38	16.18	14.90	15.00	14.95
17	383.3	20.65	20.81	20.49	20.65	19.02	516.9	7.15	0.882E+07	0.03410	14.73	13.41	16.35	14.73	14.83	14.78
18	403.3	20.88	21.04	20.65	20.86	19.19	519.0	7.12	0.893E+07	0.03590	14.23	12.99	16.45	14.42	14.56	14.49
19	423.3	21.06	21.26	20.91	21.07	19.36	521.1	7.09	0.904E+07	0.03770	14.17	12.67	15.49	14.02	14.11	14.07
20	443.3	21.13	21.32	20.98	21.14	19.53	523.3	7.06	0.915E+07	0.03950	14.99	13.45	16.54	14.89	14.99	14.94
21	463.3	21.26	21.46	21.17	21.30	19.70	525.4	7.02	0.926E+07	0.04130	15.46	13.65	16.33	15.06	15.15	15.10
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 20.71 20.91 20.56 20.73 19.09 517.8 7.14 0.887E+07 0.03485 14.83 13.17 16.31 14.66 14.77 14.71																

XP045-05

INPUT ELECTRIC POWER = 585.0 W HEAT RATE GAINED BY WATER = 571.0 W HEAT BALANCE ERROR = 2.40%  
 MASS FLOW RATE = 52.3660 G/S PRESSURE DROP = 1.3743 MM H2O FRICTION FACTOR = 0.014377 FREM = 21.9849  
 REM = 1529.1 GRM = 0.20658E+07 UPSTREAM BULK TEMPERATURE = 15.93 DEG C DOWN STREAM BULK TEMPERATURE = 18.54 DEG C  
 PRM = 7.488 RAM = 0.15469E+08 INLET BULK TEMPERATURE = 15.93 DEG C OUTLET BULK TEMPERATURE = 18.54 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVER-AGE						A	B	C	T	H	T+H
0	0.1	17.14	17.13	17.01	17.08	15.93	1479.5	7.79	0.138E+08	0.00000	40.24	40.32	45.12	42.06	42.70	42.38
1	1.5	17.37	17.37	17.22	17.32	15.94	1479.8	7.79	0.139E+08	0.00004	33.82	33.89	37.82	35.23	35.84	35.53
2	2.5	17.54	17.53	17.37	17.48	15.94	1480.0	7.79	0.139E+08	0.00007	30.46	30.53	34.02	31.68	32.26	31.97
3	5.5	18.63	18.02	17.82	17.96	15.96	1480.7	7.78	0.139E+08	0.00016	23.47	23.54	26.14	24.32	24.38	24.35
4	15.5	18.46	18.63	18.29	18.46	16.02	1482.8	7.77	0.139E+08	0.00044	19.89	18.54	21.35	19.86	19.92	19.89
5	25.5	18.81	18.96	18.79	18.86	16.07	1485.0	7.75	0.140E+08	0.00073	17.71	16.79	17.82	17.43	17.44	17.43
6	45.5	18.97	19.13	18.83	18.98	16.18	1489.3	7.73	0.142E+08	0.00130	17.42	16.44	18.34	17.36	17.40	17.38
7	75.5	19.37	19.54	19.10	19.34	16.35	1495.9	7.68	0.144E+08	0.00216	16.04	15.20	17.62	16.23	16.29	16.26
8	105.5	19.51	19.86	19.62	19.66	16.52	1502.6	7.64	0.146E+08	0.00302	16.18	14.49	15.60	15.39	15.42	15.41
9	135.5	19.85	20.08	19.63	19.85	16.68	1509.3	7.60	0.148E+08	0.00389	15.29	14.27	16.42	15.27	15.33	15.30
10	165.2	20.09	20.51	20.07	20.23	16.85	1516.0	7.56	0.150E+08	0.00474	14.91	13.22	15.02	14.33	14.38	14.36
11	205.2	20.19	20.59	19.98	20.26	17.07	1523.6	7.52	0.153E+08	0.00589	15.49	13.73	16.61	15.18	15.28	15.23
12	245.2	20.46	20.79	20.24	20.50	17.29	1531.2	7.48	0.155E+08	0.00705	15.29	13.81	16.41	15.09	15.17	15.13
13	275.2	20.57	20.97	20.36	20.63	17.46	1537.0	7.45	0.157E+08	0.00791	15.53	13.76	16.69	15.23	15.33	15.28
14	305.2	20.53	20.97	20.36	20.62	17.63	1542.9	7.41	0.159E+08	0.00878	16.65	14.46	17.70	16.16	16.27	16.21
15	333.3	21.14	21.59	20.86	21.20	17.78	1548.4	7.38	0.161E+08	0.00959	14.36	12.67	15.69	14.13	14.24	14.19
16	363.3	21.07	21.53	20.78	21.13	17.95	1554.3	7.35	0.163E+08	0.01046	15.46	13.47	17.03	15.18	15.32	15.25
17	383.3	21.21	21.64	20.88	21.24	18.06	1558.3	7.33	0.165E+08	0.01104	15.34	13.46	17.13	15.16	15.31	15.24
18	403.3	21.32	21.76	20.98	21.36	18.17	1562.3	7.31	0.166E+08	0.01161	15.31	13.44	17.16	15.15	15.30	15.23
19	423.3	21.44	21.87	21.19	21.50	18.28	1566.3	7.29	0.167E+08	0.01219	15.25	13.46	16.56	14.98	15.09	15.04
20	443.3	21.53	21.93	21.21	21.56	18.39	1570.4	7.27	0.169E+08	0.01277	15.39	13.62	17.15	15.25	15.39	15.32
21	463.3	21.53	22.02	21.23	21.59	18.51	1574.4	7.25	0.170E+08	0.01335	15.92	13.70	17.70	15.60	15.78	15.69
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 21.29 21.72 20.98 21.33 18.11 1560.0 7.32 0.165E+08 0.01128 15.19 13.35 16.79 14.98 15.11 15.04																

XP045-06

INPUT ELECTRIC POWER = 565.0 W HEAT RATE GAINED BY WATER = 548.4 W HEAT BALANCE ERROR = 2.94%  
 MASS FLOW RATE = 34.3160 G/S PRESSURE DROP = 0.9037 MM H2O FRICTION FACTOR = 0.022012 FREM = 22.3940  
 REM = 1017.3 GRM = 0.21222E+07 UPSTREAM BULK TEMPERATURE = 15.98 DEG C DOWN STREAM BULK TEMPERATURE = 19.81 DEG C  
 PRM = 7.363 RAM = 0.15626E+08 INLET BULK TEMPERATURE = 15.99 DEG C OUTLET BULK TEMPERATURE = 19.81 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVER-AGE						A	B	C	T	H	T+H
0	0.1	17.38	17.40	17.26	17.33	15.99	971.0	7.78	0.134E+08	0.00000	33.46	32.94	36.62	34.61	34.91	34.76
1	1.5	17.63	17.66	17.51	17.59	16.00	971.3	7.77	0.134E+08	0.00007	28.64	28.11	30.82	29.32	29.60	29.46
2	2.5	17.80	17.83	17.69	17.76	16.01	971.5	7.77	0.134E+08	0.00011	26.05	25.52	27.78	26.53	26.78	26.66
3	5.5	18.31	18.36	18.21	18.29	16.03	972.1	7.76	0.134E+08	0.00024	20.50	20.01	21.44	20.63	20.65	20.64
4	15.5	19.01	19.19	18.79	19.00	16.11	974.2	7.74	0.135E+08	0.00068	16.08	15.12	17.41	16.15	16.20	16.18
5	25.5	19.48	19.63	19.41	19.50	16.20	976.3	7.72	0.136E+08	0.00111	14.19	13.57	14.51	14.08	14.09	14.08
6	45.5	19.63	19.86	19.50	19.66	16.36	980.5	7.68	0.138E+08	0.00199	14.21	13.31	14.82	14.09	14.11	14.10
7	75.5	19.87	20.09	19.49	19.82	16.60	986.9	7.62	0.141E+08	0.00330	14.23	13.32	16.12	14.47	14.56	14.51
8	105.5	19.96	20.31	19.62	19.96	16.85	993.4	7.56	0.144E+08	0.00462	14.96	13.42	16.75	14.92	15.04	14.98
9	135.5	20.18	20.57	20.02	20.26	17.09	998.9	7.51	0.147E+08	0.00594	15.03	13.35	15.85	14.67	14.74	14.70
10	165.2	20.43	20.84	20.35	20.54	17.33	1004.3	7.47	0.150E+08	0.00724	15.00	13.22	15.39	14.48	14.54	14.51
11	205.2	20.58	21.04	20.43	20.69	17.66	1011.8	7.41	0.153E+08	0.00901	15.87	13.73	16.72	15.33	15.44	15.38
12	245.2	21.01	21.41	20.68	21.03	17.99	1019.4	7.35	0.157E+08	0.01077	15.32	13.55	17.18	15.21	15.35	15.28
13	275.2	21.07	21.53	20.86	21.15	18.23	1025.2	7.30	0.160E+08	0.01210	16.30	14.04	17.64	15.85	15.99	15.92
14	305.2	21.19	21.58	20.97	21.25	18.47	1031.0	7.25	0.163E+08	0.01342	17.02	14.91	18.57	16.70	16.83	16.77
15	333.3	21.70	22.16													

XP045-07

INPUT ELECTRIC POWER = 485.0 W HEAT RATE GAINED BY WATER = 473.9 W HEAT BALANCE ERROR = 2.28%  
 MASS FLOW RATE = 16.2660 G/S PRESSURE DROP = 0.4416 MM H2O FRICTION FACTOR = 0.047861 FREM = 23.8212  
 REM = 497.7 GRM = 0.20983E+07 UPSTREAM BULK TEMPERATURE = 15.76 DEG C DOWN STREAM BULK TEMPERATURE = 22.74 DEG C  
 PRM = 7.108 RAM = 0.14915E+08 INLET BULK TEMPERATURE = 15.77 DEG C OUTLET BULK TEMPERATURE = 22.74 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	17.37	17.41	17.28	17.35	15.77	457.7	7.83	0.113E+08	0.00000	25.23	24.55	26.67	25.58	25.78	25.68
1	1.5	17.63	17.69	17.54	17.62	15.79	457.9	7.82	0.113E+08	0.00014	21.90	21.19	23.03	22.08	22.29	22.18
2	2.5	17.81	17.89	17.72	17.80	15.81	458.1	7.82	0.114E+08	0.00023	20.07	19.36	21.05	20.18	20.38	20.28
3	5.5	18.36	18.47	18.26	18.36	15.85	458.6	7.81	0.114E+08	0.00051	16.06	15.38	16.72	16.03	16.05	16.04
4	15.5	19.01	19.25	18.90	19.05	16.00	460.4	7.77	0.116E+08	0.00142	13.38	12.39	13.89	13.19	13.22	13.21
5	25.5	19.37	19.74	19.29	19.47	16.15	462.2	7.74	0.117E+08	0.00235	12.51	11.21	12.80	12.13	12.18	12.16
6	45.5	19.63	19.86	19.39	19.53	16.45	465.9	7.66	0.120E+08	0.00419	12.62	11.80	13.67	12.65	12.70	12.67
7	75.5	19.82	20.09	19.49	19.80	16.89	471.4	7.55	0.125E+08	0.00698	13.74	12.55	15.47	13.82	13.92	13.87
8	105.5	20.07	20.48	19.73	20.09	17.34	476.1	7.47	0.129E+08	0.00976	14.71	12.78	16.75	14.57	14.75	14.66
9	135.5	20.52	20.79	20.24	20.52	17.78	481.0	7.38	0.134E+08	0.01255	14.67	13.31	16.29	14.66	14.76	14.71
10	165.2	20.97	21.23	20.68	20.93	18.23	485.9	7.30	0.138E+08	0.01532	15.12	13.31	16.29	14.80	14.91	14.85
11	205.2	21.25	21.65	21.04	21.31	18.82	492.7	7.19	0.145E+08	0.01906	16.45	14.12	17.97	16.02	16.16	16.10
12	245.2	21.84	22.18	21.52	21.85	19.41	499.6	7.08	0.151E+08	0.02281	16.42	14.41	18.93	16.38	16.59	16.48
13	275.2	22.13	22.53	21.97	22.21	19.86	505.0	6.99	0.156E+08	0.02563	17.58	14.94	18.92	16.98	17.15	17.06
14	305.2	22.53	22.92	22.31	22.58	20.31	510.4	6.91	0.161E+08	0.02846	17.92	15.25	19.91	17.48	17.69	17.59
15	333.3	23.20	23.61	22.99	23.27	20.72	515.7	6.83	0.165E+08	0.03112	16.06	13.77	17.55	15.64	15.79	15.71
16	363.3	23.57	23.98	23.29	23.61	21.17	521.4	6.75	0.170E+08	0.03397	16.57	14.14	18.74	16.27	16.48	16.37
17	383.3	23.87	24.22	23.65	23.91	21.47	525.2	6.69	0.174E+08	0.03587	16.50	14.43	18.18	16.22	16.37	16.29
18	403.3	24.22	24.55	23.99	24.25	21.76	529.2	6.64	0.177E+08	0.03778	16.13	14.23	17.82	15.93	16.06	15.99
19	423.3	24.55	24.87	24.31	24.58	22.06	532.8	6.59	0.180E+08	0.03968	15.92	14.10	17.61	15.74	15.88	15.81
20	443.3	24.76	25.12	24.49	24.79	22.36	536.2	6.54	0.184E+08	0.04158	16.52	14.33	18.58	16.29	16.48	16.38
21	463.3	24.98	25.41	24.73	25.04	22.66	539.7	6.49	0.187E+08	0.04349	17.00	14.36	19.04	16.58	16.80	16.69
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
	391.6	24.03	24.39	23.79	24.07	21.59	526.7	6.67	0.175E+08	0.03667	16.28	14.17	18.08	16.01	16.18	16.09

XP045-08

INPUT ELECTRIC POWER = 985.0 W HEAT RATE GAINED BY WATER = 960.4 W HEAT BALANCE ERROR = 2.50%  
 MASS FLOW RATE = 31.4280 G/S PRESSURE DROP = 0.7877 MM H2O FRICTION FACTOR = 0.022862 FREM = 22.8180  
 REM = 998.1 GRM = 0.49351E+07 UPSTREAM BULK TEMPERATURE = 17.13 DEG C DOWN STREAM BULK TEMPERATURE = 24.46 DEG C  
 PRM = 6.818 RAM = 0.33649E+08 INLET BULK TEMPERATURE = 17.14 DEG C OUTLET BULK TEMPERATURE = 24.45 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	19.50	19.58	19.36	19.47	17.14	915.8	7.51	0.258E+08	0.00000	34.48	33.42	36.67	35.00	35.31	35.15
1	1.5	19.94	20.04	19.79	19.91	17.16	916.3	7.50	0.259E+08	0.00007	29.36	28.29	30.99	29.60	29.91	29.76
2	2.5	20.24	20.36	20.09	20.22	17.18	916.6	7.50	0.259E+08	0.00012	26.63	25.57	28.00	26.76	27.05	26.91
3	5.5	21.13	21.32	20.97	21.14	17.23	917.6	7.49	0.260E+08	0.00026	20.82	19.86	21.72	20.77	20.80	20.79
4	15.5	22.17	22.63	21.85	22.21	17.38	920.9	7.46	0.263E+08	0.00074	17.00	15.50	18.20	16.83	16.90	16.86
5	25.5	22.83	23.29	22.67	22.93	17.54	924.1	7.43	0.266E+08	0.00122	15.35	14.12	15.83	15.06	15.10	15.08
6	45.5	23.01	23.41	22.62	23.01	17.85	930.7	7.37	0.273E+08	0.00218	15.73	14.60	17.04	15.73	15.79	15.76
7	75.5	23.31	23.76	22.72	23.26	18.32	940.8	7.28	0.282E+08	0.00362	16.26	14.91	18.41	16.41	16.53	16.47
8	105.5	23.52	24.32	23.07	23.64	18.79	951.1	7.20	0.292E+08	0.00507	17.12	14.63	18.90	16.70	16.88	16.79
9	135.5	24.13	24.74	23.68	24.18	19.25	961.6	7.11	0.302E+08	0.00652	16.60	14.75	18.26	16.41	16.54	16.47
10	165.2	24.37	25.12	24.02	24.50	19.72	972.3	7.02	0.312E+08	0.00796	17.37	14.95	18.76	16.88	17.03	16.95
11	205.2	24.58	25.43	24.10	24.70	20.34	987.0	6.90	0.326E+08	0.00991	19.01	15.85	21.45	18.48	18.77	18.63
12	245.2	25.44	26.08	24.81	25.44	20.96	1002.2	6.79	0.340E+08	0.01186	17.97	15.75	20.93	17.97	18.22	18.09
13	275.2	25.73	26.48	25.36	25.86	21.43	1013.9	6.70	0.351E+08	0.01333	18.70	15.94	20.49	18.18	18.38	18.28
14	305.2	25.97	26.76	25.59	26.11	21.90	1025.7	6.61	0.362E+08	0.01480	19.75	16.54	21.74	19.10	19.34	19.22
15	333.3	26.81	27.69	26.46	26.98	22.34	1035.6	6.54	0.372E+08	0.01618	17.96	14.99	19.48	17.27	17.48	17.37
16	363.3	27.17	27.97	26.69	27.28	22.80	1046.3	6.47	0.382E+08	0.01766	18.38	15.51	20.61	17.92	18.17	18.04
17	383.3	27.42	28.13	26.81	27.45	23.12	1053.6	6.42	0.389E+08	0.01864	18.60	15.98	21.68	18.47	18.76	18.61
18	403.3	27.62	28.34	26.99	27.65	23.43	1061.0	6.37	0.396E+08	0.01963	19.09	16.28	22.45	18.94	19.27	19.11
19	423.3	27.99	28.77	27.43	28.06	23.74	1068.4	6.32	0.403E+08	0.02062	18.83	15.91	21.66	18.50	18.80	18.65
20	443.3	28.27	29.04	27.66	28.32	24.05	1076.0	6.27	0.410E+08	0.02161	18.96	16.03	22.15	18.72	19.05	18.88
21	463.3	28.72	29.39	28.24	28.78	24.36	1083.7	6.22	0.418E+08	0.02260	18.35	15.89	20.58	18.07	18.27	18.17
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
	391.6	27.55	28.32	27.01	27.63	23.25	1056.8	6.40	0.392E+08	0.01905	18.64	15.79	21.34	18.30	18.59	18.44

XP045-09

INPUT ELECTRIC POWER = 1250.0 W HEAT RATE GAINED BY WATER = 1207.6 W HEAT BALANCE ERROR = 3.39%  
 MASS FLOW RATE = 50.2000 G/S PRESSURE DROP = 1.2907 MM H2O FRICTION FACTOR = 0.014688 FREM = 22.3555  
 REM = 1522.0 GRM = 0.51455E+08 UPSTREAM BULK TEMPERATURE = 15.98 DEG C DOWN STREAM BULK TEMPERATURE = 21.74 DEG C  
 PRM = 7.181 RAM = 0.36952E+08 INLET BULK TEMPERATURE = 15.99 DEG C OUTLET BULK TEMPERATURE = 21.74 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	18.47	18.45	18.23	18.36	15.99	1420.5	7.78	0.294E+08	0.00000	41.39	41.68	45.87	43.22	43.70	43.46
1	1.5	18.96	18.93	18.69	18.85	16.01	1421.1	7.77	0.295E+08	0.00004	34.78	35.07	38.23	36.13	36.58	36.36
2	2.5	19.29	19.26	19.01	19.18	16.02	1421.6	7.77	0.295E+08	0.00007	31.33	31.61	34.29	32.46	32.88	32.67
3	5.5	20.31	20.26	19.97	20.18	16.06	1423.0	7.76	0.296E+08	0.00016	24.14	24.40	26.20	24.88	24.91	24.90
4	15.5	21.23	21.73	20.79	21.25	16.18	1427.6	7.73	0.299E+08	0.00046	20.31	18.48	22.25	20.23	20.35	20.29
5	25.5	21.94	22.29	21.68	21.97	16.30	1432.2	7.70	0.303E+08	0.00076	18.19	17.11	19.07	18.08	18.12	18.10
6	45.5	22.07	22.41	21.67	22.05	16.55	1441.6	7.64	0.309E+08	0.00136	18.54	17.47	19.99	18.61	18.67	18.64
7	75.5	22.75	23.20	22.16	22.70	16.92	1455.4	7.55	0.319E+08	0.00226	17.54	16.28	19.51	17.68	17.78	17.73
8	105.5	23.07	23.87	22.46	23.13	17.28	1467.5	7.48	0.328E+08	0.00316	17.66	15.52	19.77	17.48	17.65	17.56
9	135.5	23.68	24.29	23.13	23.70	17.65	1479.9	7.41	0.338E+08	0.00406	16.93	15.38	18.63	16.88	16.98	16.93
10	165.2	23.98	24.73	23.58	24.10	18.02	1492.3	7.34	0.347E+08	0.00496	17.12	15.19	18.35	16.78	16.88	16.83
11	205.2	23.92	24.82	23.49	24.07	18.51	1509.3	7.25	0.360E+08	0.00617	18.84	16.15	20.45	18.30	18.48	18.39
12	245.2	24.56	25.41	24.03	24.66	19.00	1526.8	7.16	0.373E+08	0.00738	18.31	15.87	20.23	17.96	18.14	18.05
13	275.2	24.79	25.76	24.30	24.95	19.36	1540.1	7.09	0.383E+08	0.00829	18.75	15.91	20.60	18.21	18.42	18.32
14	305.2	24.91	25.81	24.42	25.05	19.73	1553.7	7.02	0.393E+0							

XP045-10

INPUT ELECTRIC POWER = 545.0 W HEAT RATE GAINED BY WATER = 531.1 W HEAT BALANCE ERROR = 2.54%  
 MASS FLOW RATE = 13.1350 G/S PRESSURE DROP = 0.2950 MM H2O FRICTION FACTOR = 0.048923 FREM = 24.3521  
 REM = 497.8 GRM = 0.51270E+07 UPSTREAM BULK TEMPERATURE = 23.74 DEG C DOWN STREAM BULK TEMPERATURE = 33.44 DEG C  
 PRM = 5.611 RAM = 0.28766E+08 INLET BULK TEMPERATURE = 23.75 DEG C OUTLET BULK TEMPERATURE = 33.43 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-----NUSELT NUMBER-----			-----AVERAGE-----		
		A	B	C	AVER- AGE						A	B	C	T	H	T+H
0	0.1	25.37	25.41	25.25	25.33	23.75	446.7	6.32	0.223E+08	0.00001	27.40	26.68	29.64	28.07	28.34	28.20
1	1.5	25.66	25.72	25.54	25.63	23.78	447.0	6.31	0.224E+08	0.00017	23.59	22.83	25.23	23.96	24.22	24.09
2	2.5	25.86	25.93	25.74	25.84	23.80	447.2	6.31	0.224E+08	0.00029	21.52	20.77	22.88	21.75	22.01	21.89
3	5.5	26.46	26.57	26.34	26.46	23.87	447.8	6.30	0.225E+08	0.00064	17.04	16.34	17.88	17.06	17.09	17.08
4	15.5	26.99	27.24	26.74	26.99	24.07	449.9	6.27	0.227E+08	0.00181	15.16	13.93	16.54	15.14	15.21	15.17
5	25.5	27.47	27.84	27.22	27.51	24.28	452.1	6.24	0.230E+08	0.00298	13.83	12.39	15.04	13.67	13.75	13.71
6	45.5	27.61	27.97	27.36	27.64	24.69	456.4	6.17	0.235E+08	0.00531	15.12	13.48	16.57	14.95	15.06	15.00
7	75.5	27.96	28.30	27.63	27.96	25.31	463.0	6.07	0.244E+08	0.00883	16.67	14.75	19.02	16.63	16.81	16.72
8	105.5	28.36	28.83	27.97	28.39	25.93	469.8	5.98	0.252E+08	0.01236	18.15	15.16	21.61	17.93	18.31	18.12
9	135.5	28.91	29.29	28.62	28.94	26.55	476.8	5.88	0.261E+08	0.01590	18.65	16.01	21.20	18.38	18.62	18.50
10	165.2	29.41	29.85	29.14	29.47	27.16	483.4	5.79	0.269E+08	0.01941	19.52	16.33	22.14	19.03	19.33	19.18
11	205.2	30.20	30.65	29.89	30.25	27.99	491.6	5.69	0.280E+08	0.02415	19.81	16.46	23.05	19.40	19.77	19.59
12	245.2	31.32	31.74	31.00	31.35	28.81	500.1	5.58	0.291E+08	0.02889	17.47	14.92	20.00	17.22	17.46	17.34
13	275.2	31.90	32.27	31.63	31.93	29.43	506.7	5.50	0.299E+08	0.03246	17.70	15.41	19.85	17.46	17.65	17.56
14	305.2	32.35	32.77	32.06	32.39	30.05	513.4	5.43	0.308E+08	0.03604	18.98	16.03	21.71	18.62	18.91	18.76
15	333.3	33.02	33.45	32.77	33.08	30.63	519.9	5.35	0.316E+08	0.03939	18.22	15.46	20.35	17.78	18.01	17.90
16	363.3	33.76	34.14	33.44	33.78	31.25	527.0	5.28	0.325E+08	0.04299	17.37	15.06	19.88	17.21	17.44	17.33
17	383.3	34.09	34.49	33.85	34.14	31.66	531.8	5.22	0.331E+08	0.04539	17.92	15.35	19.88	17.52	17.72	17.62
18	403.3	34.48	34.82	34.12	34.47	32.08	536.4	5.17	0.337E+08	0.04781	18.08	15.84	21.22	18.12	18.38	18.25
19	423.3	34.92	35.39	34.68	35.00	32.49	540.7	5.13	0.342E+08	0.05024	17.83	14.96	19.77	17.29	17.52	17.40
20	443.3	35.24	35.75	35.01	35.33	32.90	545.1	5.08	0.348E+08	0.05268	18.54	15.21	20.60	17.83	18.12	17.98
21	463.3	35.68	36.22	35.37	35.76	33.32	549.6	5.03	0.353E+08	0.05513	18.32	14.89	21.04	17.72	18.08	17.90
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
	391.6	34.25	34.67	33.98	34.30	31.84	533.5	5.20	0.333E+08	0.04642	17.99	15.31	20.28	17.63	17.86	17.75

XP045-11

INPUT ELECTRIC POWER = 875.0 W HEAT RATE GAINED BY WATER = 841.5 W HEAT BALANCE ERROR = 3.83%  
 MASS FLOW RATE = 15.5440 G/S PRESSURE DROP = 0.3870 MM H2O FRICTION FACTOR = 0.045903 FREM = 23.4677  
 REM = 511.2 GRM = 0.49426E+07 UPSTREAM BULK TEMPERATURE = 15.76 DEG C DOWN STREAM BULK TEMPERATURE = 28.74 DEG C  
 PRM = 6.556 RAM = 0.32406E+08 INLET BULK TEMPERATURE = 15.78 DEG C OUTLET BULK TEMPERATURE = 28.73 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-----NUSELT NUMBER-----			-----AVERAGE-----		
		A	B	C	AVER- AGE						A	B	C	T	H	T+H
0	0.1	18.41	18.48	16.24	18.36	15.78	437.4	7.83	0.201E+08	0.00000	27.15	26.45	29.08	27.71	27.94	27.82
1	1.5	18.86	18.96	18.69	18.82	15.82	437.9	7.82	0.202E+08	0.00014	23.54	22.80	24.94	23.83	24.05	23.94
2	2.5	19.16	19.28	19.00	19.14	15.85	438.2	7.81	0.202E+08	0.00024	21.56	20.82	22.71	21.74	21.95	21.84
3	5.5	20.08	20.26	19.92	20.09	15.93	439.2	7.79	0.204E+08	0.00053	17.22	16.51	17.91	17.20	17.21	17.20
4	15.5	21.01	21.50	20.73	21.08	16.21	442.4	7.72	0.209E+08	0.00149	14.89	13.50	15.78	14.66	14.72	14.69
5	25.5	21.66	22.18	21.46	21.77	16.48	445.6	7.65	0.214E+08	0.00246	13.79	12.53	14.36	13.51	13.56	13.54
6	45.5	21.96	22.41	21.56	21.98	17.04	451.9	7.53	0.224E+08	0.00440	14.47	13.26	15.75	14.42	14.50	14.46
7	75.5	22.42	22.98	21.88	22.43	17.86	460.5	7.37	0.239E+08	0.00732	15.62	13.91	17.70	15.59	15.74	15.67
8	105.5	22.85	23.59	22.29	22.91	18.69	465.4	7.21	0.254E+08	0.01025	17.07	14.48	19.74	16.83	17.10	16.96
9	135.5	23.63	24.24	23.19	23.69	19.52	478.7	7.06	0.270E+08	0.01319	17.25	15.01	19.31	17.01	17.19	17.10
10	165.2	24.37	25.12	24.02	24.50	20.34	488.2	6.90	0.286E+08	0.01612	17.56	14.78	19.20	16.96	17.18	17.08
11	205.2	25.19	25.87	24.77	25.28	21.45	501.7	6.70	0.308E+08	0.02009	18.80	15.92	21.18	18.38	18.63	18.51
12	245.2	26.44	27.02	25.92	26.46	22.55	514.6	6.51	0.330E+08	0.02408	18.05	15.72	20.84	17.96	18.20	18.08
13	275.2	27.12	27.81	26.69	27.21	23.38	524.2	6.38	0.346E+08	0.02708	18.74	15.83	21.19	18.32	18.59	18.46
14	305.2	27.94	28.37	27.32	27.81	24.21	534.1	6.25	0.363E+08	0.03009	19.80	16.81	22.48	19.42	19.70	19.56
15	333.3	28.81	29.54	28.47	28.94	24.98	543.7	6.12	0.379E+08	0.03292	18.29	15.34	20.07	17.68	17.90	17.79
16	363.3	29.54	30.31	28.98	29.61	25.81	554.4	5.99	0.397E+08	0.03596	18.69	15.53	22.04	18.37	18.75	18.56
17	383.3	30.09	30.69	29.64	30.14	26.37	561.8	5.91	0.409E+08	0.03800	18.71	16.09	21.28	18.45	18.69	18.57
18	403.3	30.69	31.36	30.23	30.76	26.92	569.2	5.82	0.421E+08	0.04004	18.45	15.68	21.02	18.12	18.38	18.25
19	423.3	31.43	32.05	31.01	31.49	27.47	575.6	5.75	0.432E+08	0.04206	17.56	15.17	19.65	17.27	17.46	17.36
20	443.3	31.84	32.51	31.39	31.91	28.02	582.2	5.68	0.444E+08	0.04408	18.18	15.48	20.61	17.85	18.09	17.97
21	463.3	32.28	33.08	31.92	32.43	28.57	588.9	5.61	0.455E+08	0.04611	18.72	15.37	20.74	18.00	18.28	18.14
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
	391.6	30.40	31.08	29.95	30.48	26.60	564.5	5.88	0.414E+08	0.03884	18.31	15.55	20.78	17.96	18.21	18.08

XP045-12

INPUT ELECTRIC POWER = 1180.0 W HEAT RATE GAINED BY WATER = 1142.6 W HEAT BALANCE ERROR = 3.17%  
 MASS FLOW RATE = 40.8140 G/S PRESSURE DROP = 0.8934 MM H2O FRICTION FACTOR = 0.015353 FREM = 23.1197  
 REM = 1505.8 GRM = 0.10079E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 30.64 DEG C  
 PRM = 5.775 RAM = 0.58204E+08 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 30.64 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-----NUSELT NUMBER-----			-----AVERAGE-----		
		A	B	C	AVER- AGE						A	B	C	T	H	T+H
0	0.1	26.03	26.03	25.78	25.93	23.94	1393.9	6.29	0.485E+08	0.00000	45.56	45.47	51.64	47.82	48.58	48.20
1	1.5	26.50	26.50	26.22	26.40	23.96	1394.5	6.29	0.486E+08	0.00006	37.49	37.40	42.02	39.07	39.74	39.40
2	2.5	26.82	26.83	26.53	26.72	23.98	1395.0	6.28	0.486E+08	0.00009	33.41	33.33	37.24	34.69	35.30	35.00
3	5.5	27.79	27.81	27.44	27.68	24.02	1396.3	6.28	0.487E+08	0.00021	25.19	25.11	27.76	25.96	26.02	25.99
4	15.5	28.54	29.04	28.19	28.59	24.16	1400.9	6.25	0.491E+08	0.00058	21.72	19.47	23.57	21.46	21.59	21.52
5	25.5	29.20	29.73	28.88	29.27	24.30	1405.5	6.23	0.495E+08	0.00096	19.41	17.51	20.76	19.13	19.22	19.18
6	45.5	29.38	29.86	29.03	29.42	24.59	1414.7	6.19	0.503E+08	0.00171	19.82	18.04	21.40	19.66	19.75	19.70
7	75.5	29.84	30.36	29.19	29.79	25.02	1428.9	6.12	0.516E+08	0.00284	19.69	17.78	22.75	19.87	20.07	19.97
8	105.5	30.14	31.01	29.63	30.26	25.45	1443.3	6.05	0.528E+08	0.00397	20.21	17.05	22.65	19.70	19.97	19.83
9	135.5	30.91	31.63	30.51	31.01	25.88	1458.0	5.98	0.541E+08	0.00511	18.83	16.46	20.43	18.43	18.58	18.50
10	165.2	31.02	31.91	30.59	31.17	26.30	1472.8	5.92	0.553E+08	0.00624	20.06	16.88	22.06	19.43	19.67	19.55
11	205.2	31.42	32.26	30.78	31.49	26.87	1493.2	5.83	0.571E+08	0.00776	20.77	17.53	24.19	20.47	20.83	20.65
12	245.2	32.09	32.97	31.61	32.22	27.44	1510.6	5.76	0.586E+08	0.00928	20.29	17.09	22.64	19.74	20.01	19.87
13	275.2	32.46	33.32	32.02	32.60	27.87	1523.9	5.70	0.598E+08	0.01042	20.57	17.30	22.72	19.94	20.19	20.07
14	305.2	32.68	33.49	32.12	32.76	28.30	1537									

XP045-13

INPUT ELECTRIC POWER = 1065.0 W HEAT RATE GAINED BY WATER = 1015.0 W HEAT BALANCE ERROR = 4.70%  
 MASS FLOW RATE = 26.3740 G/S PRESSURE DROP = 0.5746 MM H2O FRICTION FACTOR = 0.023637 FREM = 23.5491  
 REM = 996.3 GRM = 0.96939E+07 UPSTREAM BULK TEMPERATURE = 23.82 DEG C DOWN STREAM BULK TEMPERATURE = 33.05 DEG C  
 PRM = 5.630 RAM = 0.54577E+08 INLET BULK TEMPERATURE = 23.83 DEG C OUTLET BULK TEMPERATURE = 33.04 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSSLET NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	26.14	26.17	25.90	26.05	23.83	898.5	6.31	0.428E+08	0.00000	36.65	36.24	40.82	38.20	38.64	38.42
1	1.5	26.61	26.65	26.37	26.53	23.86	899.1	6.30	0.429E+08	0.00009	30.76	30.35	33.66	31.73	32.11	31.92
2	2.5	26.93	26.98	26.70	26.86	23.88	899.5	6.30	0.430E+08	0.00015	27.69	27.30	30.02	28.41	28.76	28.59
3	5.5	27.91	27.97	27.67	27.85	23.94	900.7	6.29	0.431E+08	0.00032	21.31	20.96	22.68	21.62	21.65	21.64
4	15.5	28.76	28.76	28.42	28.78	24.14	904.8	6.26	0.435E+08	0.00090	18.27	16.83	19.74	18.20	18.28	18.24
5	25.5	29.37	29.84	29.05	29.42	24.33	908.8	6.23	0.441E+08	0.00148	16.75	15.32	17.90	16.59	16.66	16.62
6	45.5	29.55	30.02	29.08	29.55	24.73	917.1	6.17	0.451E+08	0.00265	17.49	15.93	19.36	17.48	17.59	17.53
7	75.5	29.89	30.52	29.41	29.94	25.32	929.8	6.07	0.466E+08	0.00440	18.40	16.18	20.57	18.20	18.38	18.29
8	105.5	30.36	31.23	29.74	30.44	25.91	942.8	5.98	0.481E+08	0.00616	18.88	15.80	21.91	18.53	18.86	18.70
9	135.5	30.85	31.52	30.29	30.89	26.49	956.2	5.89	0.497E+08	0.00792	19.29	16.73	22.14	19.13	19.38	19.26
10	165.2	31.07	31.85	30.53	31.15	27.08	968.9	5.80	0.512E+08	0.00967	21.00	17.58	24.28	20.59	20.95	20.77
11	205.2	31.64	32.54	31.11	31.76	27.86	984.5	5.70	0.531E+08	0.01202	22.15	17.91	25.79	21.47	21.95	21.71
12	245.2	32.76	33.52	32.22	32.84	28.65	1000.7	5.60	0.551E+08	0.01438	20.33	17.16	23.40	19.97	20.29	20.13
13	275.2	33.23	34.10	32.80	33.38	29.24	1013.2	5.53	0.567E+08	0.01616	20.90	17.17	23.44	20.17	20.50	20.34
14	305.2	33.63	34.44	33.12	33.73	29.83	1025.9	5.45	0.582E+08	0.01794	21.94	18.08	25.35	21.38	21.79	21.59
15	333.3	34.19	35.07	33.73	34.33	30.38	1038.2	5.39	0.597E+08	0.01961	21.86	17.75	24.87	21.08	21.49	21.29
16	363.3	34.70	35.64	34.28	34.87	30.97	1051.6	5.31	0.613E+08	0.02140	22.29	17.81	25.14	21.31	21.75	21.53
17	383.3	35.31	35.89	34.63	35.28	31.36	1060.8	5.26	0.624E+08	0.02259	21.04	18.35	25.44	21.23	21.61	21.42
18	403.3	35.59	36.27	35.07	35.64	31.76	1070.1	5.21	0.636E+08	0.02379	21.62	18.40	25.07	21.35	21.70	21.53
19	423.3	36.14	36.89	35.74	36.26	32.15	1078.5	5.17	0.646E+08	0.02499	20.78	17.47	23.06	20.17	20.44	20.30
20	443.3	36.46	37.32	36.01	36.59	32.54	1086.8	5.12	0.656E+08	0.02621	21.13	17.35	23.91	20.44	20.80	20.62
21	463.3	36.90	37.84	36.37	37.04	32.93	1095.2	5.07	0.665E+08	0.02742	20.87	16.85	24.07	20.16	20.60	20.38
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 35.40 36.18 34.91 35.50 31.53 1064.3 5.24 0.629E+08 0.02310 21.45 17.85 24.58 20.93 21.30 21.11																

XP045-14

INPUT ELECTRIC POWER = 1055.0 W HEAT RATE GAINED BY WATER = 1018.0 W HEAT BALANCE ERROR = 3.50%  
 MASS FLOW RATE = 26.3740 G/S PRESSURE DROP = 0.5780 MM H2O FRICTION FACTOR = 0.023775 FREM = 23.7759  
 REM = 1000.0 GRM = 0.98454E+07 UPSTREAM BULK TEMPERATURE = 23.99 DEG C DOWN STREAM BULK TEMPERATURE = 33.24 DEG C  
 PRM = 5.607 RAM = 0.55205E+08 INLET BULK TEMPERATURE = 24.00 DEG C OUTLET BULK TEMPERATURE = 33.24 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSSLET NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	26.28	26.35	26.03	26.20	24.00	901.9	6.28	0.434E+08	0.00000	37.10	36.00	41.82	38.55	39.18	38.87
1	1.5	26.75	26.84	26.48	26.67	24.03	902.5	6.28	0.434E+08	0.00009	31.19	30.10	34.60	32.06	32.62	32.34
2	2.5	27.06	27.18	26.79	27.00	24.05	902.9	6.27	0.435E+08	0.00015	28.11	27.04	30.92	28.73	29.25	28.99
3	5.5	28.02	28.19	27.72	27.98	24.11	904.1	6.26	0.436E+08	0.00032	21.67	20.73	23.44	21.89	21.95	21.92
4	15.5	28.76	29.38	28.53	28.89	24.30	908.2	6.23	0.441E+08	0.00090	19.00	16.67	20.05	18.46	18.58	18.52
5	25.5	29.54	29.96	29.22	29.57	24.50	912.3	6.20	0.446E+08	0.00148	16.80	15.52	17.95	16.70	16.76	16.73
6	45.5	29.66	30.19	29.31	29.72	24.90	920.7	6.14	0.456E+08	0.00265	17.75	15.98	19.18	17.53	17.63	17.58
7	75.5	29.95	30.52	29.36	29.94	25.49	933.5	6.05	0.471E+08	0.00440	18.92	16.77	21.83	18.95	19.17	19.01
8	105.5	30.36	31.23	29.86	30.48	26.08	946.7	5.95	0.487E+08	0.00616	19.69	16.38	22.33	19.15	19.46	19.31
9	135.5	31.07	31.79	30.62	31.16	26.67	960.2	5.86	0.503E+08	0.00792	19.13	16.43	21.31	18.74	18.96	18.85
10	165.2	31.46	32.29	31.03	31.60	27.25	972.4	5.78	0.518E+08	0.00967	19.99	16.69	22.26	19.37	19.65	19.51
11	205.2	32.20	32.98	31.50	32.23	28.04	988.2	5.68	0.537E+08	0.01203	20.20	16.99	24.28	20.06	20.49	20.28
12	245.2	32.90	33.69	32.39	33.02	28.83	1004.5	5.58	0.558E+08	0.01439	20.18	17.25	23.55	20.00	20.33	20.17
13	275.2	33.51	34.27	33.03	33.60	29.42	1017.1	5.51	0.573E+08	0.01617	20.47	17.28	23.21	20.02	20.32	20.17
14	305.2	33.85	34.67	33.34	33.95	30.01	1030.0	5.43	0.589E+08	0.01795	21.78	17.96	25.13	21.22	21.63	21.42
15	333.3	34.47	35.41	34.01	34.63	30.57	1042.4	5.36	0.604E+08	0.01962	21.41	17.25	24.28	20.57	20.98	20.77
16	363.3	35.03	35.86	34.61	35.17	31.16	1056.0	5.29	0.621E+08	0.02140	21.51	17.73	24.14	20.79	21.13	20.96
17	383.3	35.59	36.23	34.96	35.59	31.55	1065.2	5.24	0.632E+08	0.02260	20.63	17.81	24.43	20.61	20.96	20.79
18	403.3	35.93	36.66	35.46	36.01	31.95	1074.3	5.19	0.643E+08	0.02380	20.90	17.67	23.68	20.45	20.75	20.60
19	423.3	36.53	37.28	36.02	36.61	32.34	1082.5	5.14	0.652E+08	0.02501	19.85	16.82	22.58	19.47	19.75	19.61
20	443.3	36.80	37.60	36.28	36.89	32.73	1090.9	5.10	0.662E+08	0.02622	20.43	17.07	23.40	19.96	20.30	20.13
21	463.3	37.24	38.07	36.65	37.32	33.13	1099.4	5.05	0.672E+08	0.02744	20.18	16.78	23.56	19.79	20.17	19.98
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 35.72 36.51 35.22 35.82 31.72 1068.6 5.22 0.636E+08 0.02311 20.79 17.39 23.75 20.31 20.64 20.48																

XP045-15

INPUT ELECTRIC POWER = 995.0 W HEAT RATE GAINED BY WATER = 965.9 W HEAT BALANCE ERROR = 2.93%  
 MASS FLOW RATE = 20.5980 G/S PRESSURE DROP = 0.4569 MM H2O FRICTION FACTOR = 0.030800 FREM = 24.5650  
 REM = 797.6 GRM = 0.10006E+08 UPSTREAM BULK TEMPERATURE = 23.99 DEG C DOWN STREAM BULK TEMPERATURE = 35.23 DEG C  
 PRM = 5.482 RAM = 0.54850E+08 INLET BULK TEMPERATURE = 24.00 DEG C OUTLET BULK TEMPERATURE = 35.22 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSSLET NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	26.37	26.47	26.17	26.32	24.00	704.5	6.28	0.412E+08	0.00000	33.92	32.65	37.11	34.73	35.20	34.96
1	1.5	26.83	26.95	26.61	26.78	24.04	705.0	6.27	0.412E+08	0.00011	28.84	27.56	31.25	29.28	29.72	29.50
2	2.5	27.14	27.29	26.91	27.10	24.06	705.4	6.27	0.413E+08	0.00019	26.14	24.88	28.17	26.41	26.84	26.63
3	5.5	28.07	28.31	27.83	28.07	24.13	706.6	6.26	0.415E+08	0.00041	20.41	19.26	21.76	20.43	20.48	20.45
4	15.5	28.87	29.44	28.53	28.95	24.37	710.4	6.22	0.420E+08	0.00115	17.85	15.86	19.33	17.56	17.68	17.62
5	25.5	29.54	30.01	29.16	29.57	24.61	714.4	6.18	0.426E+08	0.00190	16.30	14.87	17.65	16.19	16.27	16.23
6	45.5	29.66	30.19	29.19	29.68	25.09	722.3	6.11	0.438E+08	0.00339	17.55	15.73	19.54	17.47	17.61	17.54
7	75.5	29.95	30.58	29.41	29.98	25.81	734.6	5.99	0.455E+08	0.00564	19.34	16.79	22.23	19.20	19.45	19.32
8	105.5	30.31	31.23	29.80	30.44	26.53	747.4	5.88	0.473E+08	0.00790	21.16	17.01	24.42	20.41	20.86	20.63
9	135.5	31.13	31.91	30.68	31.24	27.25	759.3	5.78	0.491E+08	0.01016	20.56	17.12	23.25	19.99	20.31	20.15
10	165.2	31.74	32.63	31.37	31.91	27.96	770.4	5.69	0.508E+08	0.01240	21.06	17.06	23.36	20.15	20.49	20.32
11	205.2	32.76	33.54	32.22	32.84	28.91	785.9	5.57	0.531E+08	0.01542	20.69	17.19	24.03	20.25	20.64	20.45
12	245.2	33.81	34.52	33.28	33.87	29.87	802.0	5.45	0.555E+08	0.01846	20.14	17.06	23.25	19.83	20.15	19.99
13	275.2	34.40	35.16	33.97	34.51	30.59	814.5	5.36	0.574E+08	0.02074	20.78	17.35	23.41	20.21	20.51	20.36
14	305.2	34.91	35.61	34.46	34.99	31.31	827.4	5.27	0.593E+08	0.02303	21.98	18.37	25.12	21.47	21.82	21.65
15	333.3	35.69	36.53	35.29	35.84	31.98	839									

XP045-16

INPUT ELECTRIC POWER = 865.0 W HEAT RATE GAINED BY WATER = 830.2 W HEAT BALANCE ERROR = 4.02%
MASS FLOW RATE = 12.2970 G/S PRESSURE DROP = 0.2626 MM H2O FRICTION FACTOR = 0.049621 FREM = 24.8573
REM = 500.9 GRM = 0.10103E+08 UPSTREAM BULK TEMPERATURE = 23.85 DEG C DOWN STREAM BULK TEMPERATURE = 40.04 DEG C
PRM = 5.188 RAM = 0.52416E+08 INLET BULK TEMPERATURE = 23.87 DEG C OUTLET BULK TEMPERATURE = 40.03 DEG C

Table with columns: STA-TION NO., Z CM, WALL TEMPERATURE (DEG C) A, B, C, AVERAGE, TB (C), RE, PR, RA, Z, NUSSELT NUMBER A, B, C, AVERAGE T, H, T+H. Rows 0-21 and average values.

XP045-17

INPUT ELECTRIC POWER = 1080.0 W HEAT RATE GAINED BY WATER = 1052.1 W HEAT BALANCE ERROR = 2.58%
MASS FLOW RATE = 13.3780 G/S PRESSURE DROP = 0.3205 MM H2O FRICTION FACTOR = 0.051258 FREM = 25.4930
REM = 497.3 GRM = 0.95243E+07 UPSTREAM BULK TEMPERATURE = 18.23 DEG C DOWN STREAM BULK TEMPERATURE = 37.09 DEG C
PRM = 5.728 RAM = 0.54551E+08 INLET BULK TEMPERATURE = 18.25 DEG C OUTLET BULK TEMPERATURE = 37.07 DEG C

Table with columns: STA-TION NO., Z CM, WALL TEMPERATURE (DEG C) A, B, C, AVERAGE, TB (C), RE, PR, RA, Z, NUSSELT NUMBER A, B, C, AVERAGE T, H, T+H. Rows 0-21 and average values.

XP045-18

INPUT ELECTRIC POWER = 1930.0 W HEAT RATE GAINED BY WATER = 1848.6 W HEAT BALANCE ERROR = 4.22%
MASS FLOW RATE = 38.6480 G/S PRESSURE DROP = 0.8934 MM H2O FRICTION FACTOR = 0.017107 FREM = 25.5998
REM = 1496.5 GRM = 0.19150E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 35.34 DEG C
PRM = 5.482 RAM = 0.10498E+09 INLET BULK TEMPERATURE = 23.89 DEG C OUTLET BULK TEMPERATURE = 35.34 DEG C

Table with columns: STA-TION NO., Z CM, WALL TEMPERATURE (DEG C) A, B, C, AVERAGE, TB (C), RE, PR, RA, Z, NUSSELT NUMBER A, B, C, AVERAGE T, H, T+H. Rows 0-21 and average values.

XP045-19

INPUT ELECTRIC POWER = 1695.0 W HEAT RATE GAINED BY WATER = 1594.7 W HEAT BALANCE ERROR = 5.92%  
 MASS FLOW RATE = 24.9300 G/S PRESSURE DROP = 0.6002 MM H2O FRICTION FACTOR = 0.027599 FREM = 27.7541  
 REM = 1005.6 GRM = 0.18818E+08 UPSTREAM BULK TEMPERATURE = 23.82 DEG C DOWN STREAM BULK TEMPERATURE = 39.16 DEG C  
 PRM = 5.245 RAM = 0.98696E+08 INLET BULK TEMPERATURE = 23.84 DEG C OUTLET BULK TEMPERATURE = 39.15 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C	AVER- AGE						A	B	C	T	H	T+H	
0	0.1	27.43	27.53	27.04	27.30	23.84	849.5	6.31	0.673E+08	0.00000	37.00	36.01	41.55	38.43	39.03	38.73	
1	1.5	28.13	28.27	27.72	28.01	23.89	850.4	6.30	0.675E+08	0.00009	31.29	30.30	34.66	32.18	32.73	32.45	
2	2.5	28.61	28.78	28.19	28.51	23.92	851.0	6.29	0.676E+08	0.00015	28.28	27.31	31.11	28.93	29.45	29.19	
3	5.5	30.07	30.32	29.60	29.99	24.02	852.9	6.28	0.680E+08	0.00034	21.95	21.08	23.78	22.21	22.27	22.24	
4	15.5	31.09	31.92	30.48	31.16	24.35	859.3	6.23	0.693E+08	0.00095	19.67	17.52	21.63	19.46	19.60	19.53	
5	25.5	32.05	32.89	31.49	32.14	24.67	865.8	6.23	0.706E+08	0.00157	17.96	16.12	19.44	17.74	17.84	17.79	
6	45.5	32.27	33.14	31.64	32.35	25.32	879.0	6.07	0.732E+08	0.00280	19.06	16.94	20.94	18.84	18.98	18.91	
7	75.5	32.99	33.96	32.14	33.05	26.30	899.7	5.92	0.772E+08	0.00467	19.74	17.25	22.63	19.63	19.87	19.75	
8	105.5	33.59	34.96	32.64	33.73	27.28	919.7	5.78	0.812E+08	0.00653	20.90	17.16	24.61	20.44	20.89	20.66	
9	135.5	34.52	35.68	33.62	34.61	28.26	938.4	5.65	0.851E+08	0.00840	21.02	17.72	24.56	20.73	21.10	20.92	
10	165.2	35.07	36.36	34.21	35.21	29.23	957.6	5.53	0.890E+08	0.01026	22.49	18.42	26.38	21.95	22.43	22.19	
11	205.2	36.03	37.26	35.06	36.12	30.54	984.8	5.36	0.945E+08	0.01278	23.81	19.46	28.93	23.45	24.07	23.76	
12	245.2	37.58	38.75	36.74	37.69	31.85	1013.5	5.20	0.100E+09	0.01530	22.73	18.89	26.65	22.31	22.76	22.54	
13	275.2	38.34	39.67	37.64	38.55	32.83	1033.0	5.09	0.104E+09	0.01723	23.60	19.01	27.03	22.73	23.21	22.97	
14	305.2	39.07	40.29	38.24	39.20	33.81	1053.3	4.97	0.108E+09	0.01917	24.66	20.01	29.23	24.05	24.64	24.35	
15	333.3	40.02	41.50	39.21	40.24	34.72	1073.0	4.87	0.112E+09	0.02100	24.46	19.10	28.88	23.46	24.15	23.80	
16	363.3	41.07	42.36	40.19	41.21	35.70	1094.8	4.75	0.116E+09	0.02297	24.07	19.39	28.79	23.47	24.08	23.78	
17	383.3	41.58	42.65	40.67	41.64	36.36	1109.9	4.68	0.119E+09	0.02429	24.66	20.48	29.87	24.42	25.00	24.71	
18	403.3	42.17	43.35	41.36	42.29	37.01	1124.9	4.61	0.122E+09	0.02561	24.93	20.30	29.58	24.35	24.94	24.64	
19	423.3	43.18	44.46	42.49	43.38	37.66	1138.6	4.55	0.125E+09	0.02691	23.28	18.91	26.64	22.49	22.94	22.72	
20	443.3	43.66	44.98	42.79	43.81	38.32	1152.7	4.49	0.128E+09	0.02821	24.04	19.26	28.70	23.37	24.00	23.69	
21	463.3	44.48	45.80	43.61	44.63	38.97	1167.1	4.42	0.131E+09	0.02951	23.28	18.77	27.62	22.65	23.22	22.94	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	41.95	43.22	41.12	42.09	36.63	1115.6	4.66	0.120E+09	0.02483	24.24	19.57	28.74	23.59	24.19	23.89

XP045-20

INPUT ELECTRIC POWER = 1810.0 W HEAT RATE GAINED BY WATER = 1738.9 W HEAT BALANCE ERROR = 3.93%  
 MASS FLOW RATE = 29.9840 G/S PRESSURE DROP = 0.7042 MM H2O FRICTION FACTOR = 0.022391 FREM = 26.6897  
 REM = 1192.0 GRM = 0.19599E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 37.78 DEG C  
 PRM = 5.328 RAM = 0.10443E+09 INLET BULK TEMPERATURE = 23.89 DEG C OUTLET BULK TEMPERATURE = 37.77 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C	AVER- AGE						A	B	C	T	H	T+H	
0	0.1	27.68	27.81	27.29	27.56	23.89	1022.9	6.30	0.736E+08	0.00000	38.20	37.00	42.62	39.50	40.11	39.81	
1	1.5	28.42	28.59	28.01	28.32	23.94	1023.9	6.29	0.738E+08	0.00008	32.28	31.08	35.55	33.06	33.62	33.34	
2	2.5	28.93	29.14	28.50	28.84	23.97	1024.6	6.29	0.739E+08	0.00013	29.16	27.99	31.90	29.72	30.24	29.98	
3	5.5	30.46	30.77	29.99	30.40	24.06	1026.7	6.27	0.743E+08	0.00028	22.61	21.56	24.39	22.80	22.86	22.83	
4	15.5	31.48	32.48	30.92	31.63	24.35	1033.6	6.22	0.756E+08	0.00079	20.29	17.80	22.01	19.88	20.03	19.96	
5	25.5	32.50	33.39	31.98	32.63	24.65	1040.7	6.18	0.768E+08	0.00130	18.40	16.52	19.70	18.11	18.21	18.16	
6	45.5	32.60	33.58	31.98	32.72	25.24	1055.1	6.08	0.794E+08	0.00233	19.61	17.30	21.40	19.29	19.44	19.36	
7	75.5	33.27	34.34	32.42	33.34	26.13	1077.5	5.94	0.834E+08	0.00388	20.16	17.53	22.90	19.96	20.20	20.08	
8	105.5	33.87	35.41	32.92	34.06	27.02	1100.2	5.81	0.874E+08	0.00543	20.98	17.13	24.36	20.40	20.83	20.61	
9	135.5	34.79	35.96	33.89	34.88	27.90	1120.2	5.70	0.912E+08	0.00698	20.82	17.81	23.95	20.55	20.86	20.71	
10	165.2	35.12	36.63	34.32	35.36	28.78	1140.9	5.59	0.950E+08	0.00853	22.59	18.24	25.87	21.78	22.23	22.01	
11	205.2	36.09	37.43	35.12	36.21	29.97	1169.9	5.44	0.100E+09	0.01061	23.33	19.14	27.73	22.87	23.40	23.13	
12	245.2	37.31	38.58	36.46	37.45	31.15	1200.4	5.29	0.106E+09	0.01271	23.14	19.16	26.82	22.61	23.04	22.83	
13	275.2	37.89	39.39	37.19	38.16	32.04	1223.6	5.18	0.110E+09	0.01429	24.27	19.34	27.57	23.22	23.73	23.48	
14	305.2	38.62	39.96	37.80	38.79	32.93	1244.9	5.07	0.114E+09	0.01589	24.90	20.18	29.10	24.18	24.73	24.45	
15	333.3	39.57	41.17	38.82	39.85	33.76	1265.6	4.98	0.118E+09	0.01740	24.34	19.10	27.88	23.23	23.81	23.52	
16	363.3	40.57	41.97	39.80	40.78	34.65	1288.5	4.88	0.122E+09	0.01902	23.83	19.27	27.40	23.02	23.50	23.26	
17	383.3	41.08	42.37	40.12	41.19	35.24	1304.2	4.81	0.124E+09	0.02011	24.12	19.76	28.91	23.69	24.26	23.98	
18	403.3	41.72	42.96	40.81	41.83	35.83	1320.3	4.74	0.127E+09	0.02121	23.90	19.74	28.30	23.47	23.98	23.72	
19	423.3	42.63	43.91	41.87	42.80	36.43	1336.8	4.67	0.130E+09	0.02231	22.65	18.78	25.80	22.04	22.41	22.22	
20	443.3	43.04	44.48	42.12	43.21	37.02	1353.1	4.60	0.133E+09	0.02341	23.30	18.81	27.49	22.65	23.20	22.92	
21	463.3	43.81	45.24	42.83	43.96	37.61	1368.0	4.55	0.136E+09	0.02448	22.62	18.37	26.83	22.07	22.61	22.34	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	41.44	42.81	40.59	41.61	35.49	1311.4	4.78	0.126E+09	0.02058	23.69	19.25	27.65	23.01	23.53	23.27

XP045-21

INPUT ELECTRIC POWER = 1650.0 W HEAT RATE GAINED BY WATER = 1599.5 W HEAT BALANCE ERROR = 3.06%  
 MASS FLOW RATE = 24.5690 G/S PRESSURE DROP = 0.5610 MM H2O FRICTION FACTOR = 0.026558 FREM = 26.4653  
 REM = 996.5 GRM = 0.19201E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 39.54 DEG C  
 PRM = 5.214 RAM = 0.10011E+09 INLET BULK TEMPERATURE = 23.95 DEG C OUTLET BULK TEMPERATURE = 39.53 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVER- AGE						A	B	C	T	H	T+H
0	0.1	27.65	27.82	27.26	27.54	23.95	839.3	6.29	0.680E+08	0.00000	36.00	34.45	40.27	37.10	37.75	37.42
1	1.5	28.35	28.59	27.94	28.27	24.00	840.2	6.28	0.682E+08	0.00009	30.59	29.02	33.79	31.20	31.80	31.50
2	2.5	28.84	29.12	28.41	28.77	24.03	840.9	6.27	0.683E+08	0.00016	27.72	26.18	30.41	28.11	28.68	28.40
3	5.5	30.29	30.71	29.82	30.27	24.13	842.8	6.26	0.687E+08	0.00034	21.62	20.23	23.40	21.68	21.75	21.71
4	15.5	31.20	32.26	30.64	31.37	24.47	849.2	6.21	0.700E+08	0.00097	19.75	17.07	21.53	19.27	19.45	19.36
5	25.5	32.27	33.17	31.82	32.42	24.80	855.7	6.15	0.713E+08	0.00159	17.78	15.87	18.94	17.44	17.53	17.48
6	45.5	32.43	33.36	31.82	32.54	25.46	869.1	6.05	0.740E+08	0.00285	19.04	16.80	20.88	18.76	18.91	18.83
7	75.5	33.05	34.07	32.08	33.07	26.46	890.0	5.89	0.781E+08	0.00474	20.09	17.39	23.54	20.03	20.34	20.19
8	105.5	33.59	35.02	32.64	33.75	27.46	909.6	5.75	0.821E+08	0.00663	21.54	17.47	25.49	21.00	21.50	21.25
9	135.5	34.52	35.74	33.73	34.66	28.45	928.4	5.63	0.861E+08	0.00853	21.74	18.09	24.99	21.23	21.61	21.42
10	165.2	35.29	36.63	34.54	35.49	29.44	947.9	5.50	0.901E+08	0.01042	22.49	18.28	25.80	21.75	22.19	21.97
11	205.2	36.31	37.65	35.34	36.43	30.77	975.4	5.34	0.958E+08	0.01297	23.66	19.06	28.70	23.15	23.81	23.48
12	245.2	37.86	39.08	37.02	37.99	32.10	1003.8	5.17	0.102E+09	0.01554	22.69	18.72	26.59	22.20	22.66	22.43
13	275.2	38.67	39.94	37.97	38.86	33.10	1023.5	5.06	0.106E+09	0.01750	23.39	19.04	26.74	22.61	23.06	22.83
14	305.2	39.46	40.68</													

XP045-22

INPUT ELECTRIC POWER = 1505.0 W HEAT RATE GAINED BY WATER = 1436.8 W HEAT BALANCE ERROR = 4.53%  
 MASS FLOW RATE = 19.1540 G/S PRESSURE DROP = 0.4552 MM H2O FRICTION FACTOR = 0.035439 FREM = 26.1852  
 REM = 795.3 GRM = 0.18561E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 41.92 DEG C  
 PRM = 5.075 RAM = 0.94191E+08 INLET BULK TEMPERATURE = 23.95 DEG C OUTLET BULK TEMPERATURE = 41.91 DEG C

STATION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSSLETT NUMBER			AVERAGE				
		A	B	C						A	B	C	T	H	T+H		
0	0.1	27.49	27.67	27.15	27.41	23.96	654.4	6.29	0.611E+08	0.00000	33.88	32.18	37.46	34.66	35.25	34.95	
1	1.5	28.14	28.41	27.79	28.09	24.01	655.2	6.28	0.613E+08	0.00012	28.95	27.22	31.68	29.32	29.88	29.60	
2	2.5	28.60	28.91	28.23	28.56	24.05	655.8	6.27	0.614E+08	0.00020	26.30	24.60	28.64	26.50	27.04	26.77	
3	5.5	29.96	30.43	29.54	29.98	24.16	657.5	6.25	0.618E+08	0.00044	20.65	19.09	22.22	20.57	20.65	20.61	
4	15.5	30.92	31.86	30.37	31.05	24.55	663.3	6.19	0.631E+08	0.00124	18.74	16.33	20.53	18.37	18.53	18.45	
5	25.5	31.88	32.78	31.43	32.03	24.93	669.2	6.13	0.645E+08	0.00204	17.17	15.20	18.35	16.80	16.91	16.65	
6	45.5	32.19	33.03	31.59	32.24	25.70	681.3	6.01	0.673E+08	0.00365	18.60	16.25	20.22	18.21	18.36	18.28	
7	75.5	32.78	33.74	31.86	32.79	26.85	700.4	5.83	0.717E+08	0.00608	20.03	17.24	23.69	19.98	20.32	20.15	
8	105.5	33.48	34.79	32.58	33.62	27.99	716.9	5.69	0.757E+08	0.00851	21.61	17.43	25.83	21.07	21.62	21.35	
9	135.5	34.57	35.74	33.84	34.72	29.14	734.3	5.54	0.798E+08	0.01095	21.77	17.92	25.17	21.21	21.62	21.42	
10	165.2	35.57	36.80	34.93	35.76	30.28	752.4	5.40	0.841E+08	0.01338	22.30	18.08	25.37	21.50	21.92	21.71	
11	205.2	37.03	38.15	36.17	37.12	31.81	778.1	5.20	0.902E+08	0.01667	22.51	18.54	26.95	22.14	22.67	22.40	
12	245.2	38.63	39.75	37.97	38.78	33.34	801.8	5.03	0.957E+08	0.02001	22.13	18.27	25.32	21.52	21.91	21.71	
13	275.2	39.56	40.72	38.92	39.73	34.49	820.5	4.89	0.999E+08	0.02255	23.03	18.73	26.38	22.27	22.71	22.49	
14	305.2	40.46	41.62	39.81	40.63	35.64	840.1	4.76	0.104E+09	0.02511	24.17	19.46	27.95	23.34	23.86	23.60	
15	333.3	41.91	43.18	41.16	42.08	36.72	859.3	4.64	0.109E+09	0.02753	22.37	17.96	26.12	21.64	22.15	21.89	
16	363.3	42.90	44.19	42.20	43.10	37.87	878.2	4.53	0.113E+09	0.03007	23.00	18.30	26.72	22.13	22.67	22.40	
17	383.3	43.58	44.66	42.72	43.66	38.63	891.0	4.46	0.116E+09	0.03176	23.35	19.18	28.27	23.02	23.60	23.31	
18	403.3	44.34	45.47	43.70	44.51	39.40	904.2	4.39	0.119E+09	0.03346	23.34	19.01	26.84	22.61	23.06	22.83	
19	423.3	45.46	46.58	44.83	45.62	40.17	917.7	4.31	0.123E+09	0.03516	21.79	17.97	24.72	21.13	21.49	21.31	
20	443.3	45.99	47.22	45.24	46.15	40.93	931.7	4.24	0.126E+09	0.03687	22.73	18.29	26.72	22.05	22.58	22.31	
21	463.3	46.82	48.10	46.12	47.01	41.70	946.2	4.17	0.129E+09	0.03859	22.45	17.95	26.01	21.63	22.13	21.88	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:		391.6	44.03	45.22	43.31	44.19	38.95	897.0	4.43	0.118E+09	0.03247	22.76	18.45	26.56	22.09	22.59	22.34

XP045-23

INPUT ELECTRIC POWER = 2340.0 W HEAT RATE GAINED BY WATER = 2246.2 W HEAT BALANCE ERROR = 4.01%  
 MASS FLOW RATE = 27.0960 G/S PRESSURE DROP = 0.6922 MM H2O FRICTION FACTOR = 0.026990 FREM = 26.9820  
 REM = 999.7 GRM = 0.19815E+08 UPSTREAM BULK TEMPERATURE = 17.35 DEG C DOWN STREAM BULK TEMPERATURE = 37.22 DEG C  
 PRM = 5.775 RAM = 0.11442E+09 INLET BULK TEMPERATURE = 17.37 DEG C OUTLET BULK TEMPERATURE = 37.21 DEG C

STATION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSSLETT NUMBER			AVERAGE				
		A	B	C						A	B	C	T	H	T+H		
0	0.1	22.35	22.59	21.90	22.24	17.37	793.7	7.46	0.615E+08	0.00000	38.22	36.46	42.00	39.08	39.67	39.38	
1	1.5	23.31	23.65	22.85	23.24	17.43	794.9	7.45	0.618E+08	0.00008	32.36	30.59	35.14	32.76	33.31	33.03	
2	2.5	23.97	24.38	23.50	23.93	17.48	795.6	7.44	0.620E+08	0.00014	29.27	27.54	31.58	29.47	29.99	29.73	
3	5.5	25.96	26.57	25.45	25.99	17.60	797.9	7.42	0.626E+08	0.00031	22.74	21.19	24.22	22.65	22.72	22.68	
4	15.5	27.38	28.82	26.74	27.65	18.03	805.7	7.34	0.646E+08	0.00086	20.30	17.58	21.77	19.73	19.89	19.81	
5	25.5	28.76	29.96	28.16	28.96	18.45	813.6	7.26	0.667E+08	0.00142	18.40	16.48	19.52	18.04	18.13	18.09	
6	45.5	28.94	30.19	28.14	29.09	19.30	829.9	7.10	0.709E+08	0.00254	19.61	17.37	21.39	19.31	19.46	19.39	
7	75.5	29.89	31.29	28.85	30.01	20.57	855.7	6.86	0.775E+08	0.00423	20.21	17.57	22.76	19.96	20.18	20.07	
8	105.5	30.64	32.56	29.30	30.83	21.84	883.1	6.62	0.844E+08	0.00593	21.34	17.52	25.17	20.88	21.35	21.11	
9	135.5	31.74	33.46	30.62	31.94	23.10	908.1	6.42	0.909E+08	0.00764	21.70	18.09	24.92	21.20	21.57	21.39	
10	165.2	32.46	34.41	31.26	32.71	24.36	934.3	6.22	0.977E+08	0.00935	23.06	18.59	27.10	22.38	22.92	22.65	
11	205.2	33.42	35.21	32.22	33.62	26.05	972.0	5.96	0.107E+09	0.01166	25.26	20.33	30.17	24.61	25.25	24.93	
12	245.2	35.42	37.08	34.29	35.60	27.75	1009.1	5.72	0.117E+09	0.01398	24.15	19.85	28.33	23.61	24.11	23.86	
13	275.2	36.51	38.33	35.64	36.82	29.02	1036.0	5.56	0.124E+09	0.01572	24.68	19.85	27.91	23.67	24.15	23.91	
14	305.2	37.35	39.01	36.24	37.53	30.29	1064.4	5.40	0.132E+09	0.01748	26.09	21.14	30.96	25.44	26.06	25.75	
15	333.3	38.80	40.83	37.75	39.13	31.47	1092.5	5.25	0.139E+09	0.01912	25.09	19.64	29.29	24.02	24.67	24.35	
16	363.3	40.07	41.92	38.91	40.30	32.74	1121.0	5.10	0.146E+09	0.02092	25.00	19.98	29.74	24.26	24.91	24.58	
17	383.3	40.64	42.37	39.45	40.82	33.59	1139.9	5.00	0.151E+09	0.02213	25.94	20.82	31.20	25.29	25.99	25.64	
18	403.3	41.61	43.35	40.53	41.83	34.44	1159.4	4.90	0.156E+09	0.02335	25.43	20.47	29.95	24.68	25.28	24.98	
19	423.3	42.85	44.57	41.82	43.08	35.28	1179.6	4.80	0.161E+09	0.02458	24.06	19.60	27.86	23.35	23.84	23.59	
20	443.3	43.48	45.32	42.29	43.70	36.13	1200.6	4.70	0.166E+09	0.02582	24.70	19.76	29.49	24.00	24.65	24.32	
21	463.3	44.31	46.31	43.06	44.56	36.98	1221.8	4.61	0.171E+09	0.02707	24.71	19.43	29.81	23.91	24.65	24.28	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:		391.6	41.24	43.06	40.12	41.48	33.94	1148.8	4.96	0.153E+09	0.02266	25.04	20.04	29.59	24.26	24.89	24.58

XP045-24

INPUT ELECTRIC POWER = 1245.0 W HEAT RATE GAINED BY WATER = 1200.1 W HEAT BALANCE ERROR = 3.60%  
 MASS FLOW RATE = 11.2120 G/S PRESSURE DROP = 0.2523 MM H2O FRICTION FACTOR = 0.057238 FREM = 28.7910  
 REM = 503.0 GRM = 0.19567E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 49.55 DEG C  
 PRM = 4.635 RAM = 0.90695E+08 INLET BULK TEMPERATURE = 23.91 DEG C OUTLET BULK TEMPERATURE = 49.53 DEG C

STATION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSSLETT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	27.49	27.72	27.12	27.41	23.91	382.7	6.29	0.509E+08	0.00001	27.94	26.25	31.17	28.59	29.13	28.86
1	1.5	28.06	28.38	27.68	28.01	23.99	383.4	6.28	0.511E+08	0.00020	24.58	22.76	27.12	24.85	25.39	25.12
2	2.5	28.45	28.84	28.06	28.43	24.05	383.8	6.27	0.513E+08	0.00034	22.69	20.85	24.89	22.79	23.33	23.06
3	5.5	29.62	30.21	29.21	29.68	24.21	385.3	6.25	0.517E+08	0.00075	18.45	16.66	19.97	18.26	18.36	18.31
4	15.5	30.53	31.52	30.03	30.69	24.76	390.1	6.16	0.534E+08	0.00212	17.26	14.74	18.92	16.80	16.97	16.89
5	25.5	31.55	32.39	31.10	31.68	25.30	395.1	6.07	0.550E+08	0.00349	15.94	14.05	17.18	15.62	15.72	15.67
6	45.5	31.93	32.86	31.42	32.07	26.40	405.5	5.90	0.584E+08	0.00625	17.94	15.37	19.77	17.50	17.69	17.60
7	75.5	33.11	33.96	32.19	33.09	28.04	420.0	5.68	0.633E+08	0.01041	19.52	16.71	23.80	19.60	20.01	19.80
8	105.5	34.31	35.46	33.47	34.41	29.68	434.7	5.47	0.684E+08	0.01458	21.27	17.04	25.97	20.81	21.43	21.12
9	135.5	35.79	36.85	35.12	35.92	31.31	450.5	5.27	0.737E+08	0.01878	21.93	17.75	25.84	21.33	21.84	21.59
10	165.2	37.56	38.58	36.93	37.69	32.94	465.6	5.07	0.787E+08	0.02300	21.17	17.35	24.49	20.59	21.00	20.80
11	205.2	39.48	40.54	38.79	39.60	35.12	486.5	4.82	0.855E+08	0.02878	22.35	17.97	26.55	21.73	22.29	22.01
12	245.2	41.74	42.81	41.03	41.86	37.31	508.7	4.58	0.926E+08	0.03464	21.86	17.61	26.00	21.88	21.82	21.55
13	275.2	43.23	44.28	42.58	43.36	38.95	524.7	4.43	0.981E+08	0.03897	22.55	18.09	26.55	21.25	22.40	22.13
14	305.2	44.62	45.63	44.04	44.76	40.59	541.7	4.28	0.104E+09	0.04334	23.88	19.07	27.84	23.04	23.60	

XP045-25

INPUT ELECTRIC POWER = 3345.0 W HEAT RATE GAINED BY WATER = 3208.2 W HEAT BALANCE ERROR = 4.09%  
 MASS FLOW RATE = 34.3160 G/S PRESSURE DROP = 0.8917 MM H2O FRICTION FACTOR = 0.021608 FREM = 32.2616  
 REM = 1493.0 GRM = 0.47779E+08 UPSTREAM BULK TEMPERATURE = 24.04 DEG C DOWN STREAM BULK TEMPERATURE = 46.46 DEG C  
 PRM = 4.605 RAM = 0.22959E+09 INLET BULK TEMPERATURE = 24.07 DEG C OUTLET BULK TEMPERATURE = 46.44 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE		
		A	B	C	AVERAGE						T	H	T+H			
0	0.1	30.83	30.30	29.36	29.84	24.07	1175.3	6.27	0.137E+09	0.00000	44.80	42.88	50.47	46.24	47.15	46.70
1	1.5	31.25	31.63	30.51	31.09	24.14	1177.2	6.26	0.138E+09	0.00007	37.54	35.65	41.88	38.39	39.24	38.81
2	2.5	32.09	32.54	31.31	31.95	24.18	1178.5	6.25	0.139E+09	0.00011	33.76	31.94	37.48	34.37	35.16	34.76
3	5.5	34.62	35.29	33.69	34.54	24.33	1182.4	6.23	0.139E+09	0.00025	25.94	24.33	28.49	26.14	26.25	26.20
4	15.5	35.85	37.72	34.82	36.13	24.81	1195.4	6.15	0.143E+09	0.00069	24.13	20.64	26.63	23.54	23.80	23.67
5	25.5	37.36	39.00	36.47	37.61	25.28	1208.8	6.08	0.147E+09	0.00114	22.04	19.41	23.80	21.60	21.75	21.67
6	45.5	37.48	39.19	36.38	37.69	26.24	1236.5	5.93	0.155E+09	0.00204	23.64	20.50	26.19	23.21	23.44	23.32
7	75.5	38.59	40.61	37.43	38.88	27.67	1275.9	5.73	0.166E+09	0.00340	24.25	20.46	27.12	23.63	23.94	23.79
8	105.5	39.76	42.37	37.98	40.04	29.10	1314.4	5.55	0.178E+09	0.00476	24.76	19.90	29.74	24.14	24.80	24.47
9	135.5	40.96	43.29	39.44	41.23	30.53	1355.3	5.37	0.190E+09	0.00613	25.23	20.62	29.52	24.59	25.12	24.86
10	165.2	41.72	44.47	40.27	42.16	31.95	1397.9	5.19	0.203E+09	0.00749	26.84	20.94	31.51	25.70	26.43	26.07
11	205.2	42.98	45.59	41.24	43.27	33.86	1451.4	4.97	0.218E+09	0.00936	28.62	22.24	35.37	27.73	28.74	28.24
12	245.2	44.95	47.42	43.32	45.23	35.77	1509.1	4.75	0.234E+09	0.01126	28.29	22.29	34.41	27.45	28.33	27.89
13	275.2	46.17	48.84	44.86	46.63	37.20	1553.8	4.59	0.247E+09	0.01270	28.85	22.23	33.79	27.46	28.29	27.87
14	305.2	46.95	49.42	45.49	47.29	38.63	1596.2	4.46	0.259E+09	0.01411	31.03	23.93	37.62	29.82	30.86	30.34
15	333.3	48.96	51.73	47.48	49.39	39.97	1638.1	4.33	0.272E+09	0.01545	28.66	21.89	34.28	27.34	28.28	27.81
16	363.3	50.32	52.97	48.67	50.65	41.41	1685.2	4.20	0.286E+09	0.01688	28.80	22.21	35.36	27.77	28.79	28.28
17	383.3	51.08	53.26	49.32	51.22	42.36	1715.4	4.12	0.295E+09	0.01784	29.40	23.51	36.84	28.93	29.92	29.43
18	403.3	52.09	54.45	50.55	52.36	43.32	1744.1	4.05	0.304E+09	0.01880	29.16	22.98	35.36	28.28	29.17	28.72
19	423.3	53.38	55.87	52.02	53.76	44.27	1773.9	3.97	0.313E+09	0.01977	28.02	22.01	32.94	26.91	27.66	27.28
20	443.3	53.97	56.57	52.42	54.32	45.23	1804.6	3.90	0.322E+09	0.02074	29.16	22.47	35.44	28.04	29.03	28.53
21	463.3	55.28	57.62	53.75	55.55	46.18	1836.5	3.82	0.332E+09	0.02171	27.97	22.24	33.61	27.15	27.94	27.55
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 51.63 54.14 50.08 51.95 42.76 1726.9 4.09 0.299E+09 0.01825 28.87 22.51 35.04 27.88 28.81 28.34																

XP045-26

INPUT ELECTRIC POWER = 3000.0 W HEAT RATE GAINED BY WATER = 2903.8 W HEAT BALANCE ERROR = 3.21%  
 MASS FLOW RATE = 26.3740 G/S PRESSURE DROP = 0.6718 MM H2O FRICTION FACTOR = 0.027540 FREM = 32.8148  
 REM = 1191.6 GRM = 0.48340E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 50.27 DEG C  
 PRM = 4.595 RAM = 0.22231E+09 INLET BULK TEMPERATURE = 23.90 DEG C OUTLET BULK TEMPERATURE = 50.26 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE		
		A	B	C	AVERAGE						T	H	T+H			
0	0.1	29.78	30.15	29.06	29.60	23.91	900.0	6.30	0.123E+09	0.00000	41.20	38.72	46.92	42.47	43.44	42.95
1	1.5	30.92	31.44	30.16	30.79	23.99	901.7	6.28	0.124E+09	0.00009	34.89	32.42	39.17	35.52	36.41	35.96
2	2.5	31.70	32.34	30.92	31.62	24.04	902.8	6.27	0.124E+09	0.00015	31.56	29.15	35.16	31.91	32.76	32.34
3	5.5	34.06	35.01	33.19	34.09	24.21	906.3	6.25	0.125E+09	0.00032	24.53	22.37	26.90	24.46	24.60	24.53
4	15.5	35.30	37.21	34.37	35.63	24.78	918.1	6.16	0.129E+09	0.00090	22.92	19.41	25.14	22.24	22.49	22.36
5	25.5	36.80	38.45	36.03	37.09	25.34	930.2	6.07	0.133E+09	0.00149	21.02	18.38	22.53	20.50	20.64	20.57
6	45.5	37.14	38.81	36.16	37.37	26.46	955.4	5.89	0.142E+09	0.00266	22.50	19.47	24.78	22.03	22.25	22.14
7	75.5	38.21	40.06	36.66	38.31	28.15	990.3	5.67	0.154E+09	0.00443	23.81	20.10	28.14	23.57	24.02	23.79
8	105.5	39.15	41.76	37.59	39.50	29.83	1026.1	5.45	0.167E+09	0.00620	25.60	20.91	30.76	24.68	25.46	25.07
9	135.5	40.68	42.97	39.11	40.92	31.52	1064.5	5.24	0.180E+09	0.00799	25.93	20.77	31.30	25.28	26.00	25.64
10	165.2	42.00	44.53	40.81	42.38	33.19	1100.7	5.04	0.192E+09	0.00979	26.86	20.87	31.91	25.75	26.54	26.15
11	205.2	43.64	46.04	41.85	43.84	35.44	1151.9	4.78	0.209E+09	0.01225	28.67	22.19	36.69	27.99	29.18	28.59
12	245.2	46.17	48.42	44.54	46.38	37.69	1205.1	4.54	0.227E+09	0.01473	27.58	21.80	34.13	26.92	27.84	27.38
13	275.2	47.67	50.18	46.48	48.11	39.37	1244.3	4.39	0.241E+09	0.01658	28.13	21.58	32.84	26.70	27.52	27.11
14	305.2	48.78	51.03	47.22	49.01	41.06	1286.2	4.23	0.256E+09	0.01844	30.11	23.31	37.73	29.24	30.39	29.81
15	333.3	50.90	53.47	49.55	51.31	42.64	1324.7	4.10	0.269E+09	0.02019	28.07	21.41	33.55	26.75	27.68	27.21
16	363.3	52.37	54.80	51.01	52.73	44.33	1364.7	3.97	0.284E+09	0.02208	28.72	22.06	34.57	27.51	28.45	27.98
17	383.3	53.24	55.38	51.59	53.41	45.45	1392.7	3.88	0.294E+09	0.02334	29.59	23.22	37.53	28.98	30.11	29.55
18	403.3	54.54	56.74	53.33	54.87	46.57	1421.8	3.79	0.304E+09	0.02461	28.87	22.64	34.05	27.73	28.52	28.13
19	423.3	55.94	58.21	54.64	56.26	47.70	1450.0	3.71	0.314E+09	0.02588	27.88	21.86	33.10	26.83	27.61	27.22
20	443.3	56.64	59.08	55.25	56.99	48.82	1478.7	3.63	0.325E+09	0.02715	29.34	22.35	35.68	28.07	29.12	28.60
21	463.3	58.06	60.26	56.42	58.25	49.95	1508.5	3.55	0.335E+09	0.02843	28.21	22.20	35.35	27.58	28.59	28.08
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 53.94 56.28 52.56 54.26 45.92 1405.4 3.85 0.298E+09 0.02387 28.74 22.26 34.74 27.65 28.58 28.11																

XP045-27

INPUT ELECTRIC POWER = 2755.0 W HEAT RATE GAINED BY WATER = 2612.3 W HEAT BALANCE ERROR = 5.18%  
 MASS FLOW RATE = 21.3200 G/S PRESSURE DROP = 0.5354 MM H2O FRICTION FACTOR = 0.033565 FREM = 33.1179  
 REM = 992.6 GRM = 0.47543E+08 UPSTREAM BULK TEMPERATURE = 23.99 DEG C DOWN STREAM BULK TEMPERATURE = 53.37 DEG C  
 PRM = 4.451 RAM = 0.21163E+09 INLET BULK TEMPERATURE = 24.02 DEG C OUTLET BULK TEMPERATURE = 53.34 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE		
		A	B	C	AVERAGE						T	H	T+H			
0	0.1	29.58	30.00	28.96	29.46	24.02	729.5	6.28	0.111E+09	0.00000	39.14	36.40	44.00	40.02	40.89	40.45
1	1.5	30.64	31.23	30.00	30.58	24.11	731.0	6.26	0.112E+09	0.00011	33.32	30.56	36.91	33.61	34.43	34.02
2	2.5	31.37	32.08	30.72	31.36	24.18	732.0	6.25	0.112E+09	0.00018	30.22	27.51	33.22	30.27	31.04	30.66
3	5.5	33.56	34.62	32.87	33.68	24.36	735.2	6.22	0.114E+09	0.00040	23.62	21.18	25.55	23.31	23.45	23.38
4	15.5	34.86	36.70	33.98	35.18	24.99	745.9	6.12	0.118E+09	0.00112	21.99	18.53	24.12	21.29	21.55	21.42
5	25.5	36.36	38.01	35.64	36.67	25.61	756.9	6.03	0.122E+09	0.00184	20.17	17.48	21.60	19.60	19.75	19.68
6	45.5	36.70	38.36	35.66	36.91	26.87	779.9	5.83	0.130E+09	0.00329	21.97	18.79	24.56	21.51	21.77	21.64
7	75.5	37.93	39.78	36.60	38.10	28.74	810.5	5.59	0.142E+09	0.00548	23.42	19.48	27.37	22.98	23.42	23.20
8	105.5	39.21	41.70	37.64	39.52	30.62	843.6	5.36	0.155E+09	0.00768	24.94	19.33	30.49	24.07	24.92	24.50
9	135.5	40.79	42.91	39.33	41.01	32.50	877.7	5.13	0.168E+09	0.00991	25.69	20.48	31.18	25.04	25.79	25.41
10	165.2	42.39	44.86	41.22	42.82	34.35	910.7	4.91	0.181E+09	0.01215	26.41	20.20	30.92	25.06	25.84	25.45
11	205.2	44.59	46.76	43.13	44.83	36.86	959.2	4.62	0.198E+09	0.01523	27.25	21.29	33.62	26.45	27.38	26.92
12	245.2	47.22	49.37	45.72	47.44	39.36	1005.6	4.39	0.217E+09	0.01827	26.68	20.97	33.00	25.98	26.88	26.43
13	275.2	49.00	51.18	47.76	49.31	41.23	1043.4	4.22	0.231E+09	0.02058	26.93	21.02	32.07	25.88	26.67	26.28
14	305.2	50.12	52.15	48.89	50.39	43.11	1079.7	4.06	0.246E+09	0.02289	29.74	23.05	36.03	28.64	29.61	29.12
15	333.3	52.45	54.92	51.23	52.87	44.87										

XP045-28

INPUT ELECTRIC POWER = 2755.0 W HEAT RATE GAINED BY WATER = 2617.6 W HEAT BALANCE ERROR = 4.99%  
 MASS FLOW RATE = 21.3200 G/S PRESSURE DROP = 0.5456 MM H2O FRICTION FACTOR = 0.034205 FREM = 33.9364  
 REM = 992.1 GRM = 0.47570E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 53.37 DEG C  
 PRM = 4.454 RAM = 0.21187E+09 INLET BULK TEMPERATURE = 23.96 DEG C OUTLET BULK TEMPERATURE = 53.35 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE				
		A	B	C						A	B	C	T	H	T+H		
0	0.1	29.60	30.00	29.01	29.48	23.97	728.6	6.29	0.111E+09	0.00000	38.69	36.14	43.23	39.54	40.32	39.93	
1	1.5	30.67	31.23	30.06	30.61	24.06	730.1	6.27	0.112E+09	0.00011	22.96	30.39	36.27	33.24	33.97	33.61	
2	2.5	31.41	32.08	30.79	31.39	24.12	731.1	6.26	0.112E+09	0.00018	29.90	27.38	32.65	29.95	30.65	30.30	
3	5.5	33.62	34.62	32.98	33.74	24.31	734.3	6.23	0.113E+09	0.00040	23.39	21.11	25.12	23.09	23.21	23.15	
4	15.5	34.91	36.70	34.04	35.22	24.94	745.0	6.13	0.118E+09	0.00112	21.80	18.48	23.88	21.15	21.39	21.27	
5	25.5	36.41	38.06	35.64	36.71	25.56	756.0	6.03	0.122E+09	0.00184	20.01	17.37	21.53	19.48	19.64	19.56	
6	45.5	36.76	38.42	35.88	37.02	26.82	779.0	5.84	0.130E+09	0.00329	21.78	18.66	23.88	21.22	21.44	21.33	
7	75.5	37.98	39.89	36.66	38.18	28.70	809.7	5.60	0.142E+09	0.00548	23.21	19.25	27.08	22.74	23.18	22.96	
8	105.5	39.26	41.76	37.76	39.59	30.58	842.8	5.36	0.155E+09	0.00768	24.72	19.20	29.90	23.81	24.61	24.21	
9	135.5	40.91	43.07	39.44	41.14	32.46	877.1	5.13	0.169E+09	0.00991	25.29	20.13	30.58	24.61	25.34	24.97	
10	165.2	42.50	44.86	41.22	42.86	34.32	910.1	4.91	0.181E+09	0.01215	25.99	20.17	30.83	24.90	25.67	25.28	
11	205.2	44.54	46.71	42.96	44.74	36.83	958.6	4.62	0.199E+09	0.01523	27.40	21.39	34.44	26.72	27.74	27.23	
12	245.2	47.22	49.42	45.77	47.47	39.33	1005.1	4.39	0.217E+09	0.01827	26.66	20.84	32.66	25.84	26.72	26.28	
13	275.2	49.00	51.24	47.98	49.41	41.21	1042.9	4.22	0.232E+09	0.02058	26.91	20.90	30.98	25.58	26.26	25.92	
14	305.2	50.12	52.15	48.89	50.39	43.09	1079.4	4.06	0.246E+09	0.02289	29.73	23.06	36.00	28.63	29.00	29.11	
15	333.3	52.51	54.92	51.34	52.92	44.86	1113.7	3.93	0.260E+09	0.02508	27.20	20.67	32.10	25.80	26.66	26.23	
16	363.3	54.26	56.47	52.96	54.56	46.74	1152.8	3.78	0.275E+09	0.02743	27.58	21.31	33.31	26.50	27.40	26.95	
17	383.3	55.30	57.12	53.81	55.41	47.99	1178.1	3.69	0.286E+09	0.02900	28.31	22.67	35.55	27.89	28.84	28.37	
18	403.3	56.66	58.69	55.45	56.93	49.24	1204.2	3.60	0.296E+09	0.03058	27.84	21.87	33.28	26.86	27.66	27.26	
19	423.3	58.21	60.32	56.98	58.51	50.50	1231.6	3.51	0.307E+09	0.03217	26.72	20.98	31.78	25.74	26.49	26.12	
20	443.3	58.93	61.27	57.64	59.28	51.75	1260.2	3.43	0.318E+09	0.03377	28.67	21.62	34.91	27.33	28.40	27.86	
21	463.3	60.40	62.55	59.21	60.72	53.00	1284.4	3.36	0.329E+09	0.03534	27.76	21.51	33.11	26.62	27.46	27.04	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	55.98	58.13	54.70	56.27	48.51	1190.1	3.66	0.291E+09	0.02967	27.72	21.52	33.49	26.69	27.58	27.13

XP045-29

INPUT ELECTRIC POWER = 2405.0 W HEAT RATE GAINED BY WATER = 2334.6 W HEAT BALANCE ERROR = 2.93%  
 MASS FLOW RATE = 16.2660 G/S PRESSURE DROP = 0.4228 MM H2O FRICTION FACTOR = 0.045493 FREM = 36.1085  
 REM = 793.7 GRM = 0.48654E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 58.28 DEG C  
 PRM = 4.230 RAM = 0.20581E+09 INLET BULK TEMPERATURE = 23.92 DEG C OUTLET BULK TEMPERATURE = 58.26 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE				
		A	B	C						A	B	C	T	H	T+H		
0	0.1	29.74	30.20	29.08	29.60	23.92	555.2	6.29	0.990E+08	0.00000	33.40	30.94	37.67	34.19	34.92	34.56	
1	1.5	30.68	31.33	30.03	30.63	24.02	556.6	6.28	0.996E+08	0.00014	29.20	26.59	32.38	29.42	30.13	29.78	
2	2.5	31.33	32.11	30.68	31.34	24.10	557.5	6.26	0.100E+09	0.00024	26.86	24.24	29.51	26.84	27.53	27.18	
3	5.5	33.28	34.46	32.64	33.46	24.32	560.3	6.23	0.101E+09	0.00052	21.66	19.16	23.32	21.24	21.38	21.31	
4	15.5	34.58	36.36	33.76	34.90	25.05	569.9	6.11	0.106E+09	0.00146	20.35	17.14	22.26	19.68	19.92	19.80	
5	25.5	36.02	37.62	35.26	36.30	25.78	579.8	6.00	0.110E+09	0.00241	18.91	16.36	20.43	18.41	18.57	18.49	
6	45.5	36.26	38.03	35.44	36.57	27.25	599.6	5.78	0.119E+09	0.00432	21.42	17.89	23.55	20.68	20.95	20.82	
7	75.5	37.87	39.62	36.43	37.97	29.45	627.6	5.50	0.132E+09	0.00719	22.78	18.87	27.47	22.51	23.04	22.77	
8	105.5	39.37	41.70	37.87	39.65	31.64	658.3	5.23	0.145E+09	0.01069	24.71	18.99	30.69	23.87	24.80	24.33	
9	135.5	41.29	43.29	40.00	41.53	33.84	687.7	4.97	0.158E+09	0.01304	25.48	20.09	30.84	24.70	25.47	25.09	
10	165.2	43.44	45.33	42.22	43.73	36.02	719.0	4.72	0.172E+09	0.01602	25.45	19.86	30.44	24.49	25.25	24.87	
11	205.2	46.09	48.09	44.68	46.29	38.95	761.2	4.43	0.191E+09	0.02003	26.27	20.52	32.73	25.57	26.51	26.04	
12	245.2	49.17	51.26	47.78	49.40	41.88	806.4	4.16	0.212E+09	0.02406	25.62	19.91	31.65	24.82	25.72	25.27	
13	275.2	51.11	53.18	50.14	51.48	44.08	837.9	3.99	0.226E+09	0.02710	26.43	20.41	30.64	25.11	25.83	25.47	
14	305.2	52.61	54.49	51.46	52.85	46.28	872.0	3.81	0.242E+09	0.03018	29.22	22.54	35.74	28.15	29.16	28.66	
15	333.3	55.39	57.72	54.42	55.84	48.33	904.2	3.67	0.257E+09	0.03307	26.13	19.66	30.33	24.57	25.38	24.97	
16	363.3	57.41	59.52	56.25	57.73	50.53	940.2	3.51	0.274E+09	0.03619	26.73	20.45	32.16	25.55	26.45	26.00	
17	383.3	58.63	60.36	57.25	58.74	52.00	965.3	3.41	0.286E+09	0.03828	27.67	21.95	34.92	27.19	28.18	27.68	
18	403.3	60.12	62.15	58.96	60.41	53.46	986.7	3.33	0.297E+09	0.04034	27.51	21.07	33.32	26.36	27.30	26.83	
19	423.3	61.87	63.88	60.66	62.14	54.93	1009.1	3.25	0.308E+09	0.04241	26.30	20.39	31.85	25.32	26.18	25.75	
20	443.3	62.77	64.90	61.59	63.09	56.39	1032.5	3.17	0.319E+09	0.04449	28.56	21.42	35.03	27.21	28.34	27.78	
21	463.3	64.13	66.28	62.99	64.47	57.86	1055.7	3.10	0.330E+09	0.04663	28.96	21.59	35.39	27.49	28.64	28.07	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	59.37	61.42	58.19	59.66	52.61	973.0	3.39	0.290E+09	0.03913	27.15	20.82	32.94	26.03	26.97	26.50

XP045-30

INPUT ELECTRIC POWER = 3500.0 W HEAT RATE GAINED BY WATER = 3330.7 W HEAT BALANCE ERROR = 4.84%  
 MASS FLOW RATE = 22.7640 G/S PRESSURE DROP = 0.6513 MM H2O FRICTION FACTOR = 0.035869 FREM = 35.3071  
 REM = 984.3 GRM = 0.48693E+08 UPSTREAM BULK TEMPERATURE = 17.41 DEG C DOWN STREAM BULK TEMPERATURE = 52.48 DEG C  
 PRM = 4.840 RAM = 0.23568E+09 INLET BULK TEMPERATURE = 17.44 DEG C OUTLET BULK TEMPERATURE = 52.46 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	24.95	25.47	24.09	24.75	17.45	667.9	7.45	0.917E+08	0.00000	37.54	35.12	42.42	38.61	39.38	38.99
1	1.5	26.30	27.03	25.45	26.19	17.55	669.6	7.43	0.925E+08	0.00010	32.22	29.75	35.68	32.62	33.33	32.98
2	2.5	27.23	28.10	26.39	27.19	17.63	670.7	7.41	0.930E+08	0.00017	29.35	26.90	32.15	29.47	30.14	29.80
3	5.5	30.01	31.32	29.21	30.18	17.85	674.2	7.37	0.946E+08	0.00036	23.16	20.91	24.79	22.84	22.95	22.90
4	15.5	31.70	34.00	30.48	32.06	18.60	685.9	7.23	0.100E+09	0.00103	21.45	18.25	23.66	20.88	21.12	21.00
5	25.5	33.39	35.51	32.59	33.83	19.35	698.1	7.09	0.106E+09	0.00169	19.97	17.36	21.17	19.36	19.50	19.43
6	45.5	33.76	35.86	32.48	34.04	20.84	723.8	6.81	0.117E+09	0.00304	21.62	18.60	24.00	21.17	21.41	21.29
7	75.5	35.16	37.39	33.42	35.32	23.08	762.6	6.43	0.135E+09	0.00507	23.01	19.41	26.87	22.70	23.10	22.90
8	105.5	36.59	39.69	34.75	37.01	25.32	802.6	6.07	0.153E+09	0.00712	24.54	19.23	29.32	23.65	24.36	24.01
9	135.5	38.52	41.18	36.78	38.83	27.56	844.6	5.74	0.172E+09	0.00919	25.11	20.19	29.83	24.41	25.04	24.73
10	165.2	40.22	43.19	38.61	40.67	29.78	884.7	5.46	0.191E+09	0.01125	26.21	20.40	31.01	25.12	25.87	25.50
11	205.2	42.53	45.32	40.57	42.81	32.77	942.3	5.09	0.217E+09	0.01407	27.83	21.66	34.85	27.08	28.11	27.59
12	245.2	45.73	48.64	44.21	46.19	35.76	1000.8	4.75	0.243E+09	0.01698	27.05	20.92	31.90	25.84	26.62	26.23
13	275.2	47.67	50.63	46.26	48.18	38.00	1046.3	4.51	0.263E+09	0.01917	27.75	21.24	32.50	26.34	27.16	26.75
14	305.2	48.84	51.64	47.28	49.25	40.24	1092.3	4.31	0.2							

## Appendix 2

### Experimental Data for $\alpha = 90^\circ$

XP090-01

INPUT ELECTRIC POWER = 170.0 W HEAT RATE GAINED BY WATER = 168.2 W HEAT BALANCE ERROR = 1.08%  
 MASS FLOW RATE = 52.9430 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 1776.7 GRM = 0.10651E+07 UPSTREAM BULK TEMPERATURE = 22.78 DEG C DOWN STREAM BULK TEMPERATURE = 23.54 DEG C  
 PRM = 6.412 RAM = 0.68299E+07 INLET BULK TEMPERATURE = 22.78 DEG C OUTLET BULK TEMPERATURE = 23.54 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----				
		A	B	C						A	B	C	T	H	T+H		
0	0.1	23.28	23.29	23.24	23.26	22.78	1761.8	6.47	0.668E+07	0.00000	28.40	27.67	31.08	29.39	29.56	29.47	
1	1.5	23.33	23.35	23.30	23.32	22.79	1761.9	6.47	0.668E+07	0.00004	25.90	25.04	27.56	26.37	26.52	26.45	
2	2.5	23.36	23.39	23.34	23.36	22.79	1762.0	6.47	0.668E+07	0.00007	24.42	23.51	25.57	24.63	24.77	24.70	
3	5.5	23.47	23.50	23.46	23.48	22.79	1762.1	6.47	0.668E+07	0.00016	20.83	19.85	21.01	20.55	20.56	20.56	
4	15.5	23.56	23.58	23.52	23.55	22.81	1762.8	6.47	0.669E+07	0.00045	18.80	18.13	19.84	18.90	18.92	18.91	
5	25.5	23.62	23.63	23.62	23.62	22.83	1763.4	6.47	0.670E+07	0.00074	17.74	17.49	17.74	17.65	17.66	17.65	
6	45.5	23.68	23.69	23.68	23.68	22.86	1764.7	6.46	0.671E+07	0.00131	17.11	16.89	17.11	17.04	17.04	17.04	
7	75.5	23.69	23.69	23.72	23.74	22.91	1766.6	6.45	0.673E+07	0.00218	17.81	15.91	17.20	16.78	16.84	16.81	
8	105.5	23.85	23.93	23.85	23.88	22.95	1768.5	6.45	0.675E+07	0.00304	15.68	14.42	15.68	15.24	15.26	15.25	
9	135.5	23.91	24.02	24.02	23.98	23.00	1770.4	6.44	0.677E+07	0.00391	15.56	13.85	13.85	14.37	14.42	14.40	
10	165.2	23.98	24.07	24.08	24.04	23.05	1772.3	6.43	0.679E+07	0.00477	15.15	13.82	13.67	14.18	14.21	14.20	
11	205.2	23.97	24.09	24.04	24.04	23.12	1774.9	6.42	0.681E+07	0.00592	16.39	14.34	15.12	15.24	15.28	15.26	
12	245.2	24.06	24.18	24.08	24.11	23.18	1777.4	6.41	0.684E+07	0.00708	15.94	14.00	15.55	15.11	15.16	15.14	
13	275.2	24.12	24.25	24.13	24.17	23.23	1779.4	6.40	0.686E+07	0.00795	15.72	13.75	15.52	14.94	15.00	14.97	
14	305.2	24.13	24.31	24.26	24.23	23.28	1781.3	6.39	0.688E+07	0.00881	16.41	13.65	14.35	14.72	14.80	14.76	
15	333.3	24.26	24.39	24.33	24.33	23.32	1783.1	6.39	0.689E+07	0.00963	15.05	13.10	13.97	14.00	14.04	14.02	
16	363.3	24.28	24.42	24.35	24.35	23.37	1785.1	6.38	0.691E+07	0.01049	15.40	13.36	14.35	14.32	14.37	14.34	
17	383.3	24.32	24.44	24.37	24.37	23.41	1786.4	6.37	0.693E+07	0.01107	15.38	13.56	14.58	14.47	14.51	14.49	
18	403.3	24.33	24.55	24.38	24.42	23.44	1787.7	6.37	0.694E+07	0.01165	15.65	12.60	14.91	14.26	14.39	14.32	
19	423.3	24.44	24.54	24.43	24.47	23.47	1789.0	6.36	0.695E+07	0.01223	14.47	13.11	14.63	14.04	14.07	14.05	
20	443.3	24.37	24.51	24.43	24.44	23.50	1790.3	6.36	0.696E+07	0.01281	16.22	13.97	15.05	15.02	15.08	15.05	
21	463.3	24.48	24.49	24.51	24.49	23.53	1791.6	6.35	0.698E+07	0.01339	14.77	14.69	14.35	14.60	14.61	14.60	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	24.33	24.48	24.38	24.40	23.42	1786.9	6.37	0.693E+07	0.01131	15.36	13.28	14.58	14.35	14.41	14.38

XP090-02

INPUT ELECTRIC POWER = 165.0 W HEAT RATE GAINED BY WATER = 170.2 W HEAT BALANCE ERROR = -3.16%  
 MASS FLOW RATE = 29.2620 G/S PRESSURE DROP = 0.7127 MM H2O FRICTION FACTOR = 0.023848 FREM = 23.5711  
 REM = 988.4 GRM = 0.11040E+07 UPSTREAM BULK TEMPERATURE = 22.76 DEG C DOWN STREAM BULK TEMPERATURE = 24.15 DEG C  
 PRM = 6.367 RAM = 0.70290E+07 INLET BULK TEMPERATURE = 22.76 DEG C OUTLET BULK TEMPERATURE = 24.15 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----				
		A	B	C						A	B	C	T	H	T+H		
0	0.1	23.34	23.38	23.39	23.38	22.76	973.2	6.48	0.675E+07	0.00000	24.22	22.85	22.43	22.97	22.98	22.98	
1	1.5	23.42	23.47	23.47	23.46	22.76	973.3	6.48	0.675E+07	0.00008	21.55	20.05	20.10	20.44	20.45	20.45	
2	2.5	23.47	23.53	23.52	23.51	22.76	973.3	6.48	0.675E+07	0.00013	20.02	18.48	18.75	19.00	19.00	19.00	
3	5.5	23.63	23.72	23.68	23.68	22.77	973.5	6.47	0.676E+07	0.00029	16.52	14.98	15.62	15.68	15.70	15.69	
4	15.5	23.89	23.98	23.85	23.91	22.80	974.2	6.47	0.677E+07	0.00081	13.08	12.09	13.57	12.89	12.92	12.90	
5	25.5	24.01	24.07	24.06	24.04	22.83	974.8	6.46	0.678E+07	0.00133	12.11	11.51	11.57	11.72	11.73	11.73	
6	45.5	24.07	24.19	24.12	24.13	22.89	976.1	6.46	0.681E+07	0.00238	12.09	10.95	11.55	11.51	11.53	11.52	
7	75.5	24.08	24.09	23.94	24.04	22.98	978.0	6.44	0.684E+07	0.00394	12.95	12.82	14.74	13.45	13.50	13.48	
8	105.5	23.96	24.09	24.02	24.02	23.07	980.0	6.43	0.688E+07	0.00551	15.94	13.86	15.00	14.89	14.93	14.91	
9	135.5	24.07	24.18	24.18	24.15	23.16	981.9	6.41	0.691E+07	0.00708	15.55	13.86	13.86	14.38	14.42	14.40	
10	165.2	24.14	24.29	24.30	24.25	23.25	983.9	6.40	0.695E+07	0.00863	15.82	13.63	13.48	14.24	14.31	14.27	
11	205.2	24.14	24.32	24.21	24.22	23.37	986.5	6.38	0.699E+07	0.01072	18.36	14.93	16.79	16.57	16.69	16.63	
12	245.2	24.28	24.41	24.36	24.35	23.48	989.1	6.36	0.704E+07	0.01282	17.77	15.41	16.19	16.40	16.45	16.43	
13	275.2	24.34	24.47	24.41	24.41	23.57	991.1	6.35	0.708E+07	0.01439	18.41	15.79	16.94	16.98	17.04	17.01	
14	305.2	24.47	24.53	24.48	24.49	23.66	993.1	6.33	0.711E+07	0.01596	17.64	16.40	17.40	17.13	17.15	17.14	
15	333.3	24.59	24.78	24.72	24.70	23.75	995.0	6.32	0.715E+07	0.01743	16.83	13.67	14.53	14.89	15.01	14.95	
16	363.3	24.62	24.81	24.74	24.72	23.84	997.0	6.31	0.718E+07	0.01901	18.14	14.53	15.69	15.98	16.12	16.05	
17	383.3	24.76	25.00	24.87	24.88	23.89	998.3	6.30	0.721E+07	0.02006	16.36	12.82	14.58	14.44	14.59	14.52	
18	403.3	24.89	24.99	24.93	24.94	23.95	999.7	6.29	0.723E+07	0.02111	15.16	13.62	14.47	14.39	14.42	14.40	
19	423.3	24.88	25.09	24.98	24.99	24.01	1001.0	6.28	0.726E+07	0.02216	16.29	13.11	14.61	14.55	14.67	14.61	
20	443.3	24.98	25.12	24.99	25.03	24.07	1002.4	6.27	0.728E+07	0.02321	15.56	13.50	15.46	14.78	14.84	14.81	
21	463.3	24.98	25.19	25.01	25.06	24.13	1003.7	6.26	0.731E+07	0.02426	16.64	13.40	16.12	15.25	15.39	15.32	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	24.79	24.97	24.87	24.88	23.92	998.9	6.29	0.722E+07	0.02050	16.39	13.54	14.89	14.84	14.94	14.89

XP090-03

INPUT ELECTRIC POWER = 165.0 W HEAT RATE GAINED BY WATER = 175.3 W HEAT BALANCE ERROR = -6.22%  
 MASS FLOW RATE = 41.5360 G/S PRESSURE DROP = 0.9633 MM H2O FRICTION FACTOR = 0.015999 FREM = 22.3355  
 REM = 1396.1 GRM = 0.11164E+07 UPSTREAM BULK TEMPERATURE = 22.73 DEG C DOWN STREAM BULK TEMPERATURE = 23.74 DEG C  
 PRM = 6.401 RAM = 0.71466E+07 INLET BULK TEMPERATURE = 22.73 DEG C OUTLET BULK TEMPERATURE = 23.74 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	23.27	23.26	23.24	23.25	22.73	1380.5	6.48	0.694E+07	0.00000	27.21	27.66	28.86	27.95	28.15	28.05
1	1.5	23.33	23.32	23.30	23.32	22.73	1380.6	6.48	0.694E+07	0.00006	24.28	24.79	25.95	25.02	25.24	25.13
2	2.5	23.38	23.37	23.34	23.36	22.73	1380.7	6.48	0.694E+07	0.00009	22.60	23.13	24.27	23.33	23.56	23.45
3	5.5	23.52	23.50	23.46	23.49	22.74	1380.9	6.48	0.695E+07	0.00020	18.72	19.26	20.30	19.41	19.43	19.42
4	15.5	23.67	23.76	23.63	23.68	22.76	1361.5	6.48	0.695E+07	0.00057	16.17	14.73	16.90	15.88	15.93	15.91
5	25.5	23.73	23.85	23.84	23.81	22.78	1382.2	6.47	0.696E+07	0.00094	15.49	13.72	13.86	14.31	14.36	14.34
6	45.5	23.84	23.86	23.84	23.85	22.83	1383.5	6.47	0.698E+07	0.00167	14.37	14.22	14.37	14.32	14.32	14.32
7	75.5	23.75	23.87	23.78	23.80	22.89	1385.5	6.46	0.701E+07	0.00278	17.03	14.99	16.50	16.13	16.17	16.15
8	105.5	23.79	23.93	23.85	23.86	22.96	1387.5	6.45	0.703E+07	0.00388	17.44	15.05	16.35	16.22	16.28	16.25
9	135.5	23.85	23.96	23.96	23.92	23.02	1389.5	6.44	0.706E+07	0.00498	17.62	15.54	15.54	16.18	16.24	16.21
10	165.2	23.98	24.01	24.02	24.00	23.08	1391.5	6.42	0.709E+07	0.00608	16.36	15.77	15.59	15.90	15.91	15.91
11	205.2	23.97	24.09	24.10	24.06	23.17	1394.1	6.41	0.712E+07	0.00755	18.23	15.82	15.72	16.51	16.59	16.55
12	245.2	24.12	24.29	24.25	24.22	23.26	1396.8	6.40	0.716E+07	0.00903	16.99	14.08	14.71	15.16	15.26	15.21
13	275.2	24.18	24.31	24.24	24.22	23.32	1398.8	6.39	0.718E+07	0.01013	17.06	14.84	15.83	15.86	15.91	15.88
14	305.2	24.24	24.42	24.37	24.34	23.39	1400.9									

XP090-04

INPUT ELECTRIC POWER = 305.0 W HEAT RATE GAINED BY WATER = 315.3 W HEAT BALANCE ERROR = -3.38%  
 MASS FLOW RATE = 25.6520 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 740.4 GRM = 0.10823E+07 UPSTREAM BULK TEMPERATURE = 15.27 DEG C DOWN STREAM BULK TEMPERATURE = 18.21 DEG C  
 PRM = 7.588 RAM = 0.82120E+07 INLET BULK TEMPERATURE = 15.27 DEG C OUTLET BULK TEMPERATURE = 18.21 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVERAGE						A	B	C	T	H	T+H
0	0.1	16.38	16.42	16.40	16.40	15.27	712.4	7.95	0.719E+07	0.00000	24.12	23.30	23.80	23.78	23.76	23.77
1	1.5	16.54	16.60	16.57	16.57	15.28	712.6	7.95	0.720E+07	0.00009	21.30	20.40	20.80	20.85	20.82	20.84
2	2.5	16.65	16.71	16.69	16.68	15.29	712.7	7.95	0.720E+07	0.00015	19.71	18.79	19.13	19.22	19.19	19.21
3	5.5	16.97	17.07	17.04	17.03	15.30	713.1	7.95	0.721E+07	0.00032	16.10	15.19	15.43	15.56	15.57	15.57
4	15.5	17.46	17.62	17.45	17.51	15.37	714.2	7.93	0.726E+07	0.00090	12.82	11.93	12.89	12.53	12.55	12.54
5	25.5	17.81	17.91	17.74	17.82	15.43	715.4	7.91	0.730E+07	0.00148	11.30	10.84	11.62	11.24	11.25	11.25
6	45.5	17.81	17.96	17.83	17.86	15.56	717.7	7.88	0.738E+07	0.00265	11.92	11.15	11.80	11.62	11.63	11.62
7	75.5	17.77	17.93	17.71	17.80	15.74	721.2	7.84	0.751E+07	0.00439	13.25	12.24	13.66	13.02	13.05	13.04
8	105.5	17.79	17.97	17.73	17.83	15.93	724.8	7.79	0.764E+07	0.00615	14.43	13.13	14.92	14.12	14.16	14.14
9	135.5	17.91	18.08	18.03	18.00	16.12	728.3	7.74	0.777E+07	0.00790	15.00	13.68	14.03	14.21	14.24	14.22
10	165.2	17.99	18.23	18.18	18.13	16.31	731.9	7.70	0.790E+07	0.00965	15.90	13.89	14.30	14.65	14.70	14.67
11	205.2	18.03	18.32	18.21	18.18	16.56	736.8	7.63	0.808E+07	0.01200	18.18	15.20	16.22	16.44	16.53	16.49
12	245.2	18.46	18.79	18.62	18.63	16.81	741.8	7.57	0.826E+07	0.01436	16.16	13.45	14.73	14.70	14.78	14.74
13	275.2	18.68	18.97	18.79	18.81	17.00	745.1	7.53	0.839E+07	0.01613	15.83	13.55	14.85	14.68	14.74	14.71
14	305.2	18.75	19.02	18.91	18.89	17.18	748.2	7.50	0.851E+07	0.01790	17.04	14.56	15.50	15.64	15.70	15.67
15	333.3	19.09	19.42	19.24	19.25	17.36	751.2	7.46	0.862E+07	0.01955	15.38	12.97	14.20	14.12	14.18	14.15
16	363.3	19.19	19.53	19.33	19.35	17.55	754.4	7.43	0.875E+07	0.02132	16.25	13.43	14.94	14.78	14.87	14.83
17	383.3	19.32	19.58	19.49	19.46	17.67	756.6	7.41	0.883E+07	0.02250	16.22	14.00	14.68	14.91	14.97	14.94
18	403.3	19.43	19.70	19.53	19.55	17.80	758.7	7.38	0.892E+07	0.02369	16.36	14.02	15.36	15.18	15.24	15.21
19	423.3	19.56	19.81	19.63	19.67	17.92	760.9	7.36	0.900E+07	0.02487	16.27	14.12	15.58	15.27	15.32	15.30
20	443.3	19.63	19.92	19.71	19.75	18.05	763.1	7.33	0.909E+07	0.02605	16.87	14.22	16.08	15.64	15.72	15.68
21	463.3	19.69	20.03	19.78	19.84	18.17	765.3	7.31	0.917E+07	0.02724	17.51	14.32	16.60	16.03	16.15	16.09
AVERAGE VALUES THROUGH STATIONS 15 TO 20:		391.6	19.37	19.66	19.49	19.51	757.5	7.40	0.887E+07	0.02300	16.22	13.79	15.14	14.98	15.05	15.02

XP090-05

INPUT ELECTRIC POWER = 185.0 W HEAT RATE GAINED BY WATER = 193.3 W HEAT BALANCE ERROR = -4.47%  
 MASS FLOW RATE = 53.0880 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 1778.3 GRM = 0.12158E+07 UPSTREAM BULK TEMPERATURE = 22.64 DEG C DOWN STREAM BULK TEMPERATURE = 23.52 DEG C  
 PRM = 6.426 RAM = 0.78120E+07 INLET BULK TEMPERATURE = 22.65 DEG C OUTLET BULK TEMPERATURE = 23.52 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVERAGE						A	B	C	T	H	T+H
0	0.1	23.19	23.22	23.24	23.22	22.65	1761.2	6.49	0.762E+07	0.00000	29.67	27.88	27.33	28.04	28.05	28.05
1	1.5	23.25	23.30	23.30	23.28	22.65	1761.3	6.49	0.762E+07	0.00004	26.89	24.84	24.91	25.38	25.39	25.38
2	2.5	23.29	23.35	23.34	23.33	22.65	1761.4	6.49	0.762E+07	0.00007	25.25	23.11	23.48	23.82	23.83	23.83
3	5.5	23.41	23.50	23.46	23.46	22.66	1761.6	6.49	0.762E+07	0.00016	21.35	19.11	20.03	20.12	20.16	20.14
4	15.5	23.50	23.64	23.52	23.55	22.67	1762.3	6.49	0.763E+07	0.00045	19.53	16.72	19.15	18.38	18.47	18.42
5	25.5	23.62	23.63	23.67	23.64	22.69	1763.0	6.49	0.764E+07	0.00073	17.46	17.25	16.47	17.05	17.06	17.06
6	45.5	23.68	23.80	23.73	23.74	22.73	1764.5	6.48	0.765E+07	0.00131	17.02	15.07	16.08	16.02	16.06	16.04
7	75.5	23.69	23.81	23.72	23.74	22.79	1766.7	6.47	0.768E+07	0.00217	17.75	15.73	17.22	16.85	16.90	16.88
8	105.5	23.74	23.87	23.74	23.78	22.84	1768.9	6.46	0.770E+07	0.00304	17.97	15.64	17.97	17.12	17.19	17.16
9	135.5	23.85	23.91	23.91	23.89	22.90	1771.0	6.45	0.773E+07	0.00390	16.92	15.99	15.99	16.29	16.30	16.29
10	165.2	23.87	24.01	24.02	23.97	22.95	1773.2	6.45	0.775E+07	0.00475	17.63	15.23	15.07	15.89	15.98	15.93
11	205.2	23.97	24.04	23.99	24.00	23.03	1776.1	6.43	0.779E+07	0.00591	17.05	15.92	16.75	16.56	16.57	16.57
12	245.2	24.06	24.18	24.08	24.11	23.10	1779.1	6.42	0.782E+07	0.00706	16.78	14.89	16.40	15.98	16.03	16.00
13	275.2	24.12	24.25	24.24	24.21	23.16	1781.3	6.41	0.785E+07	0.00792	16.69	14.74	14.81	15.36	15.41	15.39
14	305.2	24.19	24.31	24.26	24.25	23.21	1783.5	6.40	0.787E+07	0.00879	16.50	14.74	15.45	15.53	15.56	15.54
15	333.3	24.26	24.45	24.33	24.34	23.26	1785.6	6.40	0.790E+07	0.00960	16.26	13.59	15.15	14.92	15.00	14.96
16	363.3	24.28	24.48	24.35	24.37	23.32	1787.8	6.39	0.792E+07	0.01046	16.73	13.92	15.64	15.34	15.43	15.38
17	383.3	24.37	24.55	24.42	24.45	23.36	1789.3	6.38	0.794E+07	0.01104	15.87	13.51	15.13	14.77	14.84	14.80
18	403.3	24.44	24.55	24.54	24.51	23.39	1790.8	6.38	0.795E+07	0.01162	15.34	13.94	14.01	14.40	14.43	14.42
19	423.3	24.44	24.54	24.54	24.51	23.43	1792.3	6.37	0.797E+07	0.01220	15.99	14.55	14.55	15.00	15.03	15.01
20	443.3	24.48	24.56	24.54	24.53	23.47	1793.8	6.36	0.799E+07	0.01277	15.96	14.74	14.97	15.21	15.23	15.22
21	463.3	24.43	24.57	24.46	24.49	23.51	1795.3	6.36	0.801E+07	0.01335	17.47	15.10	16.96	16.44	16.51	16.48
AVERAGE VALUES THROUGH STATIONS 15 TO 20:		391.6	24.38	24.52	24.45	23.37	1790.0	6.38	0.794E+07	0.01128	16.02	14.04	14.91	14.94	14.99	14.97

XP090-06

INPUT ELECTRIC POWER = 285.0 W HEAT RATE GAINED BY WATER = 295.0 W HEAT BALANCE ERROR = -3.52%  
 MASS FLOW RATE = 16.9880 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 498.0 GRM = 0.10861E+07 UPSTREAM BULK TEMPERATURE = 15.32 DEG C DOWN STREAM BULK TEMPERATURE = 19.48 DEG C  
 PRM = 7.456 RAM = 0.80981E+07 INLET BULK TEMPERATURE = 15.33 DEG C OUTLET BULK TEMPERATURE = 19.48 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVERAGE						A	B	C	T	H	T+H
0	0.1	16.55	16.57	16.51	16.53	15.33	472.5	7.94	0.677E+07	0.00000	20.56	20.26	21.26	20.82	20.84	20.83
1	1.5	16.71	16.73	16.68	16.70	15.34	472.7	7.94	0.677E+07	0.00013	18.39	18.06	18.74	18.47	18.48	18.48
2	2.5	16.81	16.84	16.80	16.82	15.35	472.8	7.93	0.678E+07	0.00022	17.14	16.80	17.32	17.14	17.15	17.14
3	5.5	17.14	17.18	17.16	17.16	15.38	473.1	7.93	0.680E+07	0.00048	14.25	13.90	14.12	14.09	14.09	14.09
4	15.5	17.57	17.73	17.51	17.60	15.47	474.2	7.91	0.685E+07	0.00136	11.91	11.07	12.30	11.74	11.76	11.75
5	25.5	17.75	17.96	17.91	17.87	15.55	475.3	7.88	0.691E+07	0.00224	11.43	10.43	10.67	10.83	10.84	10.83
6	45.5	17.86	18.02	17.88	17.92	15.73	477.5	7.84	0.702E+07	0.00400	11.78	10.97	11.65	11.46	11.47	11.46
7	75.5	17.88	18.04	17.87	17.93	16.00	480.8	7.77	0.719E+07	0.00665	13.33	12.24	13.36	12.96	12.98	12.97
8	105.5	17.96	18.08	17.96	18.00	16.26	484.2	7.71	0.737E+07	0.00930	14.79	13.75	14.79	14.43	14.45	14.44
9	135.5	18.13	18.41	18.25	18.26	16.53	487.6	7.64	0.754E+07	0.01196	15.65	13.29	14.54	14.43	14.49	14.46
10	165.2	18.27	18.57	18.46	18.43	16.79	491.0	7.57	0.772E+07	0.01461	16.95	14.09	15.03	15.26	15.35	15.31
11	205.2	18.53	18.76	18.65	18.65	17.15	495.1	7.50	0.794E+07	0.01817	18.07	15.46	16.61	16.65	16.71	16.68
12	245.2	18.91	19.29	19.18	19.13	17.50	499.1	7.44	0.816E+07	0.02173	17.75	13.91	14.87	15.35	15.51	15.43
13	275.2	19.29	19.58	19.46	19.44	17.76	502.1	7.39	0.832E+07	0.02440	16.31	13.76	14.71	14.85	14.93	14.89
14	305.2	19.47	19.80	19.63	19.63	18.03	505.2	7.34	0.849E+07	0.02708	17.29	14.09	15.61			

XP090-07

INPUT ELECTRIC POWER = 305.0 W HEAT RATE GAINED BY WATER = 319.6 W HEAT BALANCE ERROR = -4.80%  
 MASS FLOW RATE = 41.5360 G/S PRESSURE DROP = 0.9497 MM H2O FRICTION FACTOR = 0.015771 FREM = 22.3058  
 REM = 1414.4 GRM = 0.21351E+07 UPSTREAM BULK TEMPERATURE = 22.89 DEG C DOWN STREAM BULK TEMPERATURE = 24.73 DEG C  
 PRM = 6.310 RAM = 0.13473E+08 INLET BULK TEMPERATURE = 22.89 DEG C OUTLET BULK TEMPERATURE = 24.73 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVER-AGE						A	B	C	T	H	T+H
0	0.1	23.73	23.72	23.66	23.70	22.89	1385.5	6.46	0.128E+08	0.00000	31.73	31.98	34.47	32.96	33.16	33.06
1	1.5	23.85	23.84	23.79	23.82	22.90	1385.7	6.45	0.128E+08	0.00006	27.88	28.16	29.97	28.80	29.00	28.90
2	2.5	23.94	23.93	23.87	23.91	22.90	1385.8	6.45	0.128E+08	0.00005	25.73	26.01	27.50	26.49	26.69	26.59
3	5.5	24.19	24.17	24.12	24.16	22.91	1386.1	6.45	0.128E+08	0.00020	20.89	21.17	22.04	21.36	21.37	21.36
4	15.5	24.38	24.60	24.36	24.45	22.95	1387.3	6.45	0.128E+08	0.00057	18.63	18.18	19.00	17.84	17.93	17.89
5	25.5	24.57	24.68	24.61	24.62	22.99	1388.6	6.44	0.129E+08	0.00094	16.92	15.81	16.46	16.38	16.40	16.39
6	45.5	24.73	24.80	24.74	24.76	23.07	1391.0	6.43	0.129E+08	0.00167	16.02	15.41	15.97	15.79	15.80	15.80
7	75.5	24.80	24.87	24.73	24.80	23.19	1394.6	6.41	0.130E+08	0.00278	16.53	15.87	17.30	16.55	16.57	16.56
8	105.5	24.85	25.10	24.85	24.93	23.30	1396.3	6.39	0.131E+08	0.00388	17.25	14.85	17.25	16.36	16.45	16.41
9	135.5	25.02	25.18	25.13	25.11	23.42	1402.0	6.37	0.132E+08	0.00499	16.71	15.13	15.62	15.80	15.82	15.81
10	165.2	25.14	25.40	25.31	25.28	23.54	1405.7	6.35	0.133E+08	0.00608	16.65	14.31	15.08	15.29	15.35	15.32
11	205.2	25.14	25.43	25.27	25.28	23.70	1410.7	6.33	0.134E+08	0.00756	18.46	15.38	16.90	16.82	16.91	16.86
12	245.2	25.33	25.52	25.37	25.41	23.85	1415.7	6.30	0.135E+08	0.00904	17.99	15.95	17.59	17.13	17.17	17.15
13	275.2	25.46	25.70	25.63	25.60	23.97	1419.5	6.28	0.136E+08	0.01015	17.93	15.39	16.01	16.37	16.44	16.41
14	305.2	25.58	25.87	25.76	25.74	24.09	1423.3	6.27	0.137E+08	0.01126	17.87	14.96	15.91	16.16	16.25	16.20
15	333.3	25.64	25.96	25.78	25.79	24.20	1426.9	6.25	0.138E+08	0.01230	18.40	15.14	16.79	16.67	16.78	16.73
16	363.3	25.78	26.08	26.02	25.96	24.32	1430.8	6.23	0.139E+08	0.01341	18.20	15.05	15.99	16.17	16.28	16.23
17	383.3	25.87	26.23	26.03	26.04	24.40	1433.3	6.22	0.139E+08	0.01415	18.00	14.51	16.23	16.12	16.25	16.19
18	403.3	26.01	26.28	26.16	26.15	24.47	1435.9	6.21	0.140E+08	0.01489	17.36	14.74	15.76	15.88	15.95	15.91
19	423.3	26.11	26.32	26.32	26.25	24.55	1438.5	6.19	0.141E+08	0.01563	17.11	15.02	15.02	15.66	15.72	15.69
20	443.3	26.15	26.47	26.21	26.28	24.63	1441.1	6.18	0.141E+08	0.01637	17.49	14.48	16.82	16.16	16.26	16.21
21	463.3	26.21	26.56	26.24	26.34	24.71	1443.8	6.17	0.142E+08	0.01711	17.69	14.35	17.37	16.33	16.47	16.40
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 25.93 26.22 26.09 26.08 24.43 1434.4 6.21 0.140E+08 0.01446 17.76 14.82 16.03 16.11 16.21 16.16																

XP090-08

INPUT ELECTRIC POWER = 295.0 W HEAT RATE GAINED BY WATER = 309.2 W HEAT BALANCE ERROR = -4.83%  
 MASS FLOW RATE = 29.2620 G/S PRESSURE DROP = 0.6871 MM H2O FRICTION FACTOR = 0.025288 FREM = 23.1440  
 REM = 1006.8 GRM = 0.21443E+07 UPSTREAM BULK TEMPERATURE = 23.00 DEG C DOWN STREAM BULK TEMPERATURE = 25.53 DEG C  
 PRM = 6.238 RAM = 0.13376E+08 INLET BULK TEMPERATURE = 23.00 DEG C OUTLET BULK TEMPERATURE = 25.53 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVER-AGE						A	B	C	T	H	T+H
0	0.1	23.89	23.90	23.82	23.86	23.00	978.5	6.44	0.124E+08	0.00000	29.21	28.71	31.59	30.02	30.28	30.15
1	1.5	24.03	24.05	23.96	24.01	23.01	978.7	6.44	0.124E+08	0.00008	25.42	24.89	27.20	25.93	26.18	26.05
2	2.5	24.12	24.15	24.06	24.10	23.02	978.8	6.44	0.125E+08	0.00013	23.33	22.79	24.82	23.70	23.94	23.82
3	5.5	24.41	24.45	24.34	24.40	23.03	979.1	6.43	0.125E+08	0.00029	18.71	18.20	19.66	18.84	18.86	18.85
4	15.5	24.72	24.94	24.69	24.78	23.09	980.3	6.42	0.125E+08	0.00081	15.82	13.92	16.10	15.22	15.28	15.25
5	25.5	24.96	25.12	25.06	25.04	23.14	981.5	6.42	0.125E+08	0.00133	14.21	13.01	13.47	13.55	13.56	13.55
6	45.5	25.06	25.19	25.13	25.13	23.25	983.9	6.40	0.126E+08	0.00238	14.22	13.29	13.72	13.73	13.74	13.74
7	75.5	25.08	25.14	25.06	25.09	23.41	987.4	6.37	0.127E+08	0.00395	15.45	14.91	15.61	15.32	15.32	15.32
8	105.5	25.07	25.32	25.13	25.17	23.57	991.0	6.35	0.129E+08	0.00552	17.17	14.72	16.56	16.08	16.15	16.12
9	135.5	25.24	25.52	25.41	25.39	23.73	994.7	6.32	0.130E+08	0.00709	17.11	14.44	15.40	15.58	15.65	15.61
10	165.2	25.42	25.62	25.58	25.54	23.89	998.3	6.30	0.131E+08	0.00864	16.91	14.90	15.24	15.63	15.68	15.66
11	205.2	25.42	25.76	25.61	25.59	24.11	1003.2	6.26	0.132E+08	0.01074	19.69	15.58	17.20	17.33	17.49	17.41
12	245.2	25.78	25.97	25.81	25.85	24.33	1008.2	6.23	0.134E+08	0.01284	17.71	15.67	17.31	16.85	16.90	16.87
13	275.2	25.79	26.14	26.02	25.99	24.49	1011.9	6.20	0.135E+08	0.01442	19.75	15.51	16.75	17.16	17.34	17.25
14	305.2	25.97	26.26	26.15	26.12	24.65	1015.7	6.18	0.137E+08	0.01600	19.50	16.00	17.12	17.42	17.54	17.48
15	333.3	26.20	26.57	26.51	26.43	24.80	1019.3	6.15	0.138E+08	0.01748	18.36	14.50	15.02	15.79	15.96	15.87
16	363.3	26.39	26.69	26.53	26.54	24.96	1023.1	6.13	0.139E+08	0.01906	18.00	14.83	16.40	16.31	16.41	16.36
17	383.3	26.53	26.78	26.64	26.65	25.07	1025.7	6.11	0.140E+08	0.02012	17.55	14.99	16.31	16.21	16.28	16.25
18	403.3	26.67	26.89	26.72	26.76	25.18	1028.2	6.09	0.141E+08	0.02117	17.18	14.95	16.68	16.21	16.27	16.24
19	423.3	26.71	26.99	26.88	26.86	25.29	1030.8	6.08	0.142E+08	0.02223	18.00	15.07	16.12	16.31	16.40	16.35
20	443.3	26.77	27.08	26.88	26.91	25.39	1033.5	6.06	0.142E+08	0.02328	18.69	15.18	17.22	16.91	17.03	16.97
21	463.3	26.88	27.17	27.02	27.02	25.50	1036.1	6.04	0.143E+08	0.02434	18.64	15.35	16.93	16.87	16.97	16.92
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 26.55 26.84 26.69 26.69 25.11 1026.8 6.10 0.140E+08 0.02056 17.96 14.92 16.29 16.29 16.39 16.34																

XP090-09

INPUT ELECTRIC POWER = 750.0 W HEAT RATE GAINED BY WATER = 714.6 W HEAT BALANCE ERROR = 4.71%  
 MASS FLOW RATE = 28.6840 G/S PRESSURE DROP = 0.7246 MM H2O FRICTION FACTOR = 0.025228 FREM = 24.9733  
 REM = 989.9 GRM = 0.50087E+07 UPSTREAM BULK TEMPERATURE = 21.41 DEG C DOWN STREAM BULK TEMPERATURE = 27.38 DEG C  
 PRM = 6.217 RAM = 0.31141E+08 INLET BULK TEMPERATURE = 21.42 DEG C OUTLET BULK TEMPERATURE = 27.38 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVER-AGE						A	B	C	T	H	T+H
0	0.1	23.10	23.14	22.97	23.05	21.42	925.1	6.70	0.261E+08	0.00000	35.54	34.77	38.61	36.63	36.88	36.76
1	1.5	23.42	23.47	23.30	23.39	21.44	925.5	6.70	0.261E+08	0.00008	30.16	29.39	32.04	30.69	30.91	30.80
2	2.5	23.64	23.70	23.54	23.62	21.45	925.8	6.70	0.262E+08	0.00013	27.31	26.55	28.67	27.61	27.80	27.70
3	5.5	24.30	24.39	24.23	24.31	21.49	926.7	6.69	0.262E+08	0.00029	21.28	20.59	21.79	21.21	21.22	21.21
4	15.5	24.83	25.27	24.86	24.99	21.61	929.7	6.66	0.265E+08	0.00082	18.62	16.36	18.46	17.75	17.81	17.78
5	25.5	25.23	25.62	25.39	25.41	21.74	932.6	6.64	0.267E+08	0.00135	17.13	15.41	16.40	16.28	16.31	16.30
6	45.5	25.39	25.80	25.52	25.57	22.00	938.2	6.60	0.271E+08	0.00242	17.59	15.71	16.98	16.72	16.76	16.74
7	75.5	25.58	26.14	25.67	25.80	22.38	946.0	6.54	0.277E+08	0.00402	18.66	15.87	18.12	17.47	17.55	17.51
8	105.5	25.74	26.44	25.85	26.01	22.76	954.0	6.48	0.284E+08	0.00562	19.98	16.21	19.30	18.34	18.50	18.42
9	135.5	26.18	26.74	26.41	26.44	23.14	962.1	6.42	0.290E+08	0.00722	19.59	16.56	18.26	18.05	18.14	18.09
10	165.2	26.42	27.02	26.81	26.75	23.52	970.3	6.36	0.296E+08	0.00881	20.55	17.02	18.12	18.45	18.56	18.50
11	205.2	26.70	27.43	27.05	27.06	24.03	981.5	6.28	0.305E+08	0.01096	22.25	17.49	19.67	19.61	19.81	19.71
12	245.2	27.11	27.86	27.48	27.48	24.54	993.0	6.20	0.314E+08	0.01311	23.12	17.89	20.15	20.17	20.39	20.28
13	275.2	27.46	28.21	27.86	27.84	24.92	1001.8	6.14	0.321E+08	0.01473	23.38	18.05	20.20	20.31	20.54	20.43
14	305.2	27.91	28.59	28.27	28.26	25.30	1010.8	6.08	0.327E+08	0.01635	22.70	17.99	19.98	20.04	20.22	20.13
15	333.3	28.36														

XP090-10

INPUT ELECTRIC POWER = 1290.0 W HEAT RATE GAINED BY WATER = 1245.7 W HEAT BALANCE ERROR = 3.44%  
 MASS FLOW RATE = 50.5610 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 1516.8 GRM = 0.50753E+07 UPSTREAM BULK TEMPERATURE = 15.46 DEG C DOWN STREAM BULK TEMPERATURE = 21.36 DEG C  
 PRM = 7.266 RAM = 0.36879E+08 INLET BULK TEMPERATURE = 15.47 DEG C OUTLET BULK TEMPERATURE = 21.36 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-----			NUSSELT NUMBER			-----		
		A	B	C	AVER- AGE						A	B	C	T	AVERAGE	H	T+H		
0	0.1	18.16	18.21	18.02	18.11	15.47	1411.4	7.91	0.289E+08	0.00000	39.36	38.70	41.46	40.09	40.24	40.16			
1	1.5	18.66	18.72	18.54	18.63	15.49	1412.0	7.90	0.290E+08	0.00004	33.42	32.75	34.67	33.74	33.88	33.81			
2	2.5	19.00	19.08	18.90	18.98	15.50	1412.5	7.90	0.290E+08	0.00007	30.27	29.61	31.16	30.42	30.55	30.48			
3	5.5	20.03	20.14	19.97	20.05	15.54	1413.9	7.89	0.291E+08	0.00016	23.59	23.00	23.89	23.49	23.49	23.49			
4	15.5	21.12	21.56	21.07	21.25	15.65	1418.5	7.86	0.295E+08	0.00046	19.42	17.96	19.60	18.96	18.99	18.98			
5	25.5	21.77	22.35	22.17	22.10	15.79	1423.2	7.83	0.298E+08	0.00075	17.69	16.14	16.59	16.78	16.81	16.79			
6	45.5	21.63	22.41	21.84	21.96	16.04	1432.5	7.76	0.305E+08	0.00135	18.93	16.61	18.25	17.87	17.93	17.90			
7	75.5	21.98	22.70	22.05	22.24	16.42	1446.9	7.67	0.315E+08	0.00224	19.01	16.82	18.76	18.14	18.20	18.17			
8	105.5	22.07	23.09	22.29	22.49	16.79	1461.5	7.57	0.326E+08	0.00313	20.00	16.76	19.21	18.55	18.66	18.61			
9	135.5	22.52	23.41	22.91	22.94	17.17	1474.3	7.50	0.336E+08	0.00403	19.73	16.92	18.38	18.27	18.34	18.31			
10	165.2	22.81	23.84	23.30	23.32	17.54	1486.8	7.43	0.346E+08	0.00492	20.01	16.73	18.31	18.25	18.35	18.30			
11	205.2	22.81	23.98	23.32	23.37	18.05	1504.0	7.34	0.359E+08	0.00612	22.11	17.73	19.95	19.77	19.93	19.85			
12	245.2	23.39	24.41	23.81	23.87	18.55	1521.6	7.24	0.372E+08	0.00732	21.69	17.94	19.99	19.79	19.88	19.82			
13	275.2	23.57	24.64	24.13	24.11	18.92	1535.1	7.17	0.383E+08	0.00822	22.62	18.36	20.16	20.23	20.38	20.31			
14	305.2	23.86	24.92	24.37	24.38	19.30	1548.8	7.10	0.393E+08	0.00913	23.04	18.66	20.71	20.65	20.80	20.73			
15	333.3	24.37	25.62	25.06	25.01	19.65	1561.9	7.03	0.403E+08	0.00998	22.24	17.56	19.41	19.55	19.74	19.64			
16	363.3	24.39	25.75	24.96	25.04	20.03	1576.1	6.96	0.414E+08	0.01089	22.00	18.31	21.24	20.92	21.18	21.05			
17	383.3	24.65	25.89	25.20	25.25	20.28	1585.7	6.91	0.421E+08	0.01150	23.96	18.65	21.28	21.07	21.29	21.18			
18	403.3	24.89	26.06	25.49	25.48	20.53	1595.5	6.87	0.429E+08	0.01211	24.01	18.94	21.08	21.14	21.34	21.24			
19	423.3	25.16	26.32	25.76	25.75	20.79	1605.3	6.82	0.436E+08	0.01272	23.88	18.87	21.00	21.05	21.25	21.15			
20	443.3	25.26	26.47	25.82	25.85	21.04	1615.3	6.77	0.444E+08	0.01333	24.72	19.23	21.82	21.69	21.92	21.81			
21	463.3	25.88	26.67	26.46	26.34	21.29	1625.4	6.73	0.451E+08	0.01394	22.74	19.38	20.17	20.67	20.76	20.72			
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																			
		391.6	24.79	26.02	25.38	25.40	20.39	1590.0	6.89	0.425E+08	0.01175	23.80	18.59	20.97	20.91	21.12	21.01		

XP090-11

INPUT ELECTRIC POWER = 640.0 W HEAT RATE GAINED BY WATER = 621.0 W HEAT BALANCE ERROR = 2.97%  
 MASS FLOW RATE = 14.1000 G/S PRESSURE DROP = 0.3461 MM H2O FRICTION FACTOR = 0.049849 FREM = 25.4517  
 REM = 510.6 GRM = 0.51412E+07 UPSTREAM BULK TEMPERATURE = 21.16 DEG C DOWN STREAM BULK TEMPERATURE = 31.72 DEG C  
 PRM = 5.894 RAM = 0.30303E+08 INLET BULK TEMPERATURE = 21.18 DEG C OUTLET BULK TEMPERATURE = 31.71 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-----			NUSSELT NUMBER			-----		
		A	B	C	AVER- AGE						A	B	C	T	AVERAGE	H	T+H		
0	0.1	23.10	23.21	23.11	23.14	21.18	452.0	6.75	0.223E+08	0.00001	27.02	25.65	26.87	26.51	26.60	26.55			
1	1.5	23.42	23.57	23.44	23.46	21.21	452.4	6.74	0.224E+08	0.00016	23.52	22.07	23.31	22.96	23.06	23.01			
2	2.5	23.64	23.81	23.67	23.71	21.23	452.6	6.74	0.224E+08	0.00027	21.59	20.14	21.36	21.02	21.12	21.07			
3	5.5	24.30	24.56	24.34	24.40	21.30	453.4	6.72	0.225E+08	0.00059	17.33	15.95	17.08	16.76	16.79	16.78			
4	15.5	24.88	25.33	24.97	25.06	21.52	455.9	6.68	0.229E+08	0.00167	15.47	13.67	15.10	14.70	14.75	14.73			
5	25.5	25.29	25.73	25.50	25.51	21.75	458.5	6.64	0.232E+08	0.00276	16.48	13.04	13.85	13.82	13.85	13.84			
6	45.5	25.39	25.86	25.57	25.61	22.20	463.2	6.57	0.238E+08	0.00492	16.24	14.19	15.38	15.22	15.27	15.25			
7	75.5	25.74	26.25	25.84	25.95	22.87	470.1	6.46	0.248E+08	0.00818	18.04	15.34	17.44	16.86	16.94	16.90			
8	105.5	26.19	26.77	26.41	26.46	23.55	477.3	6.35	0.258E+08	0.01145	19.58	16.04	18.10	17.79	17.91	17.85			
9	135.5	26.79	27.35	27.07	27.07	24.22	484.6	6.25	0.268E+08	0.01473	20.08	16.51	18.12	18.12	18.24	18.18			
10	165.2	27.36	28.02	27.69	27.69	24.89	492.1	6.14	0.278E+08	0.01798	20.86	16.49	18.38	18.41	18.58	18.49			
11	205.2	28.03	28.71	28.38	28.37	25.79	502.6	6.00	0.292E+08	0.02239	22.92	17.64	19.83	19.90	20.13	20.02			
12	245.2	29.10	29.74	29.44	29.43	26.69	513.6	5.86	0.307E+08	0.02682	21.29	16.80	18.67	18.74	18.92	18.83			
13	275.2	29.79	30.43	30.13	30.12	27.36	521.0	5.77	0.317E+08	0.03014	21.13	16.73	18.50	18.62	18.79	18.70			
14	305.2	30.41	30.99	30.67	30.69	28.04	528.2	5.68	0.328E+08	0.03346	21.56	17.34	19.47	19.30	19.46	19.38			
15	333.3	31.25	31.88	31.54	31.56	28.67	535.2	5.60	0.338E+08	0.03658	19.80	15.90	17.77	17.68	17.83	17.76			
16	363.3	31.76	32.42	32.11	32.09	29.34	542.8	5.52	0.348E+08	0.03991	21.11	16.61	18.48	18.55	18.73	18.64			
17	383.3	32.26	32.82	32.52	32.53	29.79	548.0	5.46	0.356E+08	0.04214	20.70	16.86	18.72	18.63	18.76	18.69			
18	403.3	32.75	33.37	33.01	33.04	30.24	553.4	5.40	0.363E+08	0.04437	20.31	16.30	18.39	18.19	18.34	18.26			
19	423.3	33.20	33.78	33.57	33.51	30.69	558.8	5.35	0.371E+08	0.04661	20.29	16.49	17.70	18.02	18.16	18.09			
20	443.3	33.57	34.19	33.83	33.86	31.14	564.3	5.29	0.378E+08	0.04885	20.96	16.68	18.88	18.68	18.84	18.76			
21	463.3	33.89	34.59	34.14	34.21	31.59	570.0	5.23	0.386E+08	0.05110	22.04	16.91	19.88	19.38	19.61	19.50			
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																			
		391.6	32.46	33.07	32.76	32.77	29.98	550.4	5.44	0.359E+08	0.04308	20.53	16.47	18.32	18.29	18.44	18.37		

XP090-12

INPUT ELECTRIC POWER = 1165.0 W HEAT RATE GAINED BY WATER = 1109.9 W HEAT BALANCE ERROR = 4.73%  
 MASS FLOW RATE = 32.8720 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 1019.2 GRM = 0.51860E+07 UPSTREAM BULK TEMPERATURE = 15.76 DEG C DOWN STREAM BULK TEMPERATURE = 23.85 DEG C  
 PRM = 7.004 RAM = 0.36323E+08 INLET BULK TEMPERATURE = 15.77 DEG C OUTLET BULK TEMPERATURE = 23.84 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-----			NUSSELT NUMBER			-----		
		A	B	C	AVER- AGE						A	B	C	T	AVERAGE	H	T+H		
0	0.1	18.57	18.57	18.39	18.49	15.77	924.9	7.83	0.265E+08	0.00000	33.74	33.68	36.01	34.68	34.86	34.77			
1	1.5	19.07	19.08	18.90	19.01	15.80	925.5	7.82	0.266E+08	0.00007	28.78	28.72	30.39	29.40	29.57	29.49			
2	2.5	19.42	19.43	19.25	19.36	15.81	925.9	7.82	0.266E+08	0.00011	26.13	26.08	27.44	26.61	26.77	26.69			
3	5.5	20.47	20.48	20.31	20.42	15.86	927.1	7.81	0.267E+08	0.00025	20.48	20.43	21.24	20.71	20.72	20.71			
4	15.5	21.34	21.84	21.35	21.51	16.04	931.3	7.76	0.272E+08	0.00071	17.79	16.25	17.75	17.23	17.26	17.25			
5	25.5	21.83	22.41	22.17	22.14	16.21	935.6	7.72	0.276E+08	0.00116	16.78	15.21	15.81	15.91	15.93	15.92			
6	45.5	21.74	22.36	21.95	22.01	16.55	944.1	7.63	0.284E+08	0.00208	18.16	16.23	17.45	17.24	17.28	17.26			
7	75.5	22.20	22.87	22.39	22.49	17.07	956.4	7.52	0.297E+08	0.00345	18.33	16.22	17.68	17.37	17.41	17.39			
8	105.5	22.29	23.37	22.51	22.73	17.59	967.6	7.42	0.309E+08	0.00483	19.95	16.23	19.07	18.27	18.42	18.35			
9	135.5	22.85	23.74	23.19	23.26	18.10	979.2	7.32	0.321E+08	0.00622	19.76	16.64	18.44	18.19	18.28	18.23			
10	165.2	23.03	24.12	23.80	23.65	18.62	990.8	7.23	0.334E+08	0.00759	21.20	17.01	18.06	18.59	18.75	18.67			
11	205.2	23.36	24.48	23.88	23.91	19.30	1007.0	7.10	0.351E+08	0.00944	23.04	18.05	20.44	20.31	20.51	20.41			
12	245.2	24.17	25.19	24.53	24.63	19.99	1023.8	6.97	0.368E+08	0.01131	22.33	17.96	20.58	20.12	20.29	20.20			
13	275.2	24.46	25.53	25.02	25.00	20.51	1036.7	6.87	0.381E+08	0.01271	23.61	18.55	20.65	20.7					

XP090-13

INPUT ELECTRIC POWER = 905.0 W HEAT RATE GAINED BY WATER = 859.6 W HEAT BALANCE ERROR = 5.01%  
 MASS FLOW RATE = 15.5440 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 512.8 GRM = 0.51082E+07 UPSTREAM BULK TEMPERATURE = 15.76 DEG C DOWN STREAM BULK TEMPERATURE = 29.02 DEG C  
 PRM = 6.534 RAM = 0.33379E+08 INLET BULK TEMPERATURE = 15.78 DEG C OUTLET BULK TEMPERATURE = 29.01 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-----			NUSSLET NUMBER	-----		
		A	B	C	AVERAGE						T	AVERAGE	T+H				
0	0.1	18.52	18.61	18.35	18.47	15.78	437.4	7.83	0.206E+08	0.00000	26.71	25.84	28.44	27.20	27.36	27.28	
1	1.5	18.93	19.06	18.80	18.91	15.82	437.9	7.82	0.206E+08	0.00014	23.48	22.54	24.56	23.63	23.79	23.71	
2	2.5	19.22	19.38	19.10	19.22	15.85	438.2	7.81	0.207E+08	0.00024	21.67	20.72	22.45	21.67	21.82	21.75	
3	5.5	20.08	20.32	20.03	20.14	15.93	439.2	7.79	0.208E+08	0.00053	17.60	16.67	17.84	17.35	17.37	17.36	
4	15.5	20.67	21.22	20.73	20.88	16.22	442.5	7.72	0.214E+08	0.00149	16.38	14.58	16.16	15.66	15.71	15.68	
5	25.5	21.16	21.79	21.51	21.49	16.50	445.8	7.65	0.219E+08	0.00246	15.66	13.77	14.55	14.62	14.66	14.64	
6	45.5	21.29	21.97	21.56	21.61	17.06	452.2	7.52	0.230E+08	0.00440	17.21	14.85	16.19	16.02	16.08	16.05	
7	75.5	21.76	22.42	21.94	22.04	17.91	460.9	7.36	0.245E+08	0.00732	18.89	16.10	18.03	17.59	17.67	17.63	
8	105.5	22.18	23.09	22.40	22.56	18.75	470.1	7.20	0.261E+08	0.01025	21.15	16.71	19.89	19.06	19.25	19.15	
9	135.5	23.02	23.85	23.46	23.44	19.60	479.6	7.04	0.277E+08	0.01320	21.18	17.03	18.74	18.83	18.98	18.91	
10	165.2	23.76	24.62	24.24	24.21	20.44	489.4	6.89	0.294E+08	0.01613	21.77	17.26	18.97	19.16	19.33	19.24	
11	205.2	24.64	25.54	25.05	25.08	21.57	503.2	6.67	0.317E+08	0.02010	23.44	18.12	20.67	20.52	20.74	20.63	
12	245.2	25.89	26.74	26.26	26.30	22.70	516.3	6.49	0.340E+08	0.02409	22.48	17.73	20.14	19.93	20.12	20.02	
13	275.2	26.62	27.48	27.08	27.06	23.54	526.1	6.35	0.357E+08	0.02709	23.26	18.21	20.27	20.37	20.58	20.48	
14	305.2	27.30	28.15	27.66	27.70	24.39	536.3	6.22	0.374E+08	0.03010	24.57	19.01	21.89	21.59	21.82	21.70	
15	333.3	28.42	29.31	28.92	28.88	25.18	546.3	6.09	0.391E+08	0.03294	22.07	17.29	19.11	19.29	19.49	19.39	
16	363.3	29.11	30.03	29.59	29.58	26.03	557.3	5.96	0.410E+08	0.03598	23.15	17.81	19.98	20.08	20.31	20.20	
17	383.3	29.70	30.47	30.08	30.08	26.59	564.9	5.87	0.423E+08	0.03802	22.90	18.34	20.42	20.38	20.55	20.47	
18	403.3	30.24	31.08	30.67	30.66	27.16	572.0	5.79	0.435E+08	0.04005	23.06	18.12	20.21	20.27	20.46	20.37	
19	423.3	30.93	31.72	31.39	31.35	27.72	578.6	5.72	0.447E+08	0.04207	22.13	17.76	19.32	19.58	19.74	19.66	
20	443.3	31.33	32.17	31.78	31.76	28.29	585.3	5.65	0.459E+08	0.04410	23.26	18.24	20.30	20.39	20.60	20.49	
21	463.3	32.00	32.72	32.42	32.38	28.85	592.3	5.58	0.471E+08	0.04613	22.47	18.28	19.85	20.05	20.20	20.13	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
391.6 29.95 30.80 30.41 30.39 26.83 567.4 5.85 0.428E+08 0.03886 22.76 17.93 19.89 20.00 20.19 20.10																	

XP090-14

INPUT ELECTRIC POWER = 1415.0 W HEAT RATE GAINED BY WATER = 1389.0 W HEAT BALANCE ERROR = 1.84%  
 MASS FLOW RATE = 42.9800 G/S PRESSURE DROP = 1.0827 MM H2O FRICTION FACTOR = 0.016788 FREM = 25.2531  
 REM = 1504.3 GRM = 0.10230E+08 UPSTREAM BULK TEMPERATURE = 21.13 DEG C DOWN STREAM BULK TEMPERATURE = 28.88 DEG C  
 PRM = 6.121 RAM = 0.62624E+08 INLET BULK TEMPERATURE = 21.14 DEG C OUTLET BULK TEMPERATURE = 28.87 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-----			NUSSLET NUMBER	-----		
		A	B	C	AVERAGE						T	AVERAGE	T+H				
0	0.1	23.74	23.80	23.58	23.69	21.14	1376.7	6.75	0.498E+08	0.00000	44.85	43.74	47.71	45.69	46.00	45.85	
1	1.5	24.27	24.36	24.12	24.24	21.17	1377.5	6.75	0.499E+08	0.00005	37.50	36.41	39.36	37.88	38.15	38.02	
2	2.5	24.64	24.75	24.50	24.62	21.18	1378.0	6.75	0.500E+08	0.00009	33.69	32.64	35.12	33.88	34.14	34.01	
3	5.5	25.74	25.91	25.62	25.75	21.23	1379.7	6.74	0.501E+08	0.00019	25.82	24.90	26.54	25.73	25.75	25.74	
4	15.5	26.43	27.19	26.58	26.73	21.40	1385.4	6.71	0.507E+08	0.00055	23.09	20.08	22.45	21.80	21.88	21.84	
5	25.5	27.13	27.73	27.44	27.44	21.56	1391.1	6.67	0.512E+08	0.00090	20.87	18.84	19.78	19.80	19.83	19.81	
6	45.5	27.33	28.08	27.63	27.68	21.89	1402.5	6.61	0.524E+08	0.00161	21.35	18.78	20.23	20.06	20.12	20.09	
7	75.5	27.73	28.52	27.79	28.02	22.39	1417.8	6.54	0.539E+08	0.00268	21.70	18.91	21.46	20.61	20.69	20.65	
8	105.5	27.97	28.89	28.19	28.35	22.88	1433.3	6.46	0.555E+08	0.00375	22.79	19.29	21.84	21.20	21.31	21.25	
9	135.5	28.52	29.41	28.84	28.92	23.38	1449.2	6.38	0.571E+08	0.00482	22.52	19.20	21.17	20.87	20.96	20.92	
10	165.2	28.69	29.56	29.42	29.36	23.87	1465.3	6.30	0.587E+08	0.00589	23.98	18.97	20.81	21.06	21.25	21.16	
11	205.2	29.09	30.43	29.67	29.73	24.52	1487.6	6.20	0.610E+08	0.00732	25.30	19.56	22.46	22.19	22.44	22.32	
12	245.2	29.66	30.97	30.27	30.30	25.18	1510.5	6.09	0.632E+08	0.00876	25.79	19.94	22.66	22.55	22.80	22.67	
13	275.2	30.07	31.37	30.69	30.71	25.68	1528.2	6.01	0.650E+08	0.00985	26.25	20.23	22.99	22.90	23.16	23.03	
14	305.2	30.47	31.77	31.06	31.10	26.17	1546.3	5.94	0.668E+08	0.01094	26.80	20.57	23.57	23.37	23.65	23.51	
15	333.3	30.97	32.39	31.82	31.73	26.64	1563.6	5.86	0.685E+08	0.01196	26.51	19.98	22.17	22.58	22.89	22.73	
16	363.3	31.32	32.81	32.11	32.08	27.13	1580.7	5.79	0.702E+08	0.01305	27.40	20.23	23.08	23.21	23.57	23.39	
17	383.3	31.76	33.04	32.41	32.40	27.46	1591.3	5.75	0.713E+08	0.01377	26.72	20.55	23.18	23.21	23.48	23.35	
18	403.3	32.13	33.48	32.79	32.80	27.79	1602.1	5.71	0.725E+08	0.01450	26.40	20.16	22.94	22.89	23.17	23.03	
19	423.3	32.53	33.83	33.29	33.22	28.12	1613.0	5.67	0.736E+08	0.01523	25.96	20.05	22.17	22.47	22.73	22.60	
20	443.3	32.67	34.07	33.33	33.36	28.45	1624.1	5.63	0.747E+08	0.01595	27.12	20.36	23.44	23.32	23.64	23.48	
21	463.3	33.12	34.37	33.81	33.77	28.78	1635.3	5.59	0.759E+08	0.01668	26.38	20.46	22.74	22.94	23.19	23.07	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
391.6 31.90 33.27 32.63 32.60 27.60 1595.8 5.74 0.718E+08 0.01408 26.68 20.22 22.83 22.95 23.24 23.10																	

XP090-15

INPUT ELECTRIC POWER = 1255.0 W HEAT RATE GAINED BY WATER = 1212.7 W HEAT BALANCE ERROR = 3.37%  
 MASS FLOW RATE = 27.8180 G/S PRESSURE DROP = 0.7280 MM H2O FRICTION FACTOR = 0.026940 FREM = 26.8890  
 REM = 998.1 GRM = 0.97318E+07 UPSTREAM BULK TEMPERATURE = 20.83 DEG C DOWN STREAM BULK TEMPERATURE = 31.28 DEG C  
 PRM = 5.955 RAM = 0.57951E+08 INLET BULK TEMPERATURE = 20.84 DEG C OUTLET BULK TEMPERATURE = 31.28 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-----			NUSSLET NUMBER	-----		
		A	B	C	AVERAGE						T	AVERAGE	T+H				
0	0.1	23.58	23.71	23.46	23.56	20.85	884.5	6.81	0.426E+08	0.00000	37.17	35.47	38.93	37.46	37.62	37.54	
1	1.5	24.10	24.28	24.02	24.12	20.88	885.2	6.80	0.427E+08	0.00008	31.60	29.88	32.39	31.42	31.56	31.49	
2	2.5	24.45	24.67	24.40	24.50	20.90	885.7	6.80	0.428E+08	0.00014	28.64	26.95	29.03	28.28	28.41	28.34	
3	5.5	25.52	25.85	25.56	25.64	20.97	887.2	6.79	0.430E+08	0.00030	22.35	20.82	22.13	21.75	21.77	21.76	
4	15.5	26.38	27.13	26.41	26.64	21.19	892.1	6.74	0.436E+08	0.00085	19.59	17.10	19.46	18.64	18.72	18.68	
5	25.5	26.97	27.68	27.27	27.31	21.41	897.0	6.70	0.443E+08	0.00140	18.28	16.21	17.33	17.23	17.27	17.25	
6	45.5	27.06	27.97	27.47	27.50	21.86	907.1	6.62	0.456E+08	0.00249	19.51	16.60	18.08	17.99	18.07	18.03	
7	75.5	27.35	28.24	27.57	27.72	22.52	920.4	6.51	0.475E+08	0.00414	20.99	17.71	20.07	19.49	19.59	19.54	
8	105.5	27.63	28.69	27.86	28.13	23.19	934.1	6.41	0.493E+08	0.00580	22.77	17.75	21.65	20.50	20.74	20.62	
9	135.5	28.24	29.29	28.73	28.76	23.86	948.3	6.30	0.513E+08	0.00746	23.06	18.58	20.72	20.62	20.78	20.70	
10	165.2	28.69	29.91	29.42	29.34	24.52	962.7	6.20	0.532E+08	0.00911	24.19	18.72	20.57	20.92	21.16	21.04	
11	205.2	29.26	30.54	29.89	29.89	25.41	982.9	6.06	0.559E+08	0.01134	26.17	19.62	22.47	22.44	22.75	22.59	
12	245.2	30.21	31.41	30.77	30.80	26.30	1003.9	5.92	0.587E+08	0.01358	25.68	19.65	22.45	22.32	22.59	22.46	
13	275.2	30.84	32.04	31.47	31.45	26.97	1019.7	5.82	0.608E+08	0.01527	25.86	19.75	22.29	22.36	22.63	22.50	
14	305.2	31.57	32.77	32.12	32.15	27.64	1033.6	5.73	0.628E+08	0.01695	25.43	19.49	22.34	22.16	22.42	22.29	
15	333.3	32.31	33.62	33.00	32												

XP090-16

INPUT ELECTRIC POWER = 2100.0 W HEAT RATE GAINED BY WATER = 2016.6 W HEAT BALANCE ERROR = 3.97%  
 MASS FLOW RATE = 47.3120 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 1511.4 GRM = 0.10603E+08 UPSTREAM BULK TEMPERATURE = 15.93 DEG C DOWN STREAM BULK TEMPERATURE = 26.14 DEG C  
 PRM = 6.773 RAM = 0.71814E+08 INLET BULK TEMPERATURE = 15.94 DEG C OUTLET BULK TEMPERATURE = 26.13 DEG C

STATION NO.	Z CM	WALL TEMPERATURE (DEG C)			TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	20.00	20.12	19.79	19.93	15.94	1337.0	7.75	0.489E+08	0.00000	42.21	40.97	44.56	42.90	43.07	42.99
1	1.5	20.82	20.95	20.66	20.80	15.97	1338.1	7.78	0.491E+08	0.00005	35.35	34.13	36.54	35.50	35.64	35.57
2	2.5	21.38	21.55	21.26	21.39	15.99	1338.9	7.77	0.492E+08	0.00008	31.79	30.61	32.51	31.73	31.85	31.79
3	5.5	23.08	23.39	23.07	23.18	16.06	1341.2	7.76	0.495E+08	0.00017	24.40	23.37	24.42	24.05	24.07	24.06
4	15.5	24.55	25.61	24.80	24.99	16.28	1348.9	7.70	0.504E+08	0.00049	20.69	18.34	20.08	19.65	19.71	19.68
5	25.5	25.40	26.40	25.94	25.91	16.49	1356.7	7.65	0.514E+08	0.00081	19.21	17.27	18.10	18.16	18.20	18.18
6	45.5	25.17	26.58	25.91	25.89	16.93	1372.1	7.55	0.534E+08	0.00145	20.73	17.71	19.04	19.08	19.16	19.12
7	75.5	25.63	26.92	25.90	26.15	17.58	1392.5	7.42	0.561E+08	0.00240	21.19	18.27	20.51	19.91	19.99	19.95
8	105.5	25.91	27.61	26.29	26.60	18.23	1413.5	7.30	0.589E+08	0.00336	22.18	18.17	21.13	20.35	20.49	20.42
9	135.5	26.52	28.18	27.18	27.29	18.88	1435.1	7.18	0.618E+08	0.00433	22.28	18.28	20.49	20.22	20.35	20.28
10	165.2	26.92	28.63	27.53	27.69	19.53	1457.2	7.06	0.647E+08	0.00529	22.98	18.66	21.22	20.80	20.95	20.88
11	205.2	26.92	28.82	27.66	27.80	20.40	1488.1	6.89	0.688E+08	0.00658	25.96	20.12	23.32	22.88	23.14	23.01
12	245.2	27.88	29.69	28.71	28.76	21.27	1520.3	6.73	0.730E+08	0.00788	25.55	20.07	22.70	22.55	22.77	22.66
13	275.2	28.34	30.26	29.36	29.32	21.92	1544.9	6.61	0.762E+08	0.00887	26.26	20.23	22.69	22.80	23.06	22.93
14	305.2	29.02	30.93	29.88	29.95	22.57	1567.1	6.51	0.791E+08	0.00985	26.12	20.15	23.04	22.85	23.10	22.97
15	333.3	29.81	31.83	30.87	30.84	23.19	1588.5	6.41	0.820E+08	0.01077	25.40	19.46	21.88	21.98	22.25	22.12
16	363.3	29.99	32.08	31.10	31.06	23.84	1612.0	6.31	0.851E+08	0.01176	27.30	20.37	23.12	23.26	23.60	23.43
17	383.3	30.48	32.26	31.30	31.35	24.27	1628.1	6.24	0.873E+08	0.01242	27.03	21.00	23.87	23.72	23.97	23.84
18	403.3	30.86	32.75	31.78	31.80	24.71	1644.4	6.17	0.894E+08	0.01308	27.26	20.84	23.68	23.64	23.93	23.78
19	423.3	31.43	33.39	32.51	32.44	25.14	1661.1	6.10	0.916E+08	0.01374	26.64	20.30	22.72	22.93	23.22	23.08
20	443.3	31.56	33.63	32.56	32.58	25.58	1678.2	6.03	0.938E+08	0.01441	27.98	20.78	23.97	23.89	24.24	24.06
21	463.3	32.22	34.07	33.14	33.14	26.01	1695.6	5.96	0.961E+08	0.01507	26.91	20.74	23.43	23.43	23.69	23.56
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
	391.6	30.69	32.66	31.69	31.68	24.45	1635.4	6.21	0.882E+08	0.01270	26.94	20.46	23.21	23.24	23.53	23.38

XP090-17

INPUT ELECTRIC POWER = 1800.0 W HEAT RATE GAINED BY WATER = 1700.4 W HEAT BALANCE ERROR = 5.53%  
 MASS FLOW RATE = 28.5400 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 954.6 GRM = 0.10638E+08 UPSTREAM BULK TEMPERATURE = 15.87 DEG C DOWN STREAM BULK TEMPERATURE = 30.15 DEG C  
 PRM = 6.436 RAM = 0.68464E+08 INLET BULK TEMPERATURE = 15.89 DEG C OUTLET BULK TEMPERATURE = 30.14 DEG C

STATION NO.	Z CM	WALL TEMPERATURE (DEG C)			TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	19.94	20.12	19.78	19.92	15.89	805.5	7.80	0.411E+08	0.00000	35.72	34.13	37.18	35.90	36.05	35.98
1	1.5	20.68	20.95	20.58	20.72	15.93	806.4	7.79	0.412E+08	0.00008	30.43	28.82	31.10	30.22	30.36	30.29
2	2.5	21.20	21.52	21.13	21.27	15.96	807.1	7.78	0.413E+08	0.00013	27.61	26.02	27.95	27.25	27.38	27.32
3	5.5	22.74	23.22	22.79	22.92	16.06	809.0	7.76	0.417E+08	0.00029	21.60	20.16	21.44	21.04	21.06	21.05
4	15.5	23.89	24.88	24.13	24.30	16.36	815.5	7.68	0.428E+08	0.00081	19.17	16.94	18.58	18.18	18.23	18.21
5	25.5	24.62	25.73	25.22	25.19	16.66	822.1	7.61	0.440E+08	0.00134	18.12	15.90	16.85	16.91	16.96	16.93
6	45.5	24.68	26.08	25.45	25.39	17.27	834.1	7.48	0.462E+08	0.00240	19.44	16.35	17.70	17.74	17.83	17.79
7	75.5	25.19	26.42	25.45	25.69	18.18	851.7	7.31	0.495E+08	0.00399	20.50	17.45	19.77	19.15	19.24	19.19
8	105.5	25.58	27.27	25.85	26.23	19.10	870.0	7.14	0.529E+08	0.00559	22.10	17.52	21.21	20.07	20.28	20.18
9	135.5	26.41	27.91	27.07	27.13	20.01	889.2	6.97	0.564E+08	0.00720	22.34	18.10	20.23	20.07	20.22	20.15
10	165.2	26.86	28.63	27.81	27.76	20.91	908.9	6.80	0.600E+08	0.00880	23.96	18.48	20.68	20.80	21.04	20.92
11	205.2	27.48	29.21	28.22	28.30	22.13	936.1	6.58	0.650E+08	0.01056	26.56	20.08	23.34	23.03	23.33	23.18
12	245.2	28.77	30.47	29.61	29.61	23.34	961.6	6.38	0.698E+08	0.01314	26.11	19.90	22.63	22.60	22.88	22.74
13	275.2	29.57	31.32	30.58	30.49	24.25	981.7	6.24	0.735E+08	0.01478	26.63	20.03	22.37	22.70	23.01	22.85
14	305.2	30.47	32.10	31.28	31.28	25.17	1002.6	6.10	0.774E+08	0.01643	26.63	20.36	23.10	23.08	23.36	23.22
15	333.3	31.58	33.45	32.55	32.53	26.02	1023.0	5.96	0.811E+08	0.01798	25.33	18.96	21.58	21.65	21.96	21.80
16	363.3	32.26	34.14	33.22	33.21	26.93	1045.4	5.82	0.852E+08	0.01964	26.39	19.51	22.37	22.41	22.76	22.58
17	383.3	32.87	34.44	33.63	33.64	27.54	1058.4	5.74	0.877E+08	0.02074	26.36	20.35	23.06	23.00	23.26	23.13
18	403.3	33.42	35.04	34.23	34.23	28.15	1071.7	5.67	0.902E+08	0.02185	26.61	20.35	23.04	23.06	23.33	23.20
19	423.3	33.98	35.67	34.79	34.81	28.76	1085.3	5.59	0.928E+08	0.02295	26.82	20.26	23.19	23.12	23.42	23.27
20	443.3	34.62	36.20	35.34	35.39	29.36	1099.3	5.51	0.955E+08	0.02406	26.59	20.45	23.40	23.22	23.48	23.35
21	463.3	35.29	36.78	35.93	36.00	29.97	1113.6	5.44	0.982E+08	0.02517	26.26	20.50	23.45	23.16	23.40	23.28
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
	391.6	33.12	34.82	33.96	33.97	27.79	1063.8	5.72	0.887E+08	0.02120	26.35	19.98	22.77	22.74	23.04	22.89

XP090-18

INPUT ELECTRIC POWER = 1300.0 W HEAT RATE GAINED BY WATER = 1276.8 W HEAT BALANCE ERROR = 1.78%  
 MASS FLOW RATE = 14.1000 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 513.5 GRM = 0.10774E+08 UPSTREAM BULK TEMPERATURE = 15.82 DEG C DOWN STREAM BULK TEMPERATURE = 37.53 DEG C  
 PRM = 5.857 RAM = 0.63103E+08 INLET BULK TEMPERATURE = 15.84 DEG C OUTLET BULK TEMPERATURE = 37.51 DEG C

STATION NO.	Z CM	WALL TEMPERATURE (DEG C)			TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	19.54	19.75	19.46	19.56	15.85	397.5	7.81	0.307E+08	0.00001	29.37	27.80	30.01	29.22	29.30	29.26
1	1.5	20.13	20.42	20.11	20.20	15.91	398.2	7.79	0.309E+08	0.00016	25.75	24.07	25.83	25.29	25.37	25.33
2	2.5	20.53	20.88	20.56	20.65	15.96	398.7	7.78	0.310E+08	0.00027	23.73	22.03	23.57	23.15	23.22	23.19
3	5.5	21.74	22.27	21.91	21.98	16.10	400.1	7.75	0.314E+08	0.00058	19.21	17.56	18.66	18.45	18.48	18.46
4	15.5	22.67	23.64	22.85	23.05	16.56	405.0	7.63	0.327E+08	0.00165	17.74	15.30	17.22	16.69	16.75	16.72
5	25.5	23.44	24.40	23.95	23.93	17.02	409.8	7.53	0.340E+08	0.00272	16.85	14.66	15.62	15.66	15.71	15.68
6	45.5	23.62	24.97	24.46	24.35	17.95	418.5	7.35	0.365E+08	0.00486	19.01	15.37	16.58	16.86	16.99	16.92
7	75.5	24.36	25.47	24.67	24.83	19.33	432.2	7.09	0.404E+08	0.00810	21.41	17.51	20.14	19.55	19.69	19.62
8	105.5	25.24	26.66	25.68	25.86	20.72	447.0	6.83	0.445E+08	0.01136	23.68	18.03	21.58	20.83	21.10	20.96
9	135.5	26.74	27.91	27.35	27.33	22.11	462.3	6.58	0.487E+08	0.01465	23.04	18.41	20.36	20.43	20.60	20.52
10	165.2	27.86	29.24	28.64	28.58	23.48	476.5	6.36	0.528E+08	0.01792	24.31	18.47	20.60	20.86	21.13	21.00
11	205.2	29.31	30.65	29.89	29.95	25.33	497.2	6.07	0.586E+08	0.02236	26.60	19.91	23.23	22.92	23.25	23.09
12	245.2	31.32	32.63	32.00	31.98	27.18	519.0	5.79	0.647E+08	0.02684	25.49	19.34	21.88	21.95	22.23	22.09
13	275.2	32.62	33.93	33.36	33.31	28.56	534.0	5.61	0.691E+08	0.03019	25.91	19.59	21.92	21.98	22.47	22.33
14	305.2	34.02	35.22	34.51	34.58	29.95	549.9	5.44	0.736E+08	0.03356	25.78	19.89	22.99			

XP090-22

INPUT ELECTRIC POWER = 4310.0 W HEAT RATE GAINED BY WATER = 4142.8 W HEAT BALANCE ERROR = 3.88%  
 MASS FLOW RATE = 37.9260 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 1549.4 GRM = 0.50867E+08 UPSTREAM BULK TEMPERATURE = 19.00 DEG C DOWN STREAM BULK TEMPERATURE = 45.18 DEG C  
 PRM = 5.171 RAM = 0.26305E+09 INLET BULK TEMPERATURE = 19.02 DEG C OUTLET BULK TEMPERATURE = 45.17 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-NUSSELT NUMBER-			-AVERAGE-		
		A	B	C	AVER-AGE						A	B	C	T	H	T+H
0	0.1	26.65	27.17	26.41	26.68	19.03	1154.3	7.15	0.128E+09	0.00000	45.82	42.88	47.29	45.63	45.82	45.72
1	1.5	28.12	28.86	28.04	28.29	19.11	1156.5	7.14	0.129E+09	0.00006	38.74	35.81	39.09	38.01	38.18	38.10
2	2.5	29.13	30.02	29.16	29.40	19.16	1158.0	7.12	0.130E+09	0.00010	35.01	32.16	34.91	34.09	34.24	34.17
3	5.5	32.18	33.51	32.53	32.74	19.33	1162.6	7.09	0.131E+09	0.00022	27.16	24.61	26.43	26.02	26.07	26.04
4	15.5	33.42	36.14	34.04	34.53	19.89	1178.2	6.99	0.136E+09	0.00062	25.75	21.44	24.62	23.79	23.94	23.86
5	25.5	34.90	37.39	36.09	36.13	20.45	1194.2	6.88	0.142E+09	0.00102	24.07	20.52	22.24	22.18	22.28	22.23
6	45.5	34.54	37.47	35.99	36.00	21.56	1227.6	6.67	0.153E+09	0.00183	26.73	21.80	24.03	24.02	24.19	24.10
7	75.5	35.38	38.34	36.27	36.66	23.24	1274.8	6.40	0.169E+09	0.00304	28.45	22.87	26.51	25.73	25.94	25.84
8	105.5	36.20	39.86	36.98	37.68	24.91	1324.4	6.14	0.186E+09	0.00427	30.48	23.01	28.51	26.95	27.33	27.14
9	135.5	37.35	41.02	38.72	39.03	26.58	1377.9	5.87	0.204E+09	0.00551	31.83	23.75	28.24	27.54	27.94	27.74
10	165.2	38.51	42.63	40.38	40.51	28.24	1426.7	5.66	0.221E+09	0.00674	33.26	23.72	28.12	27.83	28.37	28.10
11	205.2	39.53	43.59	40.90	41.34	30.47	1495.8	5.37	0.245E+09	0.00840	37.49	25.89	32.57	31.25	31.98	31.62
12	245.2	41.68	45.64	43.65	43.66	32.70	1567.6	5.10	0.269E+09	0.01009	37.62	26.12	30.87	30.84	31.54	31.19
13	275.2	43.12	47.23	45.31	45.22	34.37	1620.7	4.91	0.287E+09	0.01138	38.49	26.17	30.78	31.03	31.81	31.42
14	305.2	44.62	48.53	46.44	46.53	36.05	1677.5	4.71	0.305E+09	0.01270	39.10	26.85	32.24	31.97	32.73	32.35
15	333.3	46.01	50.73	48.32	48.35	37.61	1730.5	4.55	0.323E+09	0.01392	39.76	25.46	31.18	31.09	32.14	31.61
16	363.3	47.94	52.19	49.95	50.03	39.29	1786.3	4.40	0.343E+09	0.01522	38.47	25.80	31.21	30.99	31.83	31.41
17	383.3	48.86	52.65	50.54	50.68	40.40	1825.6	4.29	0.357E+09	0.01609	39.29	27.12	32.77	32.31	33.06	32.68
18	403.3	49.97	53.89	51.66	51.84	41.52	1866.6	4.19	0.371E+09	0.01696	39.20	26.78	32.67	32.10	32.88	32.49
19	423.3	51.39	55.37	53.25	53.34	42.63	1904.8	4.10	0.384E+09	0.01783	37.77	25.97	31.15	30.90	31.63	31.27
20	443.3	51.90	56.01	53.64	53.85	43.75	1942.3	4.01	0.398E+09	0.01871	40.49	26.93	33.37	32.68	33.59	33.14
21	463.3	53.44	57.06	55.03	55.18	44.86	1981.4	3.92	0.412E+09	0.01960	38.41	27.00	32.40	31.94	32.60	32.27
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 49.34 53.47 51.23 51.35 40.87 1842.7 4.26 0.363E+09 0.01646 39.16 26.34 32.06 31.68 32.52 32.10																

XP090-23

INPUT ELECTRIC POWER = 3100.0 W HEAT RATE GAINED BY WATER = 2961.2 W HEAT BALANCE ERROR = 4.48%  
 MASS FLOW RATE = 22.0420 G/S PRESSURE DROP = 0.0000 MM H2O FRICTION FACTOR = 0.000000 FREM = 0.0000  
 REM = 991.8 GRM = 0.48697E+08 UPSTREAM BULK TEMPERATURE = 20.75 DEG C DOWN STREAM BULK TEMPERATURE = 52.95 DEG C  
 PRM = 4.619 RAM = 0.22493E+09 INLET BULK TEMPERATURE = 20.78 DEG C OUTLET BULK TEMPERATURE = 52.94 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-NUSSELT NUMBER-			-AVERAGE-		
		A	B	C	AVER-AGE						A	B	C	T	H	T+H
0	0.1	27.21	27.69	26.96	27.24	20.75	699.8	6.82	0.104E+09	0.00000	38.64	35.97	40.23	38.50	38.77	38.63
1	1.5	28.33	29.00	28.18	28.46	20.89	701.6	6.80	0.104E+09	0.00010	33.37	30.60	34.05	32.76	33.02	32.89
2	2.5	29.10	29.90	29.02	29.31	20.95	702.7	6.79	0.105E+09	0.00017	30.49	27.74	30.79	29.71	29.95	29.83
3	5.5	31.40	32.61	31.54	31.85	21.16	706.3	6.75	0.106E+09	0.00038	24.23	21.67	23.91	23.21	23.27	23.24
4	15.5	32.47	34.67	32.92	33.36	21.85	718.6	6.62	0.111E+09	0.00107	23.31	19.31	22.36	21.52	21.66	21.59
5	25.5	33.67	35.84	34.70	34.74	22.53	729.4	6.51	0.116E+09	0.00177	22.20	18.59	20.32	20.26	20.37	20.32
6	45.5	33.65	36.64	35.44	35.24	23.90	752.1	6.30	0.125E+09	0.00316	25.29	19.36	21.37	21.74	22.01	21.88
7	75.5	34.82	37.45	35.71	35.99	25.96	789.0	5.97	0.141E+09	0.00527	27.69	21.36	25.18	24.46	24.74	24.60
8	105.5	36.03	39.19	37.03	37.42	28.02	825.5	5.68	0.156E+09	0.00740	30.47	21.85	27.09	25.98	26.47	26.22
9	135.5	37.68	40.68	39.22	39.20	30.08	862.0	5.42	0.172E+09	0.00953	31.96	22.92	26.58	26.65	27.15	26.90
10	165.2	39.73	43.03	41.49	41.42	32.11	900.8	5.17	0.188E+09	0.01167	31.78	22.17	25.79	26.01	26.58	26.29
11	205.2	41.70	44.93	43.13	43.25	34.86	951.2	4.85	0.209E+09	0.01463	35.11	23.86	29.05	28.62	29.34	28.98
12	245.2	44.51	47.53	45.99	46.01	37.60	1005.5	4.55	0.231E+09	0.01763	34.54	24.03	28.44	28.37	29.00	28.69
13	275.2	46.34	49.51	48.09	47.98	39.66	1045.7	4.36	0.248E+09	0.01985	35.59	24.13	28.20	28.57	29.31	28.94
14	305.2	47.67	50.64	49.17	49.16	41.72	1089.2	4.17	0.267E+09	0.02209	39.76	26.53	31.76	31.80	32.68	32.24
15	333.3	50.12	53.69	51.90	51.91	43.64	1126.8	4.02	0.283E+09	0.02420	36.42	23.48	28.58	28.56	29.49	29.03
16	363.3	52.15	55.41	53.63	53.73	45.70	1169.3	3.86	0.302E+09	0.02648	36.45	24.21	29.63	29.27	30.10	29.68
17	383.3	53.30	56.22	54.59	54.70	47.07	1198.9	3.75	0.315E+09	0.02801	37.65	25.63	31.20	30.73	31.49	31.11
18	403.3	54.77	57.74	56.12	56.21	48.44	1227.7	3.66	0.327E+09	0.02954	37.01	25.16	30.50	30.13	30.89	30.51
19	423.3	56.05	59.38	57.87	57.77	49.82	1257.8	3.56	0.341E+09	0.03108	37.45	24.42	28.98	29.37	30.28	29.82
20	443.3	57.14	60.31	58.59	58.68	51.19	1289.4	3.47	0.354E+09	0.03263	39.11	25.54	31.48	31.09	32.04	31.57
21	463.3	58.79	61.60	60.15	60.18	52.56	1319.1	3.38	0.368E+09	0.03417	37.32	25.72	30.63	30.59	31.22	30.87
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 53.92 57.13 55.45 55.50 47.64 1211.6 3.72 0.320E+09 0.02866 37.35 24.74 30.06 29.86 30.72 30.29																

XP090-24

INPUT ELECTRIC POWER = 3045.0 W HEAT RATE GAINED BY WATER = 2941.9 W HEAT BALANCE ERROR = 3.39%  
 MASS FLOW RATE = 22.0420 G/S PRESSURE DROP = 0.7110 MM H2O FRICTION FACTOR = 0.041721 FREM = 41.9280  
 REM = 1005.0 GRM = 0.50328E+08 UPSTREAM BULK TEMPERATURE = 21.57 DEG C DOWN STREAM BULK TEMPERATURE = 53.56 DEG C  
 PRM = 4.554 RAM = 0.22917E+09 INLET BULK TEMPERATURE = 21.60 DEG C OUTLET BULK TEMPERATURE = 53.54 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-NUSSELT NUMBER-			-AVERAGE-		
		A	B	C	AVER-AGE						A	B	C	T	H	T+H
0	0.1	31.04	31.52	27.45	29.39	21.60	714.2	6.67	0.109E+09	0.00000	26.08	24.83	42.13	31.61	33.79	32.70
1	1.5	31.23	31.90	28.64	30.14	21.70	716.0	6.65	0.110E+09	0.00010	25.85	24.14	35.49	29.18	30.24	29.71
2	2.5	31.35	32.16	29.46	30.65	21.77	717.2	6.64	0.110E+09	0.00017	25.68	23.69	32.01	27.70	28.35	28.03
3	5.5	31.73	32.94	31.92	32.20	21.97	720.6	6.60	0.111E+09	0.00038	25.21	22.43	24.73	24.06	24.12	24.09
4	15.5	32.86	35.07	33.26	33.73	22.66	731.4	6.49	0.116E+09	0.00107	24.07	19.79	23.16	22.18	22.34	22.26
5	25.5	34.12	36.17	34.98	35.09	23.34	742.6	6.39	0.121E+09	0.00177	22.75	19.11	21.07	20.87	20.98	20.92
6	45.5	34.04	36.81	35.61	35.48	24.70	766.0	6.17	0.130E+09	0.00317	26.18	20.20	22.42	22.67	22.93	22.80
7	75.5	35.27	37.89	36.16	36.44	26.74	804.0	5.85	0.146E+09	0.00528	28.55	21.82	25.86	25.10	25.41	25.26
8	105.5	36.53	39.81	37.53	37.96	28.79	838.8	5.59	0.161E+09	0.00741	31.27	21.98	27.70	26.42	26.98	26.70
9	135.5	38.24	41.18	39.72	39.71	30.83	876.3	5.33	0.177E+09	0.00955	32.55	23.29	27.12	27.14	27.65	27.40
10	165.2	40.22	43.53	42.00	41.92	32.86	913.9	5.08	0.192E+09	0.01170	32.57	22.48	26.24	26.48	27.10	26.79
11	205.2	42.26	45.43	43.68	43.79	35.58	965.5	4.77	0.213E+09	0.01466	35.70	24.20	29.41	29.03	29.77	29.40
12	245.2	45.01	47.98	46.44	46.47	38.31	1019.0	4.49	0.235E+09	0.01765	35.36	24.49	29.12	29.00	29.66	29.33
13	275.2	46.56	49.73	48.31	48.20	40.35	1060.0	4.30	0.253E+09	0.01987	37.99	25.14	29.64	30.05	30.92	30.49
14	305.2	48.84	51.64	50.23	50.24	42.40	1102.5	4.12	0.271E+09	0.02212	36.47	25.41	30.00	29.97	30.63	30.30
15	333.3	51.07														

## Appendix 3

### Experimental Data for $\alpha = 135^\circ$

XP135-01

INPUT ELECTRIC POWER = 145.0 W HEAT RATE GAINED BY WATER = 144.4 W HEAT BALANCE ERROR = 0.43%  
 MASS FLOW RATE = 14.1000 G/S PRESSURE DROP = 0.3069 MM H2O FRICTION FACTOR = 0.044214 FREM = 21.8983  
 REM = 495.3 GRM = 0.10770E+07 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 26.39 DEG C  
 PRM = 6.097 RAM = 0.65660E+07 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 26.39 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-NUSSLETT NUMBER-			-AVERAGE-			
		A	B	C	AVERAGE						T	H	T+H				
0	0.1	24.45	24.47	24.47	24.46	23.94	481.5	6.29	0.613E+07	0.00001	23.30	22.53	22.72	22.89	22.82	22.86	
1	1.5	24.53	24.55	24.55	24.54	23.94	481.6	6.29	0.613E+07	0.00016	20.56	19.73	19.79	20.05	19.97	20.01	
2	2.5	24.58	24.61	24.61	24.60	23.95	481.6	6.29	0.613E+07	0.00027	19.03	18.17	18.17	18.47	18.39	18.43	
3	5.5	24.74	24.78	24.79	24.77	23.97	481.8	6.29	0.614E+07	0.00060	15.54	14.69	14.60	14.93	14.94	14.94	
4	15.5	24.99	24.99	25.02	25.00	24.02	482.4	6.28	0.616E+07	0.00168	12.30	12.30	11.96	12.19	12.19	12.19	
5	25.5	25.07	25.12	25.17	25.12	24.07	483.0	6.27	0.618E+07	0.00277	12.06	11.42	10.96	11.46	11.48	11.47	
6	45.5	25.12	25.19	25.18	25.16	24.17	484.1	6.25	0.621E+07	0.00494	12.75	11.84	11.91	12.15	12.17	12.16	
7	75.5	25.02	25.14	25.12	25.09	24.33	485.9	6.23	0.627E+07	0.00821	17.38	14.87	15.25	15.77	15.85	15.81	
8	105.5	25.13	25.21	25.18	25.18	24.49	487.6	6.20	0.632E+07	0.01147	18.60	16.60	17.26	17.45	17.49	17.47	
9	135.5	25.24	25.35	25.52	25.37	24.64	489.4	6.18	0.638E+07	0.01474	20.20	17.02	13.76	16.58	16.99	16.79	
10	165.2	25.47	25.57	25.69	25.58	24.80	491.1	6.15	0.644E+07	0.01798	17.85	15.65	13.41	15.42	15.63	15.53	
11	205.2	25.64	25.76	25.77	25.72	25.01	493.5	6.12	0.651E+07	0.02235	19.04	15.94	15.71	16.77	16.90	16.83	
12	245.2	25.89	26.02	26.03	25.98	25.22	495.9	6.09	0.659E+07	0.02672	17.87	14.90	14.70	15.70	15.82	15.76	
13	275.2	26.07	26.14	26.19	26.13	25.37	497.8	6.06	0.664E+07	0.03000	17.32	15.57	14.72	15.79	15.87	15.83	
14	305.2	26.19	26.31	26.32	26.27	25.53	499.6	6.04	0.670E+07	0.03328	18.22	15.36	15.25	16.17	16.28	16.22	
15	333.3	26.42	26.57	26.57	26.52	25.68	501.3	6.02	0.676E+07	0.03636	16.10	13.39	13.48	14.22	14.32	14.27	
16	363.3	26.56	26.69	26.75	26.67	25.84	503.2	5.99	0.681E+07	0.03965	16.61	13.93	13.08	14.39	14.54	14.47	
17	383.3	26.70	26.78	26.81	26.76	25.94	504.5	5.97	0.685E+07	0.04184	15.74	14.18	13.73	14.50	14.55	14.53	
18	403.3	26.78	26.84	26.88	26.84	26.04	505.7	5.96	0.689E+07	0.04404	16.19	15.05	14.26	15.12	15.16	15.14	
19	423.3	26.88	26.99	26.99	26.95	26.15	507.0	5.94	0.693E+07	0.04623	16.40	14.24	14.24	14.89	14.96	14.92	
20	443.3	26.93	26.97	27.05	26.98	26.25	508.2	5.92	0.697E+07	0.04843	17.58	16.76	15.01	16.38	16.45	16.41	
21	463.3	27.04	27.03	27.24	27.11	26.36	509.5	5.91	0.701E+07	0.05063	17.41	17.69	13.57	15.99	16.22	16.10	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:		391.6	26.71	26.81	26.84	26.79	25.98	505.0	5.97	0.687E+07	0.04276	16.44	14.59	13.97	14.92	15.00	14.96

XP135-02

INPUT ELECTRIC POWER = 294.0 W HEAT RATE GAINED BY WATER = 292.3 W HEAT BALANCE ERROR = 0.56%  
 MASS FLOW RATE = 28.5400 G/S PRESSURE DROP = 0.6581 MM H2O FRICTION FACTOR = 0.023141 FREM = 23.1989  
 REM = 1002.5 GRM = 0.21807E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 26.39 DEG C  
 PRM = 6.097 RAM = 0.13295E+08 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 26.39 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-NUSSLETT NUMBER-			-AVERAGE-			
		A	B	C	AVERAGE						T	H	T+H				
0	0.1	24.70	24.69	24.66	24.68	23.94	974.6	6.29	0.124E+08	0.00000	32.05	32.14	33.41	32.57	32.75	32.66	
1	1.5	24.83	24.82	24.79	24.81	23.94	974.8	6.29	0.124E+08	0.00008	27.61	27.70	28.88	28.08	28.27	28.17	
2	2.5	24.91	24.91	24.87	24.90	23.95	974.9	6.29	0.124E+08	0.00013	25.20	25.30	26.41	25.64	25.83	25.74	
3	5.5	25.18	25.18	25.12	25.16	23.96	975.2	6.29	0.124E+08	0.00030	19.97	20.06	21.02	20.34	20.35	20.35	
4	15.5	25.49	25.56	25.52	25.52	24.02	976.4	6.28	0.125E+08	0.00083	16.47	15.82	16.17	16.15	16.15	16.15	
5	25.5	25.74	25.79	25.83	25.79	24.07	977.5	6.27	0.125E+08	0.00137	15.57	14.15	13.79	14.16	14.17	14.17	
6	45.5	25.78	25.86	25.91	25.85	24.17	979.9	6.25	0.126E+08	0.00244	14.11	14.46	14.05	14.53	14.54	14.53	
7	75.5	25.80	25.86	25.84	25.84	24.33	983.4	6.23	0.127E+08	0.00405	16.55	15.89	16.06	16.16	16.17	16.16	
8	105.5	25.74	25.88	25.91	25.84	24.49	986.9	6.20	0.128E+08	0.00567	19.34	17.48	17.14	17.94	17.99	17.96	
9	135.5	25.91	26.07	26.24	26.07	24.64	990.9	6.18	0.129E+08	0.00728	19.27	17.02	15.24	17.02	17.17	17.09	
10	165.2	26.03	26.24	26.36	26.21	24.80	994.1	6.15	0.130E+08	0.00888	19.78	16.88	15.55	17.23	17.40	17.32	
11	205.2	26.14	26.43	26.44	26.34	25.01	998.9	6.12	0.132E+08	0.01104	21.38	17.11	16.98	18.28	18.49	18.38	
12	245.2	26.50	26.63	26.71	26.61	25.22	1003.8	6.09	0.133E+08	0.01320	18.93	17.14	16.31	17.39	17.46	17.43	
13	275.2	26.62	26.81	26.91	26.78	25.37	1007.5	6.06	0.135E+08	0.01482	19.44	16.89	15.79	17.24	17.37	17.31	
14	305.2	26.69	26.92	26.99	26.87	25.53	1011.2	6.04	0.136E+08	0.01644	20.95	17.44	16.64	18.16	18.34	18.25	
15	333.3	27.03	27.30	27.41	27.24	25.68	1014.8	6.02	0.137E+08	0.01796	17.96	14.95	14.03	15.48	15.65	15.56	
16	363.3	27.11	27.42	27.47	27.33	25.84	1018.5	5.99	0.138E+08	0.01959	18.99	15.32	14.80	16.17	16.37	16.27	
17	383.3	27.31	27.51	27.58	27.47	25.94	1021.1	5.97	0.139E+08	0.02067	17.67	15.42	14.74	15.85	15.94	15.89	
18	403.3	27.40	27.62	27.66	27.56	26.04	1023.6	5.96	0.140E+08	0.02176	17.87	15.40	14.98	15.99	16.08	16.04	
19	423.3	27.54	27.77	27.83	27.71	26.15	1026.2	5.94	0.140E+08	0.02284	17.35	14.97	14.42	15.48	15.58	15.53	
20	443.3	27.54	27.69	27.88	27.71	26.25	1028.7	5.92	0.141E+08	0.02393	18.75	16.80	14.85	16.65	16.80	16.73	
21	463.3	27.71	27.73	28.02	27.82	26.36	1031.3	5.91	0.142E+08	0.02501	17.89	17.60	14.59	16.55	16.69	16.62	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:		391.6	27.32	27.55	27.64	27.50	25.98	1022.1	5.97	0.139E+08	0.02112	18.10	15.48	14.64	15.94	16.07	16.00

XP135-03

INPUT ELECTRIC POWER = 316.0 W HEAT RATE GAINED BY WATER = 311.8 W HEAT BALANCE ERROR = 1.31%  
 MASS FLOW RATE = 42.9800 G/S PRESSURE DROP = 0.9616 MM H2O FRICTION FACTOR = 0.014311 FREM = 22.2952  
 REM = 1495.2 GRM = 0.22489E+07 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 25.62 DEG C  
 PRM = 6.162 RAM = 0.13858E+08 INLET BULK TEMPERATURE = 23.88 DEG C OUTLET BULK TEMPERATURE = 25.62 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	-NUSSLETT NUMBER-			-AVERAGE-		
		A	B	C	AVERAGE						T	H	T+H			
0	0.1	24.62	24.62	24.59	24.61	23.88	1465.8	6.30	0.132E+08	0.00000	35.16	35.26	36.69	35.75	35.95	35.85
1	1.5	24.74	24.74	24.70	24.72	23.88	1466.0	6.30	0.132E+08	0.00005	30.41	30.52	31.86	30.94	31.16	31.05
2	2.5	24.82	24.82	24.78	24.81	23.89	1466.1	6.30	0.132E+08	0.00009	27.81	27.92	29.20	28.32	28.53	28.43
3	5.5	25.07	25.07	25.01	25.05	23.90	1466.5	6.30	0.132E+08	0.00020	22.14	22.25	23.36	22.57	22.59	22.58
4	15.5	25.27	25.33	25.24	25.28	23.94	1467.7	6.29	0.132E+08	0.00055	19.44	18.66	19.85	19.31	19.32	19.31
5	25.5	25.46	25.46	25.50	25.47	23.97	1468.9	6.28	0.133E+08	0.00091	17.52	17.52	17.01	17.35	17.35	17.35
6	45.5	25.56	25.58	25.63	25.59	24.05	1471.4	6.27	0.133E+08	0.00162	17.15	16.96	16.43	16.84	16.85	16.85
7	75.5	25.69	25.69	25.67	25.69	24.16	1475.2	6.26	0.134E+08	0.00269	16.96	16.90	17.15	17.00	17.00	17.00
8	105.5	25.63	25.88	25.79	25.77	24.27	1478.9	6.24	0.135E+08	0.00376	19.03	16.14	17.02	17.31	17.39	17.35
9	135.5	25.79	25.85	26.07	25.91	24.38	1482.7	6.22	0.136E+08	0.00483	18.35	17.66	15.34	17.01	17.11	17.06
10	165.2	25.86	26.02	26.14	26.01	24.49	1486.4	6.20	0.137E+08	0.00589	18.93	17.00	15.74	17.12	17.22	17.17
11	205.2	25.98	26.15	26.22	26.11	24.64	1491.5	6.18	0.138E+08	0.00732	19.36	17.16	16.43	17.57	17.65	17.61
12	245.2	26.17	26.36	26.37	26.30	24.79	1496.6	6.16	0.139E+08	0.00875	18.78	16.52	16.35	17.15	17.22	17.18
13	275.2	26.23	26.48	26.58	26.43	24.90	1500.5	6.14	0.140E+08	0.00983	19.40	16.40	15.42	16.92	17.08	17.00
14	305.2	26.30	26.48	26.59	26.46	25.01	1504.4	6.12	0.141E+08	0.01090	20.07	17.64	16.34	17.85	18.02	17.95

XP135-04

INPUT ELECTRIC POWER = 268.0 W HEAT RATE GAINED BY WATER = 264.6 W HEAT BALANCE ERROR = 1.28%  
 MASS FLOW RATE = 14.1000 G/S PRESSURE DROP = 0.3274 MM H2O FRICTION FACTOR = 0.047158 FREM = 23.8965  
 REM = 506.7 GRM = 0.21350E+07 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 28.38 DEG C  
 PRM = 5.944 RAM = 0.12690E+08 INLET BULK TEMPERATURE = 23.88 DEG C OUTLET BULK TEMPERATURE = 28.37 DEG C

STA-TION NO.	Z CM	WALL TEMPERATURE (DEG C)			TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE				
		A	B	C						A	B	C	T	H	T+H		
0	0.1	24.83	24.83	24.86	24.84	23.88	480.9	6.30	0.112E+08	0.00001	23.33	23.38	22.55	23.11	22.95	23.03	
1	1.5	24.97	24.96	25.02	24.98	23.90	481.1	6.30	0.112E+08	0.00016	20.60	20.66	19.67	20.32	20.15	20.24	
2	2.5	25.06	25.06	25.13	25.08	23.91	481.2	6.29	0.112E+08	0.00027	19.07	19.13	18.08	18.76	18.59	18.67	
3	5.5	25.35	25.34	25.45	25.38	23.94	481.5	6.29	0.112E+08	0.00060	15.58	15.64	14.55	15.24	15.26	15.25	
4	15.5	25.66	25.78	25.74	25.73	24.03	482.5	6.28	0.113E+08	0.00168	13.52	12.62	12.86	12.99	13.00	12.99	
5	25.5	25.85	25.96	26.06	25.95	24.13	483.6	6.26	0.114E+08	0.00277	12.79	12.05	11.42	12.06	12.08	12.07	
6	45.5	25.84	26.02	26.07	25.98	24.32	485.7	6.23	0.115E+08	0.00495	14.48	12.92	12.56	13.27	13.32	13.30	
7	75.5	25.91	26.08	26.07	26.02	24.61	488.9	6.18	0.117E+08	0.00821	16.93	14.89	15.06	15.58	15.63	15.60	
8	105.5	26.02	26.27	26.18	26.16	24.89	492.2	6.14	0.119E+08	0.01149	19.48	16.01	17.05	17.40	17.51	17.46	
9	135.5	26.29	26.52	26.63	26.48	25.18	495.5	6.09	0.120E+08	0.01476	19.73	16.45	15.19	16.92	17.12	17.02	
10	165.2	26.58	26.79	26.86	26.75	25.47	498.8	6.05	0.122E+08	0.01801	19.64	16.52	15.73	17.14	17.30	17.22	
11	205.2	26.87	27.09	27.11	27.02	25.85	503.4	5.99	0.125E+08	0.02239	21.55	17.61	17.45	18.69	18.87	18.78	
12	245.2	27.38	27.63	27.59	27.54	26.23	508.0	5.93	0.128E+08	0.02679	19.03	15.64	16.08	16.79	16.92	16.85	
13	275.2	27.57	27.92	27.97	27.82	26.52	511.5	5.88	0.130E+08	0.03009	20.91	15.61	15.13	16.85	17.21	17.03	
14	305.2	27.91	28.09	28.21	28.07	26.81	515.1	5.84	0.132E+08	0.03339	19.81	16.99	15.58	17.29	17.46	17.38	
15	333.3	28.36	28.64	28.64	28.55	27.08	518.0	5.80	0.133E+08	0.03648	17.01	13.94	13.99	14.85	14.98	14.92	
16	363.3	28.61	28.86	28.87	28.78	27.36	521.0	5.77	0.135E+08	0.03979	17.59	14.59	14.53	15.45	15.57	15.51	
17	383.3	28.81	29.02	29.08	28.97	27.55	523.0	5.74	0.136E+08	0.04199	17.38	14.88	14.29	15.41	15.52	15.46	
18	403.3	29.02	29.18	29.22	29.14	27.75	525.1	5.72	0.138E+08	0.04419	17.19	15.19	14.79	15.66	15.72	15.69	
19	423.3	29.21	29.44	29.44	29.36	27.94	527.2	5.69	0.139E+08	0.04640	17.14	14.54	14.48	15.29	15.39	15.34	
20	443.3	29.33	29.43	29.61	29.46	28.13	529.2	5.67	0.140E+08	0.04860	18.20	16.73	14.78	16.45	16.57	16.51	
21	463.3	29.49	29.56	29.74	29.60	28.32	531.3	5.64	0.141E+08	0.05081	18.58	17.66	15.32	17.07	17.18	17.13	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	28.89	29.10	29.14	29.04	27.63	523.9	5.73	0.137E+08	0.04291	17.42	14.98	14.48	15.52	15.62	15.57

XP135-05

INPUT ELECTRIC POWER = 665.0 W HEAT RATE GAINED BY WATER = 657.2 W HEAT BALANCE ERROR = 1.17%  
 MASS FLOW RATE = 42.2580 G/S PRESSURE DROP = 0.9991 MM H2O FRICTION FACTOR = 0.016023 FREM = 24.1423  
 REM = 1506.7 GRM = 0.51628E+07 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 27.66 DEG C  
 PRM = 5.996 RAM = 0.30958E+08 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 27.66 DEG C

STA-TION NO.	Z CM	WALL TEMPERATURE (DEG C)			TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE				
		A	B	C						A	B	C	T	H	T+H		
0	0.1	25.25	25.22	25.16	25.20	23.94	1443.1	6.29	0.279E+08	0.00000	41.79	42.51	44.83	43.23	43.49	43.36	
1	1.5	25.51	25.48	25.41	25.46	23.95	1443.5	6.29	0.279E+08	0.00005	35.04	35.74	37.37	36.14	36.38	36.26	
2	2.5	25.69	25.66	25.59	25.64	23.96	1443.7	6.29	0.279E+08	0.00009	31.52	32.21	33.52	32.47	32.69	32.58	
3	5.5	26.24	26.18	26.12	26.18	23.98	1444.5	6.28	0.280E+08	0.00020	24.23	24.84	25.61	24.88	24.89	24.88	
4	15.5	26.60	26.79	26.69	26.69	24.06	1447.1	6.27	0.281E+08	0.00056	21.53	20.00	20.81	20.76	20.78	20.77	
5	25.5	26.97	27.12	27.16	27.08	24.14	1449.7	6.26	0.282E+08	0.00092	19.34	18.33	18.10	18.58	18.59	18.58	
6	45.5	27.11	27.19	27.41	27.24	24.30	1455.0	6.23	0.285E+08	0.00165	19.43	18.91	17.56	18.60	18.64	18.62	
7	75.5	27.29	27.47	27.52	27.43	24.54	1463.0	6.20	0.289E+08	0.00274	19.81	18.61	16.33	18.90	18.92	18.91	
8	105.5	27.36	27.77	27.74	27.62	24.77	1471.0	6.16	0.293E+08	0.00383	21.16	18.22	18.39	19.16	19.25	19.21	
9	135.5	27.74	28.07	28.29	28.03	25.01	1479.2	6.12	0.296E+08	0.00492	20.02	17.84	16.66	18.07	18.17	18.12	
10	165.2	27.86	28.24	28.48	28.18	25.25	1487.4	6.08	0.300E+08	0.00601	20.92	18.24	16.89	18.54	18.68	18.61	
11	205.2	27.98	28.43	28.55	28.32	25.57	1498.5	6.03	0.306E+08	0.00747	22.60	19.05	18.23	19.79	19.96	19.87	
12	245.2	28.38	28.80	28.99	28.72	25.88	1509.8	5.98	0.311E+08	0.00893	21.79	18.68	17.54	19.17	19.33	19.25	
13	275.2	28.57	29.09	29.30	28.99	26.12	1518.4	5.95	0.315E+08	0.01003	22.26	18.31	17.13	19.00	19.23	19.12	
14	305.2	28.63	29.09	29.33	29.02	26.36	1527.1	5.91	0.319E+08	0.01113	23.93	19.90	18.33	20.46	20.72	20.59	
15	333.3	29.25	29.76	29.98	29.66	26.58	1535.3	5.87	0.323E+08	0.01216	20.39	17.11	16.02	17.65	17.84	17.75	
16	363.3	29.27	29.86	30.04	29.72	26.82	1544.2	5.83	0.327E+08	0.01326	22.17	17.88	16.89	18.72	18.98	18.85	
17	383.3	29.48	30.03	30.19	29.90	26.98	1549.4	5.81	0.330E+08	0.01400	21.75	17.82	16.93	18.62	18.83	18.73	
18	403.3	29.68	30.13	30.34	30.05	27.14	1554.3	5.79	0.332E+08	0.01473	21.34	18.17	16.97	18.65	18.83	18.74	
19	423.3	29.87	30.38	30.56	30.27	27.30	1559.3	5.77	0.335E+08	0.01547	21.08	17.59	16.66	18.26	18.45	18.35	
20	443.3	29.94	30.38	30.67	30.33	27.46	1564.4	5.75	0.337E+08	0.01620	21.86	18.54	16.90	18.89	19.10	18.99	
21	463.3	30.17	30.51	30.86	30.51	27.61	1569.5	5.73	0.340E+08	0.01694	21.26	18.73	16.71	18.72	18.90	18.81	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	29.58	30.09	30.29	29.99	27.05	1551.2	5.81	0.331E+08	0.01430	21.43	17.85	16.73	18.47	18.67	18.57

XP135-06

INPUT ELECTRIC POWER = 625.0 W HEAT RATE GAINED BY WATER = 612.4 W HEAT BALANCE ERROR = 2.02%  
 MASS FLOW RATE = 27.8180 G/S PRESSURE DROP = 0.6564 MM H2O FRICTION FACTOR = 0.024288 FREM = 24.5424  
 REM = 1010.5 GRM = 0.51233E+07 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 29.21 DEG C  
 PRM = 5.874 RAM = 0.30093E+08 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 29.21 DEG C

STA-TION NO.	Z CM	WALL TEMPERATURE (DEG C)			TB (C)	RE	PR	RA	Z	NUSELT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	25.35	25.35	25.32	25.33	23.94	950.0	6.29	0.260E+08	0.00000	36.26	36.26	36.86	36.63	36.56	36.60
1	1.5	25.63	25.63	25.62	25.62	23.96	950.4	6.29	0.260E+08	0.00008	30.50	30.50	30.59	30.62	30.54	30.58
2	2.5	25.82	25.82	25.83	25.82	23.97	950.6	6.29	0.260E+08	0.00014	27.48	27.48	27.37	27.50	27.43	27.47
3	5.5	26.41	26.41	26.45	26.42	24.00	951.3	6.28	0.261E+08	0.00030	21.20	21.20	20.81	21.07	21.07	21.07
4	15.5	26.88	27.08	27.08	27.01	24.11	953.8	6.26	0.263E+08	0.00085	18.43	17.19	17.19	17.59	17.61	17.60
5	25.5	27.24	27.40	27.66	27.44	24.23	956.2	6.24	0.264E+08	0.00140	16.88	16.05	14.83	15.87	15.92	15.90
6	45.5	27.33	27.52	27.74	27.53	24.45	961.2	6.21	0.268E+08	0.00251	17.66	16.58	15.46	16.52	16.57	16.54
7	75.5	27.41	27.69	27.74	27.61	24.79	968.7	6.16	0.273E+08	0.00416	19.44	17.50	17.24	18.01	18.06	18.03
8	105.5	27.52	27.99	27.97	27.83	25.12	976.3	6.10	0.278E+08	0.00582	21.21	17.72	17.89	18.81	18.94	18.87
9	135.5	27.85	28.24	28.51	28.20	25.46	984.0	6.05	0.283E+08	0.00749	21.27	18.29	16.66	18.55	18.74	18.65
10	165.2	28.08	28.52	28.76	28.45	25.80	991.8	6.00	0.288E+08	0.00914	22.24	18.65	17.15	19.12	19.35	19.24
11	205.2	28.31	28.87	28.94	28.71	26.24	1002.5	5.93	0.296E+08	0.01136	24.54	19.30	18.79	20.58	20.88	20.73
12	245.2	28.88	29.41	29.55	29.28	26.69	1013.4	5.85	0.303E+08	0.01359	23.14	18.65	17.74	19.58	19.84	19.71
13	275.2	29.18	29.71	29.97	29.62	27.03	1021.0	5.81	0.308E+08	0.01527	23.58	18.93	17.25	19.58	19.92	19.75
14	305.2	29.47	29.93	30.17	29.86	27.37	1028.0	5.76	0.313E+08	0.01694	24.11	19.72	18.08			

XP135-07

INPUT ELECTRIC POWER = 535.0 W HEAT RATE GAINED BY WATER = 525.6 W HEAT BALANCE ERROR = 1.75%  
 MASS FLOW RATE = 13.3780 G/S PRESSURE DROP = 0.3035 MM H2O FRICTION FACTOR = 0.048520 FREM = 24.6270  
 REM = 507.6 GRM = 0.50936E+07 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 33.36 DEG C  
 PRM = 5.604 RAM = 0.28542E+08 INLET BULK TEMPERATURE = 23.95 DEG C OUTLET BULK TEMPERATURE = 33.35 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	NUSSLELT NUMBER			AVERAGE		
		A	B	C	AVERAGE						T	H	T+H			
0	0.1	25.52	25.57	25.51	25.52	23.95	456.9	6.29	0.223E+08	0.00001	27.74	26.99	28.00	27.79	27.68	27.73
1	1.5	25.79	25.85	25.82	25.81	23.98	457.2	6.28	0.224E+08	0.00017	24.14	23.35	23.76	23.86	23.75	23.81
2	2.5	25.97	26.04	26.03	26.01	24.00	457.5	6.28	0.224E+08	0.00029	22.16	21.36	21.51	21.74	21.64	21.69
3	5.5	26.52	26.63	26.67	26.60	24.06	458.1	6.27	0.225E+08	0.00063	17.78	17.01	16.76	17.17	17.18	17.18
4	15.5	26.93	27.19	27.19	27.11	24.26	460.2	6.24	0.227E+08	0.00178	16.34	14.91	14.88	15.35	15.38	15.36
5	25.5	27.19	27.51	27.66	27.45	24.46	462.3	6.21	0.230E+08	0.00292	16.00	14.31	13.64	14.59	14.65	14.62
6	45.5	27.39	27.63	27.80	27.61	24.86	466.6	6.14	0.235E+08	0.00522	17.26	15.74	14.64	15.88	15.95	15.91
7	75.5	27.62	27.97	27.96	27.85	25.46	473.2	6.05	0.243E+08	0.00868	20.17	17.36	17.44	18.23	18.32	18.28
8	105.5	27.97	28.50	28.47	28.31	26.06	480.0	5.95	0.251E+08	0.01214	22.86	17.86	18.11	19.36	19.61	19.48
9	135.5	28.63	29.07	29.29	29.00	26.66	487.0	5.86	0.260E+08	0.01562	22.14	18.05	16.56	18.64	18.92	18.78
10	165.2	29.24	29.68	29.81	29.58	27.26	493.3	5.78	0.267E+08	0.01906	21.87	17.91	17.02	18.71	18.94	18.83
11	205.2	29.87	30.32	30.39	30.19	28.06	501.4	5.68	0.278E+08	0.02371	24.01	19.22	18.62	20.36	20.62	20.49
12	245.2	30.77	31.19	31.33	31.10	28.86	509.9	5.58	0.288E+08	0.02837	22.74	18.61	17.52	19.38	19.62	19.50
13	275.2	31.73	31.76	31.97	31.82	29.47	516.4	5.50	0.296E+08	0.03187	19.05	18.82	17.28	18.35	18.38	18.37
14	305.2	31.85	32.27	32.45	32.19	30.07	523.1	5.42	0.305E+08	0.03538	24.20	19.57	18.11	20.32	20.63	20.47
15	333.3	32.69	33.17	33.33	33.07	30.63	529.5	5.35	0.313E+08	0.03868	20.88	16.96	15.95	17.69	17.93	17.81
16	363.3	33.21	33.69	33.83	33.58	31.23	536.5	5.28	0.321E+08	0.04220	21.81	17.48	16.55	18.35	18.61	18.48
17	383.3	33.64	33.99	34.18	33.94	31.63	541.3	5.23	0.327E+08	0.04456	21.38	18.21	16.86	18.64	18.82	18.73
18	403.3	34.03	34.48	34.57	34.36	32.03	545.9	5.18	0.333E+08	0.04693	21.49	17.58	16.97	18.48	18.68	18.58
19	423.3	34.48	34.89	35.02	34.79	32.44	550.1	5.13	0.338E+08	0.04932	21.02	17.49	16.63	18.19	18.38	18.29
20	443.3	34.79	34.91	35.39	35.03	32.84	554.5	5.09	0.343E+08	0.05172	21.96	20.67	16.76	19.53	19.80	19.67
21	463.3	35.23	35.13	35.76	35.37	33.24	558.9	5.04	0.349E+08	0.05412	21.46	22.66	16.97	20.05	20.37	20.21
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 33.81 34.19 34.39 34.13 31.80 543.0 5.21 0.329E+08 0.04557 21.43 18.07 16.62 18.48 18.70 18.59																

XP135-08

INPUT ELECTRIC POWER = 1170.0 W HEAT RATE GAINED BY WATER = 1152.2 W HEAT BALANCE ERROR = 1.52%  
 MASS FLOW RATE = 40.8140 G/S PRESSURE DROP = 1.0128 MM H2O FRICTION FACTOR = 0.017406 FREM = 26.1953  
 REM = 1505.0 GRM = 0.10145E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 30.64 DEG C  
 PRM = 5.778 RAM = 0.58619E+08 INLET BULK TEMPERATURE = 23.88 DEG C OUTLET BULK TEMPERATURE = 30.64 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	NUSSLELT NUMBER			AVERAGE		
		A	B	C	AVERAGE						T	H	T+H			
0	0.1	26.00	26.02	25.96	25.98	23.88	1392.1	6.30	0.488E+08	0.00000	45.44	44.88	46.25	45.78	45.70	45.74
1	1.5	26.43	26.47	26.43	26.43	23.91	1392.8	6.30	0.488E+08	0.00006	38.00	37.45	38.03	37.96	37.88	37.92
2	2.5	26.73	26.77	26.75	26.75	23.92	1393.2	6.29	0.489E+08	0.00009	34.15	33.61	33.88	33.96	33.88	33.92
3	5.5	27.63	27.69	27.72	27.68	23.96	1394.6	6.29	0.490E+08	0.00021	26.18	25.71	25.52	25.80	25.80	25.80
4	15.5	28.21	28.59	28.53	28.44	24.11	1399.2	6.26	0.494E+08	0.00058	23.40	21.37	21.69	22.12	22.15	22.14
5	25.5	28.76	29.07	29.32	29.05	24.25	1403.8	6.24	0.498E+08	0.00096	21.28	19.91	18.90	19.98	20.03	20.01
6	45.5	28.83	29.13	29.42	29.13	24.54	1413.1	6.20	0.506E+08	0.00171	22.31	20.85	19.64	20.88	20.94	20.91
7	75.5	29.12	29.52	29.74	29.46	24.87	1427.3	6.13	0.519E+08	0.00284	23.06	21.03	20.05	21.31	21.38	21.35
8	105.5	29.31	30.06	30.02	29.79	25.40	1441.8	6.06	0.531E+08	0.00397	24.51	20.55	20.70	21.78	21.92	21.85
9	135.5	29.96	30.52	30.90	30.46	25.84	1456.6	5.99	0.544E+08	0.00511	23.16	20.41	18.86	20.66	20.81	20.74
10	165.2	30.08	30.91	31.20	30.73	26.26	1471.6	5.92	0.557E+08	0.00624	25.02	20.56	19.33	21.38	21.64	21.51
11	205.2	30.37	31.15	31.33	30.95	26.84	1492.2	5.83	0.575E+08	0.00776	27.03	22.11	21.21	23.19	23.45	23.32
12	245.2	30.88	31.69	32.06	31.54	27.42	1509.8	5.76	0.591E+08	0.00928	27.50	22.28	20.52	23.08	23.43	23.26
13	275.2	31.29	32.16	32.52	31.99	27.85	1523.2	5.70	0.603E+08	0.01042	27.65	22.08	20.35	22.97	23.36	23.16
14	305.2	31.46	32.33	32.67	32.15	28.28	1536.8	5.65	0.615E+08	0.01156	29.88	23.48	21.63	24.53	25.00	24.76
15	333.3	32.36	33.39	33.73	33.16	28.69	1549.8	5.60	0.627E+08	0.01264	25.83	20.16	18.82	21.21	21.60	21.41
16	363.3	32.54	33.58	33.89	33.34	29.12	1563.9	5.54	0.640E+08	0.01378	27.72	21.24	19.88	22.48	22.95	22.71
17	383.3	32.92	33.77	34.07	33.59	29.41	1573.5	5.51	0.648E+08	0.01455	26.96	21.73	20.31	22.67	23.00	22.83
18	403.3	33.31	34.14	34.46	33.97	29.70	1583.2	5.47	0.657E+08	0.01532	26.23	21.28	19.89	22.16	22.47	22.31
19	423.3	33.59	34.50	34.79	34.29	29.98	1593.0	5.44	0.666E+08	0.01608	26.25	20.95	19.67	21.95	22.29	22.12
20	443.3	33.68	34.41	34.89	34.33	30.27	1602.9	5.40	0.674E+08	0.01685	27.76	22.84	20.46	23.31	23.69	23.50
21	463.3	34.17	34.54	35.37	34.70	30.56	1612.9	5.36	0.683E+08	0.01762	26.16	23.72	19.64	22.85	23.17	23.01
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 33.07 33.97 34.31 33.78 29.53 1577.7 5.49 0.652E+08 0.01487 26.79 21.37 19.84 22.30 22.67 22.48																

XP135-09

INPUT ELECTRIC POWER = 1115.0 W HEAT RATE GAINED BY WATER = 1081.9 W HEAT BALANCE ERROR = 2.97%  
 MASS FLOW RATE = 32.1500 G/S PRESSURE DROP = 0.8133 MM H2O FRICTION FACTOR = 0.022519 FREM = 27.0845  
 REM = 1202.7 GRM = 0.10003E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 32.00 DEG C  
 PRM = 5.689 RAM = 0.56907E+08 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 31.99 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	NUSSLELT NUMBER			AVERAGE		
		A	B	C	AVERAGE						T	H	T+H			
0	0.1	26.18	26.21	26.13	26.16	23.94	1098.0	6.29	0.459E+08	0.00000	40.17	39.70	41.10	40.64	40.52	40.58
1	1.5	26.63	26.66	26.63	26.63	23.97	1098.6	6.29	0.460E+08	0.00007	33.85	33.39	33.84	33.86	33.73	33.79
2	2.5	26.93	26.98	26.97	26.95	23.98	1099.1	6.28	0.461E+08	0.00012	30.54	30.09	30.16	30.36	30.24	30.30
3	5.5	27.85	27.92	27.99	27.92	24.04	1100.4	6.27	0.462E+08	0.00026	23.61	23.20	22.75	23.18	23.19	23.19
4	15.5	28.43	28.93	28.86	28.74	24.21	1104.7	6.25	0.466E+08	0.00074	21.33	19.05	19.34	19.86	19.91	19.88
5	25.5	28.98	29.34	29.66	29.33	24.38	1109.0	6.22	0.471E+08	0.00122	19.57	18.12	17.05	18.19	18.25	18.22
6	45.5	29.06	29.36	29.69	29.37	24.72	1117.9	6.17	0.480E+08	0.00217	20.75	15.41	18.09	19.35	19.42	19.38
7	75.5	29.29	29.74	29.97	29.67	25.24	1131.3	6.08	0.494E+08	0.00361	22.17	19.93	18.99	20.28	20.36	20.32
8	105.5	29.42	30.33	30.30	30.02	25.75	1145.2	6.00	0.508E+08	0.00505	24.49	19.59	19.73	21.04	21.27	21.15
9	135.5	30.02	30.63	31.12	30.59	26.27	1159.3	5.92	0.523E+08	0.00649	23.91	20.55	18.46	20.74	20.97	20.86
10	165.2	30.30	31.13	31.48	30.97	26.78	1173.7	5.84	0.538E+08	0.00793	25.42	20.58	19.05	21.36	21.68	21.52
11	205.2	30.76	31.59	31.78	31.38	27.47	1190.5	5.75	0.556E+08	0.00986	27.16	21.64	20.72	22.85	23.18	23.01
12	245.2	31.59	32.41	32.67	32.23	28.15	1207.4	5.67	0.574E+08	0.01179	25.92	20.95	19.74	21.90	22.21	22.06
13	275.2	32.01	32.93	33.31	32.75	28.67	1220.3	5.60	0.588E+08	0.01324	26.66	20.89	19.22	21.83	22.26	22.04
14	305.2	32.35	33.22	33.51	33.03	29.18	1233.6	5.54	0.602E+08	0.01470	28.11	22.07	20.57	23.17	23.58	23.37
15	333.3	33.36	34.29	34.57	34.07	29.67	1246.3	5.48	0.616E+0							

XP135-10

INPUT ELECTRIC POWER = 1065.0 W HEAT RATE GAINED BY WATER = 1033.5 W HEAT BALANCE ERROR = 2.95%  
 MASS FLOW RATE = 26.3740 G/S PRESSURE DROP = 0.6053 MM H2O FRICTION FACTOR = 0.024898 FREM = 24.8921  
 REM = 999.8 GRM = 0.99859E+07 UPSTREAM BULK TEMPERATURE = 23.91 DEG C DOWN STREAM BULK TEMPERATURE = 33.30 DEG C  
 FRM = 5.609 RAM = 0.56010E+08 INLET BULK TEMPERATURE = 23.92 DEG C OUTLET BULK TEMPERATURE = 33.29 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVERAGE						A	B	C	T	H	T+H
0	0.1	26.16	26.19	26.17	26.16	23.92	900.2	6.29	0.438E+08	0.00000	38.31	37.86	38.22	38.34	38.16	38.25
1	1.5	26.60	26.63	26.65	26.62	23.95	900.8	6.29	0.439E+08	0.00005	32.45	32.01	31.77	32.18	32.00	32.09
2	2.5	26.90	26.94	26.99	26.94	23.97	901.2	6.29	0.439E+08	0.00015	29.36	28.92	28.46	28.97	28.80	28.89
3	5.5	27.79	27.86	27.99	27.88	24.03	902.5	6.28	0.441E+08	0.00032	22.83	22.43	21.68	22.30	22.31	22.30
4	15.5	28.43	28.82	28.75	28.67	24.23	906.6	6.24	0.446E+08	0.00090	20.46	18.71	19.01	19.36	19.39	19.38
5	25.5	28.87	29.23	29.60	29.23	24.43	910.8	6.21	0.451E+08	0.00148	19.35	17.88	16.61	17.88	17.95	17.91
6	45.5	28.94	29.36	29.64	29.31	24.83	919.2	6.15	0.461E+08	0.00265	20.85	18.96	17.84	19.14	19.22	19.18
7	75.5	29.18	29.74	29.86	29.59	25.43	932.2	6.06	0.477E+08	0.00440	22.86	19.86	19.36	20.58	20.69	20.63
8	105.5	29.36	30.22	30.24	29.94	26.03	945.5	5.96	0.493E+08	0.00616	25.68	20.41	20.30	21.86	22.13	22.00
9	135.5	30.13	30.74	31.12	30.66	26.63	959.2	5.87	0.509E+08	0.00792	24.42	20.79	19.02	21.18	21.41	21.30
10	165.2	30.52	31.41	31.71	31.21	27.22	971.7	5.78	0.525E+08	0.00967	25.86	20.40	19.04	21.40	21.77	21.58
11	205.2	31.09	31.98	32.11	31.73	28.02	987.7	5.68	0.545E+08	0.01203	27.78	21.51	20.84	22.99	23.38	23.18
12	245.2	31.98	32.80	33.06	32.61	28.82	1004.3	5.58	0.566E+08	0.01439	26.90	21.38	20.07	22.43	22.78	22.60
13	275.2	32.57	33.38	33.69	33.21	29.42	1017.1	5.51	0.582E+08	0.01617	27.01	21.47	19.88	22.41	22.79	22.60
14	305.2	33.18	33.99	34.23	33.80	30.02	1030.2	5.43	0.598E+08	0.01795	26.83	21.36	20.14	22.43	22.78	22.60
15	333.3	33.69	34.57	34.90	34.39	30.58	1042.8	5.36	0.614E+08	0.01962	27.29	21.27	19.63	22.29	22.73	22.51
16	363.3	34.26	35.19	35.45	34.97	31.18	1056.6	5.28	0.631E+08	0.02141	27.55	21.10	19.84	22.37	22.83	22.60
17	383.3	34.70	35.50	35.73	35.31	31.58	1066.0	5.23	0.642E+08	0.02260	27.13	21.59	20.37	22.68	23.03	22.86
18	403.3	35.14	35.93	36.13	35.74	31.98	1075.1	5.18	0.653E+08	0.02380	26.72	21.39	20.38	22.52	22.83	22.67
19	423.3	35.58	36.34	36.69	36.20	32.38	1083.5	5.14	0.664E+08	0.02501	26.37	21.34	19.60	22.09	22.44	22.26
20	443.3	35.79	36.31	36.85	36.33	32.78	1091.9	5.09	0.674E+08	0.02623	28.00	23.90	20.51	23.75	24.13	23.94
21	463.3	36.34	36.51	37.49	36.78	33.18	1100.6	5.05	0.684E+08	0.02745	26.64	25.35	19.56	23.42	23.85	23.64
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 34.86 35.64 35.97 35.49 31.75 1069.3 5.22 0.646E+08 0.02311 27.18 21.76 20.06 22.62 23.00 22.81																

XP135-11

INPUT ELECTRIC POWER = 995.0 W HEAT RATE GAINED BY WATER = 961.2 W HEAT BALANCE ERROR = 3.39%  
 MASS FLOW RATE = 20.5980 G/S PRESSURE DROP = 0.4808 MM H2O FRICTION FACTOR = 0.032412 FREM = 25.8054  
 REM = 796.2 GRM = 0.99011E+07 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 35.12 DEG C  
 PRM = 5.492 RAM = 0.54380E+08 INLET BULK TEMPERATURE = 23.95 DEG C OUTLET BULK TEMPERATURE = 35.11 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVERAGE						A	B	C	T	H	T+H
0	0.1	26.20	26.24	26.22	26.21	23.95	703.6	6.29	0.408E+08	0.00000	35.58	34.83	35.16	35.38	35.18	35.28
1	1.5	26.62	26.69	26.71	26.66	23.98	704.1	6.28	0.409E+08	0.00011	30.31	29.55	29.34	29.83	29.64	29.73
2	2.5	26.91	27.00	27.04	26.98	24.01	704.5	6.28	0.410E+08	0.00019	27.50	26.75	26.33	26.92	26.73	26.82
3	5.5	27.79	27.92	28.05	27.92	24.08	705.6	6.27	0.411E+08	0.00041	21.52	20.84	20.14	20.82	20.83	20.82
4	15.5	28.32	28.82	28.81	28.65	24.32	709.5	6.23	0.417E+08	0.00115	19.98	17.74	17.81	18.45	18.51	18.48
5	25.5	28.87	29.18	29.49	29.18	24.55	713.4	6.19	0.423E+08	0.00190	18.53	17.28	16.19	17.28	17.33	17.31
6	45.5	28.89	29.36	29.64	29.29	25.03	721.3	6.12	0.434E+08	0.00339	20.69	18.45	17.32	18.72	18.82	18.77
7	75.5	29.12	29.69	29.91	29.57	25.74	733.5	6.00	0.451E+08	0.00564	23.60	20.21	19.13	20.81	20.98	20.89
8	105.5	29.53	30.33	30.36	30.07	26.46	746.2	5.89	0.469E+08	0.00789	25.93	20.54	20.42	22.02	22.30	22.16
9	135.5	30.35	31.07	31.46	30.96	27.17	758.2	5.79	0.487E+08	0.01015	25.01	20.38	18.55	20.95	21.31	21.15
10	165.2	31.02	31.79	32.15	31.65	27.88	769.2	5.70	0.504E+08	0.01239	25.30	20.27	18.58	21.03	21.38	21.20
11	205.2	31.70	32.59	32.73	32.34	28.83	784.6	5.58	0.527E+08	0.01542	27.62	21.05	20.33	22.57	23.00	22.78
12	245.2	32.82	33.58	33.84	33.41	29.79	800.6	5.46	0.550E+08	0.01845	26.07	20.83	19.49	21.79	22.13	21.96
13	275.2	33.40	34.27	34.58	34.08	30.50	813.0	5.37	0.569E+08	0.02074	27.21	20.94	19.32	22.01	22.49	22.25
14	305.2	34.02	34.78	35.07	34.62	31.22	825.8	5.28	0.587E+08	0.02303	28.11	22.10	20.45	23.13	23.55	23.34
15	333.3	35.02	35.86	36.13	35.67	31.89	838.1	5.20	0.605E+08	0.02518	25.06	19.80	18.53	20.78	21.13	20.95
16	363.3	35.58	36.42	36.62	36.21	32.60	849.8	5.11	0.622E+08	0.02751	26.30	20.56	19.51	21.75	22.12	21.94
17	383.3	36.03	36.73	37.07	36.61	33.08	857.7	5.06	0.634E+08	0.02906	26.51	21.46	19.64	22.18	22.54	22.36
18	403.3	36.54	37.33	37.52	37.13	33.55	865.9	5.00	0.645E+08	0.03063	26.22	20.74	19.75	21.90	22.23	22.06
19	423.3	37.08	37.89	38.14	37.71	34.03	874.2	4.95	0.657E+08	0.03220	25.60	20.23	19.02	21.27	21.62	21.44
20	443.3	37.36	37.88	38.45	37.89	34.51	882.6	4.89	0.669E+08	0.03377	27.40	23.16	19.80	23.04	23.45	23.25
21	463.3	38.02	38.16	39.04	38.41	34.98	891.2	4.84	0.681E+08	0.03536	25.70	24.58	19.20	22.78	23.16	22.97
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 36.27 37.02 37.32 36.87 33.28 861.4 5.03 0.639E+08 0.02972 26.18 20.99 19.37 21.82 22.18 22.00																

XP135-12

INPUT ELECTRIC POWER = 850.0 W HEAT RATE GAINED BY WATER = 816.1 W HEAT BALANCE ERROR = 3.99%  
 MASS FLOW RATE = 12.2950 G/S PRESSURE DROP = 0.2779 MM H2O FRICTION FACTOR = 0.052531 FREM = 26.2546  
 REM = 499.8 GRM = 0.98638E+07 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 39.79 DEG C  
 PRM = 5.201 RAM = 0.51303E+08 INLET BULK TEMPERATURE = 23.90 DEG C OUTLET BULK TEMPERATURE = 39.78 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVERAGE						A	B	C	T	H	T+H
0	0.1	26.05	26.15	26.15	26.11	23.90	419.5	6.30	0.346E+08	0.00001	31.54	30.25	30.19	30.67	30.55	30.61
1	1.5	26.44	26.57	26.58	26.53	23.95	420.0	6.29	0.347E+08	0.00019	27.24	25.89	25.79	26.30	26.18	26.24
2	2.5	26.71	26.87	26.88	26.82	23.98	420.3	6.28	0.347E+08	0.00031	24.89	23.55	23.43	23.95	23.83	23.89
3	5.5	27.52	27.75	27.78	27.68	24.08	421.3	6.27	0.349E+08	0.00069	19.78	18.53	18.39	18.88	18.90	18.89
4	15.5	28.09	28.54	28.53	28.39	24.42	424.5	6.21	0.356E+08	0.00193	18.48	16.49	16.53	17.12	17.17	17.14
5	25.5	28.59	29.01	29.32	28.97	24.76	427.9	6.16	0.363E+08	0.00318	17.72	15.96	14.87	16.10	16.18	16.14
6	45.5	28.72	29.19	29.53	29.15	25.44	434.7	6.05	0.377E+08	0.00569	20.62	18.06	16.56	18.26	18.41	18.34
7	75.5	29.34	29.97	30.02	29.78	26.46	445.3	5.89	0.398E+08	0.00946	23.39	19.24	18.94	20.33	20.52	20.43
8	105.5	30.08	30.95	30.91	30.65	27.47	455.3	5.75	0.419E+08	0.01325	25.81	19.38	19.60	21.22	21.60	21.41
9	135.5	31.18	31.96	32.23	31.79	28.49	464.9	5.62	0.440E+08	0.01705	24.96	19.37	17.96	20.35	20.76	20.56
10	165.2	32.18	32.96	33.21	32.78	29.50	474.9	5.50	0.461E+08	0.02082	24.96	19.36	18.08	20.40	20.80	20.60
11	205.2	33.31	34.09	34.17	33.86	30.85	489.0	5.33	0.491E+08	0.02592	27.19	20.62	20.14	22.23	22.65	22.44
12	245.2	34.81	35.52	35.68	35.34	32.21	503.4	5.16	0.520E+08	0.03106	25.60	20.11	19.20	21.30	21.64	21.47
13	275.2	35.73	36.44	36.69	36.29	33.22	513.5	5.04	0.541E+08	0.03498	26.55	20.68	19.15	21.70	22.13	21.91
14	305.2	36.63	37.28	37.58	37.16	34.24	524.0	4.92	0.562E+08	0.03892	27.77	21.79	19.87	22.69	23.14	22.91
15	333.3	37.86	38.65	38.93	38.48	35.										

XP135-13

INPUT ELECTRIC POWER = 1950.0 W HEAT RATE GAINED BY WATER = 1875.8 W HEAT BALANCE ERROR = 3.80%  
 MASS FLOW RATE = 38.6480 G/S PRESSURE DROP = 0.9514 MM H2O FRICTION FACTOR = 0.018217 FREM = 27.3095  
 REM = 1499.2 GRM = 0.19544E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 35.51 DEG C  
 PRM = 5.471 RAM = 0.10694E+09 INLET BULK TEMPERATURE = 23.89 DEG C OUTLET BULK TEMPERATURE = 35.50 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSSLELT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	27.22	27.23	27.16	27.18	23.89	1318.4	6.30	0.794E+08	0.00000	46.92	46.76	47.81	47.50	47.33	47.41
1	1.5	27.90	27.92	27.91	27.89	23.93	1319.4	6.29	0.796E+08	0.00006	39.27	39.12	39.20	39.36	39.19	39.28
2	2.5	28.37	28.39	28.43	28.39	23.95	1320.2	6.29	0.797E+08	0.00010	35.30	35.15	34.86	35.21	35.04	35.13
3	5.5	29.79	29.82	29.99	29.86	24.02	1322.4	6.28	0.800E+08	0.00022	27.09	26.96	26.18	26.73	26.74	26.74
4	15.5	30.48	31.18	31.09	30.92	24.27	1329.9	6.24	0.812E+08	0.00051	25.15	22.96	22.89	23.49	23.54	23.51
5	25.5	31.27	31.73	32.37	31.79	24.52	1337.5	6.20	0.823E+08	0.00101	23.12	21.64	19.86	21.46	21.54	21.50
6	45.5	31.32	31.86	32.54	31.91	25.02	1352.9	6.12	0.846E+08	0.00181	24.70	22.78	20.71	22.61	22.73	22.67
7	75.5	31.89	32.63	32.98	32.50	25.76	1376.8	6.00	0.882E+08	0.00301	25.37	22.64	21.54	23.08	23.16	23.13
8	105.5	32.08	33.40	33.53	33.00	26.50	1401.5	5.88	0.918E+08	0.00421	27.82	22.51	22.10	23.88	24.14	24.01
9	135.5	33.07	34.02	34.73	33.94	27.25	1424.7	5.78	0.954E+08	0.00541	26.60	22.89	20.71	23.16	23.40	23.28
10	165.2	33.41	34.74	35.32	34.49	27.98	1446.2	5.69	0.987E+08	0.00661	28.53	22.88	21.08	23.77	24.16	23.97
11	205.2	33.87	35.26	35.67	34.93	28.97	1476.4	5.56	0.103E+09	0.00822	31.54	24.55	23.04	25.90	26.38	26.14
12	245.2	34.87	36.19	36.79	35.95	29.96	1507.8	5.44	0.108E+09	0.00984	31.42	24.72	22.55	25.72	26.23	25.98
13	275.2	35.56	36.99	37.64	36.73	30.71	1532.2	5.34	0.112E+09	0.01106	31.68	24.46	22.18	25.53	26.11	25.82
14	305.2	35.91	37.34	37.86	37.03	31.45	1557.5	5.25	0.116E+09	0.01228	34.46	26.07	23.97	27.50	28.17	27.83
15	333.3	37.24	38.82	39.37	38.48	32.15	1580.4	5.17	0.119E+09	0.01343	30.06	22.98	21.21	24.21	24.75	24.48
16	363.3	37.58	39.19	39.74	38.84	32.89	1603.5	5.08	0.123E+09	0.01467	32.63	24.26	22.32	25.71	26.40	26.06
17	383.3	38.19	39.52	40.12	39.28	33.39	1619.2	5.02	0.125E+09	0.01551	31.77	24.89	22.70	25.93	26.45	26.19
18	403.3	38.77	40.17	40.64	39.86	33.88	1635.3	4.96	0.127E+09	0.01634	31.23	24.25	22.58	25.52	26.02	25.77
19	423.3	39.30	40.84	41.48	40.54	34.38	1651.7	4.91	0.130E+09	0.01718	30.95	23.56	21.44	24.71	25.32	25.02
20	443.3	39.64	40.79	41.68	40.70	34.87	1668.4	4.85	0.132E+09	0.01802	31.93	25.72	22.36	26.11	26.67	26.39
21	463.3	40.41	41.09	42.39	41.30	35.37	1685.5	4.79	0.135E+09	0.01887	30.14	26.54	21.65	25.63	26.11	25.87
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 38.45 39.89 40.51 39.62 33.59 1626.4 5.00 0.126E+09 0.01586 31.43 24.28 22.10 25.37 25.94 25.65																

XP135-14

INPUT ELECTRIC POWER = 1810.0 W HEAT RATE GAINED BY WATER = 1738.6 W HEAT BALANCE ERROR = 3.94%  
 MASS FLOW RATE = 29.9840 G/S PRESSURE DROP = 0.7519 MM H2O FRICTION FACTOR = 0.023908 FREM = 28.4965  
 REM = 1191.9 GRM = 0.19594E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 37.78 DEG C  
 PRM = 5.328 RAM = 0.10440E+09 INLET BULK TEMPERATURE = 23.89 DEG C OUTLET BULK TEMPERATURE = 37.77 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSSLELT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	27.29	27.36	27.31	27.30	23.89	1022.9	6.30	0.736E+08	0.00000	42.67	41.72	42.35	42.53	42.27	42.40
1	1.5	27.95	28.06	28.08	28.01	23.94	1023.9	6.29	0.738E+08	0.00008	36.04	35.10	34.91	35.49	35.24	35.36
2	2.5	28.41	28.54	28.61	28.51	23.97	1024.6	6.29	0.739E+08	0.00013	32.56	31.63	31.13	31.85	31.61	31.73
3	5.5	29.79	29.98	30.21	29.99	24.05	1026.7	6.27	0.743E+08	0.00028	25.23	24.41	23.50	24.36	24.38	24.37
4	15.5	30.53	31.29	31.31	31.05	24.35	1033.6	6.22	0.756E+08	0.00079	23.39	20.82	20.77	21.60	21.66	21.63
5	25.5	31.33	31.84	32.48	31.88	24.65	1040.7	6.18	0.768E+08	0.00130	21.63	20.09	18.44	19.97	20.05	20.01
6	45.5	31.32	31.91	32.54	31.92	25.24	1055.1	6.08	0.794E+08	0.00233	23.72	21.63	19.77	21.59	21.71	21.65
7	75.5	31.78	32.74	33.14	32.55	26.13	1077.5	5.94	0.834E+08	0.00388	25.48	21.78	20.52	22.41	22.60	22.50
8	105.5	32.25	33.62	33.69	33.19	27.01	1100.1	5.81	0.874E+08	0.00543	27.45	21.75	21.52	23.28	23.57	23.43
9	135.5	33.29	34.35	35.06	34.24	27.90	1120.2	5.70	0.912E+08	0.00698	26.60	22.25	20.04	22.65	22.96	22.81
10	165.2	33.85	35.13	35.66	34.88	28.78	1140.8	5.59	0.950E+08	0.00853	28.29	22.54	20.83	23.48	23.87	23.67
11	205.2	34.48	35.93	36.17	35.53	29.97	1169.8	5.44	0.100E+09	0.01061	31.64	23.95	23.00	25.68	26.20	25.94
12	245.2	35.81	37.14	37.63	36.86	31.15	1200.3	5.29	0.106E+09	0.01271	30.58	23.78	21.96	24.94	25.44	25.19
13	275.2	36.51	37.94	38.53	37.66	32.04	1223.5	5.18	0.110E+09	0.01429	31.81	24.08	21.90	25.29	25.93	25.61
14	305.2	37.18	38.51	39.03	38.24	32.93	1244.9	5.08	0.114E+09	0.01589	33.30	25.41	23.23	26.69	27.32	27.00
15	333.3	38.41	39.99	40.55	39.65	33.76	1265.6	4.98	0.118E+09	0.01740	30.40	22.68	20.83	24.00	24.64	24.32
16	363.3	38.97	40.53	41.03	40.17	34.65	1288.4	4.88	0.122E+09	0.01902	32.67	24.00	22.12	25.53	26.26	25.90
17	383.3	39.69	40.92	41.50	40.70	35.24	1304.2	4.81	0.124E+09	0.02011	31.63	24.82	22.51	25.79	26.32	26.05
18	403.3	40.38	41.73	42.08	41.40	35.83	1320.2	4.74	0.127E+09	0.02121	30.91	23.84	22.50	25.26	25.75	25.51
19	423.3	40.97	42.52	43.04	42.18	36.42	1336.7	4.67	0.130E+09	0.02231	30.92	23.06	21.22	24.42	25.06	24.74
20	443.3	41.37	42.41	43.41	42.39	37.01	1353.0	4.60	0.133E+09	0.02340	32.24	26.00	21.95	26.08	26.73	26.40
21	463.3	42.25	42.75	44.23	43.08	37.61	1368.0	4.55	0.136E+09	0.02448	30.18	27.25	21.17	25.62	26.20	25.91
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 39.96 41.35 41.94 41.08 35.49 1311.4 4.78 0.126E+09 0.02058 31.46 24.07 21.85 25.18 25.79 25.49																

XP135-15

INPUT ELECTRIC POWER = 1695.0 W HEAT RATE GAINED BY WATER = 1619.7 W HEAT BALANCE ERROR = 4.44%  
 MASS FLOW RATE = 24.5690 G/S PRESSURE DROP = 0.5746 MM H2O FRICTION FACTOR = 0.027200 FREM = 27.1804  
 REM = 999.3 GRM = 0.19612E+08 UPSTREAM BULK TEMPERATURE = 23.96 DEG C DOWN STREAM BULK TEMPERATURE = 39.77 DEG C  
 PRM = 5.198 RAM = 0.10194E+09 INLET BULK TEMPERATURE = 23.98 DEG C OUTLET BULK TEMPERATURE = 39.76 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	NUSSLELT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	27.49	27.56	27.59	27.53	23.98	839.8	6.28	0.689E+08	0.00000	38.45	37.63	37.38	38.00	37.71	37.86
1	1.5	28.14	28.25	28.36	28.24	24.03	840.7	6.28	0.691E+08	0.00009	32.76	31.93	31.14	32.03	31.74	31.89
2	2.5	28.60	28.73	28.89	28.73	24.06	841.4	6.27	0.693E+08	0.00016	29.73	28.90	27.93	28.90	28.62	28.76
3	5.5	29.96	30.15	30.48	30.20	24.16	843.3	6.25	0.697E+08	0.00034	23.27	22.51	21.32	22.34	22.37	22.35
4	15.5	30.70	31.47	31.42	31.20	24.50	849.9	6.20	0.710E+08	0.00097	21.72	19.33	19.45	20.11	20.17	20.14
5	25.5	31.38	31.95	32.59	31.98	24.84	856.6	6.15	0.723E+08	0.00159	20.55	18.92	17.34	18.85	18.94	18.89
6	45.5	31.38	32.08	32.59	32.02	25.51	870.1	6.04	0.751E+08	0.00285	22.90	20.46	18.96	20.65	20.77	20.71
7	75.5	31.72	32.63	32.98	32.44	26.52	891.3	5.88	0.794E+08	0.00474	25.76	21.94	20.75	22.63	22.82	22.73
8	105.5	32.19	33.57	33.69	33.15	27.53	910.9	5.74	0.835E+08	0.00663	28.67	22.15	21.69	23.78	24.17	23.98
9	135.5	33.41	34.52	35.12	34.35	28.54	930.1	5.62	0.875E+08	0.00853	27.42	22.32	20.28	22.98	23.34	23.16
10	165.2	34.18	35.52	35.93	35.21	29.54	949.8	5.49	0.917E+08	0.01042	28.70	22.25	20.82	23.47	23.92	23.69
11	205.2	35.09	36.48	36.67	36.08	30.88	977.8	5.32	0.975E+08	0.01297	31.57	23.71	22.93	25.54	26.07	25.80
12	245.2	36.58	37.92	38.36	37.62	32.23	1006.3	5.16	0.103E+09	0.01555	30.40	23.27	21.60	24.56	25.09	24.82
13	275.2	37.45	38.83	39.36	38.55	33.24	1026.4	5.04	0.107E+09	0.01750	31.35	23.62	21.56	24.87	25.51	25.19
14	305.2	38.23	39.56	40.14	39.31	34.25	1047.3	4.92	0.112E+09	0.01948	33.05	24.78	22.35	26.01	26.73	26.37
15	333.3	39.57	41.00	41.61	40											

XP135-16

INPUT ELECTRIC POWER = 1550.0 W HEAT RATE GAINED BY WATER = 1503.3 W HEAT BALANCE ERROR = 3.01%  
 MASS FLOW RATE = 19.1540 G/S PRESSURE DROP = 0.4399 MM H2O FRICTION FACTOR = 0.034242 FREM = 27.4572  
 REM = 801.9 GRM = 0.19919E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 42.75 DEG C  
 PRM = 5.026 RAM = 0.10012E+09 INLET BULK TEMPERATURE = 23.95 DEG C OUTLET BULK TEMPERATURE = 42.74 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	27.35	27.52	27.51	27.45	23.96	654.4	6.29	0.639E+08	0.00000	36.81	35.12	35.18	35.21	35.57	35.69
1	1.5	27.96	28.19	28.23	28.12	24.01	655.2	6.28	0.641E+08	0.00012	31.72	29.96	29.67	30.49	30.26	30.37
2	2.5	28.37	28.65	28.73	28.58	24.05	655.8	6.27	0.642E+08	0.00020	28.96	27.21	26.78	27.66	27.43	27.54
3	5.5	29.62	30.04	30.21	29.96	24.17	657.6	6.25	0.647E+08	0.00044	22.96	21.33	20.72	21.63	21.67	21.65
4	15.5	30.37	31.18	31.20	30.92	24.57	663.7	6.19	0.662E+08	0.00124	21.58	18.91	18.86	19.70	19.78	19.74
5	25.5	31.10	31.73	32.26	31.70	24.98	669.9	6.13	0.677E+08	0.00204	20.38	18.49	17.14	18.58	18.67	18.62
6	45.5	31.16	31.86	32.37	31.80	25.78	682.6	6.00	0.707E+08	0.00365	23.15	20.50	18.90	20.70	20.85	20.77
7	75.5	31.61	32.57	32.92	32.37	26.98	702.3	5.81	0.755E+08	0.00608	26.83	22.22	20.91	23.06	23.32	23.19
8	105.5	32.42	33.84	33.86	33.37	28.18	719.7	5.66	0.799E+08	0.00852	29.26	21.88	21.82	23.87	24.32	24.09
9	135.5	33.85	35.07	35.51	34.81	29.38	738.0	5.51	0.845E+08	0.01096	27.67	21.73	20.19	22.78	23.20	22.99
10	165.2	34.79	36.08	36.49	35.79	30.57	757.1	5.36	0.892E+08	0.01339	29.24	22.39	20.84	23.65	24.16	23.90
11	205.2	35.03	37.32	37.57	36.97	32.18	783.7	5.16	0.957E+08	0.01669	31.84	23.89	22.78	25.61	26.17	25.89
12	245.2	37.86	39.08	39.42	38.79	33.78	808.8	4.98	0.102E+09	0.02004	29.96	23.06	21.69	24.42	24.90	24.66
13	275.2	38.89	40.22	40.69	39.94	34.98	828.7	4.84	0.106E+09	0.02258	31.16	23.26	21.34	24.60	25.25	24.93
14	305.2	39.90	41.07	41.53	40.83	36.18	849.6	4.70	0.111E+09	0.02516	32.71	24.89	22.72	26.14	26.77	26.46
15	333.3	41.35	42.62	43.23	42.40	37.31	869.0	4.58	0.116E+09	0.02756	30.01	22.82	20.47	23.81	24.43	24.12
16	363.3	42.29	43.69	44.15	43.38	38.51	888.9	4.47	0.121E+09	0.03010	32.02	23.34	21.45	24.86	25.61	25.23
17	383.3	43.25	44.44	44.88	44.19	39.31	902.6	4.39	0.125E+09	0.03179	30.68	23.56	21.68	24.76	25.31	25.03
18	403.3	44.07	45.30	45.59	44.99	40.11	916.8	4.32	0.128E+09	0.03350	30.51	23.26	22.01	24.75	25.26	25.00
19	423.3	44.96	46.24	46.72	45.97	40.92	931.4	4.25	0.132E+09	0.03521	29.81	22.60	20.74	23.81	24.38	24.09
20	443.3	45.55	46.38	47.29	46.41	41.72	946.5	4.17	0.135E+09	0.03692	31.37	25.77	21.56	25.62	26.23	25.93
21	463.3	46.48	46.92	48.18	47.19	42.52	960.1	4.11	0.139E+09	0.03864	30.28	27.26	21.21	25.67	26.25	25.96
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 43.58 44.78 45.31 44.56 39.65 909.2 4.36 0.126E+09 0.03251 30.73 23.56 21.32 24.60 25.20 24.90																

XP135-17

INPUT ELECTRIC POWER = 1240.0 W HEAT RATE GAINED BY WATER = 1210.9 W HEAT BALANCE ERROR = 2.35%  
 MASS FLOW RATE = 11.2120 G/S PRESSURE DROP = 0.2626 MM H2O FRICTION FACTOR = 0.059571 FREM = 30.0536  
 REM = 504.5 GRM = 0.19914E+08 UPSTREAM BULK TEMPERATURE = 23.91 DEG C DOWN STREAM BULK TEMPERATURE = 49.80 DEG C  
 PRM = 4.619 RAM = 0.91983E+08 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 49.79 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	26.99	27.13	27.22	27.11	23.94	382.9	6.29	0.514E+08	0.00001	33.03	31.58	30.76	31.77	31.53	31.65
1	1.5	27.51	27.71	27.84	27.69	24.02	383.6	6.28	0.516E+08	0.00020	28.84	27.30	26.40	27.48	27.24	27.36
2	2.5	27.87	28.11	28.26	28.08	24.07	384.1	6.27	0.518E+08	0.00034	26.52	24.96	24.06	25.14	24.90	25.02
3	5.5	28.96	29.31	29.54	29.27	24.24	385.5	6.24	0.523E+08	0.00075	21.36	19.86	18.99	20.02	20.07	20.05
4	15.5	29.71	30.39	30.53	30.21	24.79	390.4	6.16	0.539E+08	0.00212	20.47	17.95	17.52	18.56	18.65	18.60
5	25.5	30.43	31.12	31.66	31.07	25.34	395.5	6.07	0.556E+08	0.00350	19.74	17.38	15.92	17.54	17.68	17.61
6	45.5	30.72	31.41	31.93	31.35	26.44	406.0	5.89	0.591E+08	0.00625	23.46	20.18	18.28	20.43	20.64	20.53
7	75.5	31.83	32.85	33.09	32.59	28.10	420.6	5.67	0.641E+08	0.01041	26.74	21.02	20.01	22.23	22.59	22.41
8	105.5	33.31	34.57	34.53	34.14	29.75	435.4	5.46	0.692E+08	0.01459	27.96	20.67	20.83	22.70	23.15	22.93
9	135.5	35.07	36.13	36.51	35.90	31.41	451.4	5.26	0.746E+08	0.01879	27.04	21.00	19.44	22.05	22.49	22.27
10	165.2	36.51	37.69	37.93	37.38	33.05	466.6	5.06	0.797E+08	0.02301	28.48	21.25	20.19	22.78	23.31	23.04
11	205.2	38.26	39.43	39.57	39.08	35.25	487.8	4.81	0.866E+08	0.02880	32.66	23.50	22.74	25.61	26.30	25.95
12	245.2	40.63	41.75	41.98	41.45	37.46	510.1	4.56	0.939E+08	0.03464	30.78	22.74	21.59	24.43	25.04	24.74
13	275.2	42.17	43.34	43.75	43.09	39.11	526.3	4.41	0.996E+08	0.03898	31.79	23.02	20.98	24.47	25.26	24.87
14	305.2	43.62	44.63	45.10	44.45	40.76	543.6	4.26	0.106E+09	0.04336	33.95	25.08	22.38	26.31	27.14	26.72
15	333.3	45.57	46.76	47.32	46.55	42.31	560.0	4.12	0.111E+09	0.04748	29.74	21.75	19.34	22.85	23.61	23.23
16	363.3	46.99	48.30	48.61	47.97	43.97	576.5	3.99	0.117E+09	0.05190	31.87	22.26	20.77	24.11	24.97	24.54
17	383.3	48.13	49.24	49.71	49.03	45.07	588.0	3.91	0.121E+09	0.05486	31.42	23.09	20.76	24.33	25.09	24.71
18	403.3	49.19	50.32	50.55	50.02	46.17	600.0	3.82	0.125E+09	0.05784	31.79	23.15	21.94	24.95	25.63	25.29
19	423.3	50.28	51.53	51.86	51.22	47.28	612.0	3.74	0.129E+09	0.06083	31.88	22.55	20.93	24.29	25.12	24.71
20	443.3	51.18	51.64	52.75	51.86	48.38	623.8	3.66	0.134E+09	0.06382	34.19	29.36	21.89	27.53	28.48	28.00
21	463.3	52.33	52.30	53.75	52.79	49.48	636.0	3.59	0.138E+09	0.06683	33.57	33.90	22.38	28.85	29.95	29.40
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6 48.56 49.63 50.13 49.44 45.53 593.4 3.88 0.123E+09 0.05612 31.82 23.69 20.94 24.68 25.48 25.08																

XP135-18

INPUT ELECTRIC POWER = 3380.0 W HEAT RATE GAINED BY WATER = 3248.4 W HEAT BALANCE ERROR = 3.89%  
 MASS FLOW RATE = 34.3160 G/S PRESSURE DROP = 0.9719 MM H2O FRICTION FACTOR = 0.023551 FREM = 35.1835  
 REM = 1493.9 GRM = 0.48463E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 46.62 DEG C  
 PRM = 4.802 RAM = 0.23271E+09 INLET BULK TEMPERATURE = 23.95 DEG C OUTLET BULK TEMPERATURE = 46.61 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	29.47	29.62	29.64	29.54	23.96	1172.4	6.29	0.138E+09	0.00000	49.06	47.73	47.58	48.41	47.99	48.20
1	1.5	30.57	30.79	30.95	30.75	24.03	1174.2	6.28	0.139E+09	0.00007	41.30	39.98	39.03	40.24	39.84	40.04
2	2.5	31.33	31.59	31.86	31.58	24.07	1175.5	6.27	0.139E+09	0.00011	37.24	35.96	34.73	36.04	35.66	35.85
3	5.5	33.62	34.01	34.58	34.07	24.22	1179.4	6.25	0.140E+09	0.00025	28.76	27.62	26.09	27.45	27.49	27.47
4	15.5	34.41	35.86	35.93	35.40	24.70	1192.6	6.17	0.144E+09	0.00069	27.81	24.21	24.05	25.24	25.35	25.30
5	25.5	35.68	36.78	37.92	36.79	25.19	1206.1	6.09	0.148E+09	0.00114	25.69	23.25	21.18	23.23	23.37	23.30
6	45.5	35.64	36.75	37.89	36.76	26.15	1234.0	5.94	0.156E+09	0.00204	28.35	25.39	22.93	25.36	25.55	25.46
7	75.5	36.27	37.89	38.61	37.59	27.60	1274.2	5.74	0.168E+09	0.00340	30.95	26.06	24.37	26.85	27.13	26.99
8	105.5	36.87	39.25	39.48	38.53	29.05	1313.1	5.55	0.180E+09	0.00476	34.21	26.21	25.64	28.20	28.69	28.44
9	135.5	38.41	40.41	41.66	40.16	30.50	1354.4	5.37	0.192E+09	0.00613	33.71	26.90	23.87	27.59	28.16	27.88
10	165.2	39.22	41.74	42.83	41.27	31.94	1397.6	5.19	0.205E+09	0.00749	36.43	27.08	24.37	28.46	29.29	28.88
11	205.2	40.59	42.98	43.68	42.42	33.87	1451.7	4.97	0.221E+09	0.00936	39.33	29.00	26.93	30.91	31.75	31.33
12	245.2	42.73	45.25	46.22	44.73	35.80	1510.2	4.74	0.237E+09	0.01226	37.94	27.83	25.25	29.45	30.34	29.89
13	275.2	44.01	46.84	47.92	46.26	37.25	1555.4	4.58	0.250E+09	0.01270	38.81	27.32	24.56	29.10	30.23	29.67
14	305.2	45.12	47.64	48.56	47.10	38.70	1598.4	4.45	0.263E+09	0.01412	40.75	29.24	26.52	31.11	32.17	31.64
15	333.3	46.96	49.72	51.06	49.25	40.06	1640									

XP135-19

INPUT ELECTRIC POWER = 3045.0 W HEAT RATE GAINED BY WATER = 2910.4 W HEAT BALANCE ERROR = 4.42%  
 MASS FLOW RATE = 26.3740 G/S PRESSURE DROP = 0.7485 MM H2O FRICTION FACTOR = 0.030683 FREM = 36.5987  
 REM = 1192.8 GRM = 0.48601E+08 UPSTREAM BULK TEMPERATURE = 23.91 DEG C DOWN STREAM BULK TEMPERATURE = 50.36 DEG C  
 PRM = 4.594 RAM = 0.22325E+09 INLET BULK TEMPERATURE = 23.93 DEG C OUTLET BULK TEMPERATURE = 50.34 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----															
		A	B	C	AVERAGE						A	B	C	T	H	T+H													
0	0.1	23.34	29.56	29.59	29.48	23.93	900.6	6.29	0.124E+09	0.00000	44.84	43.09	42.84	43.71	43.40	43.56													
1	1.5	30.39	30.70	30.81	30.62	24.02	902.2	6.28	0.124E+09	0.00009	38.02	36.26	35.63	36.68	36.38	36.53													
2	2.5	31.11	31.48	31.66	31.41	24.07	903.4	6.27	0.125E+09	0.00015	34.41	32.69	31.92	33.02	32.73	32.88													
3	5.5	33.28	33.84	34.15	33.77	24.24	906.5	6.24	0.126E+09	0.00032	26.78	25.23	24.32	25.40	25.44	25.42													
4	15.5	34.02	35.57	35.59	35.06	24.80	918.7	6.15	0.130E+09	0.00090	26.23	22.46	22.41	23.57	23.70	23.63													
5	25.5	35.24	36.34	37.47	36.35	25.37	930.9	6.06	0.134E+09	0.00149	24.46	22.01	19.95	21.99	22.14	22.07													
6	45.5	35.15	36.42	37.50	36.36	25.37	956.2	5.89	0.142E+09	0.00266	27.83	24.28	21.89	24.43	24.66	24.54													
7	75.5	35.93	37.56	38.22	37.24	28.18	991.1	5.66	0.155E+09	0.00443	30.96	25.59	23.92	26.51	26.82	26.66													
9	105.5	36.87	39.25	39.48	38.53	29.87	1026.9	5.45	0.167E+09	0.00620	34.18	25.49	24.89	27.61	28.19	27.90													
9	135.5	38.74	40.79	41.88	40.47	31.56	1065.5	5.24	0.181E+09	0.00799	33.19	25.80	23.08	26.73	27.35	27.04													
10	165.2	39.95	42.36	43.28	41.86	33.24	1101.8	5.04	0.193E+09	0.00979	35.33	26.01	23.62	27.50	28.32	27.91													
11	205.2	41.64	44.04	44.52	43.40	35.49	1153.1	4.78	0.210E+09	0.01225	38.31	27.58	26.12	29.81	30.67	30.24													
12	245.2	44.01	46.53	47.28	45.94	37.75	1206.4	4.54	0.228E+09	0.01473	37.43	26.69	24.60	28.62	29.57	29.09													
13	275.2	45.56	48.12	49.03	47.57	39.44	1245.8	4.38	0.242E+09	0.01658	38.16	26.91	24.36	28.73	29.81	29.27													
14	305.2	46.84	49.19	50.06	48.70	41.13	1287.9	4.23	0.257E+09	0.01844	40.79	28.88	26.08	30.77	31.91	31.34													
15	333.3	48.79	51.34	52.63	50.92	42.71	1326.3	4.09	0.271E+09	0.02020	38.21	26.90	23.42	28.29	29.51	28.90													
16	363.3	50.10	52.63	53.63	52.12	44.40	1366.5	3.96	0.285E+09	0.02208	40.62	28.12	25.08	29.98	31.27	30.63													
17	383.3	51.41	53.49	54.53	53.14	45.53	1394.6	3.87	0.295E+09	0.02334	39.26	29.02	25.66	30.33	31.31	30.82													
18	403.3	52.32	54.56	55.51	54.13	46.65	1423.9	3.79	0.306E+09	0.02461	40.71	29.16	26.05	30.85	31.97	31.41													
19	423.3	53.61	56.04	57.15	55.60	47.78	1452.1	3.70	0.316E+09	0.02588	39.51	27.87	24.56	29.44	30.65	30.04													
20	443.3	54.63	56.39	57.92	56.32	48.91	1480.9	3.63	0.326E+09	0.02716	40.12	30.68	25.49	31.01	32.10	31.55													
21	463.3	56.28	57.32	59.59	57.73	50.03	1510.8	3.55	0.337E+09	0.02844	36.70	31.49	23.99	29.79	30.73	30.26													
AVERAGE VALUES THROUGH STATIONS 15 TO 20:														391.6	51.81	54.08	55.23	53.70	45.99	1407.4	3.84	0.300E+09	0.02388	39.74	28.63	25.04	29.98	31.14	30.56

XP135-20

INPUT ELECTRIC POWER = 2750.0 W HEAT RATE GAINED BY WATER = 2637.9 W HEAT BALANCE ERROR = 4.08%  
 MASS FLOW RATE = 21.3200 G/S PRESSURE DROP = 0.5951 MM H2O FRICTION FACTOR = 0.037307 FREM = 37.0926  
 REM = 994.3 GRM = 0.48239E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 53.59 DEG C  
 PRM = 4.444 RAM = 0.21435E+09 INLET BULK TEMPERATURE = 23.96 DEG C OUTLET BULK TEMPERATURE = 53.57 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----															
		A	B	C	AVERAGE						A	B	C	T	H	T+H													
0	0.1	29.12	29.38	29.41	29.29	23.97	728.5	6.29	0.112E+09	0.00000	42.62	40.55	40.31	41.27	40.94	41.11													
1	1.5	30.08	30.45	30.57	30.35	24.06	730.0	6.27	0.113E+09	0.00011	36.46	34.35	33.72	34.88	34.56	34.72													
2	2.5	30.74	31.19	31.36	31.09	24.12	731.1	6.26	0.113E+09	0.00018	33.16	31.07	30.31	31.51	31.21	31.36													
3	5.5	32.73	33.39	33.75	33.29	24.31	734.3	6.23	0.114E+09	0.00040	26.07	24.15	23.24	24.43	24.49	24.46													
4	15.5	33.58	34.96	34.98	34.51	24.94	745.1	6.13	0.119E+09	0.00112	25.35	21.88	21.82	22.90	23.02	22.96													
5	25.5	34.68	35.84	36.81	35.77	25.57	756.2	6.03	0.123E+09	0.00184	24.03	21.31	19.48	21.45	21.61	21.53													
6	45.5	34.71	35.97	36.99	35.89	26.84	779.4	5.83	0.131E+09	0.00329	27.72	23.88	21.48	24.09	24.35	24.23													
7	75.5	35.60	37.23	37.94	36.92	28.73	810.3	5.59	0.144E+09	0.00548	31.62	25.55	23.59	26.51	26.92	26.72													
8	105.5	36.87	39.19	39.42	38.49	30.63	843.7	5.35	0.157E+09	0.00768	34.66	25.24	24.59	27.49	28.16	27.82													
9	135.5	39.07	41.18	42.05	40.77	32.52	878.2	5.12	0.170E+09	0.00991	32.87	24.85	22.60	26.11	26.77	26.44													
10	165.2	40.50	42.86	43.61	42.32	34.40	911.5	4.90	0.183E+09	0.01216	35.11	25.32	23.25	27.03	27.89	27.46													
11	205.2	42.42	44.71	45.18	44.10	36.92	960.4	4.61	0.201E+09	0.01523	38.72	27.36	25.78	29.65	30.62	30.13													
12	245.2	45.17	47.42	48.11	46.90	39.45	1007.4	4.38	0.220E+09	0.01828	37.03	26.58	24.46	28.43	29.36	28.89													
13	275.2	46.95	49.29	50.14	48.79	41.35	1045.7	4.21	0.235E+09	0.02058	37.67	26.58	23.99	28.34	29.41	28.88													
14	305.2	48.45	50.59	51.34	50.13	43.24	1082.2	4.05	0.249E+09	0.02290	40.38	28.62	25.96	30.54	31.65	31.10													
15	333.3	50.68	52.97	54.08	52.58	45.01	1116.9	3.91	0.263E+09	0.02509	37.02	26.37	23.12	27.73	28.84	28.28													
16	363.3	52.15	54.58	55.47	54.07	46.91	1156.5	3.77	0.279E+09	0.02744	39.87	27.25	24.40	29.19	30.50	29.85													
17	383.3	53.63	55.72	56.58	55.31	48.17	1181.8	3.68	0.289E+09	0.02901	38.19	27.62	24.79	29.20	30.20	29.70													
18	403.3	54.66	56.74	57.62	56.34	49.44	1208.4	3.59	0.300E+09	0.03059	39.87	28.50	25.42	30.15	31.26	30.71													
19	423.3	56.05	58.37	59.32	57.91	50.70	1236.1	3.50	0.311E+09	0.03218	38.82	27.07	24.09	28.78	29.99	29.39													
20	443.3	57.14	58.92	60.26	58.77	51.96	1264.6	3.41	0.323E+09	0.03378	40.00	29.81	25.00	30.44	31.60	31.02													
21	463.3	58.73	60.00	61.82	60.19	53.23	1288.7	3.35	0.333E+09	0.03535	37.56	30.54	24.07	29.73	30.72	30.23													
AVERAGE VALUES THROUGH STATIONS 15 TO 20:														391.6	54.05	56.22	57.22	55.83	48.70	1194.0	3.64	0.294E+09	0.02968	38.96	27.77	24.47	29.25	30.40	29.82

XP135-21

INPUT ELECTRIC POWER = 2435.0 W HEAT RATE GAINED BY WATER = 2333.2 W HEAT BALANCE ERROR = 4.18%  
 MASS FLOW RATE = 16.2660 G/S PRESSURE DROP = 0.4058 MM H2O FRICTION FACTOR = 0.043664 FREM = 34.6481  
 REM = 793.5 GRM = 0.48589E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 58.26 DEG C  
 PRM = 4.231 RAM = 0.20559E+09 INLET BULK TEMPERATURE = 23.91 DEG C OUTLET BULK TEMPERATURE = 58.24 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA	Z	----- NUSSELT NUMBER -----			----- AVERAGE -----		
		A	B	C	AVERAGE						A	B	C	T	H	T+H
0	0.1	28.76	29.01	29.08	28.94	23.92	555.2	6.29	0.989E+08	0.00000	40.14	38.10	37.60	38.68	38.36	38.52
1	1.5	29.64	30.00	30.15	29.92	24.02	556.6	6.28	0.995E+08	0.00014	34.59	32.48	31.71	32.93	32.62	32.78
2	2.5	30.24	30.68	30.88	30.60	24.10	557.5	6.27	0.100E+09	0.00024	31.58	29.49	28.62	29.87	29.57	29.72
3	5.5	32.07	32.72	33.08	32.62	24.32	560.3	6.23	0.101E+09	0.00052	25.04	23.09	22.14	23.36	23.42	23.39
4	15.5	32.92	34.34	34.48	33.91	25.05	569.8	6.11	0.105E+09	0.00146	24.62	20.85	20.53	21.86	22.00	21.93
5	25.5	34.06	35.28	36.14	35.16	25.78	579.7	6.00	0.110E+09	0.00241	23.36	20.35	18.66	20.61	20.79	20.70
6	45.5	34.09	35.97	36.33	35.46	27.24	599.6	5.78	0.119E+09	0.00432	28.14	22.09	21.22	23.45	23.82	23.63
7	75.5	35.49	37.28	37.72	36.83	29.44	627.5	5.50	0.131E+09	0.00719	31.72	24.46	23.18	25.96	26.45	26.21
8	105.5	37.26	39.64	39.70	38.87	31.64	658.2	5.23	0.145E+09	0.01009	33.94	23.85	23.67	26.40	27.16	26.78
9	135.5	39.74	41.85	42.55	41.38	33.83	687.6	4.97	0.158E+09	0.01304	32.14	23.68	21.78	25.15	25.87	25.51
10	165.2	41.56	43.75	44.44	43.25	36.01	718.9	4.72	0.172E+09	0.01602	34.03	24.38	22.37	26.07	26.93	26.50
11	205.2	43.93	46.09	46.41	45.48	38.94	761.0	4.43	0.191E+09	0.02003	37.59	26.21	25.10	28.68	29.63	29.16
12	245.2	47.06	49.14	49.73	48.64	41.87	806.2	4.16	0.211E+09	0.02406	35.95	25.63	23.73	27.53	28.44	27.98
13	275.2	49.17	51.35	52.09	50.87	44.06	837.7	3.99	0.226E+09	0.02710	36.40	25.49	23.15	27.30	28.35	27.82
14	305.2	51.01	52.87	53.63	52.50	46.26	871.8									

## Appendix 4

### Experimental Data for $\alpha = 180^\circ$

XP180-01

INPUT ELECTRIC POWER = 155.0 W HEAT RATE GAINED BY WATER = 154.1 W HEAT BALANCE ERROR = 0.56%  
 MASS FLOW RATE = 14.1000 G/S PRESSURE DROP = 0.2847 MM H2O FRICTION FACTOR = 0.041015 FREM = 20.3534  
 REM = 496.2 GRM = 0.11576E+07 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 26.56 DEG C  
 PRM = 6.084 RAM = 0.70422E+07 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 26.55 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER			AVERAGE						
		A	B	C						A	B	C	T	H	T+H				
0	0.1	24.47	24.47	24.49	24.48	23.94	481.5	6.29	0.654E+07	0.00001	23.87	24.07	23.31	23.63	23.64	23.63			
1	1.5	24.56	24.55	24.58	24.57	23.95	481.6	6.29	0.655E+07	0.00016	20.88	21.09	20.13	20.55	20.56	20.55			
2	2.5	24.62	24.61	24.65	24.63	23.95	481.7	6.29	0.655E+07	0.00027	19.22	19.44	16.40	18.85	18.87	18.86			
3	5.5	24.79	24.78	24.84	24.82	23.97	481.8	6.29	0.656E+07	0.00060	15.52	15.73	14.63	15.11	15.13	15.12			
4	15.5	25.05	25.05	25.08	25.06	24.02	482.4	6.28	0.658E+07	0.00168	12.50	12.50	12.17	12.33	12.33	12.33			
5	25.5	25.18	25.23	25.33	25.27	24.08	483.1	6.27	0.660E+07	0.00277	11.68	11.12	10.23	10.78	10.81	10.80			
6	45.5	25.23	25.24	25.35	25.29	24.19	484.3	6.25	0.664E+07	0.00494	12.37	12.17	11.06	11.63	11.67	11.65			
7	75.5	25.19	25.14	25.23	25.20	24.36	486.2	6.22	0.670E+07	0.00821	15.43	16.42	14.74	15.30	15.33	15.32			
8	105.5	25.19	25.16	25.24	25.21	24.53	488.0	6.20	0.677E+07	0.01147	19.32	20.34	17.96	18.85	18.90	18.87			
9	135.5	25.41	25.41	25.68	25.54	24.69	489.9	6.17	0.683E+07	0.01474	17.98	17.98	12.93	15.04	15.45	15.25			
10	165.2	25.69	25.62	25.86	25.76	24.86	491.8	6.14	0.689E+07	0.01798	15.32	16.77	12.77	14.21	14.41	14.31			
11	205.2	25.75	25.82	25.99	25.89	25.08	494.4	6.11	0.698E+07	0.02235	19.15	17.42	14.02	15.86	16.15	16.01			
12	245.2	26.11	26.08	26.26	26.18	25.30	496.9	6.07	0.707E+07	0.02672	15.87	16.55	13.38	14.66	14.79	14.72			
13	275.2	26.23	26.26	26.41	26.33	25.47	498.9	6.05	0.713E+07	0.03000	16.80	16.33	13.62	14.95	15.09	15.02			
14	305.2	26.36	26.42	26.54	26.46	25.64	500.9	6.02	0.720E+07	0.03329	17.86	16.33	14.21	15.51	15.65	15.58			
15	333.3	26.59	26.63	26.84	26.73	25.80	502.7	6.00	0.726E+07	0.03637	16.12	15.37	12.19	13.74	13.97	13.86			
16	363.3	26.78	26.75	26.97	26.87	25.96	504.7	5.97	0.733E+07	0.03966	15.69	16.25	12.67	14.13	14.32	14.22			
17	383.3	26.92	26.90	27.03	26.97	26.08	506.1	5.95	0.737E+07	0.04186	15.08	15.49	13.41	14.28	14.35	14.31			
18	403.3	27.01	27.01	27.11	27.06	26.19	507.4	5.93	0.742E+07	0.04405	15.60	15.60	13.90	14.70	14.75	14.72			
19	423.3	27.16	27.10	27.27	27.20	26.30	508.8	5.92	0.746E+07	0.04625	14.89	15.93	13.18	14.20	14.30	14.25			
20	443.3	27.21	27.25	27.38	27.31	26.41	510.2	5.90	0.751E+07	0.04845	15.93	15.19	13.11	14.23	14.34	14.28			
21	463.3	27.32	27.37	27.57	27.46	26.52	511.5	5.88	0.755E+07	0.05065	15.94	15.00	12.14	13.60	13.81	13.70			
AVERAGE VALUES THROUGH STATIONS 15 TO 20:					20:														
391.6					26.94	26.94	27.10	27.02	26.12	506.7	5.95	0.739E+07	0.04277	15.55	15.64	13.08	14.21	14.34	14.27

XP180-02

INPUT ELECTRIC POWER = 285.0 W HEAT RATE GAINED BY WATER = 279.1 W HEAT BALANCE ERROR = 2.08%  
 MASS FLOW RATE = 28.5400 G/S PRESSURE DROP = 0.6257 MM H2O FRICTION FACTOR = 0.022002 FREM = 22.0283  
 REM = 1001.2 GRM = 0.20721E+07 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 26.28 DEG C  
 PRM = 6.106 RAM = 0.12652E+08 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 26.28 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER			AVERAGE						
		A	B	C						A	B	C	T	H	T+H				
0	0.1	24.62	24.62	24.58	24.60	23.94	974.6	6.29	0.118E+08	0.00000	34.07	34.18	36.34	35.20	35.23	35.21			
1	1.5	24.74	24.74	24.71	24.72	23.94	974.8	6.29	0.119E+08	0.00008	29.20	29.32	30.46	29.85	29.86	29.85			
2	2.5	24.82	24.82	24.80	24.81	23.95	974.9	6.29	0.119E+08	0.00013	26.59	26.70	27.40	27.02	27.02	27.02			
3	5.5	25.07	25.07	25.07	25.07	23.96	975.2	6.29	0.119E+08	0.00030	20.95	21.06	21.06	21.03	21.03	21.03			
4	15.5	25.44	25.44	25.47	25.45	24.01	976.3	6.28	0.119E+08	0.00083	16.29	16.23	15.98	16.12	16.12	16.12			
5	25.5	25.63	25.62	25.72	25.67	24.06	977.4	6.27	0.119E+08	0.00137	14.84	14.90	14.00	14.42	14.43	14.43			
6	45.5	25.73	25.69	25.85	25.78	24.16	979.6	6.25	0.120E+08	0.00244	14.84	15.22	13.76	14.37	14.40	14.38			
7	75.5	25.74	25.69	25.79	25.75	24.31	983.0	6.23	0.121E+08	0.00405	16.21	16.80	15.72	16.10	16.11	16.11			
8	105.5	25.74	25.77	25.91	25.83	24.46	986.4	6.21	0.122E+08	0.00567	18.10	17.79	16.08	16.96	17.01	16.99			
9	135.5	25.96	25.91	26.24	26.09	24.61	989.8	6.18	0.123E+08	0.00728	17.19	17.93	14.26	15.74	15.91	15.82			
10	165.2	26.14	26.18	26.47	26.32	24.76	993.2	6.16	0.124E+08	0.00888	16.82	16.29	13.54	14.90	15.05	14.97			
11	205.2	26.26	26.32	26.55	26.42	24.96	997.8	6.13	0.125E+08	0.01104	17.89	17.08	14.58	15.89	16.03	15.96			
12	245.2	26.44	26.41	26.71	26.57	25.16	1002.4	6.10	0.127E+08	0.01320	18.03	18.51	14.98	16.46	16.63	16.55			
13	275.2	26.62	26.70	26.91	26.79	25.31	1006.0	6.07	0.128E+08	0.01482	17.64	16.65	14.46	15.68	15.80	15.74			
14	305.2	26.69	26.70	26.99	26.84	25.46	1009.5	6.05	0.129E+08	0.01644	18.82	18.65	15.13	16.74	16.93	16.84			
15	333.3	26.92	26.97	27.29	27.12	25.60	1012.9	6.03	0.130E+08	0.01796	17.57	16.92	13.65	15.24	15.45	15.34			
16	363.3	27.06	27.03	27.36	27.20	25.75	1016.4	6.00	0.131E+08	0.01958	17.71	18.09	14.35	15.93	16.12	16.03			
17	383.3	27.20	27.18	27.47	27.33	25.85	1018.9	5.99	0.132E+08	0.02067	17.12	17.40	14.25	15.61	15.75	15.68			
18	403.3	27.34	27.23	27.55	27.42	25.95	1021.3	5.97	0.133E+08	0.02175	16.57	18.08	14.44	15.74	15.88	15.81			
19	423.3	27.43	27.49	27.77	27.62	26.05	1023.7	5.96	0.133E+08	0.02283	16.69	16.05	13.41	14.74	14.89	14.82			
20	443.3	27.54	27.58	27.83	27.70	26.15	1026.2	5.94	0.134E+08	0.02392	16.55	16.10	13.76	14.93	15.04	14.99			
21	463.3	27.77	27.76	28.02	27.89	26.25	1028.6	5.93	0.135E+08	0.02501	15.22	15.27	13.06	14.07	14.15	14.11			
AVERAGE VALUES THROUGH STATIONS 15 TO 20:					20:														
391.6					27.25	27.25	27.55	27.40	25.89	1019.9	5.98	0.132E+08	0.02112	17.03	17.11	13.98	15.36	15.52	15.44

XP180-03

INPUT ELECTRIC POWER = 295.0 W HEAT RATE GAINED BY WATER = 301.9 W HEAT BALANCE ERROR = -2.34%  
 MASS FLOW RATE = 42.9800 G/S PRESSURE DROP = 0.9395 MM H2O FRICTION FACTOR = 0.014568 FREM = 21.7136  
 REM = 1490.4 GRM = 0.21525E+07 UPSTREAM BULK TEMPERATURE = 23.77 DEG C DOWN STREAM BULK TEMPERATURE = 25.45 DEG C  
 PRM = 6.184 RAM = 0.13312E+08 INLET BULK TEMPERATURE = 23.77 DEG C OUTLET BULK TEMPERATURE = 25.45 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	24.35	24.35	24.29	24.32	23.77	1462.1	6.32	0.127E+08	0.00000	43.07	43.40	48.42	45.68	45.83	45.75
1	1.5	24.45	24.45	24.39	24.42	23.77	1462.3	6.32	0.127E+08	0.00005	36.89	37.23	40.69	38.79	38.88	38.83
2	2.5	24.53	24.52	24.46	24.49	23.78	1462.4	6.32	0.127E+08	0.00009	33.57	33.91	36.65	35.14	35.20	35.17
3	5.5	24.74	24.73	24.68	24.71	23.79	1462.8	6.31	0.127E+08	0.00020	26.43	26.75	28.25	27.39	27.42	27.41
4	15.5	24.99	24.99	24.91	24.95	23.82	1463.9	6.31	0.127E+08	0.00055	21.47	21.47	23.12	22.26	22.29	22.28
5	25.5	25.12	25.18	25.17	25.16	23.86	1465.1	6.30	0.128E+08	0.00091	19.91	19.07	19.23	19.35	19.36	19.36
6	45.5	25.23	25.24	25.29	25.27	23.93	1467.5	6.29	0.128E+08	0.00162	19.38	19.14	18.44	18.84	18.85	18.84
7	75.5	25.41	25.31	25.45	25.40	24.04	1471.1	6.27	0.129E+08	0.00269	18.31	19.83	17.80	18.40	18.44	18.42
8	105.5	25.47	25.49	25.63	25.55	24.15	1474.7	6.26	0.130E+08	0.00375	19.02	18.71	16.96	17.86	17.91	17.89
9	135.5	25.63	25.63	25.91	25.77	24.25	1478.4	6.24	0.130E+08	0.00483	18.28	18.28	15.20	16.60	16.74	16.67
10	165.2	25.81	25.79	26.14	25.97	24.36	1482.0	6.22	0.131E+08	0.00589	17.37	17.57	14.12	15.62	15.79	15.71
11	205.2	25.87	25.87	26.16	26.02	24.50	1486.9	6.20	0.132E+08	0.00732	18.41	18.34	15.14	16.60	16.76	16.68
12	245.2	26.00	26.02	26.26	26.14	24.65	1491.8	6.18	0.133E+08	0.00875	18.55	18.25	15.55	16.85	16.97	16.91
13	275.2	26.12	26.14	26.41	26.27	24.75	1495.5	6.16	0.134E+08	0.00983	18.34					

XP180-04

INPUT ELECTRIC POWER = 265.0 W HEAT RATE GAINED BY WATER = 258.0 W HEAT BALANCE ERROR = 2.64%  
 MASS FLOW RATE = 14.1000 G/S PRESSURE DROP = 0.2984 MM H2O FRICTION FACTOR = 0.042981 FREM = 21.7798  
 REM = 506.7 GRM = 0.20821E+07 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 28.32 DEG C  
 PRM = 5.944 RAM = 0.12376E+08 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 28.32 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER			AVERAGE			
		A	B	C	AVER- AGE						A	B	C	T	H	T+H	
0	0.1	24.79	24.77	24.78	24.78	23.94	481.5	6.29	0.110E+08	0.00001	25.12	25.85	25.45	25.47	25.47	25.47	
1	1.5	24.94	24.91	24.93	24.93	23.95	481.7	6.29	0.110E+08	0.00016	21.71	22.49	21.96	22.03	22.03	22.03	
2	2.5	25.04	25.00	25.03	25.03	23.96	481.0	6.29	0.110E+08	0.00027	19.86	20.63	20.06	20.15	20.15	20.15	
3	5.5	25.35	25.29	25.34	25.33	23.99	482.1	6.28	0.110E+08	0.00060	15.80	16.55	15.93	16.05	16.05	16.05	
4	15.5	25.61	25.67	25.69	25.66	24.08	483.1	6.27	0.110E+08	0.00168	14.11	13.57	13.38	13.60	13.61	13.61	
5	25.5	25.74	25.79	25.94	25.85	24.18	484.1	6.25	0.111E+08	0.00277	13.75	13.32	12.15	12.80	12.84	12.82	
6	45.5	25.78	25.80	25.96	25.88	24.36	486.2	6.22	0.112E+08	0.00495	15.12	14.95	13.44	14.19	14.24	14.21	
7	75.5	25.74	25.64	25.84	25.77	24.64	489.4	6.18	0.114E+08	0.00821	19.50	21.57	17.87	19.09	19.20	19.15	
8	105.5	25.91	25.93	26.03	24.92	24.92	492.6	6.13	0.116E+08	0.01149	21.73	21.25	17.81	19.48	19.65	19.57	
9	135.5	26.35	26.35	26.74	26.54	25.20	495.8	6.09	0.118E+08	0.01476	18.70	18.70	13.96	15.99	16.33	16.16	
10	165.2	26.69	26.68	27.08	26.89	25.48	499.0	6.05	0.119E+08	0.01801	17.66	17.82	13.37	15.25	15.55	15.40	
11	205.2	27.03	27.09	27.44	27.25	25.86	503.4	5.99	0.122E+08	0.02239	18.16	17.27	13.51	15.33	15.61	15.47	
12	245.2	27.49	27.47	27.71	27.59	26.23	508.0	5.93	0.124E+08	0.02679	16.89	17.27	14.48	15.67	15.78	15.72	
13	275.2	27.62	27.64	27.97	27.80	26.51	511.4	5.88	0.126E+08	0.03009	19.19	18.82	14.66	16.55	16.83	16.69	
14	305.2	27.91	27.93	28.21	28.07	26.79	514.9	5.84	0.128E+08	0.03339	19.03	18.75	15.02	16.73	16.95	16.84	
15	333.3	28.36	28.31	28.64	28.49	27.05	517.7	5.80	0.130E+08	0.03648	16.30	17.02	13.44	14.88	15.05	14.96	
16	363.3	28.49	28.53	28.87	28.69	27.33	520.7	5.77	0.132E+08	0.03978	18.34	17.83	13.89	15.71	15.99	15.85	
17	383.3	28.81	28.79	29.03	28.92	27.52	522.7	5.75	0.133E+08	0.04199	16.49	16.71	14.12	15.26	15.36	15.31	
18	403.3	28.96	28.96	29.22	29.09	27.71	524.7	5.72	0.134E+08	0.04419	16.97	17.04	14.05	15.39	15.53	15.46	
19	423.3	29.16	29.16	29.44	29.30	27.89	526.7	5.70	0.135E+08	0.04639	16.86	16.79	13.72	15.11	15.27	15.19	
20	443.3	29.33	29.37	29.61	29.48	28.08	528.7	5.68	0.136E+08	0.04860	17.05	16.46	13.94	15.22	15.35	15.28	
21	463.3	29.61	29.58	29.97	29.78	28.27	530.7	5.65	0.138E+08	0.05081	15.88	16.15	12.51	14.05	14.26	14.16	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:		391.6	28.85	28.85	29.13	28.99	27.60	523.5	5.74	0.133E+08	0.04291	17.00	16.98	13.86	15.26	15.43	15.34

XP180-05

INPUT ELECTRIC POWER = 665.0 W HEAT RATE GAINED BY WATER = 647.4 W HEAT BALANCE ERROR = 2.64%  
 MASS FLOW RATE = 42.2580 G/S PRESSURE DROP = 0.9088 MM H2O FRICTION FACTOR = 0.014575 FREM = 21.9459  
 REM = 1505.8 GRM = 0.50745E+07 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 27.61 DEG C  
 PRM = 6.001 RAM = 0.30451E+08 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 27.60 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER			AVERAGE			
		A	B	C	AVER- AGE						A	B	C	T	H	T+H	
0	0.1	25.22	25.20	25.16	25.19	23.94	1443.1	6.29	0.275E+08	0.00000	41.87	42.59	44.16	43.17	43.19	43.18	
1	1.5	25.48	25.45	25.41	25.44	23.95	1443.4	6.29	0.275E+08	0.00005	35.20	35.93	36.80	36.17	36.18	36.18	
2	2.5	25.66	25.62	25.59	25.61	23.96	1443.7	6.29	0.275E+08	0.00009	31.72	32.43	33.01	32.53	32.54	32.54	
3	5.5	26.18	26.13	26.12	26.14	23.98	1444.5	6.28	0.276E+08	0.00020	24.46	25.09	25.22	24.99	25.00	25.00	
4	15.5	26.66	26.68	26.63	26.65	24.06	1447.0	6.27	0.277E+08	0.00056	20.74	20.52	20.92	20.78	20.78	20.78	
5	25.5	26.97	26.96	27.16	27.06	24.14	1449.6	6.26	0.278E+08	0.00092	19.03	19.11	17.81	18.42	18.44	18.43	
6	45.5	27.00	27.02	27.36	27.18	24.29	1454.8	6.23	0.280E+08	0.00165	19.89	19.73	17.58	18.63	18.70	18.66	
7	75.5	27.24	27.19	27.52	27.37	24.53	1462.7	6.20	0.284E+08	0.00274	19.85	20.18	18.01	18.96	19.01	18.98	
8	105.5	27.30	27.38	27.89	27.57	24.76	1470.6	6.16	0.288E+08	0.00383	21.19	20.52	17.71	19.15	19.28	19.22	
9	135.5	27.57	27.57	28.29	27.93	25.00	1478.6	6.12	0.292E+08	0.00492	20.87	20.87	16.33	18.32	18.60	18.46	
10	165.2	27.80	27.85	28.59	28.21	25.23	1486.7	6.09	0.296E+08	0.00601	20.90	20.50	15.99	18.04	18.35	18.20	
11	205.2	28.09	28.09	28.67	28.38	25.54	1497.6	6.04	0.301E+08	0.00747	21.08	21.03	17.18	18.92	19.12	19.02	
12	245.2	28.38	28.41	29.04	28.72	25.85	1508.8	5.99	0.306E+08	0.00893	21.22	20.99	16.82	18.72	18.96	18.84	
13	275.2	28.68	28.76	29.41	29.07	26.09	1517.2	5.95	0.310E+08	0.01003	20.72	20.07	16.14	18.02	18.27	18.14	
14	305.2	28.80	28.76	29.44	29.11	26.32	1525.8	5.91	0.314E+08	0.01113	21.64	21.99	17.21	19.24	19.51	19.38	
15	333.3	29.25	29.26	29.98	29.62	26.54	1533.9	5.88	0.318E+08	0.01216	19.79	19.75	15.60	17.44	17.69	17.56	
16	363.3	29.33	29.36	30.04	29.69	26.78	1542.6	5.84	0.322E+08	0.01326	21.00	20.73	16.42	18.38	18.64	18.51	
17	383.3	29.64	29.58	30.24	29.63	26.93	1547.9	5.82	0.324E+08	0.01400	19.75	20.25	16.17	17.68	18.09	17.98	
18	403.3	29.74	29.68	30.34	30.03	27.09	1552.8	5.80	0.327E+08	0.01473	20.21	20.64	16.47	18.24	18.45	18.34	
19	423.3	29.98	29.88	30.61	30.27	27.25	1557.8	5.78	0.329E+08	0.01546	19.55	20.29	15.90	17.68	17.91	17.80	
20	443.3	29.99	30.04	30.67	30.34	27.40	1562.7	5.76	0.332E+08	0.01620	20.64	20.25	16.39	18.19	18.41	18.30	
21	463.3	30.33	30.23	31.03	30.65	27.56	1567.7	5.74	0.334E+08	0.01693	19.27	20.03	15.41	17.27	17.53	17.40	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:		391.6	29.66	29.63	30.31	29.98	27.00	1549.6	5.81	0.325E+08	0.01430	20.16	20.32	16.16	17.97	18.20	18.08

XP180-06

INPUT ELECTRIC POWER = 625.0 W HEAT RATE GAINED BY WATER = 605.9 W HEAT BALANCE ERROR = 3.05%  
 MASS FLOW RATE = 27.8180 G/S PRESSURE DROP = 0.6223 MM H2O FRICTION FACTOR = 0.023027 FREM = 23.2210  
 REM = 1008.4 GRM = 0.50351E+07 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 29.10 DEG C  
 PRM = 5.887 RAM = 0.29641E+08 INLET BULK TEMPERATURE = 23.88 DEG C OUTLET BULK TEMPERATURE = 29.10 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER			AVERAGE		
		A	B	C	AVER- AGE						A	B	C	T	H	T+H
0	0.1	25.32	25.32	25.30	25.31	23.88	948.8	6.30	0.256E+08	0.00000	35.04	35.04	35.60	35.32	35.32	35.32
1	1.5	25.60	25.60	25.59	25.59	23.90	949.2	6.30	0.257E+08	0.00008	29.73	29.73	29.82	29.78	29.78	29.78
2	2.5	25.79	25.79	25.79	25.79	23.91	949.4	6.29	0.257E+08	0.00014	26.92	26.92	26.82	26.87	26.87	26.87
3	5.5	26.35	26.35	26.39	26.37	23.94	950.1	6.29	0.257E+08	0.00030	20.97	20.97	20.59	20.78	20.78	20.78
4	15.5	26.82	26.96	27.08	26.98	24.06	952.5	6.27	0.259E+08	0.00085	18.23	17.36	16.69	17.22	17.24	17.23
5	25.5	27.19	27.23	27.61	27.41	24.17	954.9	6.25	0.261E+08	0.00140	16.68	16.44	14.66	15.55	15.61	15.58
6	45.5	27.22	27.24	27.69	27.46	24.39	959.8	6.22	0.264E+08	0.00251	17.79	17.65	15.27	16.40	16.50	16.45
7	75.5	27.35	27.31	27.74	27.53	24.72	967.2	6.17	0.269E+08	0.00416	19.17	19.50	16.70	17.92	18.01	17.97
8	105.5	27.52	27.61	28.02	27.79	25.06	974.7	6.11	0.274E+08	0.00582	20.41	19.74	16.97	18.39	18.52	18.45
9	135.5	27.96	27.91	28.62	28.28	25.39	982.4	6.06	0.279E+08	0.00749	19.55	19.99	15.55	17.41	17.66	17.54
10	165.2	28.24	28.24	29.03	28.64	25.72	990.1	6.01	0.284E+08	0.00913	19.90	19.95	15.16	17.22	17.54	17.38
11	205.2	28.53	28.59	29.22	28.89	26.16	1000.6	5.94	0.291E+08	0.01136	21.19	20.66	16.42	18.40	18.67	18.53
12	245.2	29.10	29.02	29.66	29.36	26.61	1011.4	5.87	0.298E+08	0.01359	20.13	20.78	16.43	18.22	18.44	18.33
13	275.2	29.29	29.32	30.08	29.69	26.94	1019.2	5.82	0.304E+08	0.01527	21.36	21.11	15.98	18.24	18.61	18.42
14	305.2	29.58	29.48	30.22	29.88	27.28	1026.1	5.7								

XP180-07

INPUT ELECTRIC POWER = 535.0 W HEAT RATE GAINED BY WATER = 522.5 W HEAT BALANCE ERROR = 2.34%  
 MASS FLOW RATE = 13.3780 G/S PRESSURE DROP = 0.2796 MM H2O FRICTION FACTOR = 0.044697 FREM = 22.7263  
 REM = 508.4 GRM = 0.50921E+07 UPSTREAM BULK TEMPERATURE = 24.04 DEG C DOWN STREAM BULK TEMPERATURE = 33.41 DEG C  
 PRM = 5.593 RAM = 0.28480E+08 INLET BULK TEMPERATURE = 24.06 DEG C OUTLET BULK TEMPERATURE = 33.40 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	----- NUSSELT NUMBER -----			----- AVERAGE -----				
		A	B	C						A	B	C	T	H	T+H		
0	0.1	25.55	25.55	25.60	25.57	24.06	458.1	6.27	0.223E+08	0.00001	29.22	29.22	28.22	28.71	28.72	28.72	
1	1.5	25.82	25.82	25.90	25.86	24.09	458.4	6.27	0.224E+08	0.00017	25.10	25.10	24.01	24.54	24.56	24.55	
2	2.5	26.01	26.01	26.10	26.06	24.11	458.6	6.26	0.224E+08	0.00029	22.88	22.88	21.77	22.31	22.32	22.32	
3	5.5	26.57	26.57	26.72	26.65	24.17	459.2	6.25	0.225E+08	0.00063	18.07	18.07	17.01	17.52	17.54	17.53	
4	15.5	26.99	27.13	27.31	27.18	24.37	461.3	6.22	0.227E+08	0.00178	16.57	15.70	14.78	15.42	15.46	15.44	
5	25.5	27.36	27.29	27.83	27.58	24.57	463.5	6.19	0.230E+08	0.00292	15.57	15.95	13.31	14.43	14.53	14.48	
6	45.5	27.39	27.36	27.86	27.61	24.96	467.8	6.13	0.235E+08	0.00522	17.89	18.14	15.01	16.38	16.51	16.44	
7	75.5	27.73	27.69	28.18	27.95	25.56	474.4	6.03	0.243E+08	0.00868	19.96	20.32	16.53	18.16	18.34	18.25	
8	105.5	28.24	28.33	28.74	28.52	26.16	481.1	5.94	0.251E+08	0.01214	20.76	19.96	16.75	18.38	18.55	18.46	
9	135.5	28.91	28.96	29.57	29.25	26.76	488.1	5.84	0.259E+08	0.01552	20.13	19.62	15.39	17.34	17.63	17.49	
10	165.2	29.41	29.41	30.14	29.78	27.35	494.2	5.77	0.267E+08	0.01907	20.94	21.00	15.45	17.79	18.21	18.00	
11	205.2	30.09	30.21	30.72	30.43	28.15	502.3	5.67	0.277E+08	0.02371	22.19	20.94	16.73	18.84	19.15	18.99	
12	245.2	30.99	30.97	31.50	31.24	28.95	510.7	5.57	0.288E+08	0.02837	21.04	21.27	16.83	18.75	19.00	18.87	
13	275.2	31.46	31.43	32.08	31.76	29.54	517.2	5.49	0.296E+08	0.03188	22.46	22.79	16.94	19.38	19.78	19.58	
14	305.2	32.07	32.04	32.62	32.34	30.14	523.9	5.42	0.304E+08	0.03539	22.21	22.54	17.33	19.53	19.85	19.69	
15	333.3	32.81	32.78	33.39	33.09	30.70	530.3	5.34	0.312E+08	0.03868	20.36	20.63	15.94	17.93	18.22	18.07	
16	363.3	33.26	33.36	33.89	33.60	31.30	537.3	5.27	0.321E+08	0.04221	21.87	20.75	16.52	18.61	18.92	18.76	
17	383.3	33.87	33.66	34.29	34.03	31.70	542.1	5.22	0.326E+08	0.04457	19.71	21.84	16.46	18.35	18.62	18.48	
18	403.3	34.14	34.03	34.62	34.36	32.10	546.5	5.17	0.332E+08	0.04694	20.85	22.05	16.91	19.91	19.18	19.04	
19	423.3	34.53	34.56	35.24	34.89	32.50	550.8	5.13	0.337E+08	0.04933	20.93	20.71	15.55	17.80	18.18	17.99	
20	443.3	34.90	34.74	35.51	35.16	32.89	555.1	5.08	0.342E+08	0.05172	21.25	23.03	16.32	18.78	19.22	19.00	
21	463.3	35.40	35.10	35.98	35.62	33.29	559.5	5.03	0.347E+08	0.05413	20.20	23.56	15.82	18.32	18.85	18.59	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	33.92	33.85	34.49	34.19	31.86	543.7	5.20	0.328E+08	0.04557	20.83	21.50	16.28	18.39	18.72	18.56

XP180-08

INPUT ELECTRIC POWER = 1175.0 W HEAT RATE GAINED BY WATER = 1142.7 W HEAT BALANCE ERROR = 2.75%  
 MASS FLOW RATE = 40.8140 G/S PRESSURE DROP = 0.9548 MM H2O FRICTION FACTOR = 0.016409 FREM = 24.7088  
 REM = 1505.8 GRM = 0.10081E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 30.64 DEG C  
 PRM = 5.775 RAM = 0.58211E+08 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 30.64 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	----- NUSSELT NUMBER -----			----- AVERAGE -----				
		A	B	C						A	B	C	T	H	T+H		
0	0.1	26.07	26.01	25.98	26.01	23.94	1393.9	6.29	0.485E+08	0.00000	44.60	45.97	46.61	45.93	45.95	45.94	
1	1.5	26.52	26.43	26.46	26.47	23.96	1394.5	6.29	0.486E+08	0.00006	37.22	38.58	38.08	37.98	37.99	37.99	
2	2.5	26.82	26.71	26.79	26.78	23.98	1395.0	6.28	0.486E+08	0.00009	33.41	34.72	33.61	33.93	33.94	33.94	
3	5.5	27.74	27.58	27.78	27.72	24.02	1396.3	6.28	0.487E+08	0.00021	25.56	26.72	25.30	25.71	25.72	25.72	
4	15.5	28.32	28.48	28.69	28.55	24.16	1400.9	6.23	0.491E+08	0.00058	22.88	22.00	20.97	21.68	21.71	21.69	
5	25.5	28.81	28.84	29.49	29.16	24.30	1405.5	6.23	0.495E+08	0.00096	21.09	20.94	18.33	19.58	19.67	19.63	
6	45.5	28.83	28.86	29.53	29.19	24.59	1414.7	6.19	0.503E+08	0.00171	22.39	22.27	19.24	20.67	20.78	20.72	
7	75.5	29.12	29.13	29.91	29.52	25.02	1428.9	6.12	0.516E+08	0.00284	23.13	23.06	19.40	21.08	21.25	21.17	
8	105.5	29.36	29.44	30.24	29.82	25.45	1443.3	6.05	0.528E+08	0.00397	24.22	23.72	19.76	21.66	21.87	21.77	
9	135.5	29.91	29.96	31.12	30.53	25.88	1458.0	5.98	0.541E+08	0.00511	23.51	23.19	18.05	20.36	20.70	20.53	
10	165.2	30.41	30.52	31.71	31.08	26.30	1472.8	5.92	0.553E+08	0.00624	23.02	22.44	17.51	19.78	20.12	19.95	
11	205.2	30.76	30.87	31.83	31.32	26.87	1493.2	5.83	0.571E+08	0.00776	24.33	23.62	19.05	21.23	21.51	21.37	
12	245.2	31.32	31.30	32.50	31.90	27.44	1510.6	5.76	0.586E+08	0.00928	24.37	24.47	18.67	21.16	21.54	21.35	
13	275.2	31.79	31.82	33.08	32.44	27.87	1523.9	5.70	0.598E+08	0.01042	24.07	23.87	18.09	20.62	21.03	20.83	
14	305.2	31.91	31.88	33.17	32.53	28.30	1537.4	5.65	0.611E+08	0.01156	26.13	26.34	19.34	22.26	22.79	22.53	
15	333.3	32.81	32.83	34.01	33.41	28.70	1550.3	5.60	0.622E+08	0.01264	22.94	22.78	17.75	19.98	20.30	20.14	
16	363.3	32.93	32.97	34.22	33.59	29.13	1564.3	5.54	0.635E+08	0.01378	24.77	24.48	18.47	21.11	21.55	21.33	
17	383.3	33.37	33.38	34.46	33.92	29.42	1573.8	5.51	0.643E+08	0.01455	23.79	23.72	18.63	20.88	21.19	21.04	
18	403.3	33.58	33.48	34.68	34.10	29.70	1583.4	5.47	0.652E+08	0.01532	24.20	24.88	18.87	21.34	21.71	21.52	
19	423.3	33.98	34.00	35.13	34.56	29.99	1593.1	5.43	0.660E+08	0.01608	23.52	23.39	18.26	20.54	20.86	20.70	
20	443.3	34.12	34.19	35.34	34.75	30.28	1603.0	5.40	0.669E+08	0.01685	24.37	23.96	18.52	20.97	21.34	21.15	
21	463.3	34.67	34.54	35.82	35.21	30.56	1612.9	5.36	0.678E+08	0.01762	22.79	23.56	17.83	20.15	20.50	20.33	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	33.46	33.48	34.64	34.05	29.54	1578.0	5.49	0.647E+08	0.01487	23.93	23.87	18.42	20.80	21.16	20.98

XP180-09

INPUT ELECTRIC POWER = 1110.0 W HEAT RATE GAINED BY WATER = 1085.6 W HEAT BALANCE ERROR = 2.20%  
 MASS FLOW RATE = 32.1500 G/S PRESSURE DROP = 0.7076 MM H2O FRICTION FACTOR = 0.019593 FREM = 23.5446  
 REM = 1201.7 GRM = 0.10008E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 31.97 DEG C  
 PRM = 5.694 RAM = 0.56989E+08 INLET BULK TEMPERATURE = 23.89 DEG C OUTLET BULK TEMPERATURE = 31.97 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	26.07	26.06	26.03	26.05	23.89	1096.6	6.30	0.460E+08	0.00000	41.33	41.67	42.08	41.79	41.79	41.79
1	1.5	26.52	26.49	26.51	26.51	23.91	1097.3	6.29	0.460E+08	0.00007	34.70	35.03	34.76	34.81	34.81	34.81
2	2.5	26.82	26.79	26.84	26.82	23.93	1097.7	6.29	0.461E+08	0.00012	31.24	31.56	31.04	31.22	31.22	31.22
3	5.5	27.74	27.69	27.83	27.77	23.98	1099.0	6.28	0.462E+08	0.00026	24.05	24.33	23.49	23.83	23.84	23.84
4	15.5	28.43	28.71	28.86	28.71	24.15	1103.3	6.26	0.467E+08	0.00074	21.13	19.84	19.19	19.81	19.84	19.82
5	25.5	28.98	28.96	29.71	29.34	24.33	1107.7	6.23	0.471E+08	0.00122	19.41	19.50	16.77	18.01	18.11	18.06
6	45.5	28.94	28.86	29.69	29.30	24.67	1116.5	6.17	0.480E+08	0.00217	21.11	21.56	17.96	19.50	19.65	19.57
7	75.5	29.18	29.19	29.97	29.58	25.19	1130.0	6.09	0.494E+08	0.00361	22.58	22.52	18.86	20.54	20.70	20.62
8	105.5	29.53	29.72	30.47	30.05	25.70	1143.8	6.01	0.509E+08	0.00505	23.54	22.40	18.90	20.73	20.94	20.83
9	135.5	30.13	30.18	31.34	30.75	26.22	1158.0	5.93	0.523E+08	0.00649	23.01	22.69	17.55	19.85	20.20	20.03
10	165.2	30.58	30.52	31.87	31.21	26.73	1172.4	5.85	0.538E+08	0.00793	23.36	23.74	17.47	20.06	20.51	20.29
11	205.2	31.76	31.80	32.89	32.34	28.11	1189.4	5.76	0.557E+08	0.00986	25.22	24.42	19.12	21.60	21.97	21.79
12	245.2	32.23	32.27	33.53	32.89	28.63	1206.3	5.67	0.575E+08	0.01179	24.53	24.27	18.72	21.89	21.56	21.37
13	275.2	32.63	32.61	33.79	33.20	29.15	1219.3	5.61	0.589E+08	0.01324	24.81	24.58	18.25	20.99	21.47	21.23
14	305.2	32.63	32.61	33.79	33.20	29.15	1232.6									

XP180-10

INPUT ELECTRIC POWER = 1065.0 W HEAT RATE GAINED BY WATER = 1024.3 W HEAT BALANCE ERROR = 3.83%  
 MASS FLOW RATE = 26.3740 G/S PRESSURE DROP = 0.6138 MM H2O FRICTION FACTOR = 0.025247 FREM = 25.2631  
 REM = 1000.6 GRM = 0.99246E+07 UPSTREAM BULK TEMPERATURE = 23.99 DEG C DOWN STREAM BULK TEMPERATURE = 33.30 DEG C  
 PRM = 5.604 RAM = 0.55614E+08 INLET BULK TEMPERATURE = 24.00 DEG C OUTLET BULK TEMPERATURE = 33.29 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	NUSELT NUMBER				AVERAGE			
		A	B	C						A	B	C	T	H	T+H		
0	0.1	26.27	26.28	26.32	26.30	24.00	901.9	6.28	0.436E+08	0.00000	37.50	37.42	36.71	37.08	37.08	37.08	
1	1.5	26.71	26.71	26.83	26.77	24.03	902.5	6.28	0.437E+08	0.00009	31.82	31.75	30.47	31.11	31.13	31.12	
2	2.5	27.01	27.01	27.17	27.09	24.05	902.9	6.27	0.438E+08	0.00015	28.81	28.74	27.28	28.01	28.03	28.02	
3	5.5	27.91	27.92	28.22	28.06	24.11	904.2	6.26	0.439E+08	0.00032	22.45	22.38	20.75	21.55	21.58	21.56	
4	15.5	28.59	28.82	29.03	28.87	24.31	908.3	6.23	0.444E+08	0.00090	19.87	18.87	18.04	18.68	18.71	18.69	
5	25.5	29.09	29.07	29.88	29.48	24.50	912.4	6.20	0.449E+08	0.00148	18.58	18.67	15.85	17.12	17.24	17.18	
6	45.5	29.06	29.02	29.86	29.45	24.90	920.8	6.14	0.459E+08	0.00265	20.48	20.65	17.15	18.70	18.86	18.78	
7	75.5	29.34	29.36	30.13	29.74	25.50	933.7	6.04	0.475E+08	0.00440	22.08	22.01	18.32	20.01	20.18	20.10	
8	105.5	29.75	29.83	30.63	30.21	26.09	947.0	5.95	0.490E+08	0.00616	23.19	22.67	18.68	20.59	20.80	20.69	
9	135.5	30.46	30.52	31.62	31.06	26.69	960.6	5.86	0.506E+08	0.00792	22.44	22.12	17.16	19.39	19.72	19.56	
10	165.2	30.96	31.02	32.26	31.63	27.27	972.8	5.78	0.521E+08	0.00967	22.95	22.61	16.97	19.45	19.87	19.66	
11	205.2	31.59	31.54	32.56	32.06	28.07	988.7	5.68	0.541E+08	0.01203	23.98	24.33	18.82	21.16	21.49	21.32	
12	245.2	32.32	32.36	33.45	32.89	28.86	1005.1	5.58	0.562E+08	0.01439	24.39	24.12	18.37	20.91	21.31	21.11	
13	275.2	32.84	32.88	34.14	33.50	29.45	1017.8	5.50	0.577E+08	0.01617	24.84	24.60	17.98	20.82	21.35	21.08	
14	305.2	33.46	33.38	34.51	33.97	30.05	1030.8	5.43	0.593E+08	0.01795	24.65	25.22	18.85	21.47	21.89	21.68	
15	333.3	34.30	34.34	35.46	34.89	30.61	1043.3	5.36	0.609E+08	0.01962	22.74	22.47	17.30	19.60	19.95	19.78	
16	363.3	34.64	34.75	35.89	35.30	31.20	1057.0	5.28	0.626E+08	0.02141	24.36	23.63	17.87	20.48	20.93	20.71	
17	383.3	35.20	35.06	36.18	35.65	31.60	1066.3	5.23	0.637E+08	0.02260	23.26	24.23	18.30	20.67	21.02	20.84	
18	403.3	35.59	35.54	36.46	36.01	31.99	1075.3	5.18	0.648E+08	0.02380	23.25	23.62	18.74	20.83	21.09	20.96	
19	423.3	35.97	36.01	37.08	36.53	32.39	1083.6	5.14	0.658E+08	0.02501	23.35	23.14	17.84	20.19	20.54	20.37	
20	443.3	36.18	36.26	37.34	36.78	32.79	1092.0	5.09	0.668E+08	0.02623	24.60	24.09	18.36	20.93	21.35	21.14	
21	463.3	36.84	36.62	37.93	37.33	33.18	1100.6	5.05	0.678E+08	0.02745	22.80	24.31	17.57	20.12	20.56	20.34	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	35.32	35.33	36.40	35.86	31.76	1069.6	5.21	0.641E+08	0.02311	23.59	23.53	18.07	20.45	20.82	20.63

XP180-11

INPUT ELECTRIC POWER = 990.0 W HEAT RATE GAINED BY WATER = 965.9 W HEAT BALANCE ERROR = 2.43%  
 MASS FLOW RATE = 20.5980 G/S PRESSURE DROP = 0.4518 MM H2O FRICTION FACTOR = 0.030456 FREM = 24.2909  
 REM = 797.6 GRM = 0.10007E+08 UPSTREAM BULK TEMPERATURE = 23.99 DEG C DOWN STREAM BULK TEMPERATURE = 35.23 DEG C  
 PRM = 5.482 RAM = 0.54855E+08 INLET BULK TEMPERATURE = 24.00 DEG C OUTLET BULK TEMPERATURE = 35.22 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	NUSELT NUMBER				AVERAGE			
		A	B	C						A	B	C	T	H	T+H		
0	0.1	26.31	26.31	26.39	26.35	24.00	704.5	6.28	0.412E+08	0.00000	34.89	34.82	33.68	34.26	34.27	34.26	
1	1.5	26.73	26.74	26.88	26.81	24.04	705.0	6.27	0.412E+08	0.00011	29.83	29.76	28.33	29.04	29.06	29.05	
2	2.5	27.03	27.03	27.21	27.12	24.06	705.4	6.27	0.413E+08	0.00019	27.12	27.05	25.53	26.28	26.31	26.29	
3	5.5	27.91	27.92	28.22	28.06	24.13	706.6	6.26	0.415E+08	0.00041	21.31	21.25	19.68	20.45	20.48	20.47	
4	15.5	28.54	28.76	29.08	28.87	24.37	710.4	6.22	0.420E+08	0.00115	19.28	18.31	17.05	17.88	17.92	17.90	
5	25.5	28.98	28.96	29.82	29.39	24.61	714.4	6.18	0.425E+08	0.00190	18.39	18.49	15.41	16.79	16.92	16.86	
6	45.5	28.94	28.97	29.81	29.38	25.09	722.3	6.11	0.438E+08	0.00339	20.81	20.65	17.01	18.70	18.88	18.79	
7	75.5	29.34	29.36	30.19	29.77	25.81	734.6	5.99	0.455E+08	0.00564	22.65	22.58	18.28	20.22	20.45	20.33	
8	105.5	29.86	30.00	30.74	30.34	26.53	747.4	5.88	0.474E+08	0.00790	23.98	23.02	18.96	20.98	21.23	21.10	
9	135.5	30.74	30.74	31.84	31.29	27.25	759.3	5.78	0.491E+08	0.01016	22.85	22.85	17.35	19.72	20.10	19.91	
10	165.2	31.35	31.57	32.65	32.06	27.96	770.4	5.69	0.508E+08	0.01240	23.48	22.04	16.98	19.44	19.87	19.65	
11	205.2	32.09	32.15	33.17	32.65	28.91	785.9	5.57	0.531E+08	0.01542	25.04	24.57	18.67	21.31	21.74	21.52	
12	245.2	33.15	33.13	34.18	33.66	29.87	802.0	5.45	0.555E+08	0.01846	24.20	24.32	18.42	20.94	21.34	21.14	
13	275.2	33.84	33.82	34.92	34.39	30.59	814.5	5.36	0.574E+08	0.02074	24.34	24.50	18.31	20.93	21.36	21.14	
14	305.2	34.46	34.44	35.46	34.96	31.31	827.5	5.27	0.593E+08	0.02303	25.08	25.26	19.04	21.68	22.10	21.89	
15	333.3	35.47	35.46	36.47	35.97	31.98	839.6	5.18	0.611E+08	0.02519	22.65	22.69	17.60	19.81	20.13	19.97	
16	363.3	36.03	36.08	37.01	36.53	32.70	851.4	5.10	0.628E+08	0.02751	23.67	23.29	18.28	20.55	20.88	20.72	
17	383.3	36.47	36.34	37.40	36.90	33.18	859.4	5.05	0.639E+08	0.02907	23.89	24.90	18.64	21.13	21.52	21.32	
18	403.3	36.93	36.88	37.80	37.35	33.66	867.7	4.99	0.651E+08	0.03064	24.03	24.40	18.97	21.28	21.60	21.44	
19	423.3	37.42	37.45	38.47	37.95	34.14	876.0	4.93	0.663E+08	0.03221	23.93	23.69	18.10	20.57	20.96	20.76	
20	443.3	37.74	37.82	38.84	38.31	34.61	884.5	4.88	0.675E+08	0.03379	25.05	24.44	18.56	21.21	21.66	21.43	
21	463.3	38.46	38.29	39.43	38.91	35.09	893.2	4.82	0.687E+08	0.03537	23.25	24.46	18.05	20.54	20.95	20.75	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	36.68	36.67	37.66	37.17	33.38	863.1	5.02	0.644E+08	0.02973	23.87	23.90	18.36	20.76	21.12	20.94

XP180-12

INPUT ELECTRIC POWER = 855.0 W HEAT RATE GAINED BY WATER = 820.2 W HEAT BALANCE ERROR = 4.07%  
 MASS FLOW RATE = 12.2950 G/S PRESSURE DROP = 0.2677 MM H2O FRICTION FACTOR = 0.050599 FREM = 25.3910  
 REM = 501.8 GRM = 0.10041E+08 UPSTREAM BULK TEMPERATURE = 24.04 DEG C DOWN STREAM BULK TEMPERATURE = 40.04 DEG C  
 PRM = 5.177 RAM = 0.51990E+08 INLET BULK TEMPERATURE = 24.06 DEG C OUTLET BULK TEMPERATURE = 40.03 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	NUSELT NUMBER				AVERAGE		
		A	B	C						A	B	C	T	H	T+H	
0	0.1	26.24	26.25	26.38	26.31	24.07	421.1	6.27	0.351E+08	0.00001	31.38	31.32	29.51	30.40	30.43	30.42
1	1.5	26.64	26.65	26.84	26.74	24.12	421.6	6.26	0.352E+08	0.00019	27.04	26.97	25.07	26.00	26.04	26.02
2	2.5	26.91	26.92	27.16	27.04	24.15	421.9	6.26	0.352E+08	0.00031	24.68	24.62	22.71	23.64	23.68	23.66
3	5.5	27.74	27.75	28.11	27.93	24.25	422.9	6.24	0.354E+08	0.00069	19.57	19.51	17.71	18.58	18.62	18.60
4	15.5	28.32	28.48	28.86	28.63	24.59	426.2	6.19	0.361E+08	0.00193	18.31	17.52	15.97	16.89	16.94	16.91
5	25.5	28.76	28.73	29.60	29.17	24.93	429.6	6.13	0.368E+08	0.00318	17.82	17.93	14.60	16.07	16.24	16.15
6	45.5	28.94	28.86	29.75	29.33	25.61	436.5	6.03	0.382E+08	0.00569	20.43	20.99	16.45	18.33	18.58	18.45
7	75.5	29.67	29.69	30.47	30.07	26.64	447.3	5.86	0.404E+08	0.00947	22.35	22.23	17.72	19.74	20.00	19.87
8	105.5	30.53	30.67	31.36	30.98	27.66	457.0	5.73	0.425E+08	0.01326	23.59	22.46	18.31	20.39	20.67	20.53
9	135.5	31.74	31.68	32.73	32.22	28.68	466.8	5.60	0.446E+08	0.01705	22.08	22.49	16.69	18.08	18.49	18.28
10	165.2	32.74	32.79	33.82	33.29	29.69	476.9	5.47	0.478E+08	0.02083	22.11	21.72	16.34	18.72	19.13	18.92
11	205.2	33.98	33.98	34.78	34.38	31.05	491.1	5.30	0.498E+08	0.02593	22.98	22.93	18.01	20.19	20.48	20.33
12	245.2	35.31	35.30	36.18	35.74	32.42	505.4	5.13	0.527E+08	0.03108	23.13	23.22	17.77	20.12	20.47	20.30
13	275.2	36.12	36.22	37.19	36.68	33.44	515.7	5.02	0.548E+08	0.03500	24.93	24.03	17.78	20.60	21.13	20.86
14	305.2	37.18	3													

XP180-13

INPUT ELECTRIC POWER = 1955.0 W HEAT RATE GAINED BY WATER = 1866.7 W HEAT BALANCE ERROR = 4.52%  
 MASS FLOW RATE = 38.6480 G/S PRESSURE DROP = 0.9003 MM H2O FRICTION FACTOR = 0.017236 FREM = 25.8576  
 REM = 1500.0 GRM = 0.19487E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 35.51 DEG C  
 PRM = 5.468 RAM = 0.10655E+09 INLET BULK TEMPERATURE = 23.94 DEG C OUTLET BULK TEMPERATURE = 35.50 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER						
		A	B	C	AVERAGE						A	B	C	AVERAGE			
0	0.1	27.32	27.24	27.26	27.28	23.95	1320.0	6.29	0.793E+08	0.00000	46.08	47.15	46.64	46.62	46.63	46.62	
1	1.5	28.02	27.91	28.06	28.01	23.98	1321.1	6.28	0.794E+08	0.00006	38.48	39.53	38.12	38.55	38.56	38.56	
2	2.5	28.50	28.37	28.60	28.52	24.01	1321.9	6.28	0.796E+08	0.00010	34.55	35.57	33.85	34.44	34.45	34.45	
3	5.5	29.96	29.76	30.21	30.03	24.08	1324.1	6.27	0.799E+08	0.00022	26.44	27.34	25.34	26.09	26.11	26.10	
4	15.5	30.70	31.13	31.53	31.22	24.33	1331.6	6.23	0.810E+08	0.00061	24.36	22.83	21.54	22.51	22.57	22.54	
5	25.5	31.38	31.34	32.71	32.03	24.57	1339.1	6.19	0.821E+08	0.00101	22.79	22.92	19.08	20.80	20.97	20.88	
6	45.5	31.38	31.41	32.76	32.08	25.07	1354.5	6.11	0.844E+08	0.00181	24.56	24.43	20.14	22.11	22.32	22.21	
7	75.5	31.89	31.79	33.31	32.58	25.81	1378.3	5.99	0.880E+08	0.00301	25.44	25.84	20.62	22.85	23.13	22.99	
8	105.5	32.42	32.68	33.97	33.26	26.55	1402.9	5.88	0.916E+08	0.00421	26.31	25.19	20.80	23.01	23.28	23.14	
9	135.5	33.29	33.24	35.17	34.27	27.29	1425.8	5.78	0.951E+08	0.00541	25.66	25.90	19.55	22.24	22.67	22.45	
10	165.2	33.96	33.96	36.04	35.00	28.02	1447.3	5.68	0.984E+08	0.00661	25.90	25.90	19.18	22.04	22.54	22.29	
11	205.2	34.59	34.43	36.23	35.37	29.00	1477.3	5.56	0.103E+09	0.00822	27.50	28.32	21.26	24.14	24.59	24.36	
12	245.2	35.42	35.36	37.41	36.40	29.99	1508.6	5.43	0.108E+09	0.00984	28.22	28.57	20.65	23.91	24.52	24.22	
13	275.2	36.12	36.16	38.19	37.17	30.73	1533.0	5.34	0.112E+09	0.01106	28.41	28.17	20.50	23.77	24.39	24.08	
14	305.2	36.46	36.44	38.41	37.43	31.47	1558.1	5.25	0.115E+09	0.01228	30.60	30.71	22.01	25.62	26.33	25.98	
15	333.3	37.74	37.81	39.82	38.80	32.16	1580.9	5.16	0.119E+09	0.01343	27.32	27.00	19.91	22.98	23.53	23.26	
16	363.3	38.30	38.36	40.36	39.34	32.90	1603.8	5.08	0.122E+09	0.01467	28.20	27.88	20.42	23.63	24.23	23.93	
17	383.3	38.86	38.63	40.62	39.68	33.40	1619.5	5.02	0.125E+09	0.01551	27.81	29.05	21.05	24.19	24.74	24.46	
18	403.3	39.38	39.28	41.08	40.21	33.89	1635.5	4.96	0.127E+09	0.01634	27.63	28.17	21.10	24.03	24.50	24.27	
19	423.3	39.86	39.90	41.93	40.90	34.38	1651.9	4.91	0.129E+09	0.01718	27.70	27.48	20.09	23.25	23.84	23.55	
20	443.3	40.20	40.28	42.23	41.24	34.87	1668.5	4.85	0.132E+09	0.01802	28.44	28.00	20.58	23.80	24.40	24.10	
21	463.3	41.08	40.79	42.94	41.94	35.37	1685.5	4.79	0.134E+09	0.01887	26.49	27.90	19.96	23.02	23.58	23.30	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	39.06	39.04	41.01	40.03	33.60	1626.7	5.00	0.126E+09	0.01586	27.85	27.93	20.53	23.65	24.21	23.93

XP180-14

INPUT ELECTRIC POWER = 1810.0 W HEAT RATE GAINED BY WATER = 1724.5 W HEAT BALANCE ERROR = 4.72%  
 MASS FLOW RATE = 29.9840 G/S PRESSURE DROP = 0.6922 MM H2O FRICTION FACTOR = 0.022010 FREM = 26.2024  
 REM = 1190.5 GRM = 0.19360E+08 UPSTREAM BULK TEMPERATURE = 23.88 DEG C DOWN STREAM BULK TEMPERATURE = 37.67 DEG C  
 PRM = 5.335 RAM = 0.10329E+09 INLET BULK TEMPERATURE = 23.89 DEG C OUTLET BULK TEMPERATURE = 37.66 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER						
		A	B	C	AVERAGE						A	B	C	AVERAGE			
0	0.1	27.34	27.35	27.38	27.36	23.89	1022.9	6.30	0.730E+08	0.00000	41.64	41.51	41.22	41.40	41.40	41.40	
1	1.5	28.01	28.02	28.18	28.10	23.94	1023.9	6.29	0.732E+08	0.00008	35.26	35.13	33.67	34.52	34.53	34.52	
2	2.5	28.47	28.49	28.73	28.60	23.97	1024.6	6.29	0.735E+08	0.00013	31.89	31.76	30.16	30.97	30.99	30.98	
3	5.5	29.84	29.87	30.38	30.12	24.05	1025.6	6.27	0.737E+08	0.00028	24.78	24.67	22.69	23.67	23.71	23.69	
4	15.5	30.64	31.02	31.59	31.21	24.35	1033.5	6.23	0.749E+08	0.00079	22.78	21.51	19.81	20.90	20.97	20.94	
5	25.5	31.38	31.23	32.82	32.06	24.64	1040.6	6.18	0.762E+08	0.00130	21.26	21.74	17.53	19.31	19.52	19.41	
6	45.5	31.38	31.30	32.82	32.08	25.23	1054.9	6.09	0.787E+08	0.00233	23.28	23.57	18.86	20.90	21.14	21.02	
7	75.5	31.94	31.85	33.31	32.60	26.11	1077.1	5.95	0.826E+08	0.00388	24.48	24.88	19.83	21.99	22.26	22.13	
8	105.5	32.53	32.73	34.08	33.36	26.99	1099.6	5.81	0.866E+08	0.00543	25.74	24.82	20.10	22.39	22.69	22.54	
9	135.5	33.41	33.41	35.45	34.43	27.87	1119.5	5.70	0.903E+08	0.00698	25.71	25.71	18.77	21.70	22.24	21.97	
10	165.2	34.23	34.29	36.27	35.27	28.74	1139.9	5.59	0.941E+08	0.00853	25.86	25.58	18.87	21.77	22.30	22.03	
11	205.2	34.98	34.98	36.62	35.80	29.92	1168.6	5.44	0.993E+08	0.01061	27.99	27.95	21.14	24.08	24.55	24.32	
12	245.2	36.25	36.19	38.13	37.18	31.09	1198.8	5.30	0.105E+09	0.01271	27.38	27.68	20.06	23.21	23.80	23.50	
13	275.2	36.95	36.99	39.03	38.00	31.97	1222.0	5.19	0.109E+09	0.01429	28.32	28.07	19.98	23.39	24.09	23.74	
14	305.2	37.68	37.51	39.42	38.50	32.85	1243.1	5.08	0.113E+09	0.01589	29.15	30.23	21.43	24.89	25.56	25.23	
15	333.3	38.97	38.93	40.94	39.94	33.68	1263.6	4.99	0.116E+09	0.01740	26.54	26.74	19.33	22.40	22.98	22.69	
16	363.3	39.63	39.75	41.53	40.61	34.56	1286.2	4.89	0.120E+09	0.01902	27.63	26.98	20.09	23.15	23.70	23.42	
17	383.3	40.19	40.08	41.94	41.04	35.15	1301.7	4.82	0.123E+09	0.02011	27.70	28.35	20.57	23.72	24.30	24.01	
18	403.3	40.89	40.84	42.53	41.70	35.73	1317.6	4.75	0.126E+09	0.02120	27.08	27.35	20.55	23.42	23.88	23.65	
19	423.3	41.30	41.34	43.49	42.41	36.32	1333.9	4.68	0.128E+09	0.02230	28.00	27.75	19.45	22.91	23.66	23.29	
20	443.3	41.81	41.91	43.79	42.83	36.91	1350.4	4.61	0.131E+09	0.02340	28.39	27.86	20.21	23.52	24.17	23.85	
21	463.3	42.75	42.44	44.67	43.63	37.50	1365.1	4.56	0.134E+09	0.02448	26.46	28.13	19.37	22.65	23.33	22.99	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	40.46	40.47	42.37	41.42	35.39	1308.9	4.79	0.124E+09	0.02057	27.56	27.50	20.03	23.19	23.78	23.48

XP180-15

INPUT ELECTRIC POWER = 1690.0 W HEAT RATE GAINED BY WATER = 1588.3 W HEAT BALANCE ERROR = 6.02%  
 MASS FLOW RATE = 24.5690 G/S PRESSURE DROP = 0.5627 MM H2O FRICTION FACTOR = 0.026639 FREM = 26.5134  
 REM = 995.3 GRM = 0.18993E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 39.43 DEG C  
 PRM = 5.221 RAM = 0.99155E+08 INLET BULK TEMPERATURE = 23.95 DEG C OUTLET BULK TEMPERATURE = 39.42 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER					
		A	B	C	AVERAGE						A	B	C	AVERAGE		
0	0.1	27.21	27.20	27.31	27.26	23.95	839.3	6.29	0.675E+08	0.00000	40.60	40.74	39.36	40.00	40.01	40.01
1	1.5	27.87	27.85	28.08	27.97	24.00	840.2	6.28	0.677E+08	0.00009	34.20	34.33	32.38	33.30	33.32	33.31
2	2.5	28.32	28.30	28.61	28.46	24.03	840.8	6.28	0.678E+08	0.00016	30.84	30.98	28.85	29.84	29.88	29.86
3	5.5	29.68	29.65	30.21	29.94	24.13	842.7	6.26	0.682E+08	0.00034	23.83	23.95	21.74	22.76	22.81	22.79
4	15.5	30.42	30.73	31.37	30.97	24.46	849.1	6.21	0.695E+08	0.00097	22.16	21.06	19.13	20.28	20.37	20.33
5	25.5	31.10	31.07	32.54	31.81	24.79	855.6	6.16	0.708E+08	0.00159	20.92	21.03	17.03	18.80	19.00	18.90
6	45.5	31.21	31.24	32.65	31.94	25.45	868.9	6.05	0.734E+08	0.00285	22.88	22.75	18.31	20.31	20.56	20.44
7	75.5	31.89	31.74	33.26	32.53	26.44	889.6	5.89	0.775E+08	0.00474	24.14	24.82	19.29	21.58	21.89	21.73
8	105.5	32.47	32.62	34.03	33.29	27.43	909.1	5.76	0.815E+08	0.00663	26.02	25.27	19.89	22.40	22.77	22.58
9	135.5	33.68	33.63	35.51	34.58	28.42	927.8	5.63	0.853E+08	0.00853	24.88	25.14	18.48	21.25	21.74	21.50
10	165.2	34.57	34.58	36.66	35.61	29.40	947.1	5.51	0.893E+08	0.01042	25.29	25.23	18.00	21.02	21.63	21.33
11	205.2	35.64	35.65	37.29	36.47	30.72	974.4	5.34	0.949E+08	0.01297	26.45	26.42	19.83	22.66	23.13	22.90
12	245.2	36.86	36.86	38.63	37.75	32.04	1002.7	5.18	0.101E+09	0.01554	26.94	26.94	19.69	22.75	23.32	23.03
13	275.2	37.73	37.72	39.69	38.71	33.03	1022.2	5.06	0.105E+09	0.01749	27.58	27.64	19.43	22.81	23.52	23.17
14																

XP180-16

INPUT ELECTRIC POWER = 1550.0 W HEAT RATE GAINED BY WATER = 1472.4 W HEAT BALANCE ERROR = 5.00%  
 MASS FLOW RATE = 19.1540 G/S PRESSURE DROP = 0.4297 MM H2O FRICTION FACTOR = 0.033450 FREM = 26.7208  
 REM = 798.8 GRM = 0.19282E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 42.37 DEG C  
 PRM = 5.049 RAM = 0.97349E+08 INLET BULK TEMPERATURE = 23.95 DEG C OUTLET BULK TEMPERATURE = 42.35 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	27.16	27.15	27.33	27.25	23.96	654.4	6.29	0.626E+08	0.00000	38.20	38.33	36.28	37.25	37.27	37.26
1	1.5	27.80	27.79	28.07	27.93	24.01	655.2	6.28	0.628E+08	0.00012	32.33	32.46	30.21	31.26	31.30	31.28
2	2.5	28.24	28.23	28.58	28.41	24.05	655.8	6.27	0.629E+08	0.00020	29.23	29.36	27.08	28.14	28.19	28.17
3	5.5	29.57	29.54	30.10	29.83	24.17	657.6	6.25	0.633E+08	0.00044	22.70	22.82	20.66	21.66	21.71	21.69
4	15.5	30.26	30.57	31.20	30.81	24.56	663.5	6.19	0.648E+08	0.00124	21.50	20.39	18.44	19.61	19.69	19.65
5	25.5	30.93	30.96	32.37	31.66	24.95	669.6	6.13	0.662E+08	0.00204	20.46	20.38	16.49	18.24	18.45	18.35
6	45.5	31.05	30.97	32.48	31.75	25.74	682.0	6.01	0.691E+08	0.00365	22.99	23.36	18.10	20.33	20.64	20.48
7	75.5	31.78	31.79	33.14	32.47	26.92	701.4	5.82	0.737E+08	0.00608	25.05	24.96	19.55	21.94	22.28	22.11
8	105.5	32.69	32.90	34.19	33.50	28.09	718.4	5.67	0.779E+08	0.00851	26.40	25.27	19.91	22.48	22.87	22.68
9	135.5	33.96	34.02	35.84	34.91	29.27	736.3	5.52	0.823E+08	0.01096	25.83	25.52	18.44	21.46	22.06	21.76
10	165.2	35.18	35.30	37.10	36.17	30.44	754.9	5.38	0.868E+08	0.01338	25.48	24.84	18.13	21.07	21.64	21.36
11	205.2	36.48	36.54	38.01	37.26	32.01	781.1	5.18	0.932E+08	0.01668	26.92	26.56	20.05	22.91	23.39	23.15
12	245.2	37.97	37.97	39.64	38.80	33.58	805.6	5.00	0.989E+08	0.02003	27.31	27.27	19.77	22.93	23.53	23.23
13	275.2	39.06	39.17	40.86	39.99	34.76	824.9	4.86	0.103E+09	0.02257	27.75	27.09	19.57	22.84	23.49	23.17
14	305.2	40.23	40.12	41.70	40.94	35.93	845.2	4.73	0.108E+09	0.02513	27.70	28.44	20.66	23.80	24.36	24.08
15	333.3	41.57	41.67	43.46	42.54	37.04	864.6	4.60	0.113E+09	0.02755	26.19	25.62	18.51	21.59	22.21	21.90
16	363.3	42.62	42.75	44.37	43.53	38.21	883.9	4.45	0.117E+09	0.03008	26.89	26.13	19.25	22.30	22.86	22.59
17	383.3	43.47	43.32	45.11	44.25	39.00	897.2	4.42	0.121E+09	0.03178	26.46	27.37	19.38	22.53	23.15	22.84
18	403.3	44.23	44.19	45.76	44.99	39.78	910.9	4.35	0.124E+09	0.03348	26.56	26.83	19.77	22.72	23.33	22.97
19	423.3	45.07	45.13	46.84	45.97	40.57	925.0	4.28	0.127E+09	0.03519	26.23	25.88	18.82	21.85	22.44	22.15
20	443.3	45.77	45.88	47.46	46.64	41.35	939.6	4.21	0.131E+09	0.03690	26.66	26.04	19.29	22.28	22.82	22.55
21	463.3	46.37	46.73	48.46	47.63	42.14	953.8	4.14	0.135E+09	0.03862	24.85	25.64	18.62	21.43	21.93	21.68
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6		43.79	43.82	45.50	44.65	39.33	903.5	4.39	0.122E+09	0.03250	26.50	26.31	19.17	22.21	22.79	22.50

XP180-17

INPUT ELECTRIC POWER = 3380.0 W HEAT RATE GAINED BY WATER = 3240.2 W HEAT BALANCE ERROR = 4.14%  
 MASS FLOW RATE = 34.3160 G/S PRESSURE DROP = 0.8201 MM H2O FRICTION FACTOR = 0.019873 FREM = 29.6713  
 REM = 1493.0 GRM = 0.48256E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 46.57 DEG C  
 PRM = 4.805 RAM = 0.23188E+09 INLET BULK TEMPERATURE = 23.95 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	29.48	29.46	29.79	29.63	23.96	1172.4	6.29	0.138E+09	0.00000	48.84	49.04	46.21	47.53	47.57	47.55
1	1.5	30.61	30.58	31.13	30.86	24.03	1174.2	6.28	0.138E+09	0.00007	40.97	41.16	37.99	39.47	39.53	39.50
2	2.5	31.39	31.35	32.04	31.71	24.07	1175.5	6.27	0.139E+09	0.00011	36.87	37.06	33.84	35.33	35.40	35.36
3	5.5	33.73	33.67	34.80	34.25	24.22	1192.4	6.25	0.140E+09	0.00025	29.35	28.52	25.48	26.87	26.96	26.92
4	15.5	34.63	35.12	36.32	35.60	24.70	1192.6	6.17	0.144E+09	0.00069	27.11	25.84	23.17	24.71	24.82	24.77
5	25.5	35.63	35.62	38.31	36.96	25.18	1206.0	6.09	0.148E+09	0.00114	25.75	25.78	20.50	22.83	23.13	22.98
6	45.5	35.59	35.53	38.22	36.89	26.15	1233.8	5.94	0.156E+09	0.00204	28.43	28.61	22.23	24.98	25.37	25.18
7	75.5	36.71	36.46	39.33	37.95	27.59	1274.0	5.74	0.167E+09	0.00340	29.36	30.19	22.80	25.82	26.28	26.05
8	105.5	37.65	38.08	40.31	39.09	29.04	1312.7	5.55	0.179E+09	0.00476	30.97	29.50	23.66	26.54	26.94	26.74
9	135.5	38.96	38.96	42.49	40.73	30.49	1353.9	5.37	0.192E+09	0.00613	31.36	31.36	22.13	25.95	26.74	26.35
10	165.2	40.22	40.08	43.83	41.99	31.92	1397.1	5.19	0.204E+09	0.00749	31.90	32.46	22.23	26.29	27.21	26.75
11	205.2	41.14	41.21	44.46	42.82	33.85	1451.0	4.97	0.220E+09	0.00936	36.12	35.82	24.83	29.38	30.40	29.89
12	245.2	43.07	43.08	46.72	44.90	35.77	1509.2	4.74	0.236E+09	0.01126	35.97	35.89	23.97	28.75	29.95	29.35
13	275.2	44.56	44.51	48.31	46.42	37.22	1554.4	4.59	0.249E+09	0.01270	35.61	35.88	23.57	28.41	29.66	29.03
14	305.2	45.67	45.52	49.01	47.30	38.67	1597.2	4.45	0.262E+09	0.01412	37.21	38.02	25.21	30.19	31.41	30.80
15	333.3	47.79	47.82	51.57	49.69	40.02	1639.6	4.33	0.275E+09	0.01545	33.47	33.33	22.52	26.90	27.96	27.43
16	363.3	48.88	48.97	52.41	50.66	41.47	1687.3	4.20	0.289E+09	0.01688	34.99	34.58	23.70	28.19	29.24	28.72
17	383.3	49.91	49.58	53.14	51.44	42.43	1717.5	4.11	0.299E+09	0.01784	34.60	36.22	24.16	28.72	29.78	29.25
18	403.3	50.92	50.60	53.94	52.35	43.40	1746.6	4.04	0.308E+09	0.01880	34.32	35.86	24.49	28.84	29.79	29.32
19	423.3	51.72	51.69	55.42	53.57	44.36	1776.7	3.96	0.317E+09	0.01977	35.03	35.16	23.31	28.01	29.20	28.61
20	443.3	52.41	52.54	56.03	54.25	45.32	1807.9	3.89	0.327E+09	0.02074	36.35	35.68	24.05	28.84	30.03	29.43
21	463.3	54.00	53.51	57.37	55.56	46.29	1840.2	3.81	0.336E+09	0.02172	33.32	35.60	23.19	27.71	28.83	28.27
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																
391.6		50.27	50.20	53.75	51.99	42.83	1729.3	4.09	0.302E+09	0.01825	34.79	35.14	23.71	28.25	29.34	28.79

XP180-18

INPUT ELECTRIC POWER = 3045.0 W HEAT RATE GAINED BY WATER = 2867.2 W HEAT BALANCE ERROR = 5.84%  
 MASS FLOW RATE = 26.3740 G/S PRESSURE DROP = 0.6343 MM H2O FRICTION FACTOR = 0.026003 FREM = 30.9210  
 REM = 1189.1 GRM = 0.47438E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 49.99 DEG C  
 PRM = 4.609 RAM = 0.21864E+09 INLET BULK TEMPERATURE = 23.96 DEG C

STA-TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-			TB (C)	RE	PR	RA+	Z+	NUSSLETT NUMBER			AVERAGE			
		A	B	C						A	B	C	T	H	T+H	
0	0.1	29.20	29.30	29.64	29.47	23.96	901.1	6.29	0.122E+09	0.00000	44.68	44.70	42.03	43.32	43.36	43.34
1	1.5	30.36	30.36	30.91	30.64	24.04	902.8	6.27	0.122E+09	0.00009	37.76	37.78	34.75	36.20	36.26	36.23
2	2.5	31.09	31.09	31.79	31.44	24.10	903.9	6.26	0.123E+09	0.00015	34.11	34.13	31.04	32.51	32.58	32.54
3	5.5	33.28	33.28	34.41	33.85	24.26	907.4	6.24	0.124E+09	0.00032	26.45	26.46	23.51	24.89	24.98	24.94
4	15.5	34.02	34.62	35.93	35.12	24.82	919.1	6.15	0.128E+09	0.00090	25.88	24.31	21.44	23.12	23.27	23.19
5	25.5	35.12	35.06	37.81	36.45	25.37	931.0	6.06	0.132E+09	0.00149	24.41	24.56	19.14	21.48	21.81	21.65
6	45.5	35.21	35.03	37.78	36.45	26.48	956.0	5.89	0.140E+09	0.00266	27.21	27.78	21.01	23.82	24.25	24.04
7	75.5	36.38	36.18	38.99	37.64	28.15	990.4	5.67	0.152E+09	0.00443	28.73	29.45	21.80	24.92	25.44	25.18
8	105.5	37.48	37.80	40.09	38.87	29.81	1025.6	5.46	0.164E+09	0.00620	30.71	29.49	22.92	26.02	26.51	26.27
9	135.5	39.02	39.18	42.44	40.77	31.48	1063.5	5.25	0.177E+09	0.00799	31.13	30.46	21.41	25.26	26.10	25.68
10	165.2	40.56	40.63	44.06	42.33	33.13	1099.4	5.05	0.189E+09	0.00978	31.46	31.13	21.38	25.41	26.34	25.87
11	205.2	42.14	42.32	45.13	43.68	35.35	1149.7	4.79	0.206E+09	0.01225	34.19	33.34	23.76	27.89	28.76	28.33
12	245.2	44.46	44.47	47.72	46.09	37.57	1202.4	4.55	0.223E+09	0.01473	33.56	33.48	22.76	27.11	28.14	27.63
13	275.2	46.01	46.06	49.48	47.76	39.23	1241.0	4.40	0.237E+09	0.01658	34.02	33.75	22.49	27.04	28.19	27.61
14	305.2	47.45	47.08	50.												

XP180-19

INPUT ELECTRIC POWER = 2750.0 W HEAT RATE GAINED BY WATER = 2632.6 W HEAT BALANCE ERROR = 4.27%  
 MASS FLOW RATE = 21.3200 G/S PRESSURE DROP = 0.5081 MM H2O FRICTION FACTOR = 0.031853 FREM = 31.6529  
 REM = 993.7 GRM = 0.48066E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 53.53 DEG C  
 PRM = 4.446 RAM = 0.21371E+09 INLET BULK TEMPERATURE = 23.96 DEG C OUTLET BULK TEMPERATURE = 53.52 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA+	Z+	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C	AVERAGE						T	H	T+H				
0	0.1	29.07	29.03	29.42	29.24	23.97	728.5	6.29	0.112E+09	0.00000	42.92	43.31	40.16	41.58	41.63	41.61	
1	1.5	30.06	30.00	30.60	30.32	24.06	730.1	6.27	0.113E+09	0.00011	36.50	36.90	33.47	35.01	35.09	35.05	
2	2.5	30.74	30.66	31.42	31.06	24.12	731.1	6.26	0.113E+09	0.00018	33.09	33.49	30.02	31.57	31.65	31.61	
3	5.5	32.78	32.67	33.86	33.29	24.31	734.3	6.23	0.114E+09	0.00040	25.84	26.20	22.93	24.38	24.47	24.43	
4	15.5	33.42	34.00	35.37	34.54	24.94	745.0	6.13	0.118E+09	0.00112	25.80	24.14	20.96	22.78	22.96	22.87	
5	25.5	34.51	34.51	37.08	35.80	25.57	756.1	6.03	0.122E+09	0.00184	24.42	24.44	18.97	21.35	21.70	21.53	
6	45.5	34.76	34.69	37.17	35.95	26.83	775.3	5.83	0.131E+09	0.00329	27.46	27.69	21.07	23.88	24.32	24.10	
7	75.5	36.16	35.96	38.44	37.25	28.72	810.2	5.69	0.143E+09	0.00548	29.16	29.97	22.31	25.43	25.94	25.68	
8	105.5	37.43	37.74	39.92	38.75	30.61	843.5	5.36	0.157E+09	0.00768	31.68	30.27	23.19	26.52	27.08	26.80	
9	135.5	39.41	39.41	42.49	40.95	32.50	877.9	5.12	0.170E+09	0.00991	31.14	31.14	21.51	25.44	26.33	25.89	
10	165.2	41.06	41.19	44.39	42.76	34.38	911.2	4.91	0.182E+09	0.01215	32.02	31.39	21.36	25.52	26.53	26.03	
11	205.2	43.03	43.26	45.97	44.56	36.90	960.0	4.62	0.200E+09	0.01523	34.64	33.40	23.43	27.75	28.72	28.24	
12	245.2	45.62	45.64	48.56	47.09	39.42	1006.8	4.38	0.219E+09	0.01828	34.13	34.01	23.13	27.56	28.60	28.08	
13	275.2	47.45	47.57	50.59	49.05	41.31	1045.0	4.21	0.234E+09	0.02058	34.33	33.69	22.71	27.23	28.36	27.80	
14	305.2	49.01	48.81	51.79	50.35	43.20	1081.4	4.05	0.249E+09	0.02290	36.18	37.47	24.45	29.29	30.64	30.01	
15	333.3	51.39	51.46	54.58	53.00	44.97	1116.1	3.92	0.262E+09	0.02508	32.59	32.29	21.78	26.06	27.11	26.58	
16	363.3	52.81	52.91	55.86	54.36	46.86	1155.6	3.77	0.278E+09	0.02744	35.07	34.49	23.18	27.82	28.98	28.40	
17	383.3	54.19	53.88	56.97	55.50	48.13	1180.9	3.68	0.288E+09	0.02901	34.32	36.18	23.52	28.21	29.39	28.80	
18	403.3	55.49	55.23	58.12	56.74	49.39	1207.3	3.59	0.299E+09	0.03059	34.01	35.52	23.78	28.23	29.27	28.75	
19	423.3	56.66	56.71	59.77	58.23	50.65	1234.9	3.50	0.310E+09	0.03218	34.50	34.22	22.72	27.35	28.54	27.95	
20	443.3	57.64	57.79	60.71	59.21	51.91	1263.6	3.42	0.322E+09	0.03377	36.07	35.15	23.52	28.32	29.56	28.94	
21	463.3	59.57	59.08	62.43	60.88	53.17	1287.6	3.35	0.332E+09	0.03534	32.28	34.95	22.29	26.79	27.95	27.37	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	54.70	54.66	57.67	56.18	48.65	1193.1	3.65	0.293E+09	0.02968	34.43	34.64	23.08	27.67	28.81	28.24

XP180-20

INPUT ELECTRIC POWER = 2430.0 W HEAT RATE GAINED BY WATER = 2319.9 W HEAT BALANCE ERROR = 4.53%  
 MASS FLOW RATE = 16.2660 G/S PRESSURE DROP = 0.3836 MM H2O FRICTION FACTOR = 0.041276 FREM = 32.7260  
 REM = 792.9 GRM = 0.48198E+08 UPSTREAM BULK TEMPERATURE = 23.93 DEG C DOWN STREAM BULK TEMPERATURE = 58.12 DEG C  
 PRM = 4.235 RAM = 0.20412E+09 INLET BULK TEMPERATURE = 23.97 DEG C OUTLET BULK TEMPERATURE = 56.10 DEG C

STA- TION NO.	Z CM	-WALL TEMPERATURE (DEG C)-				TB (C)	RE	PR	RA+	Z+	----- NUSSELT NUMBER -----			----- AVERAGE -----			
		A	B	C	AVERAGE						T	H	T+H				
0	0.1	28.86	28.85	29.25	29.05	23.97	555.9	6.28	0.987E+08	0.00000	39.55	39.60	36.61	38.03	38.09	38.06	
1	1.5	29.75	29.75	30.36	30.05	24.08	557.3	6.27	0.993E+08	0.00014	34.02	34.07	30.75	32.31	32.40	32.35	
2	2.5	30.37	30.36	31.12	30.75	24.15	558.2	6.26	0.997E+08	0.00024	31.02	31.08	27.69	29.27	29.37	29.32	
3	5.5	32.23	32.22	33.42	32.82	24.37	561.0	6.22	0.101E+09	0.00052	24.54	24.59	21.33	22.83	22.94	22.89	
4	15.5	32.86	33.32	34.82	33.95	25.10	570.5	6.11	0.105E+09	0.00146	24.81	23.42	19.82	21.75	21.97	21.86	
5	25.5	34.12	34.17	36.47	35.31	25.83	580.4	5.99	0.109E+09	0.00241	23.19	23.04	18.06	20.28	20.59	20.43	
6	45.5	34.48	34.19	36.77	35.56	27.28	600.0	5.78	0.118E+09	0.00432	26.61	27.73	20.20	23.16	23.68	23.42	
7	75.5	36.04	36.07	38.44	37.25	29.47	627.9	5.50	0.131E+09	0.00719	28.99	28.89	21.25	24.51	25.10	24.80	
8	105.5	37.87	38.24	40.20	39.13	31.65	658.4	5.23	0.145E+09	0.01009	30.50	28.78	22.19	25.37	25.92	25.64	
9	135.5	40.07	40.24	42.99	41.58	33.83	687.6	4.97	0.157E+09	0.01304	30.25	29.46	20.60	24.38	25.23	24.80	
10	165.2	42.17	42.24	45.12	43.66	36.00	718.7	4.72	0.171E+09	0.01602	30.41	30.04	20.58	24.48	25.40	24.94	
11	205.2	44.65	44.71	47.13	45.91	38.91	760.6	4.43	0.189E+09	0.02405	32.49	32.18	22.68	26.66	27.50	27.08	
12	245.2	47.61	47.59	50.29	48.94	41.82	805.5	4.16	0.210E+09	0.02710	32.03	32.16	21.90	26.04	27.00	26.52	
13	275.2	49.56	49.62	52.53	51.06	44.00	836.8	3.99	0.224E+09	0.02710	33.28	32.88	21.66	26.18	27.37	26.78	
14	305.2	51.67	51.48	54.19	52.88	46.19	870.6	3.82	0.240E+09	0.03017	33.58	34.78	23.00	27.49	28.59	28.04	
15	333.3	54.23	54.19	57.16	55.68	48.23	902.6	3.67	0.255E+09	0.03307	30.59	30.76	20.55	24.61	25.61	25.11	
16	363.3	55.92	56.08	58.76	57.38	50.42	938.3	3.52	0.272E+09	0.03618	33.23	32.29	21.90	26.25	27.33	26.79	
17	383.3	57.63	57.23	60.24	58.84	51.87	963.6	3.42	0.283E+09	0.03828	31.69	34.06	21.78	26.19	27.33	26.76	
18	403.3	59.01	58.80	61.41	60.15	53.33	984.8	3.34	0.294E+09	0.04033	32.05	33.26	22.53	26.66	27.59	27.12	
19	423.3	60.37	60.54	63.28	61.87	54.79	1006.9	3.26	0.305E+09	0.04240	32.49	31.52	21.36	25.62	26.68	26.15	
20	443.3	61.71	60.37	64.48	62.76	56.24	1030.1	3.18	0.316E+09	0.04448	33.11	43.91	21.97	27.78	30.24	29.01	
21	463.3	63.52	61.15	66.28	64.31	57.70	1053.2	3.10	0.327E+09	0.04662	31.02	52.34	21.06	27.33	31.37	29.35	
AVERAGE VALUES THROUGH STATIONS 15 TO 20:																	
		391.6	58.14	57.87	60.89	59.45	52.48	971.0	3.40	0.287E+09	0.03912	32.20	34.30	21.68	26.19	27.47	26.83

Appendix 5  
Fully Developed Nusselt Numbers,  
Experimental Data

Appendix 5.1 Fully developed Nusselt numbers for  $\alpha = 45^\circ$ .

$\alpha = 45^\circ$			
$Gr^+$	$Pr$	$Re$	$Nu/Nu_o$
2.684x10 <sup>6</sup>	7.56	1541.0	3.28
2.724x10 <sup>6</sup>	7.22	512.6	3.52
2.755x10 <sup>6</sup>	7.44	1029.1	3.37
2.864x10 <sup>6</sup>	7.14	517.8	3.60
5.197x10 <sup>6</sup>	7.32	1560.0	3.68
5.570x10 <sup>6</sup>	7.12	1048.2	3.84
6.050x10 <sup>6</sup>	6.67	526.7	3.94
1.412x10 <sup>7</sup>	6.40	1056.8	4.51
1.430x10 <sup>7</sup>	6.82	1594.4	4.58
1.477x10 <sup>7</sup>	5.20	533.5	4.34
1.623x10 <sup>7</sup>	5.88	564.5	4.42
2.717x10 <sup>7</sup>	5.49	1578.0	5.08
2.768x10 <sup>7</sup>	5.24	1064.3	5.16
2.809x10 <sup>7</sup>	5.22	1068.6	5.01
2.958x10 <sup>7</sup>	5.02	863.1	5.08
3.232x10 <sup>7</sup>	4.58	558.4	4.77
3.344x10 <sup>7</sup>	4.95	567.6	4.96
5.707x10 <sup>7</sup>	5.01	1621.9	5.75
5.938x10 <sup>7</sup>	4.66	1115.6	5.84
6.078x10 <sup>7</sup>	4.78	1311.4	5.69
6.089x10 <sup>7</sup>	4.62	1106.9	5.53
6.142x10 <sup>7</sup>	4.43	897.0	5.46
7.112x10 <sup>7</sup>	4.96	1148.8	6.01
7.172x10 <sup>7</sup>	3.89	591.1	5.41
1.686x10 <sup>8</sup>	4.09	1726.9	6.93
1.785x10 <sup>8</sup>	3.85	1405.4	6.88
1.827x10 <sup>8</sup>	3.66	1190.2	6.68
1.833x10 <sup>8</sup>	3.66	1190.1	6.64
1.972x10 <sup>8</sup>	3.39	973.0	6.48
2.135x10 <sup>8</sup>	3.79	1230.6	6.99

Appendix 5.2 Fully developed Nusselt numbers for  $\alpha = 90^\circ$ .

$\alpha = 90^\circ$			
$Gr^+$	$Pr$	$Re$	$Nu/Nu_o$
$2.508 \times 10^6$	6.37	1786.9	3.52
$2.647 \times 10^6$	6.29	998.9	3.64
$2.647 \times 10^6$	6.35	1406.7	3.51
$2.764 \times 10^6$	7.40	757.5	3.67
$2.870 \times 10^6$	6.38	1790.0	3.66
$2.880 \times 10^6$	7.19	514.3	3.79
$5.198 \times 10^6$	6.21	1434.4	3.95
$5.292 \times 10^6$	6.10	1026.8	4.00
$1.360 \times 10^7$	5.90	1037.5	4.90
$1.422 \times 10^7$	6.89	1590.0	5.14
$1.522 \times 10^7$	5.44	550.4	4.49
$1.535 \times 10^7$	6.52	1087.4	5.08
$1.687 \times 10^7$	5.85	567.4	4.92
$2.884 \times 10^7$	5.74	1595.8	5.65
$2.885 \times 10^7$	5.49	1076.2	5.42
$3.275 \times 10^7$	6.21	1635.4	5.72
$3.576 \times 10^7$	5.72	1063.8	5.60
$4.049 \times 10^7$	4.96	597.9	5.58
$1.195 \times 10^8$	4.34	1635.2	6.92
$1.377 \times 10^8$	4.27	1592.7	7.24
$1.870 \times 10^8$	4.13	1784.8	7.69
$1.965 \times 10^8$	4.26	1842.7	7.85
$1.983 \times 10^8$	3.72	1211.6	7.41
$2.036 \times 10^8$	3.67	1225.6	7.49

Appendix 5.3 Fully developed Nusselt numbers for  $\alpha = 135^\circ$ .

$\alpha = 135^\circ$			
$Gr^+$	$Pr$	$Re$	$Nu/Nu_o$
$2.653 \times 10^6$	5.97	505.0	3.66
$5.368 \times 10^6$	5.97	1022.1	3.91
$5.432 \times 10^6$	6.07	1515.7	3.97
$5.513 \times 10^6$	5.73	523.9	3.81
$1.314 \times 10^7$	5.81	1551.2	4.54
$1.341 \times 10^7$	5.64	1048.8	4.49
$1.456 \times 10^7$	5.21	543.0	4.55
$2.738 \times 10^7$	5.49	1577.7	5.50
$2.780 \times 10^7$	5.35	1273.8	5.22
$2.853 \times 10^7$	5.22	1069.3	5.53
$2.929 \times 10^7$	5.03	861.4	5.38
$3.143 \times 10^7$	4.60	556.1	5.25
$5.810 \times 10^7$	5.00	1626.4	6.27
$6.078 \times 10^7$	4.78	1311.4	6.24
$6.215 \times 10^7$	4.60	1111.1	6.09
$6.663 \times 10^7$	4.36	909.2	6.09
$7.309 \times 10^7$	3.88	593.4	6.14
$1.718 \times 10^8$	4.08	1730.8	7.60
$1.801 \times 10^8$	3.84	1407.4	7.48
$1.862 \times 10^8$	3.64	1194.0	7.29
$1.972 \times 10^8$	3.39	972.7	7.04

**Appendix 5.4 Fully developed Nusselt numbers for  $\alpha = 180^\circ$ .**

$\alpha = 180^\circ$			
$Gr^+$	$Pr$	$Re$	$Nu/Nu_o$
$2.864 \times 10^6$	5.95	506.7	3.49
$5.090 \times 10^6$	5.98	1019.9	3.78
$5.178 \times 10^6$	6.10	1510.1	3.89
$5.343 \times 10^6$	5.74	523.5	3.75
$1.290 \times 10^7$	5.81	1549.6	4.42
$1.316 \times 10^7$	5.66	1046.6	4.40
$1.454 \times 10^7$	5.20	543.7	4.54
$2.717 \times 10^7$	5.49	1578.0	5.13
$2.784 \times 10^7$	5.35	1272.9	5.17
$2.837 \times 10^7$	5.21	1069.6	5.05
$2.958 \times 10^7$	5.02	863.1	5.12
$3.197 \times 10^7$	4.58	558.7	4.95
$5.810 \times 10^7$	5.00	1626.7	5.85
$5.969 \times 10^7$	4.79	1308.9	5.74
$6.026 \times 10^7$	4.63	1104.8	5.50
$6.408 \times 10^7$	4.39	903.5	5.50
$1.703 \times 10^8$	4.09	1729.3	7.04
$1.750 \times 10^8$	3.86	1399.9	6.73
$1.851 \times 10^8$	3.65	1193.1	6.91
$1.946 \times 10^8$	3.40	971.0	6.56