EXPERIMENTAL STUDY OF EFFECTS OF ROUGHNESS
ON A SEPARATION BUBBLE
by
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#### Abstract

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BY

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A thesis submitted to the Faculty of Graduate Studies of the University of Manitoba in partial fulfillment of the requirements of the degree of

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#### Abstract

It was desired to investigate the flow within and behind a twodimensional laminar separation bubble and observe the effects of increasing the surface roughness ahead of the bubble. To this end, various grades of abrasive were attached to the leading edge of an airfoil with an elliptical nose. Measurements were made of surface pressure, streamwise mean velocity, and turbulence intensity. All tests were performed at a single Reynolds number, namely $\operatorname{Re}=2.4 \mathrm{x}$ $10^{4}$ based on the airfoil thickness and freestream velocity. The roughness was gradually made coarser until the separation bubble was eliminated. Increasing the roughness significantly beyond the grade which removed the bubble produced a flow that appeared to be much like that downstream of a backward-facing step. Data connected with the separation bubble was very similar to observations made by other researchers. Specifically, present findings exhibited several characteristics like those found by Bradshaw \& Wong (1972) and Chandrsuda \& Bradshaw (1981) regarding the reattachment and relaxation of a turbulent boundary layer.


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| C | arbitrary constant |
| :---: | :---: |
| $C_{r}$ | skin friction coefficient [ $\left.\tau_{w} /\left({ }^{1} / z p \bar{U}_{r}{ }^{2}\right)\right]$ |
| $\mathrm{C}_{\mathrm{p}}$ | pressure coefficient [ $\left.\left(\mathrm{p}-\mathrm{p}_{x}\right) /\left({ }^{1} / 2 p \bar{U}_{x}{ }^{2}\right)\right]$ |
| $\mathrm{C}_{\text {Prinin }}$ | minimum pressure coefficient in separation bubble |
| $\widetilde{C}_{P}$ | reduced pressure coefficient $\left[\left(C_{p}-C_{\text {Pmin }}\right) /\left(1-C_{\text {Pmin }}\right)\right]$ |
| H | shape factor ( $\delta^{*} / \Theta$ ) |
| $\mathrm{H}_{5}$ | step height |
| $1_{B}$ | length of separation bubble |
| p | static pressure |
| $p_{x}$ | static pressure at reference position ahead airfoil in undisturbed flow |
| R | reattachment point |
| Re | Reynolds number based on thickness ( $\mathrm{t} \mathrm{U}_{x} / \mathrm{D}$ ) |
| S | separation point |
| t | airfoil thickness |
| $\overline{\mathrm{U}}$ | mean velocity |
| $\overline{\mathrm{U}}_{\mathrm{r}}$ | mean velocity at reference position ahead of airfoil in undisturbed flow |
| $\mathrm{U}_{+}$ | non-dimensional velocity ( $\bar{U} / \mathrm{u}_{*}$ ) |
| $\left(\overline{u^{2}}\right)^{1 / 2}$ | root-mean-square fluctuating velocity |
| $\left(\overline{u^{2}}\right)^{x / 2}$ max | maximum root-mean-square fluctuating velocity |
| $u_{*}$ | friction velocity $\left(\tau_{w} / \rho\right)^{1 / 2}$ |
| X | streamwise distance from nose of leading edge |
| $\mathrm{X}_{5}$ | streamwise distance from separation point |
| X* | normalized distance from reattachment $\left.\left[\left(\mathrm{X}_{5}-\mathrm{I}_{\mathrm{B}}\right) / \mathrm{l}_{\mathrm{B}}\right)\right]$ |
| Y | distance from surface |

Greek Symbols

| $\delta^{*}$ | displacement thickness |
| :--- | :--- |
| $\theta$ | momentum thickness |
| $\nu$ | kinematic viscosity |
| $\rho$ | density |
| $\tau_{W}$ | wall shear stress |

### 1.0 INTRODUCTION

1.1 General

The separation of flow from a surface is a large and complex area of fluid mechanics. The particular phenomenon of the separation bubble arises, as do all flow separations, from the fluid's viscosity and an adverse pressure gradient. When the laminar boundary layer adjacent to the surface meets an adverse pressure gradient, the layer uses up its already reduced momentum against the increasing pressure. The free stream cannot transfer enough momentum to the boundary layer for it to overcome the pressure. Thus, the layer comes to rest at the surface and separates. The adverse pressure gradient causes reverse flow at the surface downstream of the separation. Turbulence develops in the separated flow, which enables momentum transfer to the surface and makes it possible for the flow to reattach. A separation bubble is thus formed that encloses a region of recirculating flow, downstream of which develops a turbulent boundary layer.

Separation bubbles are of practical importance in many flows. One of the major reasons for investigating them is their formation near the leading edge of airfoils. As the airfoil incidence is increased, the bubble can either gradually extend over the airfoil surface or contract and suddenly burst, completely separating flow from the airfoil. This will obviously influence the airfoil's lift and drag characteristics. Separation bubbles also occur in turbomachines, forming on turbine and compressor blades. The energy they remove from the flow reduces efficiency, and the heating they cause is undesirable.

### 1.2 Review of Other Experiments

Much of the earlier work on leading edge separation bubbles on airfoils has dealt with establishing criteria for differentiating between long and short separation bubbles. The effect of airfoil incidence on bubble development, and the bursting of separation bubbles were also investigated. Excellent summaries of this work have been written by Chang (1970) and Tani (1964).

Tani reviewed a large number of airfoil experiments. Among his conclusions were some observations of the effects of surface roughness or disturbances in the flow. He noted that bubble formation is possible for only a particular range of Reynolds number based on freestream velocity and chord length. Flows with Re belnw this range can separate but do not reattach. Flows with Re above it undergo transition to turbulent flow ahead of the separation point, and the bubble does not form. This range depends not only on the pressure distribution and surface curvature, but also on the surface roughness and freestream turbulence.

More recent work by Nakamura \& Ozono (1987) was done on a flat plate with rectangular leading-edge geometry. One of their main findings was that by increasing the freestream turbulence intensity, the leading edge's separation bubble was correspondingly shortened. The results of the present study indicated that increasing the leading edge surface roughness shortened and altered the separation bubble. A sufficient increase in roughness completely removed the bubble.

The basic structure of the flow in a two-dimensional separation bubble was put forward in a simplified model by Norbury \& Crabtree (1955) and later by Crabtree (1957). This model, shown in Fig. 1, follows the fundamental description of a bubble given at the beginning of this section. The diagram shows the streamlines of the flow, and the physical relationship between the separated flow and the recirculation within the bubble.

The separation bubble can usually be found by examining the pressure distribution over the airfoil. The pressure generally remains relatively constant after separation until turbulent mixing commences and permits a rapid pressure recovery. This is illustrated in Fig. 2 in a diagram given by Tani. The sketch shows a constant surface pressure from the beginning of the bubble. This was assumed by Norbury \& Crabtree and Tani to extend to the point of maximum bubble thickness. The bubble profile is shown at the bottom of the figure. A rapid pressure recovery takes place over the rest of the bubble. Tani approximated this with a linear recovery for theoretical calculations.

For more detailed pressure distributions, a paper by Castro \& Haque (1987) was referred to. Their measurements were conducted within the separated shear layer behind a flat plate normal to an air flow and mounted symmetrically at the leading edge of a splitter plate. In addition to their own data, they gave pressure distributions found by Roshko \& Lau (1965) behind a backward step. Pressure data from the present study revealed pressure recovery over the rear of the bubble but failed to show a constant pressure region. These results exhibited
similar trends to those reported by $\operatorname{Tani}$ and Castro \& Haque, but differed quantitatively.

In order to ascertain aspects of the separation bubble flow such as mean velocity and turbulence intensity profiles, two other studies were reviewed. One with a test geometry much like that used in the present study was by Gleyzes, Cousteix, \& Bonnet (1984). They performed hot•wire measurements of mean velocity and streamwise turbulence intensity on a two-dimensional airfoil with a leading-edge separation bubble. Another extensive study was that done by Kiya \& Sasaki (1983), which was later elaborated on by Kiya (1986). Some turbulence data measured by Kiya \& Sasaki were also presented by Castro \& Haque (1987). This research was done on a two-dimensional flat plate with a rectangular leading edge. Again, hot-wire measurements were made of the mean velocity and various turbulence quantities through the separation bubble which began at the leading edge. The findings of these two papers showed their flows to be very similar to that in the present experiment.

Concentrating on the reattaching flow at the rear of the bubble and the developing turbuient boundary layer downstream, two papers were referred to. Bradshaw \& Wong (1972) re-examined some previous experiments on the flow downstream of steps and fences, and did some new measurements downstream of a backward-facing step. They concluded that the reattaching flow had a shear layer in which the larger eddies either alternated upstream and downstream or were torn in two and moved in both directions. As well, they found that the boundary layer
subsequent to reattachment slowly relaxed back to a typical turbulent boundary layer. In a later paper, Chandrsuda \& Bradshaw (1981) did further hot-wire measurements immediately behind a backward step. Their data showed rapid changes in turbulence quantities at reattachment, and the same gradual relaxation of the turbulent boundary layer. The present study's results displayed behaviour comparable to that found in these two studies.

Further details from each of the aforementioned papers will be discussed as they relate to present findings in Section 4.0 .

### 1.3 Description of Present Experiment

In the present study interest was centered upon the effects of leading-edge roughness on the subsequent separation bubble development and on the flow structure within and downstream of the bubble. The flow examined was that just beyond the nose of a flat-plate airfoil aligned parallel to the undisturbed flow. This airfoil appeared to produce a leading-edge separation bubble despite the fact it was at zero incidence. Usually at least a small amount of incidence is required to create a bubble. It was speculated that perhaps the airfoil could not be set with sufficient precision to guarantee zero incidence.

Flow visualization was first used to establish the separation bubble's existence and approximate its length. The surface static pressure, and the mean and fluctuating components of the streamwise flow past the airfoil were then measured. Some Pitot tube measurements
were also conducted at the surface.
Sets of data were gathered for each configuration of the airfoil. They began with the bare nose and continued through each successive increase in coarseness of the roughness strip attached to the nose. No variations were made in the freestream velocity or the airfoil incidence.
The upcoming sections relate the present experiment and its results. Section 2 describes the experimental equipment and the procedures followed to obtain the results. In section 3 the raw data is reported and compared to similar research. These results are further analyzed in Section 4, and are discussed in relation to the work of others. Section 5 summarizes the present study and gives its conclusions and recommendations.

### 2.0 EXPERIMENTAL APPARATUS AND PROCEDURE

### 2.1 Wind Tunnel

The experiments were conducted in the University of Manitoba's low-speed wind tunnel, which is of the closed circuit return type, as shown in Fig. 3. The tunnel is constructed mainly of wood, with the first diffuser downstream of the lower test section being made of fibreglass. The air is driven through the tunnel by a Woods two-stage, counter-rotating tube-axial fan with a hydraulic drive. Fan speed and hence the air $£$ low rate through the tunnel are controlled by throttling the hydraulic fluid driving the fan motors. The air velocity in the lower test section was previously calibrated against the static pressure drop across the contraction just upstream of the section. The results were given by Maynard \& Starko (1982) and Dahl (1987). The more recent calibration was re-checked and found to be still correct. A T.E.M. Engineering Ltd. 513S Micro Projection manometer was used to measure this pressure and thereby monitor the air velocity in the test section. The velocity was maintained at $15 \mathrm{~ms}^{-1}$ for all tests.

All tests were performed in the lower test section. The section has a wooden frame with transparent Plexiglas windows along its top and sides. It has a rectangular cross-section, with a height of 53 cm , a width of 76 cm , and an overall length of 183 cm . The section is fitted with corner fillets which taper gradually along the section's length. These have the effect of increasing its cross-sectional area from $0.3742 \mathrm{~m}^{2}$ at the inlet to $0.3858 \mathrm{~m}^{2}$ at the outlet. This is to compensate for the growth of the boundary layer along the interior
walls of the section.


#### Abstract

Hot-wire measurements, which will be described further in upcoming sections, were complicated by the gradual increase in air temperature within the tunnel during its operation. Basically, the constant temperature anemometer circuit uses the probe as one of the arms of a Wheatstone bridge circuit. It tries to keep the hot-wire probe resistance and temperature constant. Heat transfer from the wire to the surrounding air flow will lower the wire's temperature and resistance. To bring the bridge back into balance, the circuit will increase the voltage across the wire and consequently its resistance. Changes in air flow velocity can be calibrated against the resulting voltage changes to provide a measure of the air velocity and its fluctuations. However, the anemometer responds to anything that alters the heat flux from the hot-wire probe, including changes in the ambient air temperature. It has been estimated that a $1^{\circ} \mathrm{C}$ increase in ambient temperature causes a 1 to $2.5 \%$ decrease in the linearized output voltage [see Lawn (1969), p. 12].


This problem was overcome by keeping the air inside the tunnel at a particular temperature. As the tunnel was operating at a constant speed, the air temperature within it would have to eventually reach equilibrium with the surroundings. It would only be necessary to increase the interior air temperature to this equilibrium temperature. This was accomplished by operating the tunnel at its top speed and monitoring interior temperature with a mercury thermometer fastened to the diffuser at the downstream end of the section. When the interior
temperature was 3 to $4^{\circ} \mathrm{C}$ above room temperature, the tunnel speed was reduced to the $15 \mathrm{~ms}^{-3}$ chosen for experimentation. Although the tunnel temperature would eventually begin to increase, it would stay within $1^{\circ} \mathrm{C}$ of the desired equilibrium temperature for several hours and made it possible to take a great number of reliable measurements.

### 2.2 Flat-Plate Airfoil

The Plexiglas airfoil on which the measurements were done is shown in Fig. 4. The plate-like airfoil had an overall length of 609 mm , and a thickness of 25 mm . The nose was elliptical, with a minor axis thickness of 25 mm and a semi-major axis length of 23 mm . The tail was tapered towards the airfoil's upper surface. The airfoil was positioned horizontally in the test section, midway between the upper and lower walls. It spanned the section and was fastened to the side windows.

Static pressure measurements along the length and span of the airfoil's upper surface were made possible by thirty static pressure taps that were built into the airfoil. These were used primarily to determine how uniform the flow over the airfoil was. For such measurements, a T.E.M. Engineering Ltd. inclinable multitube manometer was used.

### 2.3 Roughness Elements

The roughness of the airfoil's nose was varied by fixing a particular grade of abrasive strip to the nose. The grades used were 100-,

80-, 60-, 40-, and 4-grit. These abrasives are shown in Plate 1. The first four are manufactured by 3-M under the brand name "Three-M-ite" and consist of abrasive aluminum oxide particles glued to a fabric backing. The 4-grit consists of silicon carbide particles glued to a paper backing.

These roughness strips were attached to the nose by first placing two strips of $18-\mathrm{mm}$ wide Scotch brand transparent tape side-by-side along the very front of the nose. This created a surface onto which a roughness strip could be glued and which could also be removed later to allow a different grade to be attached. Contact cement was then applied to the tape and the backing of the abrasive which was in the form of a $25-\mathrm{mm}$ wide strip. After centering it on the nose, the abrasive strip was pressed firmly against the tape and the cement allowed to set. In this way, the airfoil roughness was varied.

Quantifying the different grades of roughness proved difficult. Direct measurement of the surface roughness using conventional methods, such as passing a stylus with an electronic pick-up, was not attempted. This was because the abrasive particles might have damaged the stylus. Abrasive manufacturers were also unwilling to provide precise data regarding their product's root-mean-square surface roughness and backing thickness. They consider such information crucial in remaining competitive in their industry. However, they indicated that the grit number used in grading the abrasive refers to the screen mesh size used in separating abrasive materials into their various sizes.
used in the experiment were found in Machinery's Handbook [see Oberg and Jones (1943), p. 997]. The average protrusion height was taken to be half of this grain size. A number of measurements of the total thickness of each strip were done with a vernier caliper and a mean value taken. These readings are defined in Fig. 5. The difference between this total thickness and half of the grain size was taken as an estimate of the average backing thickness of each roughness.

The exception to this estimation method was the 4 -grit roughness. This grade had distinct but apparently random spaces between individual roughness elements. In this case, the mean thickness of the backing and adhesive holding the grains in place was measured along with the total thickness. The difference between these two values was taken as the protrusion height estimate. The grain size indicated by Machinery's Handbook for the 4-grit abrasive seemed incorrect as it significantly exceeded the total thickness actually measured. Thus it was disregarded.

These measurements and estimates are given in Table 1, and the protrusion heights and backing thicknesses are compared in Fig. 6. There appeared to be some variation in the backing thickness, even when experimental scatter and measurement error were taken into account. The 40 -grit strip in particular seemed slightly thicker than the other four strips. Ideally, the mean height of the roughness elements should have been flush with the airfoil surface. This thickness should at least have been kept constant to make certain that any variations in the air flow arose from changes in roughness alone.

> Aside from this problem, the positioning of the roughness strip was very consistent.

### 2.4 Flow Visualization Technique

Before beginning the series of tests, it was desired to check whether the flow over the whole airfoil was parallel to the side walls and attached to the airfoil. To help visualize the flow, oil drops were placed on the airfoil surface. The oil used was a mixture of SAE 10W-30 and kerosene with dye added to make it more visible. The mixture had to be balanced to be viscous enough not to spread out in a thin layer that would be hard to observe, and yet not so viscous that it would not respond to the air flow.

After applying the drops, the tunnel was brought up to speed as rapidly as possible. This caused the drops to move in the air flow direction near the surface, leaving streaks behind them. Once it was verified that the flow over the airfoil was attached and parallel, the drops were concentrated near the nose to see whether there was a separation bubble. This process of determining the existence and extent of the leading-edge separation bubble was performed prior to all measurements for a particular roughness.

### 2.5 Traversing Mechanism

In order to take measurements of the air flow at successive positions along the airfoil's upper surface, a traverse mechanism is built into one of the Plexiglas windows in the top of the section. This traverse made it possible to position various probes both in the


#### Abstract

streamwise direction and normal to the airfoil surface. It had a positioning precision of $\pm 0.5 \mathrm{~mm}$ in the streamwise direction and $\pm$ 0.05 mm in the vertical direction. A static pressure probe, a Pitot tube, and a hot-wire probe were mounted in this traverse for the different measurements required.


### 2.5.1 Pressure Probes

Since it was found that the static taps were too widely spaced and not near enough to the airfoil nose to provide useful information, a United Sensor static pressure probe was used for all static pressure measurements. Total pressure measurements at the surface were also required to determine shear stress. Therefore, a United Sensor circular Pitot tube was used. Both of these probes are shown in Fig. 7. The pressures found by these probes were measured with a Combist micromanometer, manufactured by Combustion Instruments Ltd., which read to a precision of $\pm 0.005 \mathrm{~mm}$. The probes' readings compared extremely well with those of a reliable Pitot-static tube. The static probe's readings at the surface were identical to those of the static taps the same distance behind the nose.

For static pressure measurements, the pressure found by the static probe was measured relative to the static pressure at a reference position. This reference point was chosen to be near the test section's centerline and upstream of the airfoil so as not to be disturbed by its presence. To read this pressure difference, the probe was placed at the reference position. A reading was then taken relative to the static tap on the airfoil's centerline and 543 mm behind


#### Abstract

the nose. All subsequent static and Pitot probe measurements at the airfoil surface were made relative to this same pressure tap.


Positioning of the static and Pitot probes in the streamise direction was aided by making a light scratch in the airfoil surface 200 mm downstream from the nose. This served as a datum mark which the static probe's taps and the front of the Pitot tube could be aligned. Alignment was accomplished with the help of a vernier microscope set up outside one of the side windows and nearly level with the airfoil surface. The microscope gave a magnified side view of the probe, making more precise positioning of the probe possible. From this mark, the probe was set at successive positions upstream. For setting the probe against the surface, a light source was placed outside the opposite side window. The probe was then lowered until no light was visible between it and its reflection in the airfoil surface. Probe placement was done with the tunnel in operation. This compensated for the slight deflection of the probe caused by the drag force on the probe support normal to the air flow. There was also a small deflection of the top window upon which the traverse rests due to the lower pressure within the tunnel while it was running.

The static and Pitot pressures were used as Preston tube measurements to determine the friction velocity, $u_{\star}$. Calibrations done by Kassab (1986) on a Pitot tube of the same diameter used in the present study were utilized. It should be noted that the conditions under which the Preston tube may be used accurately demand that the inner region of the boundary layer obey the law of the wall,

$$
\begin{equation*}
\bar{U} / u_{\nu}=f\left(u_{n} Y / v\right) . \tag{1}
\end{equation*}
$$


#### Abstract

It was assumed that the Pitot tube diameter, 1.1 mm , was small enough not to be severely affected by deviations of the inner wall from typical turbulent boundary layer behaviour. To avoid any erroneous readings due to reverse flow, Pitot tube measurements were only taken downstream of reattachment.


Pressure measurements began downstream and proceeded towards the nose. No readings were taken upstream of 22 mm behind the nose. This was because further upstream from this point the elliptical nose begins to measurably slope away from the horizontal. Surface measurements tangent to the surface would have required pitching the probe downward, and the reliability of the static tube in such a disturbed flow is uncertain.

### 2.5.2 Hot-Wire Probe

Measurements were made of $\bar{U}$ and $\left(\overline{u^{2}}\right)^{1 / 2}$ using hot-wire anemometry. A 55P05 DANTEC boundary layer probe was used with a DANTEC constant temperature anemometer system consisting of a Type 55M01 main unit and a Type 55M10 standard bridge. The linearizer was a DISA Type 55D10 and root-mean-square readings were taken with a DISA Type 55D35 RMS voltmeter which passed its readings on to a Darcy Model no. 440 digital readout. Mean values were initially measured with a DISA Type 55D31 digital voltmeter. When this device began malfunctioning, mean voltages were found by passing the signal through a Linear Systems Ltd. Model no. LS7517 integrator and then reading it with a Fluke

Model no. 8000A digital multimeter. A Hewlett-Packard Model no. 1220A oscilloscope was used to help adjust the bridge's frequency response prior to taking measurements.

The setting of the wire's operating resistance, frequency response, calibration, and linearization was done within the tunnel at the chosen operating temperature described in Section 2.1. The procedure followed can be found in DANTEC manuals [see "Instruction Manual DISA 55M System...", pp. 10-14; and "Instruction...Manual for Type 55D10 Linearizer", pp. 12-19]. It should be noted that an overheat ratio of 0.8 was used to make the hot-wire as sensitive as possible without damaging it [see Lawn (1969), p. 11]. To set the frequency response, the square wave test was conducted at 30 kHz . The resulting anemometer output signal was tuned to give an undershoot of $13 \%$ of its maximum amplitude on the oscilloscope.

The hot-wire could be calibrated in situ for the 10 to $20 \mathrm{~ms}^{-1}$ range of velocities. It was put in the same reference position used by the static probe in Section 2.5.1, upstream of the airfoil. Some difficulty was encountered in maintaining test section speeds below 10 $\mathrm{ms}^{-1}$. To calibrate in the 5 to $10 \mathrm{~ms}^{-1}$ range, a DISA Type 55A60 calibration unit was used. This apparatus was basically a miniature wind tunnel with a variable speed fan drawing air through a nozzle. The hot-wire was mounted at the nozzle throat and calibrated. Even with this set-up there were fluctuations in the air velocity, and the calibration had to be performed carefully.

On completing the previous steps, the hot-wire was ready for use. The integration time constants for both the RMS voltmeter and the mean voltage reading were set at 30 s . The probe was then positioned in a similar fashion to the static pressure probe. A light source was placed on the opposite side of the tunnel and the probe observed through the microscope. It was lined up horizontally with the $200-\mathrm{mm}$ mark on the airfoil. The probe tip was lowered until it was 0.6 mm from its reflected image in the airfoil. This measurement was achieved using microscope's graticule which had been calibrated against the probe traverse. This meant that the tip was actually 0.3 mm from the surface. This vertical positioning procedure was repeated at every measurement station along the airfoil to ensure correct readings of boundary layer profiles. It was also a precaution against contact between the fragile hot-wire and the surface. All positioning was done with the tunnel operating to compensate for deflections as in the case of the static pressure probe. Measurements ahead of the $22-\mathrm{mm}$ position behind the nose were not used for the same reason given for pressure measurements. Hot-wire readings were taken near the surface with no flow in the tunnel. This was to verify that there was no significant heat transfer to the airfoil in addition to the air flow.

### 3.0 EXPERIMENTAL MEASUREMENTS

To help clarify the following results and discussion, a sketch of the airfoil's leading edge is given in Fig. 4. It shows the relative positions of the abrasive strip and the separation bubble on the airfoil. It also defines the streamwise dimensions that appear in subsequent sections.

### 3.1 Static Pressure Distribution

Rough measurements with the static taps in the airfoil indicated that there was a very slight pressure gradient along its length. The static pressure was constant across the span of the airfoil.

Static pressure probe measurements transformed into the pressure coefficient, $C_{P}$, are shown in Fig. 8. Measurements showed the pressure recovery over the downstream end of the separation bubble for each of the cases having separation. The existence of the bubble in the bare, 100-, 80-, and 60-grit cases is discussed further in Section 4.0. The curves for the 40- and 4-grit cases show the increase in pressure of the attached flow, which seemed to commence a little further upstream and proceeded slightly more gradually than in the separation bubble.

A phenomenon observed by many researchers but not found in the present study is a constant pressure region over the forward part of the separation bubble. This sort of occurrence is illustrated in Fig. 2. However, it has been noted by Tani (1964) that the presence of a bubble is not necessarily accompanied by a region of relatively constant pressure [see Tani (1964), p. 80]. He concluded this after


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reviewing a large body of bubble separation material. Furthermore, Gleyzes et al (1984) also had difficulty in some cases locating a constant pressure plateau where one should have existed. They partly attributed this to a deficiency of pressure taps in this region.


The pressure coefficient data is further analyzed in section 4.1.2.

### 3.2 Mean Velocity Profiles

The mean velocity profiles found by the hot-wire anemometer are shown in Fig. 9(a-f). They are normalized by the reference velocity, $\overrightarrow{\mathrm{U}}_{x}=15 \mathrm{~ms}^{-1}$. Fig. 9(a) compares readings made with the 10 to $20 \mathrm{~ms}^{-1}$ calibration and those made with the 5 to $10 \mathrm{~ms}^{-1}$ calibration. As can be seen in the figure, the slight difference in resulting profiles were within the anemometer's limits of precision. For this reason, other readings taken with the low-speed calibration have not been included in the presentation of results.
. The profiles for the bare, 100-, 80-, and 60-grit leading edges all showed the expected shape and trend. Namely, the upstream profiles in the separation bubble had low velocity regions next to the surface. In fact, for the bare and 100 -grit leading edges, some of the measurements closest to the airfoil showed the velocity increasing towards the surface. Measurements taken with no flow in the tunnel indicated negligible heat transfer from the hot-wire to the Plexiglas airfoil. Therefore, this apparent velocity increase was likely the hot-wire's response to the backflow adjacent to the surface in the bubble. That


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is, this type of probe cannot differentiate between forward and reverse flows and consequently registered an increasing reverse flow as a forward one. As this probe type cannot be accurately calibrated to measure reverse flows, these experimental points have been omitted from all figures and calculations.


The velocity profiles further downstream progressively filled out until they took on the appearance of typical turbulent boundary layers. This pattern of profile development through and downstream of separation bubbles was like those found by Kiya \& Sasaki (1983) and Gleyzes et al (1984).

The reattachment point for the flow in each set-up was determined from these profiles. At reattachment the mean velocity profile at the surface should be normal to the surface. However, no measurements were taken closer than 0.3 mm from the airfoil, as explained in Section 2.5.2. Instead, the reattachment point was approximated for the bare and 100 -grit cases by choosing it to be the first station downstream of the last one displaying the reverse flow effect. The 80 -grit and 60-grit cases had separation bubbles but no apparent reverse flow readings in their profile data. This was likely due to shallowness of the bubble. For these two cases it was decided to approximate reattachment with the first station that definitely showed attached flow. Thus, the first station which had no inflection point in the profile between the free stream and the surface was considered the reattachment point. These positions are recorded in Table 2 , and are the values used in all other calculations.

The profile sequence found for the 40 -grit leading edge, given in Fig. 9(e), showed what appeared to be turbulent profiles all along the surface with no separation region. This result agreed with other evidence which is reported in Section 4.0.

The arrangement with the 4 -grit abrasive gave mean velocity results that differed from the trends observed in the other cases. The data given in Fig. $9(f)$ showed a boundary layer with a lower velocity region next to the surface. This region filled out over successive stations until it was unnoticable at the station furthest downstream.

Further analysis and interpretation of all mean velocity data and is given in Section 4.1.3.

### 3.3 Mean Wall Shear Stress

To help verify that the boundary layers were becoming fully developed and turbulent, it was decided to plot the mean velocity profiles in the $U_{+}$versus $y_{+}$form. The friction velocity, $u_{m}$, is needed to normalize the data in this way. This quantity can be obtained in several ways.

### 3.3.1 Cross-Plot Method

One method is to cross-plot the mean velocity profiles. This is described by Azad \& Burhanuddin (1983). For the present study, Y+ was chosen to be 90 and used in the logarithmic law,

$$
\begin{equation*}
\mathrm{U}_{+}=(1 / 0.41) \ln \mathrm{Y}_{+}+\mathrm{C} . \tag{2}
\end{equation*}
$$

Three values of $U_{i}$ were then calculated using three values of $c$, namely 5.0, 5.5, and 6.0. The definitions of $U_{+}$and $y_{+}$provided the following equation,

$$
\begin{equation*}
\mathrm{U}_{+} \mathrm{Y}_{+}=\overline{\mathrm{U}} \mathrm{Y} / \mathrm{\nu} . \tag{3}
\end{equation*}
$$

Substituting each of the three pairs of values for $U_{+}$and $Y_{+}$into this equation gave three plots of $\vec{U}$ versus $y$. By superimposing these three plots on a velocity profile found with the hot-wire, three intersection points were produced. The velocity components of these points all gave roughly the same value, this being $u_{*}$. In this way, $u_{*}$ was found for the measuring stations downstream of reattachment.

### 3.3.2 Preston Tube Method

Another method of finding $u_{*}$ is to use a Preston tube. This approach, first mentioned in Section 2.5.1, determines $u_{*}$ at a position by measuring the total pressure at that point with a circular Pitot tube resting parallel to and against the surface. The position's static pressure is also measured, usually with a pressure tap in the wall, and the difference between the two pressures calculated. Provided the Pitot tube lies within the flow layer defined by the law of the wall, equation (1), a relationship exists between this pressure difference and $u_{*}$. This relationship can be deduced by calibrating the Pitot tube in a circular pipe with fully developed turbulent flow.

Such calibrations were done by Kassab (1986) on a Pitot tube of the same manufacture and dimensions as the one used in the present study. These calibrations were taken to be valid for this experiment.

As previously explained in Section 2.5.1, the assumption was made that the tube was small enough not to be affected significantly by deviations in the inner wall layer from the law of the wall.

### 3.3.3 Comparison of Results

Before comparing results, it should be emphasized that crossplotting only works for equilibrium turbulent boundary layers. Such layers follow the logarithmic law. However, several papers on boundary layer relaxation and plots of $U_{+}$versus $Y_{+}$using $u_{*}$ estimates irdicated that the boundary layer did not reach equilibrium until farther downstream. This meant that cross-plotted values of $u_{*}$ immediately behind the reattachment point were likely incorrect.

To determine where this method became valid, the Preston tube approach was used downstream of reattachment. Due to time limitations, measurements were only taken in the bare leading edge set-up. In order to get an estimate of $u_{*}$ for the other cases and to make comparisons of the two methods, the plot shown in Fig. 10 was made. This figure plots the $u_{*}$-values found by the two approaches against the distance downstream from reattachment normalized by that case's bubble length.

Fig. 10 shows the Preston tube values of $u_{*}$ rapidly increasing after reattachment, reaching a maximum, and then very gradually declining. The Preston tube and cross-plotted $u_{*}$-values for the bare case seemed to agree at nearly one bubble-length. The cross-plotted curves for the other cases also appeared to collapse onto the Preston tube measurement at about this point. This comparison can be extended
to the plots of $U_{+}$versus $Y_{+}$described in Section 4.2.1 and presented in Fig. 11(a-f). Namely, the bare, 100-, 80-, and $60-\mathrm{grit}$ set-ups all obeyed the logarithmic law over a region beyond the bubble. This area overlapped onto the region where the cross-plotted and Preston tube friction velocities agreed. Such agreement would be expected if the flow was in fact fully developed and turbulent.

It was finally decided to use the Preston tube readings of $u_{n}$ as estimates for the cases having separation bubbles up to one bubblelength past reattachment. For stations further downstream, the crossplotted $u_{*}$-values found for each velocity profile were used. For the 40-grit case, the cross-plotted $u_{*}$ was used exclusively as the flow was attached over all stations. Since the $U_{+}$versus $Y_{+}$profiles for this case all followed the logarithmic law, the use of cross-plotting seemed justified. The 40 -grit results are discussed further in Section 4.1.2. The cross-plotted $u_{\star}$ was also used for the entire 4 -grit case. While the $U_{+}$versus $Y_{+}$plots later showed that there were some deviations from the logarithmic law, these variations fell partly within experimental error.

The values of $u_{*}$ obtained by the Preston tube were transformed into the skin friction coefficient, $C_{f}$. This data is compared to other researchers' results in section 4.2.4.

### 3.4 Turbulence Intensity Profiles

The profiles of turbulence intensity normalized by the reference velocity are given in Fig. 12(a-f). In the cases of the bare, 100-,

80-, and 60-grit leading edges, the profile development was similar. Beginning upstream, the profiles showed a maximum about half the boundary layer thickness from the surface. This maximum increased downstream along the separation bubble. In the bare and 100-grit cases it shifted slightly away from the wall. The maximum reached its greatest magnitude near reattachment. Downstream from this point, the intensity next to the surface increased until the profile took on the shape of a typical turbulent boundary layer. Namely, this intensity was low in the free stream, increased through the boundary layer, and approached a maximum near the surface. This profile development was very similar to that shown by Gleyzes et al (1984).

The turbulence profiles for the 40 -grit and 4-grit cases, given in Fig. 12(e \& f), displayed developmental trends different from those in the other four cases. The 40 -grit case appeared to have intensity profiles typical of equilibrium turbulent boundary layers over all the stations. This supported other results that are discussed in Section 4.0.

The intensity profiles for the 4 -grit set-up began at the station furthest upstream with a region of high intensity separated from the surface by a lower intensity that increased again towards the surface. At the stations further downstream the high intensity region decreased in magnitude and the turbulence spread slightly away from the surface. However, the profiles did not assume the typical turbulent boundary layer shape until the last downstream station. A possible explanation of the 4 -grit case's behaviour is given in Section 4.0 .

### 4.0 ANALYSIS AND DISCUSSION OF RESULTS


#### Abstract

In Section 3.0 there has already been some discussion of the basic flow measurements. This section analyzes the data further to reveal more about the separation bubble and the flow downstream of it.

\subsection*{4.1 Existence of Separation Bubble}


### 4.1.1 Flow Visualization

The oil drop patterns at the nose of the airfoil are shown in Plates 2 (a-e) for the bare nose and the 100 - through 40 -grit configurations. As can be seen, the bare nose, 100-, 80-, and 60-grit arrangements each indicated a separated flow region. That is, drops placed near the leading edge moved downstream until they seemed to reach a barrier and form into a ridge normal to the flow direction. This was caused by the separation of the flow from the surface. Drops immediately downstream of this separation either remained stationary or moved upstream. It was surmised that the backward motion was induced by backflow next to the surface, commonly found in separation bubbles. Further downstream, drops again moved downstream, and in some cases a single drop flowed both upstream and downstream. This position was interpretted as being in the vicinity of reattachment. The oil drops behind the 40 -grit strip all moved downstream without interuption, showing that the flow was no longer separating.

Measurements were made from the photographs taken of the drop patterns. The positions of separation and reattachment relative to the leading edge were measured and are given in Table 3. As can be ob-
served, the oil drop method of flow visualization did not seem very precise in pinpointing where the surface flow changed, especially in the case of reattachment. The results would probably have been more conclusive had a larger number of visualization trials been performed to provide a more representative sampling. The presence of the drop itself might also have influenced the flow past it, and there could have been interference between adjacent drops. This, along with inconsistencies in drop composition and surface conditions, probably led to the variations reported in Table 3. For this reason, greater credence was given to the mean velocity profiles when it came to fixing the reattachment position. This was described in Section 3.2. Unfortunately the separation region near the leading edge could not be traversed for the velocity profile owing to the elliptical nose, as previously discussed in the experimental procedures. This, and the fact that the oil drops gave fairly consistent results for the separation position, made it possible to fix separation at 18 mm from the leading edge. All four configurations having separation bubbles had this same separation position. Aside from this instance, this flow visualization technique was mainly useful in demonstrating that the separation bubble existed and in roughly estimating its extent.

### 4.1.2 Reduced Pressure Coefficient Distribution

It was desired to collapse the pressure data given in Fig. 8 and make comparisons with other researchers' work easier. So, the pressure coefficient was renormalized in the manner used by Castro \& Haque (1987), namely the reduced pressure coefficient,

$$
\begin{equation*}
\widetilde{C}_{D}=\left(C_{P}-C_{P m i n}\right) /\left(1-C_{P_{\text {min }}}\right) \tag{4}
\end{equation*}
$$

$C_{\text {Prain }}$ is the minimum $C_{p}$ in the separation bubble.

As is shown in Fig. 8, the pressure distributions for the cases having bubble separation did not show constant pressure regions. Therefore the $C_{\text {Pmin }}$ used to calculate $\widetilde{C}_{P}$ was the pressure measured at the station furthest upstream. However Roshko \& Lau (1965) and Castro \& Haque were able to use the minimum pressure in an area of relatively constant pressure. This difference between the raw data of the present study and that of the other two papers might have contributed to later differences in calculated results.

Reduced pressure coefficient curves are shown in Fig. 13. The experimental curves seemed to collapse onto each other fairly well, but also differed from those found by Roshko \& Lau (1965) and Castro \& Haque (1987). That is, the slopes of the curves leading up to the reattachment of the flow were different, as were the values of $C_{p}$ they approached. This could have been due to the different experimental geometries used by these researchers. Castro \& Haque used a plate normal to the flow fastened to the front of a splitter plate, while Roshko \& Lau examined the flow over the backward-facing step formed by various forebodies attached to the front of a plate.

In summarizing the pressure data, it should be emphasized that the presence of the static pressure probe may have had an effect on the flow, and consequently affected the measured pressure. Such interference would have been more detrimental in the region of the separa-


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tion bubble. The curvature of the leading edge likely also affected the static probe readings. Suggestions are made to improve pressure measurement in Section 5.3.


### 4.1.3 Effect of Roughness

Using the flow visualization method described in Section 4.1.1 and the mean velocity measurements in Section 3.2 , the separation bubble lengths were determined. These lengths are summarized in Table 2. From the table it can be seen that the bubbles produced by the bare leading edge and the 100 -grit set-up had lengths slightly longer than those produced by the 80 - and 60 -grit cases. This agreed with the results of Nakamura \& Ozono (1987), who found that increasing the freestream turbulence intensity decreased the separation bubble length on a blunt plate.

The flow in the present study was disturbed by surface roughness rather than an upstream grid across the tunnel as used by Nakamura \& Ozono. It was surmised that the disturbances drew energy from the turbulence in the recirculating flow of the bubble. They continued increasing in strength and size until they were able to transfer enough momentum to the airfoil surface to permit reattachment. When the protrusion height of the roughness was increased, correspondingly greater disturbances were introduced to the flow near the surface. This meant there were larger eddies which were moving along the separating and recirculating flow, and rolling up into still larger ones. However, since they were beginning at a larger scale, they required less distance to develop sufficient energy and size to take momentum
from the free stream to the surface. So, it was expected that the bubble length would decrease with coarser abrasive strips on the leading edge. Finally, with the 40 -grit strip, the transition to turbulent flow further upstream precluded separation altogether.

### 4.2 Characteristics of Flow Development

### 4.2.1 Normalized Mean Velocity

The purpose of putting the basic mean velocity data into the normalized profiles of $U_{+}$versus $Y_{+}$was to ascertain that the boundary layers were becoming fully developed and turbulent. The values of $u_{*}$ necessary for this transformation were determined by the process discussed in Section 3.4. The resulting semi-logarithmic plots are shown in Fig. 11 (a-f).

The universal law against which all the plots were compared was that proposed by Kader \& Yaglom (1978), namely,

$$
U_{+}=\left\{\begin{array}{l}
14.5 \tanh \left(y_{+} / 14.5\right), \text { for } 0<Y_{+}<27.5  \tag{5}\\
2.44 \ln Y_{+}+5, \text { for } Y_{+}>27.5
\end{array}\right.
$$

This law was chosen for its simple mathematical form. Clearly from the plots, the velocity profiles downstream collapsed gradually toward and finally fitted the universal law for the bare, 100-, 80-, and 60-grit cases. For the 40 -grit case, the profiles over all the stations fitted the logarithmic law. This verified previous evidence indicating that the flow was attached and that there was an ordinary turbulent boundary layer over the whole region.

### 4.2.2 Displacement and Momentum Thicknesses

To get more information from the mean velocity data, the displacement thickness, $\delta^{*}$, momentum thickness, $\theta$, and shape factor, $H$, were calculated. For the cases having a separation bubble, the calculations were done in two parts. Trapezoidal integration was performed up to the reattachment point, with linear interpolation from the measurement point nearest the surface to the surface. Starting at reattachment, Kader \& Yaglom's (1978) universal law was used to extend the mean velocity profile to the experimental points. Thus, the universal law portion of the profile was integrated up to where it intersected the measured points. From there on trapezoidal integration was used as before. The 40 -grit case used the universal law and measured data exclusively, while the 4-grit case relied entirely on experimental points with linear interpolation at the surface. Plots of $\delta^{*}$, $\theta$, and $H$ are shown in Fig. $14(\mathrm{a}-\mathrm{c})$.

Fig. 14(a) shows that in the bare, 100-, 80-, and 60-grit cases, $\delta^{*}$ rose and fell as the measurements moved downstream through the separation bubble. About 10 mm downstream of reattachment, $\delta^{*}$ reached a minimum and then gradually increased. The values of $\delta^{*}$ all seemed to be increasing asymptotically towards a constant, although more measurements taken further downstream to would have made this certain.

The values of 0 for the bare, 100-, $80-$, and 60 -grit set-ups increased through the separation bubble, as shown in Fig. 14(b). Following reattachment, $\theta$ increased more gradually. It appeared that it was asymptotically approaching a constant value. Again, more read-
ings taken further downstream would have made interpretations more definite.

The plots of $H$, given in Fig. $14(\mathrm{c})$, began high in each of the four cases having a separation bubble. It then dropped quickly near reattachment and settled to a constant value of 1.6 .

In the case of the 40 -grit abrasive on the leading edge, which eliminated the bubble, $\delta^{*}$ and $\theta$ both increased over the upstream stations. They gradually approached constant values at $x=60 \mathrm{~mm} . \mathrm{H}$ maintained a roughly steady value of 1.6 throughout all measurements. An interesting feature of the results for the separation bubble cases and the attached flow of the 40 -grit case is that they all appeared to be approaching the same constants for $\delta^{*}, \theta$, and $H$ downstream.

### 4.2.3 Maximum Turbulence Intensity

The trend displayed by the turbulence intensity profiles, discussed in Section 3.3, had to be made clearer. A plot was made of the maximum turbulence intensity versus the distance from the separation point for the four cases having separation. As can be seen in Fig. 15, this plot shows the magnitude of the maximum intensity increasing along the length of the separation bubble. Interestingly, the bare, 100-, and 80 -grit cases reached a maximum magnitude at reattachment. They then declined to a relatively constant level of $\left(\overline{u^{2}}\right)^{1 / 2}=1.5$ $\mathrm{m} / \mathrm{s}$. Furthermore, the bare case attained a reattachment peak which was larger than those for the 100 -grit and 80 -grit cases. The 60 -grit intensity also increased to a maximum near reattachment. However, its maximum magnitude was was close to the final intensity level reached
by the other set-ups. After reaching it, the maximum intensity stayed at this level.

This data was re-normalized by $\overrightarrow{\mathrm{U}}_{x}{ }^{2}$ to make comparisons with other researchers' work possible. These plots of $\overline{u^{2}} / \bar{U}_{r}{ }^{2}$ versus distance from separation normalized by the bubble length are given in Fig. 16. Results from Castro \& Haque (1987) are also shown. All results exhibited the same trend, namely an increase in the non-dimensionalized normal Reynolds stress from separation to reattachment. Quantitatively, the data from the present study was much lower than that found by Castro \& Haque. The peaks of normalized intensity at reattachment were lower for the 100 -grit and 80 -grit cases, as was noted previously in describing the maximum intensity distribution. The peak was again greatest for the bare leading edge and did not exist at all for the 60-grit case. The normalized intensities in the present study seemed to all move towards a value of $\overline{u^{2}} \max / \vec{U}_{x}^{2}=0.011$.

### 4.2.4 Discussion of Results for Flow Development

Some similarities were noticed between the present mean velocity results and the work done by Bradshaw \& Wong (1972) and Chandrsuda \& Bradshaw (1981) on the reattachment and relaxation of turbulent shear layers.

In the present experiment, the rapid increase in surface shear stress following reattachment that was commented on in Section 3.4.3 was also noted in the two aforementioned papers. Plots of skin friction coefficient, $C_{F}$, versus the distance downstream from reattachment
normalized by bubble length, $X^{*}$, are shown in Fig. 17. The $C_{f}$-values for the present study were calculated from the preston tube readings taken downstream from the bare case's separation bubble. The other two sets of data were measured behind backward-facing steps. Clearly the results differ quantitatively as the backward-step measurements fell fairly close together at a lower value of $C_{f}$, while the present results were about twice the magnitude. However, all the studies showed the same rapid rise in $C_{f}$ immediately following reattachment.

Another point of comparison is that the two papers emphasized that the boundary layer beyond reattachment did not follow the universal logarithmic law. Instead they showed a slight deviation from it between the surface and the freestream which persisted some distance downstream. The present findings, which had reattaching flow and are shown in Fig. 11(a-d), did not have such variations in them. Rather, they began at reattachment by not following the logarithmic law and collapsed toward it gradually over successive stations. In addition, the portion of the profile following the logarithmic law lengthened in a direction away from the surface. This is typical for developing turbulent boundary layers.

Before proceeding with the explanation of relaxation put forward in the aforementioned papers, the displacement and momentum thickness data that complemented the velocity profiles should be discussed. These thicknesses and the shape factor for the reattaching flow are given in Fig. $14(a-c)$. The apparent tendency of $\delta^{*}$ and $\theta$ toward a constant value after reattachment seemed to correlate well with the
velocity profiles if the assumption was made that the boundary layer was developing towards equilibrium. That is, the presence of a fully developed turbulent boundary layer implies that its velocity profile obeys the logarithmic law and that its $\delta^{*}$ and $\Theta$ are constant. since both these conditions seemed to be gradually met in the bare, 100-, 80-, and 60 -grit cases after reattachment, it appeared that each flow might be approaching equilibrium.

Yet another point to consider is that $H$ was observed to have dropped quickly at reattachment and settled to a constant value of 1.6. Schlichting referred to a paper by J. Persh which gave a value of $H=1.4$ for the turbulent boundary layer of a flat plate following transition from a laminar layer [see Schlichting (1979), p. 454]. However, Gleyzes et al (1984) also found that after transition to turbulence, $H$ was near 1.6, and they considered it typical for a turbulent boundary layer. They also found the high values of $H$ in the bubble which decreased rapidly at reattachment, as in the present study. Additionally, it was notable that all the cases in the present experiment reached the same final value of $H=1.6$. This showed that all the set-ups had flows which eventually developed into ordinary boundary layers.

The two papers which Bradshaw collaborated on explained the deviation from the logarithmic law as follows. Bradshaw \& Wong (1972) surmised that at reattachment, the shear layer split in two, part of it moving upstream and part continuing downstream. They supposed this bifurcation came about either by the lateral splitting of the flow's
larger eddies or by the larger eddies being alternately deflected up and downstream. The region that was previously central in the mixing layer in the bubble was brought into close proximity to the surface. For this reason, the length scale increased swiftly above the equilibrium value moving away from the surface. As the turbulence was not in local equilibrium over the entire inner wall layer, the mean velocity did not completely agree with the logarithmic law. However, as the flow continued downstream this local-equilibrium layer spread slowly outwards from the surface until it assumed the usual thickness in an equilibrium turbulent boundary layer.

The measurements made in the present study were not extensive enough to independently draw as complete a picture of the reattaching flow. Nevertheless, the mean velocity data that was gathered showed a developing turbulent boundary layer. Its equilibrium layer spread from the surface, and the displacement and momentum thicknesses grew asymptotically beyond reattachment. These results certainly fitted the same phenomena described by these authors.

The maximum turbulence intensity data for the bare, 100-, 80-, and 60 -grit cases, described in Section 4.2 .3 and shown in Fig. 15, could have the following explanation. In the case of the bare leading edge, the eddies in the flow extracted energy from the mean flow, which was shown by the increasing intensity. When the eddies transferred enough energy to the wall layer for reattachment and the shear layer bifurcated, they broke down and the intensity dropped rapidly. The turbulence intensity then moved toward a constant value as equili-
brium was established in the relaxing boundary layer. As successively coarser abrasives were placed on the leading edge in the other cases, their roughness elements shed larger eddies. These eddies became closer in size to the freestream eddies and also were in phase with them. This meant the eddies originating from the roughness were better able to extract energy from the freestream eddies and the mean flow. Energy was lost in this interaction due to increased dissipation and the transfer of energy to the other components of turbulence. Hence, the intensity did not reach as high a level as in the bare case. The 60-grit abrasive produced eddies of a large enough size to be most efficient of the four cases in breaking up the larger eddies in the flow and distributing the energy. This was apparent in the absence of a peak in maximum intensity at reattachment. Instead, the maximum turbulence intensity rose gradually to the final level found downstream in the relaxing boundary layer. However, the disturbances added by the 60-grit abrasive were still not sufficient to cause transition to a turbulent boundary layer and completely preclude separation.

A comparison of the present maximum turbulence intensity normalized by $\overline{\mathrm{U}}_{r}{ }^{2}$ and results found by Castro \& Haque (1987) is shown in Fig. 16 and was described in Section 4.2.3. Although the present data and that given in the paper agreed in the overall trend, the data from the present experiment was quantitatively much lower. This could be attributed to the differing geometry of the two experiments used in creating separation bubbles. It was understandable that the much more severe edge of Castro \& Haque's backward step arrangement might have induced higher turbulence intensities.

Bradshaw \& Wong (1972) and Chandrsuda \& Bradshaw (1981) have made thorough measurements of the turbulence of a reattaching flow. They observed a sudden drop in the turbulence intensity and turbulent shear stress at reattaching. Their explanation of this intensity reduction was that near reattachment the flow's large eddies began transferring turbulent energy and shear stress into the inner layer where it was dissipated. The readings taken of the streamwise turbulence intensity in this experiment did not constitute a complete picture of the turbulence structure. Nevertheless, they did show a reduction in its maximum value for $u^{2}$, at least for the bare, $100-$, and $80-\mathrm{grit}$ cases. Thus, it could be speculated that the reattachment was proceeding in a manner similar to that described in the two papers mentioned.

It was suspected from the flow visualization stage of testing that the 40 -grit case had attached flow over all its stations. The mean velocity and turbulence intensity profiles in Fig. $9(e)$ and Fig. 12(e) respectively also appeared to be typical of equilibrium turbulent boundary layers. The normalized mean velocity profiles of $U_{+}$ versus $y_{+}$in Fig. $11(e)$ all fitted the logarithmic law. $\delta^{*}$ and $\theta$, shown in Fig. $14(\mathrm{a} \& \mathrm{~b})$, both increased over the upstream measurement stations and gradually approached constant values at $x=60 \mathrm{~mm} . \mathrm{H}$ maintained a roughly steady value of 1.6 throughout all readings. All these observations indicated that the 40 -grit abrasive was sufficiently coarse to cause transition, make the boundary layer turbulent, and eliminate separation.

### 4.3 Flow Downstream of 4-Grit Strip

The profiles of mean velocity and turbulence intensity for the 4-grit arrangement are given in Fig. $9(f)$ and Fig. 12(f), respectively. They were previously described in Section 3.0. Semi-logarithmic plots of $U_{+}$versus $Y_{+}$for this set-up using cross-plotted $u_{m}$, shown in Fig. 11(f), indicated that only for the last four profiles downstream did the profiles have linear portions that followed the logarithmic law.

It was anticipated that the velocity profiles following the relatively high protrusions of the 4 -grit abrasive might exhibit development like that behind a backward-facing step. Measurements done by Etheridge \& Kemp (1978) behind a backward step were chosen for comparison. One difficulty in making such a comparison was the range of profiles displayed in Etheridge $\&$ Kemp's paper. Their mean velocity profiles ended 4.0 step-heights behind the step. The 4 -grit case readings only began at 7.29 step-heights, taking the step height to be 2.21 mm from Table 1. Nevertheless, Etheridge \& Kemp's profile at 4.0 step-heights showed little if any inflection in the boundary layer, and no layer of near-uniform velocity, while the 4 -grit case profiles did. However, it could not be predicted how their flow would develop further downstream. It was also speculated that the extreme size of the abrasive's protrusions were severely disturbing the flow, which might also have made comparison difficult.

Continuing the comparison, Etheridge \& Kemp's turbulence intensity measurements were examined. Unlike their mean velocity data, they
presented all of their turbulence intensity profiles. These extended as far as 8.26 step-heights from their step. This resulted in some overlap of their data on the present readings. The streamwise turbulence intensities compared well. The intensity is a finer quantity than the mean velocity just discussed. So, it appeared that the flow in the 4 -grit case was somewhat like that behind a backward-facing step. This may have been because the 4 -grit case was making a step 2.21 mm high near the airfoil nose.

### 5.0 CONCLUSIONS AND RECOMMENDATIONS

5.1 Summary


#### Abstract

Reviewing the study's results, it was found that by increasing the coarseness of abrasive strips on the leading edge of an airfoil with a leading-edge separation bubble, the bubble could be slightly shortened and ultimately eliminated. By increasing the roughness significantly beyond this point, a flow like that downstream of a back-ward-facing step appeared. This may have occurred due to the step-like characteristics of the roughness strip rather than the scale of roughness.


The separation bubble produced by the present plate-like airfoil was discovered to have a structure resembling those generated by a range of geometries. These set-ups included an ONERA LC 100 D airfoil, uased by Gleyzes et al (1984); a flat plate with a rectangular leading edge, used by Kiya \& Sasaki (1983) and Nakamura \& Ozono (1987); a flat plate with a forebody followed by a backward-facing step, used by Roshko \& Lau (1965); and a splitter plate with a flat plate normal to the flow attached to its leading edge, used by Castro \& Haque (1987). Further, the flow produced by the 4 -grit case bore a resemblance to the reattaching flow behind a backward-facing step, the set-up used by Etheridge \& Kemp (1978).

Finally, the reattachment and relaxation of the boundary layer were investigated and gave results that were similar to the findings of Bradshaw \& Wong (1972) and Chandrsuda \& Bradshaw (1981).

### 5.2 Conclusions

This experiment showed that:

1) The introduction of disturbances to the flow by increasing surface roughness was able to alter the leading edge separation bubble and finally remove it altogether.
2) The structure of the separation bubble was similar to bubbles found on other types of leading edges, and to bubbles formed behind backward-facing steps.
3) The flow behind reattachment displayed some of the aspects of a flow relaxing to a turbulent boundary layer in equilibrium as described by Bradshaw \& Wong (1972) and Chandrsuda \& Bradshaw (1981).

### 5.3 Recommendations

To continue the research begun in this study, several recommendations are made.

1) Some provision should be made for flush-mounting abrasive strips to preclude possible interference from the backing, and more precise measurements made of the abrasive's surface roughness.
2) Static pressure taps should be incorporated in the airfoil leading edge at close intervals to determine pressure distributions more accurately.
3) Accurate measurements in the backflow region of the separation bubble are required for a more complete understanding of the flow structure.
4) Measurements of the friction velocity should be made for each
test configuration.
5) More extensive turbulence data would make the turbulence structure clearer. The shallowness of the separation bubble might make this difficult.
6) Data taken further downstream and, if possible, nearer the airfoil surface would also give a more complete picture of the flow.

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### 7.0 TABLES

## Table 1: Roughness Dimensions

(all dimensions in mm)

| Grade | Grain <br> Size | Protrusion <br> Height | Backing <br> Thickness | Total <br> Height |
| :--- | :--- | :--- | :--- | :--- |
| 100-Grit | 0.149 | 0.075 | 0.58 | 0.65 |
| 80-Grit | 0.177 | 0.089 | 0.88 | 0.97 |
| 60-Grit | 0.250 | 0.125 | 0.71 | 0.83 |
| 40 -Grit | 0.420 | 0.210 | 1.17 | 1.38 |
| 4 -Grit | $4.76^{*}$ | 1.46 | 0.75 | 2.21 |

* Not used in calculations.

Table 2: Separation Bubble Dimensions Derived

## from Velocity Profiles

## (all dimensions in mm)

| Configuration | Separation Point | Reattachment Point | Bubble Length |
| :---: | :---: | :---: | :---: |
| Bare | 18 | 42 | 24 |
| 100 -Grit | 18 | 44 | 26 |
| 80 -Grit | 18 | 36 | 18 |
| 60 -Grit | 18 | 38 | 20 |

## Table 3: Separation Bubble Dimensions

## Derived from Flow Visualization

(all dimensions in mm)

| Configuration | Separation Point | Reattachment Point |
| :---: | :---: | :---: |
| Bare | 16 | $27-37$ |
| 100-Grit | 18 | $37-43$ |
| 80-Grit | $17-18$ | $37-38$ |
| 60 -Grit | 18 | 37 |

## Plate 1:

Comparison of Grades of Abrasive
80-GRIT



## Plate 2:

## Flow Visualization Results

(a) Bare
(b) 100-Grit Separation

(c) 100-Grit Reattachment
(d) 80-Grit

(e) 60-Grit
(f) 40-Grit



Figure 1: Norbury \& Crabtree's Separation Bubble Model.


(A) UPPER TEST SECTION
(C) BALANCE
(B) LOWER TEST SECTION
(D) CONTRA-ROTATING FANS
(all dimensions are in Cm )

Figure 3: Low Speed Wind Tunnel.

NOSE OF
LEADING EDGE



Figure 4: Experimental Plate.


Figure 5: Sketch of Roughness Dimensions.


Figure 6: Comparison of Roughness Dimensions.


Figure 7: Pressure Probes (all dimensions in mm)
(a) Static Probe.

(b) Pitot Probe.


Figure 8: Pressure Coefficient Distribution.


Figure 9: Mean Velocity Profiles
(a) Bare.

(b) 100-Grit.

(c) 80-Grit.

(d) 60-Grit.

(e) 40-Grit.



Figure 10: Estimation of Friction Velocity Downstream of Reattachment.


Figure 11: Normalized Mean Velocity Profiles
(a) Bare.

(b) 100-Grit.

(c) 80-Grit.

(d) 60-Grit.

(e) 40-Grit.

(f) 4-Grit.


Figure 12: Turbulence Intensity Profiles
(a) Bare.

(b) 100-Grit.






Figure 13: Reduced Pressure Coefficient Distribution.


Figure 14: Boundary Layer Development
(a) Development of Displacement Thickness.

(b) Development of Momentum Thickness.

(c) Development of Shape Factor.


Figure 15: Distribution of RMS Values of Maximum Longitudinal Turbulence Velocity.


Figure 16: Distribution of Normalized Values of Turbulence.


Figure 17: Distribution of Skin Friction Coefficient.

## APPENDIX A:

DATA TABLES FOR PLOTS

TABLE A.1: PRESSURE COEFFICIENT DISTRIBUTION
All $C_{p}$-values are negative.

| x | Bare | 100-Grit | 80-Grit | 60-Grit | 40-Grit | 4-Grit |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 22 | 0.4350 | 0.4147 | 0.4339 | 0.4344 | 0.4118 | 0.3619 |
| 24 | 0.4450 | 0.4290 | 0.4294 | 0.4235 | 0.3641 | 0.3221 |
| 26 | 0.4190 | 0.4087 | 0.4012 | 0.3937 | 0.2854 | 0.2830 |
| 28 | 0.3950 | 0.3820 | 0.3778 | 0.3628 | 0.2453 | 0.2541 |
| 30 | 0.3520 | 0.3487 | 0.3265 | 0.2925 | 0.2183 | 0.2306 |
| 32 | 0.3400 | 0.3304 | 0.2918 | 0.2431 | 0.2017 | 0.2102 |
| 34 | 0.3190 | 0.3156 | 0.2571 | 0.2064 | 0.1865 | 0.1960 |
| 36 | 0.3120 | 0.2973 | 0.2119 | 0.1746 | 0.1736 | 0.1842 |
| 38 | 0.2900 | 0.2693 | 0.1719 | 0.1436 | 0.1622 | 0.1731 |
| 40 | 0.2560 | 0.2350 | 0.1402 | 0.1277 | 0.1543 | 0.1628 |
| 42 | 0.1970 | 0.1708 | 0.1244 | 0.1200 | 0.1466 | 0.1540 |
| 44 | 0.1370 | 0.1176 | 0.1161 | 0.1162 | 0.1389 | 0.1468 |
| 46 | 0.0980 | 0.0873 | 0.1101 | 0.1124 | 0.1327 | 0.1390 |
| 48 | 0.0803 | 0.0760 | 0.1048 | 0.1101 | 0.1265 | 0.1335 |
| 50 | 0.0742 | 0.0735 | 0.1014 | 0.1063 | 0.1211 | 0.1288 |
| 55 | 0.0657 | 0.0726 | 0.0938 | 0.0972 | 0.1095 | 0.1288 |
| 60 | 0.0680 | 0.0740 | 0.0878 | 0.0911 | 0.1005 | 0.1061 |
| 70 | 0.0684 | 0.0708 | 0.0786 | 0.0797 | 0.0866 | 0.0913 |
| 80 |  | 0.0649 | 0.0692 | 0.0721 | 0.0765 | 0.0811 |
| 90 |  | 0.0626 | 0.0632 | 0.0661 | 0.0695 | 0.0733 |
| 100 |  | 0.0587 | 0.0594 | 0.0607 | 0.0640 | 0.0683 |
| 110 |  | 0.0550 | 0.0564 | 0.0569 | 0.0601 | 0.0636 |
| 120 |  | 0.0523 |  |  | 0.0554 | 0.0605 |
| 130 |  | 0.0501 |  |  | 0.0524 | 0.0574 |
| 140 |  | 0.0479 |  |  | 0.0502 | 0.0551 |
| 150 |  | 0.0464 |  |  | 0.0479 | 0.0535 |
| 160 |  | 0.0449 |  |  | 0.0465 | 0.0512 |
| 170 |  |  |  |  | 0.0450 | 0.0496 |
| 180 |  |  |  |  | 0.0000 | 0.0481 |
| 190 |  |  |  |  | 0.0000 | 0.0473 |
| 200 |  |  |  |  | 0.0465 |  |

TABLE A. $2(\mathrm{a}):$ MEAN VELOCITY PROFILES FOR BARE CASE
All velocities given in $m / s$. Distance from surface ( $y$-value) given in mm .
y $\quad 22 \mathrm{~mm} 24 \mathrm{~mm} 26 \mathrm{~mm} 28 \mathrm{~mm} 30 \mathrm{~mm} 32 \mathrm{~mm} 34 \mathrm{~mm} 36 \mathrm{~mm} 38 \mathrm{~mm}$

| 5 to $10 \mathrm{~ms}^{-1}$ Calibration |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.3 | 5.97 | 3.28 | 1.67 | 0.83 | 0.50 | 0.47 | 0.41 |  |  |
| 0.4 | 9.17 | 5.43 | 3.35 | 1.97 | 1.54 | 1.24 | 0.60 | 0.65 | 0.84 |
| 0.5 |  | 9.87 | 6.93 | 4.86 | 4.16 | 3.66 | 2.54 | 2.04 | 1.27 |
| 0.6 |  |  |  | 7.63 | 6.79 | 6.05 | 4.80 | 4.33 | 2.93 |
| 0.7 |  |  |  |  | 9.48 | 8.66 | 7.27 | 7.14 | 5.71 |
| 0.8 |  |  |  |  |  |  | 9.64 | 9.54 | 8.63 |
| 10 to $20 \mathrm{~ms}^{-1}$ Calibration |  |  |  |  |  |  |  |  |  |


| 0.3 | 6.14 | 2.16 | 0.48 | 0.50 | 0.18 | 0.16 |  |  |  |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 0.4 | 9.26 |  |  |  |  | 0.28 | 0.22 | 0.44 |  |
| 0.5 | 13.86 | 8.90 | 4.90 |  | 3.56 | 2.00 | 0.84 | 1.02 | 0.72 |
| 0.6 | 16.06 | 12.32 | 8.40 | 8.36 |  |  | 2.86 | 3.50 | 1.70 |
| 0.7 | 17.18 | 14.96 | 11.64 | 11.24 | 9.20 | 6.40 | 5.44 | 6.18 | 3.86 |
| 0.8 | 17.58 | 16.48 | 14.30 | 13.76 | 11.82 | 9.22 | 7.28 | 8.62 | 6.86 |
| 0.9 | 17.72 | 17.22 | 15.98 | 15.54 | 13.84 | 11.78 | 10.08 | 10.86 | 9.46 |
| 1.0 | 17.72 | 17.50 | 16.90 | 16.62 | 15.66 | 13.88 | 12.32 | 12.80 | 11.36 |
| 1.0 | 18.14 | 18.04 | 17.56 | 16.96 | 16.78 | 14.22 | 13.48 | 12.58 | 11.96 |
| 1.2 | 18.10 | 18.04 | 17.72 | 17.62 | 17.60 | 15.94 | 15.74 | 14.92 | 14.40 |
| 1.4 | 18.08 | 18.02 | 17.70 | 17.68 | 17.68 | 16.36 | 16.32 | 16.02 | 15.82 |
| 1.6 | 17.98 | 17.96 | 17.66 | 17.64 | 17.68 | 16.40 | 16.48 | 16.46 | 16.50 |
| 1.8 | 17.94 | 17.92 | 17.62 | 17.60 | 17.64 | 16.38 | 16.46 | 16.50 | 16.62 |
| 2.0 | 17.90 | 17.88 | 17.58 | 17.56 | 17.58 | 16.34 | 16.42 | 16.50 | 16.60 |
| 2.2 |  | 17.84 | 17.54 | 17.52 | 17.56 | 16.30 | 16.40 | 16.46 | 16.54 |
| 2.4 | 17.82 | 17.80 | 17.52 | 17.50 | 17.52 | 16.28 | 16.36 | 16.44 | 16.48 |
| 2.6 | 17.78 | 17.76 | 17.48 | 17.46 | 17.48 | 16.26 | 16.34 | 16.40 | 16.44 |
| 3.6 | 17.60 | 17.60 | 17.34 | 17.34 | 17.36 | 16.16 | 16.24 | 16.28 | 16.30 |

TABLE A. 2(a): (cont'd)
$\mathrm{Y} \quad 40 \mathrm{~mm} 42 \mathrm{~mm} 44 \mathrm{~mm} 46 \mathrm{~mm} 48 \mathrm{~mm} 50 \mathrm{~mm} 55 \mathrm{~mm} 60 \mathrm{~mm} 70 \mathrm{~mm}$

| 5 to $10 \mathrm{~ms}^{-3}$ Calibration |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.3 |  | 2.00 | 2.88 | 4.97 | 6.72 | 8.34 | 9.55 | 9.94 | 9.79 |
| 0.4 | 1.37 | 2.11 | 3.53 | 5.92 | 7.70 | 9.09 |  |  |  |
| 0.5 | 1.42 | 2.77 | 4.49 | 7.24 | 8.61 |  |  |  |  |
| 0.6 | 2.50 | 4.09 | 5.92 | 8.51 | 9.73 |  |  |  |  |
| 0.7 | 4.45 | 5.78 | 7.41 | 9.73 |  |  |  |  |  |
| 0.8 | 7.11 | 7.51 | 8.96 |  |  |  |  |  |  |
| 0.9 | 9.02 | 9.44 |  |  |  |  |  |  |  |

10 to $20 \mathrm{~ms}^{-1}$ Calibration

| 0.3 |  | 1.86 | 2.44 | 4.28 | 6.42 | 8.08 | 9.42 | 9.86 | 9.98 |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 0.4 |  |  | 2.88 |  |  | 8.98 | 10.32 | 10.72 | 10.74 |
| 0.5 | 1.16 | 2.24 | 3.98 | 6.68 | 8.70 | 10.06 | 11.30 | 11.64 | 11.70 |
| 0.6 | 1.90 |  |  |  | 9.56 | 10.92 | 11.92 | 12.22 | 12.28 |
| 0.7 | 3.96 | 5.10 |  | 9.36 | 10.86 | 11.70 | 12.48 | 12.78 | 12.80 |
| 0.8 |  |  | 8.68 | 10.70 | 11.62 | 12.36 | 12.94 | 13.26 | 13.24 |
| 0.9 | 8.78 | 9.30 | 10.48 | 11.96 | 12.60 | 12.92 | 13.34 | 13.56 | 13.56 |
| 1.0 | 10.66 | 11.16 | 11.84 | 12.84 | 13.26 | 13.46 | 13.64 | 13.82 | 13.76 |
| 1.0 | 11.26 | 11.66 | 12.16 | 13.02 | 13.12 | 13.34 | 13.64 | 13.36 | 13.22 |
| 1.2 | 13.96 | 14.42 | 14.14 | 14.26 | 14.12 | 14.04 | 14.20 | 13.78 | 13.68 |
| 1.4 | 15.76 | 15.74 | 15.24 | 15.00 | 14.72 | 14.54 | 14.50 | 14.10 | 14.04 |
| 1.6 | 16.54 | 16.30 | 15.80 | 15.40 | 15.10 | 14.92 | 14.86 | 14.36 | 14.22 |
| 1.8 | 16.70 | 16.46 | 15.98 | 15.58 | 15.32 | 15.16 | 15.08 | 14.60 | 14.44 |
| 2.0 | 16.68 | 16.44 | 16.02 | 15.66 | 15.42 | 15.32 | 15.20 | 14.76 | 14.66 |
| 2.2 | 16.62 | 16.42 | 16.02 | 15.70 | 15.48 | 15.40 | 15.36 | 14.88 | 14.78 |
| 2.4 | 16.58 | 16.36 | 15.98 | 15.70 | 15.50 | 15.46 | 15.42 | 14.98 | 14.88 |
| 2.6 | 16.52 | 16.36 | 15.98 | 15.74 | 15.54 | 15.50 | 15.48 | 15.04 | 14.94 |
| 3.6 | 16.36 | 16.24 | 15.96 | 15.80 | 15.64 | 15.60 | 15.60 | 15.18 | 15.22 |

TABLE A. 2(b): MEAN VELOCITY PROFILES FOR 100-GRIT CASE
Y 22 mm 24 mm 26 mm 28 mm 30 mm 32 mm 34 mm 36 mm 38 mm 40 mm

| 0.3 | 2.20 | 0.78 | 0.20 | 0.12 | 0.14 | 0.12 |  |  |  |  |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 0.4 | 4.32 | 2.28 | 0.86 | 0.50 | 0.46 | 0.34 | 0.20 |  |  |  |
| 0.5 | 9.30 | 5.52 | 2.98 | 2.20 | 2.10 | 1.40 | 0.48 | 0.40 | 0.66 | 1.04 |
| 0.6 | 12.86 | 9.18 | 5.78 | 4.48 | 4.30 | 3.22 | 1.96 | 1.02 | 0.92 | 1.06 |
| 0.7 | 15.40 | 12.66 | 9.16 | 7.44 | 6.90 | 5.62 | 4.10 | 3.06 | 2.94 | 2.30 |
| 0.8 | 16.78 | 14.86 | 12.14 | 10.58 | 9.86 | 8.26 | 6.54 | 5.62 | 5.66 | 4.80 |
| 0.9 | 17.38 | 16.44 | 14.52 | 13.02 | 12.34 | 10.86 | 9.02 | 8.04 | 8.04 | 7.28 |
| 1.0 | 17.60 | 17.20 | 16.20 | 15.22 | 14.54 | 13.16 | 11.56 | 10.30 | 10.10 | 9.50 |
| 1.2 | 17.64 | 17.58 | 17.40 | 17.16 | 16.60 | 16.04 | 14.98 | 14.06 | 13.78 | 13.40 |
| 1.4 | 17.62 | 17.56 | 17.58 | 17.54 | 17.18 | 17.02 | 16.62 | 16.32 | 15.96 | 15.80 |
| 1.6 |  |  | 17.56 | 17.58 | 17.26 | 17.24 | 17.18 | 17.06 | 17.00 | 16.90 |
| 1.8 | 17.52 | 17.50 |  | 17.52 | 17.22 | 17.24 | 17.22 | 17.20 | 17.18 | 17.12 |
| 2.0 |  |  |  |  |  |  |  | 17.18 |  | 17.12 |
| 2.2 | 17.46 | 17.42 | 17.46 | 17.46 | 17.16 | 17.18 | 17.16 | 17.16 | 17.14 | 17.08 |
| 2.4 |  |  |  |  |  |  |  |  |  | 17.04 |
| 2.6 | 17.38 | 17.36 | 17.38 | 17.40 | 17.10 | 17.12 | 17.12 | 17.10 | 17.06 | 17.02 |
| 3.6 | 17.22 | 17.22 | 17.26 | 17.28 | 16.98 | 17.00 | 16.98 | 16.96 | 16.92 | 16.88 |

Y 42 mm 44 mm 46 mm 48 mm 50 mm 55 mm 60 mm 70 mm 90 mm 110 mm

|  |  |  |  |  | .48 | 4.38 | 6.28 | 7.52 | 8.22 | 8.26 |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 0.3 |  | 8.22 | 7.66 | 6.56 |  |  |  |  |  |  |
| 0.4 | 1.74 | 3.62 | 5.50 | 7.40 | 8.46 | 9.28 | 9.46 | 9.34 | 8.76 | 8.08 |
| 0.5 | 1.80 | 5.16 | 6.74 | 8.58 | 9.30 | 10.14 | 10.36 | 10.22 | 9.68 | 9.18 |
| 0.6 | 2.72 | 6.80 | 7.96 | 9.66 | 10.14 | 10.90 | 11.12 | 11.02 | 10.34 | 9.94 |
| 0.7 | 4.44 | 8.70 | 9.14 | 10.60 | 10.90 | 11.48 | 11.70 | 11.54 | 11.06 | 10.48 |
| 0.8 | 6.76 | 10.32 | 10.32 | 11.48 | 11.54 | 11.98 | 12.12 | 11.96 | 11.44 | 10.98 |
| 0.9 | 8.82 | 11.88 | 11.42 | 12.26 | 12.12 | 12.38 | 12.50 | 12.34 | 11.80 | 11.34 |
| 1.0 | 11.16 | 13.36 | 12.38 | 12.90 | 12.60 | 12.76 | 12.82 | 12.62 | 12.10 | 11.64 |
| 1.2 | 14.32 | 15.06 | 13.78 | 14.00 | 13.44 | 13.36 | 13.34 | 13.14 | 12.60 | 12.14 |
| 1.4 | 15.96 | 15.94 | 14.64 | 14.64 | 14.04 | 13.82 | 13.70 | 13.54 | 13.02 | 12.58 |
| 1.6 | 16.70 | 16.30 | 15.12 | 15.08 | 14.46 | 14.22 | 14.08 | 13.86 | 13.32 | 12.90 |
| 1.8 | 16.84 | 16.36 | 15.34 | 15.34 | 14.82 | 14.54 | 14.40 | 14.16 | 13.66 | 13.24 |
| 2.0 | 16.82 | 16.40 | 15.44 | 15.44 | 15.00 | 14.82 | 14.66 | 14.40 | 13.96 | 13.52 |
| 2.2 | 16.80 | 16.40 |  | 15.50 | 15.12 | 14.98 | 14.84 | 14.62 | 14.20 | 13.82 |
| 2.4 | 16.76 | 16.42 |  | 15.52 | 15.16 | 15.10 | 15.02 | 14.80 | 14.40 | 14.06 |
| 2.6 | 16.74 | 16.44 | 15.54 | 15.56 | 15.20 | 15.20 | 15.16 | 14.96 | 14.56 | 14.26 |
| 3.6 | 16.64 | 16.44 | 15.62 | 15.64 | 15.28 | 15.30 | 15.36 | 15.36 | 15.20 | 15.06 |

TABLE A.2(c): MEAN VELOCITY PROFILES FOR 80-GRIT CASE
22 mm 24 mm 26 mm 28 mm 30 mm 32 mm 34 mm 36 mm 38 mm 40 mm

| 0.3 | 0.78 | 0.68 | 0.44 | 0.38 | 0.62 | 1.28 | 2.38 | 3.62 | 4.66 | 6.14 |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 0.4 | 7.54 | 2.56 | 1.84 | 0.96 | 1.14 | 2.22 | 3.80 | 5.16 | 6.26 | 7.54 |
| 0.5 | 10.60 | 5.60 | 4.16 | 2.48 | 2.72 | 3.78 | 5.38 | 6.66 | 7.52 | 8.72 |
| 0.6 | 14.32 | 9.34 | 8.00 | 4.94 | 5.34 | 6.18 | 7.38 | 8.28 | 8.64 | 10.02 |
| 0.7 | 16.26 | 12.46 | 11.18 | 8.50 | 8.00 | 8.40 | 9.34 | 9.86 | 10.10 | 11.12 |
| 0.8 | 17.16 | 14.76 | 13.76 | 11.52 | 10.64 | 10.72 | 11.22 | 11.40 | 11.28 | 12.24 |
| 0.9 | 17.50 | 16.24 | 15.64 | 13.94 | 12.94 | 12.70 | 12.88 | 12.84 | 12.46 | 13.10 |
| 1.0 | 17.60 | 16.94 | 16.64 | 15.60 | 14.68 | 14.36 | 14.16 | 13.92 | 13.44 | 13.80 |
| 1.2 | 17.58 | 17.34 | 17.34 | 17.10 | 16.36 | 16.04 | 15.72 | 15.26 | 14.70 | 14.86 |
| 1.4 | 17.54 | 17.38 | 17.42 | 17.46 | 16.74 | 16.52 | 16.24 | 15.82 | 15.38 | 15.40 |
| 1.6 |  | 17.30 | 17.38 | 17.46 | 16.78 | 16.64 | 16.40 | 16.06 | 15.70 | 15.80 |
| 1.8 | 17.44 | 17.24 | 17.32 | 17.44 | 16.78 | 16.62 | 16.44 | 16.18 | 15.92 | 16.04 |
| 2.0 |  | 17.22 | 17.28 | 17.38 | 16.70 | 16.62 | 16.44 | 16.22 | 16.00 | 15.96 |
| 2.2 | 17.36 | 17.18 | 17.26 | 17.34 | 16.68 | 16.56 | 16.42 | 16.22 | 16.00 | 16.06 |
| 2.4 |  | 17.14 | 17.24 | 17.30 | 16.64 | 16.56 | 16.40 | 16.20 | 16.04 | 16.12 |
| 2.6 | 17.30 | 17.10 | 17.20 | 17.28 | 16.62 | 16.50 | 16.38 | 16.20 | 16.04 | 16.16 |
| 3.6 | 17.12 | 16.96 | 17.06 | 17.16 | 16.50 | 16.44 | 16.34 | 16.20 | 16.06 | 16.22 |

Y 42 mm 44 mm 46 mm 48 mm 50 mm 55 mm 60 mm 70 mm 90 mm 110 mm

| 0.3 | 6.14 | 7.20 | 7.18 | 7.98 | 8.06 | 8.04 | 7.94 | 7.94 | 7.46 | 7.76 |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 0.4 | 7.60 | 8.54 | 8.52 | 9.34 | 9.44 | 9.24 | 10.08 | 9.32 | 8.80 | 8.76 |
| 0.5 | 8.78 | 9.46 | 9.60 | 10.24 | 10.34 | 10.06 | 10.70 | 10.16 | 9.72 | 9.60 |
| 0.6 | 9.78 | 10.26 | 10.44 | 10.90 | 11.06 | 10.64 | 11.16 | 11.10 | 10.32 | 10.20 |
| 0.7 | 10.92 | 11.36 | 11.16 | 11.44 | 11.64 | 11.10 | 11.56 | 11.52 | 10.78 | 10.64 |
| 0.8 | 11.86 | 11.90 | 11.82 | 12.16 | 12.20 | 11.60 | 11.92 | 11.86 | 11.16 | 10.98 |
| 0.9 | 12.72 | 12.56 | 12.44 | 12.66 | 12.70 | 11.98 | 12.24 | 12.16 | 11.46 | 11.26 |
| 1.0 | 13.40 | 13.12 | 12.98 | 13.12 | 13.12 | 12.38 | 12.52 | 12.40 | 11.74 | 11.54 |
| 1.2 | 14.44 | 13.94 | 13.80 | 13.82 | 13.80 | 13.00 | 13.00 | 12.82 | 12.20 | 11.96 |
| 1.4 | 15.02 | 14.48 | 14.40 | 14.36 | 14.36 | 13.46 | 13.44 | 13.22 | 12.60 | 12.34 |
| 1.6 | 15.46 | 14.88 | 14.80 | 14.76 | 14.76 | 13.86 | 13.82 | 13.58 | 12.94 | 12.66 |
| 1.8 | 15.74 | 15.20 | 15.12 | 15.10 | 15.12 | 14.24 | 14.18 | 13.94 | 13.26 | 13.02 |
| 2.0 | 15.92 | 15.38 | 15.34 | 15.34 | 15.36 | 14.54 | 14.48 | 14.22 | 13.56 | 13.28 |
| 2.2 | 16.00 | 15.46 | 15.48 | 15.48 | 15.56 | 14.76 | 14.70 | 14.48 | 13.86 | 13.58 |
| 2.4 | 16.06 | 15.54 | 15.56 | 15.58 | 15.62 | 14.92 | 14.86 | 14.72 | 14.12 | 13.82 |
| 2.6 | 16.10 | 15.56 | 15.60 | 15.64 | 15.70 | 15.02 | 14.98 | 14.88 | 14.36 | 14.08 |
| 3.6 | 16.16 | 15.60 | 15.66 | 15.72 | 15.80 | 15.18 | 15.20 | 15.28 | 15.12 | 15.02 |

TABLE A. $2(\mathrm{~d})$ : MEAN VELOCITY PROFILES FOR 60-GRIT CASE
y 22 mm 24 mm 26 mm 28 mm 30 mm 32 mm 34 mm 36 mm 38 mm 40 mm

| 0.3 | 6.30 | 4.64 | 2.10 | 1.72 | 0.60 | 0.68 | 1.88 | 2.82 | 3.98 | 4.98 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.4 | 10.40 | 7.62 | 5.50 | 5.00 | 4.02 | 3.48 | 4.86 | 5.46 | 5.96 | 6.54 |
| 0.5 | 13.68 | 11.50 | 8.84 | 8.02 | 7.46 | 6.56 | 8.06 | 8.28 | 8.38 | 8.86 |
| 0.6 | 15.76 | 14.28 | 12.08 | 11.26 | 10.50 | 10.02 | 10.92 | 10.84 | 10.72 | 10.94 |
| 0.7 | 16.86 | 16.00 | 15.10 | 13.82 | 12.98 | 12.60 | 13.16 | 12.98 | 12.66 | 12.76 |
| 0.8 | 17.36 | 16.88 | 15.96 | 15.36 | 14.96 | 14.34 | 14.74 | 14.38 | 14.08 | 13.98 |
| 0.9 | 17.52 | 17.28 | 16.86 | 16.50 | 16.10 | 15.74 | 15.74 | 15.42 | 14.98 | 14.92 |
| 1.0 | 17.54 | 17.42 | 17.22 | 17.04 | 16.64 | 16.40 | 16.30 | 15.98 | 15.48 | 15.40 |
| 1.2 |  | 17.44 | 17.36 | 17.28 | 17.02 | 16.84 | 16.66 | 16.36 | 15.84 | 15.74 |
| 1.4 | 17.48 | 17.42 | 17.36 | 17.32 | 17.10 | 16.92 | 16.72 | 16.44 | 15.92 | 15.84 |
| 1.6 |  |  |  |  | 17.06 |  |  | 16.46 | 15.94 |  |
| 1.8 | 17.40 | 17.32 | 17.28 | 17.24 | 17.02 | 16.86 | 16.68 | 16.44 | 15.94 | 15.86 |
| 2.2 | 17.32 | 17.26 | 17.22 | 17.18 | 16.98 | 16.82 | 16.66 | 16.44 | 15.96 | 15.88 |
| 2.6 | 17.24 | 17.20 | 17.16 | 17.12 | 16.92 | 16.80 | 16.66 | 16.44 | 15.98 | 15.88 |
| 3.6 | 17.08 | 17.06 | 17.02 | 17.00 | 16.84 | 16.72 | 16.60 | 16.44 | 15.98 | 15.92 |

Y 42 mm 44 mm 46 mm 48 mm 50 mm 55 mm 60 mm 70 mm 90 mm 110 mm

| 0.3 | 5.48 | 5.78 | 6.48 | 6.22 | 6.32 | 6.70 | 7.16 | 8.30 | 8.16 | 8.22 |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 0.4 | 7.22 | 7.56 | 7.98 | 7.80 | 8.08 | 8.24 | 8.48 | 9.48 | 9.24 | 9.32 |
| 0.5 | 9.16 | 9.28 | 9.50 | 9.16 | 9.40 | 9.28 | 9.40 | 10.18 | 9.88 | 9.86 |
| 0.6 | 10.72 | 10.90 | 10.96 | 10.60 | 10.66 | 10.34 | 10.22 | 10.76 | 10.48 | 10.40 |
| 0.7 | 12.64 | 12.46 | 12.32 | 11.88 | 11.74 | 11.20 | 11.02 | 11.26 | 10.88 | 10.78 |
| 0.8 | 13.82 | 13.62 | 13.54 | 12.96 | 12.84 | 12.12 | 11.68 | 11.68 | 11.24 | 11.06 |
| 0.9 | 14.78 | 14.62 | 14.46 | 13.98 | 13.78 | 12.92 | 12.38 | 12.16 | 11.54 | 11.34 |
| 1.0 | 15.30 | 15.20 | 15.08 | 14.72 | 14.44 | 13.56 | 12.98 | 12.52 | 11.82 | 11.60 |
| 1.2 | 15.72 | 15.70 | 15.70 | 15.56 | 15.28 | 14.42 | 13.88 | 13.24 | 12.26 | 12.02 |
| 1.4 | 15.82 | 15.84 | 15.88 | 15.86 | 15.76 | 14.94 | 14.56 | 13.86 | 12.72 | 12.44 |
| 1.6 |  | 15.88 | 15.94 | 15.96 | 15.88 | 15.16 | 14.92 | 14.34 | 13.14 | 12.78 |
| 1.8 | 15.86 | 15.88 | 15.94 | 15.98 | 15.92 | 15.26 | 15.12 | 14.68 | 13.50 | 13.06 |
| 2.0 |  |  | 15.94 | 16.00 | 15.94 | 15.30 | 15.22 | 14.94 | 13.86 | 13.40 |
| 2.2 | 15.86 | 15.88 | 15.94 | 16.02 | 15.94 | 15.32 | 15.28 | 15.08 | 14.14 | 13.70 |
| 2.4 |  |  | 15.94 | 16.02 | 15.94 | 15.32 | 15.30 | 15.20 | 14.44 | 13.98 |
| 2.6 | 15.86 | 15.88 | 15.94 | 16.02 | 15.94 | 15.34 | 15.32 | 15.28 | 14.70 | 14.22 |
| 3.6 | 15.88 | 15.90 | 15.94 | 16.04 | 15.96 | 15.34 | 15.34 | 15.36 | 15.32 | 15.16 |

TABLE A. $2(\mathrm{e}):$ MEAN VELOCITY PROFILES FOR 40-GRIT CASE
Y $\quad 22 \mathrm{~mm} 24 \mathrm{~mm} 26 \mathrm{~mm} 28 \mathrm{~mm} 30 \mathrm{~mm} 36 \mathrm{~mm} 40 \mathrm{~mm}$

| 0.3 | 9.96 | 9.42 | 9.10 | 9.14 | 9.58 | 9.40 | 8.80 |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 0.4 | 11.36 | 10.86 | 10.58 | 10.32 | 10.62 | 10.30 | 9.92 |
| 0.5 | 12.08 | 11.58 | 11.34 | 11.06 | 11.18 | 10.80 | 10.46 |
| 0.6 | 12.72 | 12.22 | 11.96 | 11.72 | 11.68 | 11.30 | 10.94 |
| 0.7 | 13.28 | 12.86 | 12.48 | 12.18 | 12.20 | 11.68 | 11.40 |
| 0.8 | 13.82 | 13.36 | 12.96 | 12.64 | 12.62 | 12.08 | 11.72 |
| 0.9 | 14.36 | 13.88 | 13.44 | 13.06 | 13.00 | 12.42 | 12.08 |
| 1.0 | 14.86 | 14.40 | 13.90 | 13.46 | 13.38 | 12.76 | 12.38 |
| 1.2 | 15.64 | 15.28 | 14.78 | 14.38 | 14.08 | 13.42 | 12.96 |
| 1.4 | 16.14 | 15.92 | 15.50 | 15.12 | 14.76 | 14.02 | 13.58 |
| 1.6 | 16.52 | 16.44 | 16.08 | 15.76 | 15.26 | 14.56 | 14.08 |
| 1.8 | 16.76 | 16.74 | 16.42 | 16.24 | 15.72 | 15.04 | 14.58 |
| 2.0 | 16.84 | 16.88 | 16.70 | 16.54 | 16.02 | 15.42 | 15.04 |
| 2.2 | 16.86 | 16.92 | 16.80 | 16.68 | 16.20 | 15.74 | 15.42 |
| 2.4 | 16.84 | 16.92 | 16.84 | 16.74 | 16.28 | 15.92 | 15.70 |
| 2.6 | 16.80 | 16.90 | 16.84 | 16.76 | 16.32 | 16.04 | 15.86 |
| 3.6 | 16.66 | 16.78 | 16.78 | 16.74 | 16.32 | 16.12 | 16.08 |
| 4.6 |  |  |  |  | 16.28 | 16.10 | 16.06 |
| 5.6 |  |  |  |  | 16.24 | 16.08 | 16.04 |

Y $\quad 46 \mathrm{~mm} 50 \mathrm{~mm} 60 \mathrm{~mm} 70 \mathrm{~mm} 90 \mathrm{~mm} 110 \mathrm{~mm}$

|  |  |  |  |  |  |  |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 0.3 | 8.66 | 8.52 | 7.88 | 8.06 | 7.26 | 7.12 |
| 0.4 | 9.82 | 9.50 | 9.12 | 9.16 | 8.50 | 8.38 |
| 0.5 | 10.34 | 10.12 | 9.80 | 9.76 | 9.16 | 9.06 |
| 0.6 | 10.80 | 10.52 | 10.20 | 10.16 | 9.68 | 9.52 |
| 0.7 | 11.22 | 10.90 | 10.56 | 10.48 | 10.02 | 9.92 |
| 0.8 | 11.54 | 11.20 | 10.90 | 10.80 | 10.38 | 10.26 |
| 0.9 | 11.84 | 11.52 | 11.18 | 11.06 | 10.62 | 10.50 |
| 1.0 | 12.12 | 11.78 | 11.42 | 11.28 | 10.84 | 10.72 |
| 1.2 | 12.64 | 12.26 | 11.90 | 11.72 | 11.22 | 11.12 |
| 1.4 | 13.18 | 12.76 | 12.34 | 12.10 | 11.62 | 11.44 |
| 1.6 | 13.64 | 13.22 | 12.74 | 12.46 | 11.94 | 11.76 |
| 1.8 | 14.14 | 13.66 | 13.14 | 12.82 | 12.26 | 12.04 |
| 2.0 | 14.56 | 14.08 | 13.50 | 13.16 | 12.56 | 12.32 |
| 2.2 | 14.94 | 14.42 | 13.82 | 13.46 | 12.86 | 12.58 |
| 2.4 | 15.32 | 14.80 | 14.18 | 13.84 | 13.14 | 12.86 |
| 2.6 | 15.58 | 15.06 | 14.50 | 14.14 | 13.42 | 13.12 |
| 3.6 | 16.02 | 15.60 | 15.42 | 15.26 | 14.66 | 14.30 |
| 4.6 | 16.00 |  |  |  |  |  |
| 5.6 | 15.98 |  |  |  |  |  |

TABLE A. 2(f): MEAN VELOCITY PROFILES FOR 4-GRIT CASE
Y 22 mm 24 mm 26 mm 28 mm 31 mm 40 mm

| 0.3 | 7.72 | 7.66 | 8.40 | 8.56 | 8.90 | 9.00 |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 0.4 | 8.62 | 8.74 | 9.54 | 9.74 | 9.92 | 10.22 |
| 0.5 | 8.96 | 9.30 | 10.04 | 10.30 | 10.40 | 10.92 |
| 0.6 | 9.06 | 9.54 | 10.20 | 10.54 | 10.58 | 11.28 |
| 0.7 | 9.10 | 9.66 | 10.32 | 10.72 | 10.70 | 11.48 |
| 0.8 | 8.90 | 9.52 | 10.12 | 10.58 | 10.64 | 11.64 |
| 0.9 | 8.78 | 9.36 | 10.04 | 10.48 | 10.54 | 11.64 |
| 1.0 | 8.66 | 9.26 | 9.90 | 10.36 | 10.40 | 11.58 |
| 1.2 | 8.62 | 9.16 | 9.76 | 10.24 | 10.32 | 11.46 |
| 1.4 | 8.82 | 9.22 | 9.78 | 10.22 | 10.28 | 11.48 |
| 1.6 | 9.22 | 9.40 | 9.84 | 10.24 | 10.24 | 11.44 |
| 1.8 | 9.78 | 9.72 | 10.06 | 10.30 | 10.26 | 11.36 |
| 2.0 | 10.48 | 10.20 | 10.34 | 10.48 | 10.32 | 11.28 |
| 2.2 | 11.46 | 10.98 | 10.96 | 10.96 | 10.66 | 11.30 |
| 2.4 | 12.52 | 11.86 | 11.72 | 11.60 | 11.14 | 11.56 |
| 2.6 | 13.62 | 12.80 | 12.46 | 12.20 | 11.58 | 11.82 |
| 2.8 | 14.60 | 13.68 | 13.24 | 12.86 |  |  |
| 3.0 | 15.38 | 14.48 | 13.98 | 13.52 |  |  |
| 3.2 | 16.08 | 15.32 | 14.82 | 14.34 |  |  |
| 3.4 | 16.56 | 15.98 | 15.52 | 15.06 |  |  |
| 3.6 | 16.84 | 16.42 | 16.06 | 15.66 | 14.50 | 13.46 |
| 4.6 | 17.02 | 17.00 | 16.96 | 16.78 | 16.04 | 15.24 |
| 5.6 | 16.92 | 16.90 | 16.92 | 16.86 | 16.18 | 15.86 |
| 6.6 |  |  |  |  | 16.14 | 15.88 |

TABLE 5(f): (cont'd)
Y $\quad 50 \mathrm{~mm} 60 \mathrm{~mm} 70 \mathrm{~mm} 90 \mathrm{~mm} 110 \mathrm{~mm}$

| 0.3 | 9.04 | 9.60 | 8.58 | 8.16 | 7.62 |
| ---: | ---: | ---: | ---: | ---: | ---: |
| 0.4 | 10.40 | 10.68 | 10.22 | 9.60 | 9.22 |
| 0.5 | 11.12 | 11.32 | 11.02 | 10.50 | 10.16 |
| 0.6 | 11.60 | 11.70 | 11.52 | 11.08 | 10.78 |
| 0.7 | 11.86 | 12.02 | 11.84 | 11.46 | 11.22 |
| 0.8 | 12.06 | 12.22 | 12.12 | 11.80 | 11.52 |
| 0.9 | 12.12 | 12.34 | 12.30 | 12.04 | 11.80 |
| 1.0 | 12.18 | 12.40 | 12.44 | 12.24 | 12.02 |
| 1.2 | 12.28 | 12.60 | 12.68 | 12.54 | 12.36 |
| 1.4 | 12.36 | 12.74 | 12.88 | 12.80 | 12.66 |
| 1.6 | 12.32 | 12.78 | 12.98 | 13.00 | 12.90 |
| 1.8 | 12.26 | 12.74 | 13.04 | 13.14 | 13.08 |
| 2.0 | 12.18 | 12.72 | 13.02 | 13.24 | 13.22 |
| 2.2 | 12.24 | 12.76 | 13.08 | 13.34 | 13.32 |
| 2.4 | 12.36 | 12.86 | 13.16 | 13.44 | 13.46 |
| 2.6 | 12.44 | 12.92 | 13.26 | 13.52 | 13.60 |
| 3.6 | 13.36 | 13.56 | 13.70 | 13.96 | 14.08 |
| 4.6 | 14.70 | 14.52 | 14.46 | 14.48 | 14.54 |
| 5.6 | 15.56 | 15.34 | 15.18 | 15.00 | 14.98 |
| 6.6 | 15.74 | 15.66 | 15.56 | 15.40 | 15.34 |
| 7.6 | 15.74 | 15.68 | 15.64 | 15.58 | 15.56 |
| 8.6 |  |  |  | 15.60 | 15.62 |

TABLE A.3(a): TURBULENCE INTENSITY PROFILES FOR BARE CASE
All turbulence intensities, $\left(\mathrm{u}^{2}\right)^{1 / 2}$, normalized by $\widetilde{\mathrm{U}}_{r}$. All distances from surface ( $y$-values) in mm .
Y $22 \mathrm{~mm} \quad 24 \mathrm{~mm} \quad 26 \mathrm{~mm} \quad 28 \mathrm{~mm} \quad 30 \mathrm{~mm} \quad 32 \mathrm{~mm} \quad 34 \mathrm{~mm} \quad 36 \mathrm{~mm} \quad 38 \mathrm{~mm}$

| 5 to $10 \mathrm{~ms}^{-1}$ Calibration |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.3 | 0.0265 | 0.0246 | 0.0233 | 0.0165 | 0.0121 | 0.0123 | 0.0086 |  |  |
| 0.4 | 0.0292 | 0.0329 | 0.0338 | 0.0294 | 0.0319 | 0.0347 | 0.0176 | 0.0267 | 0.0326 |
| 0.5 |  | 0.0362 | 0.0459 | 0.0433 | 0.0478 | 0.0531 | 0.0589 | 0.0834 | 0.0570 |
| 0.6 |  |  |  | 0.0509 | 0.0551 | 0.0597 | 0.0771 | 0.1163 | 0.1323 |
| 0.7 |  |  |  |  | 0.0553 | 0.0615 | 0.0836 | 0.1146 | 0.1574 |
| 0.8 |  |  |  |  |  | 0.0615 | 0.0815 | 0.1045 | 0.1376 |

10 to $20 \mathrm{~ms}^{-1}$ Calibration

| 0.3 | 0.0299 | 0.0232 | 0.0104 | 0.0073 | 0.0053 | 0.0031 |  |  |  |  |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 0.4 | 0.0325 |  |  |  |  |  | 0.0102 | 0.0069 | 0.0174 |  |
| 0.5 | 0.0257 | 0.0420 | 0.0443 | 0.0000 | 0.0512 | 0.0411 | 0.0347 | 0.0410 |  |  |
| 0.6 | 0.0151 | 0.0421 | 0.0527 | 0.0613 | 0.0000 | 0.0000 | 0.0760 | 0.1040 | 0.0311 |  |
| 0.7 | 0.0069 | 0.0268 | 0.0476 | 0.0561 | 0.0699 | 0.0767 | 0.0907 | 0.1095 | 0.0924 |  |
| 0.8 | 0.0041 | 0.0152 | 0.0337 | 0.0459 | 0.0631 | 0.0804 | 0.0961 | 0.1048 | 0.1313 |  |
| 0.9 | 0.0037 | 0.0068 | 0.0213 | 0.0306 | 0.0551 | 0.0712 | 0.0960 | 0.1004 | 0.1313 |  |
| 1.0 | 0.0038 | 0.0039 | 0.0096 | 0.0163 | 0.0334 | 0.0600 | 0.0851 | 0.0944 | 0.1180 |  |
| 1.0 | 0.0039 | 0.0039 | 0.0047 | 0.0151 | 0.0215 | 0.0479 | 0.0677 | 0.0923 | 0.1179 |  |
| 1.2 | 0.0039 | 0.0041 | 0.0043 | 0.0045 | 0.0051 | 0.0157 | 0.0255 | 0.0669 | 0.1428 |  |
| 1.4 | 0.0038 | 0.0041 | 0.0045 | 0.0047 | 0.0050 | 0.0049 | 0.0125 | 0.0376 | 0.0936 |  |
| 1.6 | 0.0038 | 0.0040 | 0.0044 | 0.0048 | 0.0052 | 0.0051 | 0.0058 | 0.0088 | 0.0551 |  |
| 1.8 | 0.0038 | 0.0040 | 0.0043 | 0.0047 | 0.0050 | 0.0053 | 0.0065 | 0.0084 | 0.0180 |  |
| 2.0 | 0.0038 | 0.0039 | 0.0043 | 0.0047 | 0.0051 | 0.0052 | 0.0066 | 0.0087 | 0.0143 |  |
| 2.2 |  | 0.0040 | 0.0042 | 0.0046 | 0.0049 | 0.0050 | 0.0062 | 0.0085 | 0.0125 |  |
| 2.4 | 0.0037 | 0.0039 | 0.0041 | 0.0046 | 0.0049 | 0.0051 | 0.0062 | 0.0082 | 0.0116 |  |
| 2.6 | 0.0037 | 0.0039 | 0.0040 | 0.0045 | 0.0048 | 0.0050 | 0.0060 | 0.0079 | 0.0112 |  |
| 3.6 | 0.0037 | 0.0037 | 0.0038 | 0.0044 | 0.0047 | 0.0047 | 0.0055 | 0.0066 | 0.0104 |  |

TABLE A. $3(\mathrm{a}):$ (cont'd)


| 10 to $20 \mathrm{~ms}^{-7}$ Calibration |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.3 |  | 0.0747 | 0.1029 | 0.1470 | 0.1576 | 0. | 0.1268 |  |  |
| 0.4 |  |  | 0.1195 |  |  | 0. | 0 |  |  |
| 0.5 | 0.0545 | 0.1049 | 0.1529 | 0.1 | 0.1685 | 0. | 7 |  |  |
| 0.6 | 0.0995 |  |  |  | 0.1690 | 0.1576 | 0.1297 |  | 0.0997 |
| 0.7 | 0.1715 | 0.1 |  | 0.1833 | 0.1673 | 0.1559 | 0.1323 | 0.1139 | 0.0979 |
| 0.8 |  |  | 0.1947 | 0.1770 | 0.1626 | 0.1525 | 0.1309 | 0.1139 | 0 |
| 0.9 | 0.1883 | 0.1951 | 0.1858 | 0.1622 | 0.1513 | 0.1470 | 0.1287 | 0.1136 | 60 |
| 1.0 | 0.1711 | 0.1778 | 0.1685 | 0.1513 | 0.1399 | 0.1348 | 0.1249 | 0.1127 | 5 |
| 1.0 | 0.1690 | 0.1698 | 0.1622 | 0.1428 | 0.1382 | 0.1331 | 0.1193 | 0.1075 |  |
| 1.2 | 0.1235 | 0.1100 | 0.1120 | 0.1049 | 0.1058 | 0.1095 | 0.1076 | 0.1027 |  |
| 1.4 | 0.0643 | 0.0669 | 0.0807 | 0.0793 | 0.0853 | 0.0929 | 0.0992 | 0.0929 | 847 |
| 1.6 | 0.0307 | 0.0452 | 0.0596 | 0.0616 | 0.0679 | 0.0731 | . 0819 | 0.0840 | 0.0807 |
| 1.8 | 0.0242 | 0.0368 | 0.0496 | 0.0511 | 0.0528 | 0.0588 | 0.0711 | 0.0737 |  |
| 2.0 | 0.0222 | 0.0330 | 0.0435 | 0.0440 | 0.0467 | 0.0489 | 0.0589 | 0.064 | 0.0687 |
| 2.2 | 0.0194 | 0.0292 | 0.0367 | 0.0369 | 0.0375 | 0.0395 | 0.0480 | 0.056 | , |
| 2.4 | 0.0174 | 0.0259 | 0.0324 | 0.0340 | 0.0340 | 0.0338 | 0.0360 | 0.0485 | . 0.0564 |
| 2.6 | 0.0153 | 0.0228 | 0.0281 | 0.0296 | 0.0300 | 0.0308 | 0.0330 | 0.0404 | 0.0521 |
| 3.6 | 0.0098 | 0.0137 | 0.0157 | 0.0159 | 0.0169 | 0.0168 | 0.0180 | 0.0201 | 0.0276 |

TABLE A.3(b): TURBULENCE INTENSITY PROFILES FOR 100-GRIT CASE
Y $22 \mathrm{~mm} 24 \mathrm{~mm} \quad 26 \mathrm{~mm} \quad 28 \mathrm{~mm} \quad 30 \mathrm{~mm} \quad 32 \mathrm{~mm} \quad 34 \mathrm{~mm} \quad 36 \mathrm{~mm} \quad 38 \mathrm{~mm} 40 \mathrm{~mm}$

| 0.3 | 0.0129 | 0.0097 | 0.0032 | 0.0020 | 0.0023 | 0.0016 |  |  |  |  |  |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 0.4 | 0.0243 | 0.0234 | 0.0152 | 0.0110 | 0.0109 | 0.0099 | 0.0051 |  |  |  |  |
| 0.5 | 0.0355 | 0.0344 | 0.0314 | 0.0313 | 0.0327 | 0.0319 | 0.0206 | 0.0148 | 0.0256 | 0.0453 |  |
| 0.6 | 0.0286 | 0.0386 | 0.0401 | 0.0404 | 0.0421 | 0.0412 | 0.0417 | 0.0404 | 0.0385 | 0.0497 |  |
| 0.7 | 0.0188 | 0.0339 | 0.0525 | 0.0595 | 0.0676 | 0.0651 | 0.0701 | 0.0849 | 0.0999 | 0.0912 |  |
| 0.8 | 0.0101 | 0.0244 | 0.0456 | 0.0563 | 0.0609 | 0.0735 | 0.0753 | 0.0889 | 0.1056 | 0.1215 |  |
| 0.9 | 0.0052 | 0.0144 | 0.0323 | 0.0489 | 0.0548 | 0.0679 | 0.0789 | 0.0885 | 0.1037 | 0.1192 |  |
| 1.0 | 0.0037 | 0.0071 | 0.0200 | 0.0328 | 0.0412 | 0.0581 | 0.0735 | 0.0892 | 0.1031 | 0.1117 |  |
| 1.2 | 0.0037 | 0.0038 | 0.0049 | 0.0096 | 0.0152 | 0.0269 | 0.0487 | 0.0715 | 0.0791 | 0.0857 |  |
| 1.4 | 0.0037 | 0.0038 | 0.0040 | 0.0043 | 0.0044 | 0.0074 | 0.0172 | 0.0274 | 0.0384 | 0.0469 |  |
| 1.6 |  |  | 0.0041 | 0.0046 | 0.0047 | 0.0047 | 0.0050 | 0.0075 | 0.0110 | 0.0233 |  |
| 1.8 | 0.0037 | 0.0037 |  | 0.0045 | 0.0047 | 0.0050 | 0.0057 | 0.0067 | 0.0106 | 0.0207 |  |
| 2.0 |  |  |  |  |  |  |  |  |  |  | 0.0072 |

Y $\quad 42 \mathrm{~mm} \quad 44 \mathrm{~mm} \quad 46 \mathrm{~mm} \quad 48 \mathrm{~mm} \quad 50 \mathrm{~mm} \quad 55 \mathrm{~mm} \quad 60 \mathrm{~mm} \quad 70 \mathrm{~mm} \quad 90 \mathrm{~mm} \quad 110 \mathrm{~mm}$
$\begin{array}{lllllllllll}0.3 & 0.0973 & 0.1310 & 0.1316 & 0.1217 & 0.1089 & 0.1047 & 0.1051 & 0.1072 & 0.1055\end{array}$
$\begin{array}{llllllllllll}0.4 & 0.0681 & 0.1176 & 0.1327 & 0.1304 & 0.1205 & 0.1084 & 0.1039 & 0.1029 & 0.1067 & 0.1095\end{array}$
$\begin{array}{llllllllllll}0.5 & 0.0712 & 0.1331 & 0.1357 & 0.1321 & 0.1193 & 0.1068 & 0.1020 & 0.0995 & 0.1028 & 0.1064\end{array}$
$\begin{array}{lllllllllllll}0.6 & 0.0999 & 0.1420 & 0.1382 & 0.1424 & 0.1211 & 0.1061 & 0.0993 & 0.0945 & 0.0983 & 0.1016\end{array}$
$\begin{array}{llllllllllll}0.7 & 0.1263 & 0.1437 & 0.1403 & 0.1348 & 0.1227 & 0.1073 & 0.0980 & 0.0912 & 0.0911 & 0.0955\end{array}$
$\begin{array}{llllllllllll}0.8 & 0.1382 & 0.1382 & 0.1399 & 0.1348 & 0.1236 & 0.1080 & 0.0980 & 0.0885 & 0.0872 & 0.0899\end{array}$
$\begin{array}{llllllllllll}0.9 & 0.1315 & 0.1268 & 0.1344 & 0.1304 & 0.1224 & 0.1084 & 0.0981 & 0.0869 & 0.0837 & 0.0857\end{array}$
$\begin{array}{llllllllllll}1.0 & 0.1183 & 0.1108 & 0.1263 & 0.1243 & 0.1187 & 0.1079 & 0.0976 & 0.0855 & 0.0812 & 0.0821\end{array}$
$\begin{array}{lllllllllllll}1.2 & 0.0825 & 0.0816 & 0.1031 & 0.1040 & 0.1071 & 0.1037 & 0.0952 & 0.0833 & 0.0775 & 0.0773\end{array}$
$\begin{array}{lllllllllllll}1.4 & 0.0533 & 0.0591 & 0.0761 & 0.0809 & 0.0911 & 0.0953 & 0.0912 & 0.0807 & 0.0743 & 0.0743\end{array}$
$\begin{array}{llllllllllllll}1.6 & 0.0399 & 0.0485 & 0.0565 & 0.0612 & 0.0733 & 0.0833 & 0.0840 & 0.0777 & 0.0723 & 0.0715\end{array}$
$\begin{array}{lllllllllllll}1.8 & 0.0350 & 0.0416 & 0.0456 & 0.0463 & 0.0563 & 0.0719 & 0.0759 & 0.0740 & 0.0695 & 0.0693\end{array}$
$\begin{array}{llllllllllllllll}2.0 & 0.0298 & 0.0344 & 0.0368 & 0.0369 & 0.0440 & 0.0580 & 0.0667 & 0.0693 & 0.0668 & 0.0668\end{array}$
$\begin{array}{lllllllllllll}2.2 & 0.0243 & 0.0288 & 0.0314 & 0.0352 & 0.0477 & 0.0579 & 0.0628 & 0.0631 & 0.0635\end{array}$
$\begin{array}{lllllllllllll}2.4 & 0.0208 & 0.0246 & 0.0269 & 0.0301 & 0.0367 & 0.0487 & 0.0569 & 0.0595 & 0.0607\end{array}$
$\begin{array}{llllllllllllllll}2.6 & 0.0166 & 0.0205 & 0.0231 & 0.0235 & 0.0252 & 0.0307 & 0.0375 & 0.0507 & 0.0565 & 0.0580\end{array}$
$\begin{array}{lllllllllllllll}3.6 & 0.0085 & 0.0102 & 0.0114 & 0.0116 & 0.0131 & 0.0149 & 0.0171 & 0.0232 & 0.0345 & 0.0417\end{array}$

TABLE A.3(c): TURBULENCE INTENSITY PROFILES FOR 80-GRIT CASE
Y $22 \mathrm{~mm} \quad 24 \mathrm{~mm} \quad 26 \mathrm{~mm} \quad 28 \mathrm{~mm} \quad 30 \mathrm{~mm} \quad 32 \mathrm{~mm} \quad 34 \mathrm{~mm} \quad 36 \mathrm{~mm} \quad 38 \mathrm{~mm} \quad 40 \mathrm{~mm}$

| 0.3 | 0.0240 | 0.0190 | 0.0226 | 0.0302 | 0.0468 | 0.0785 | 0.1115 | 0.1289 | 0.1212 | 0.1241 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 0.4 | 0.0300 | 0.0365 | 0.0389 | 0.0616 | 0.0665 | 0.1045 | 0.1348 | 0.1399 | 0.1289 | 0.1280 |
| 0.5 | 0.0307 | 0.0496 | 0.0543 | 0.0749 | 0.0792 | 0.1192 | 0.1395 | 0.1399 | 0.1313 | 0.1291 |
| 0.6 | 0.0228 | 0.0476 | 0.0575 | 0.0800 | 0.0852 | 0.1155 | 0.1340 | 0.1378 | 0.1310 | 0.1317 |
| 0.7 | 0.0133 | 0.0377 | 0.0523 | 0.0784 | 0.0888 | 0.1143 | 0.1308 | 0.1352 | 0.1310 | 0.1327 |
| 0.8 | 0.0067 | 0.0282 | 0.0420 | 0.0696 | 0.0839 | 0.1083 | 0.1235 | 0.1293 | 0.1302 | 0.1313 |
| 0.9 | 0.0043 | 0.0190 | 0.0287 | 0.0555 | 0.0719 | 0.0961 | 0.1109 | 0.1193 | 0.1261 | 0.1280 |
| 1.0 | 0.0039 | 0.0117 | 0.0202 | 0.0424 | 0.0528 | 0.0763 | 0.0952 | 0.1067 | 0.1176 | 0.1219 |
| 1.2 | 0.0039 | 0.0064 | 0.0099 | 0.0226 | 0.0233 | 0.0405 | 0.0617 | 0.0792 | 0.0976 | 0.1047 |
| 1.4 | 0.0039 | 0.0056 | 0.0074 | 0.0136 | 0.0158 | 0.0251 | 0.0407 | 0.0581 | 0.0761 | 0.0848 |
| 1.6 |  | 0.0051 | 0.0068 | 0.0101 | 0.0120 | 0.0166 | 0.0236 | 0.0333 | 0.0532 | 0.0601 |
| 1.8 | 0.0037 | 0.0047 | 0.0061 | 0.0085 | 0.0106 | 0.0137 | 0.0171 | 0.0228 | 0.0316 | 0.0404 |
| 2.0 |  | 0.0046 | 0.0056 | 0.0076 | 0.0094 | 0.0119 | 0.0146 | 0.0169 | 0.0223 | 0.0277 |
| 2.2 | 0.0036 | 0.0043 | 0.0053 | 0.0070 | 0.0085 | 0.0107 | 0.0125 | 0.0142 | 0.0171 | 0.0205 |
| 2.4 |  | 0.0042 | 0.0050 | 0.0065 | 0.0078 | 0.0097 | 0.0114 | 0.0122 | 0.0144 | 0.0165 |
| 2.6 | 0.0035 | 0.0041 | 0.0049 | 0.0062 | 0.0071 | 0.0086 | 0.0103 | 0.0110 | 0.0117 | 0.0138 |
| 3.6 | 0.0034 | 0.0037 | 0.0040 | 0.0047 | 0.0052 | 0.0059 | 0.0067 | 0.0070 | 0.0075 | 0.0080 |

Y $42 \mathrm{~mm} \quad 44 \mathrm{~mm} \quad 46 \mathrm{~mm} \quad 48 \mathrm{~mm} \quad 50 \mathrm{~mm} \quad 55 \mathrm{~mm} \quad 60 \mathrm{~mm} \quad 70 \mathrm{~mm} \quad 90 \mathrm{~mm} \quad 110 \mathrm{~mm}$

| 0.3 | 0.1175 | 0.1084 | 0.1047 | 0.1027 | 0.1016 | 0.0960 | 0.0963 | 0.0989 | 0.1037 | 0.1068 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 0.4 | 0.1223 | 0.1108 | 0.1061 | 0.1025 | 0.1008 | 0.0925 | 0.0880 | 0.0945 | 0.1013 | 0.1044 |
| 0.5 | 0.1240 | 0.1131 | 0.1071 | 0.1031 | 0.1001 | 0.0892 | 0.0844 | 0.0887 | 0.0944 | 0.0989 |
| 0.6 | 0.1259 | 0.1151 | 0.1088 | 0.1040 | 0.1007 | 0.0868 | 0.0819 | 0.0805 | 0.0877 | 0.0917 |
| 0.7 | 0.1284 | 0.1175 | 0.1108 | 0.1053 | 0.1020 | 0.0855 | 0.0805 | 0.0769 | 0.0832 | 0.0860 |
| 0.8 | 0.1293 | 0.1189 | 0.1129 | 0.1071 | 0.1031 | 0.0853 | 0.0796 | 0.0747 | 0.0791 | 0.0821 |
| 0.9 | 0.1280 | 0.1176 | 0.1131 | 0.1075 | 0.1035 | 0.0852 | 0.0788 | 0.0732 | 0.0759 | 0.0787 |
| 1.0 | 0.1241 | 0.1145 | 0.1113 | 0.1060 | 0.1028 | 0.0845 | 0.0776 | 0.0717 | 0.0733 | 0.0757 |
| 1.2 | 0.1095 | 0.1049 | 0.1037 | 0.1005 | 0.0981 | 0.0820 | 0.0757 | 0.0696 | 0.0695 | 0.0719 |
| 1.4 | 0.0933 | 0.0901 | 0.0903 | 0.0891 | 0.0889 | 0.0780 | 0.0725 | 0.0675 | 0.0664 | 0.0685 |
| 1.6 | 0.0737 | 0.0737 | 0.0757 | 0.0772 | 0.0780 | 0.0725 | 0.0680 | 0.0644 | 0.0637 | 0.0657 |
| 1.8 | 0.0527 | 0.0547 | 0.0612 | 0.0628 | 0.0663 | 0.0640 | 0.0609 | 0.0599 | 0.0609 | 0.0632 |
| 2.0 | 0.0342 | 0.0384 | 0.0463 | 0.0487 | 0.0527 | 0.0539 | 0.0528 | 0.0549 | 0.0580 | 0.0603 |
| 2.2 | 0.0257 | 0.0296 | 0.0335 | 0.0366 | 0.0405 | 0.0451 | 0.0457 | 0.0499 | 0.0549 | 0.0577 |
| 2.4 | 0.0197 | 0.0222 | 0.0257 | 0.0280 | 0.0325 | 0.0355 | 0.0369 | 0.0441 | 0.0520 | 0.0552 |
| 2.6 | 0.0162 | 0.0177 | 0.0200 | 0.0218 | 0.0246 | 0.0285 | 0.0305 | 0.0374 | 0.0487 | 0.0525 |
| 3.6 | 0.0087 | 0.0088 | 0.0090 | 0.0092 | 0.0097 | 0.0093 | 0.0101 | 0.0140 | 0.0250 | 0.0338 |

TABLE A.3(d): TURBULENCE INTENSITY PROFILES FOR 60-GRIT CASE
Y $22 \mathrm{~mm} \quad 24 \mathrm{~mm} \quad 26 \mathrm{~mm} \quad 28 \mathrm{~mm} \quad 30 \mathrm{~mm} \quad 32 \mathrm{~mm} \quad 34 \mathrm{~mm} \quad 36 \mathrm{~mm} \quad 38 \mathrm{~mm} \quad 40 \mathrm{~mm}$

| 0.3 | 0.0286 | 0.0284 | 0.0296 | 0.0343 | 0.0184 | 0.0236 | 0.0581 | 0.0776 | 0.0817 | 0.0865 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 0.4 | 0.0394 | 0.0469 | 0.0537 | 0.0516 | 0.0488 | 0.0547 | 0.0623 | 0.0800 | 0.0859 | 0.0928 |
| 0.5 | 0.0323 | 0.0476 | 0.0589 | 0.0592 | 0.0463 | 0.0525 | 0.0599 | 0.0792 | 0.0883 | 0.0965 |
| 0.6 | 0.0194 | 0.0331 | 0.0504 | 0.0551 | 0.0386 | 0.0481 | 0.0519 | 0.0700 | 0.0793 | 0.0904 |
| 0.7 | 0.0102 | 0.0204 | 0.0229 | 0.0439 | 0.0341 | 0.0367 | 0.0405 | 0.0537 | 0.0624 | 0.0735 |
| 0.8 | 0.0060 | 0.0100 | 0.0225 | 0.0307 | 0.0242 | 0.0288 | 0.0271 | 0.0349 | 0.0441 | 0.0556 |
| 0.9 | 0.0041 | 0.0061 | 0.0103 | 0.0174 | 0.0159 | 0.0182 | 0.0187 | 0.0228 | 0.0251 | 0.0308 |
| 1.0 | 0.0041 | 0.0046 | 0.0061 | 0.0081 | 0.0095 | 0.0102 | 0.0143 | 0.0163 | 0.0180 | 0.0214 |
| 1.2 |  | 0.0041 | 0.0044 | 0.0047 | 0.0057 | 0.0070 | 0.0080 | 0.0096 | 0.0096 | 0.0134 |
| 1.4 | 0.0039 | 0.0041 | 0.0045 | 0.0046 | 0.0051 | 0.0059 | 0.0072 | 0.0083 | 0.0083 | 0.0089 |
| 1.6 |  |  |  |  | 0.0050 |  |  |  | 0.0073 | 0.0071 |
| 1.8 | 0.0040 | 0.0041 | 0.0042 | 0.0044 | 0.0048 | 0.0052 | 0.0058 | 0.0063 | 0.0063 | 0.0066 |
| 2.2 | 0.0039 | 0.0039 | 0.0041 | 0.0042 | 0.0046 | 0.0048 | 0.0050 | 0.0053 | 0.0052 | 0.0053 |
| 2.6 | 0.0038 | 0.0039 | 0.0039 | 0.0041 | 0.0043 | 0.0044 | 0.0045 | 0.0046 | 0.0046 | 0.0047 |
| 3.6 | 0.0037 | 0.0037 | 0.0038 | 0.0039 | 0.0040 | 0.0039 | 0.0040 | 0.0039 | 0.0038 | 0.0039 |

Y $\quad 42 \mathrm{~mm} \quad 44 \mathrm{~mm} \quad 46 \mathrm{~mm} \quad 48 \mathrm{~mm} \quad 50 \mathrm{~mm} \quad 55 \mathrm{~mm} \quad 60 \mathrm{~mm} \quad 70 \mathrm{~mm} \quad 90 \mathrm{~mm} \quad 110 \mathrm{~mm}$

| 0.3 | 0.0909 | 0.0959 | 0.1041 | 0.1077 | 0.1119 | 0.1140 | 0.1155 | 0.1084 | 0.1043 | 0.1055 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 0.4 | 0.0975 | 0.1007 | 0.1048 | 0.1080 | 0.1107 | 0.1100 | 0.1121 | 0.1044 | 0.0991 | 0.0987 |
| 0.5 | 0.1019 | 0.1048 | 0.1076 | 0.1091 | 0.1100 | 0.1080 | 0.1085 | 0.0999 | 0.0933 | 0.0932 |
| 0.6 | 0.1008 | 0.1040 | 0.1073 | 0.1095 | 0.1093 | 0.1049 | 0.1053 | 0.0961 | 0.0867 | 0.0863 |
| 0.7 | 0.0849 | 0.0955 | 0.1031 | 0.1076 | 0.1077 | 0.1028 | 0.1025 | 0.0925 | 0.0827 | 0.0821 |
| 0.8 | 0.0689 | 0.0828 | 0.0916 | 0.1025 | 0.1032 | 0.0991 | 0.1001 | 0.0897 | 0.0792 | 0.0785 |
| 0.9 | 0.0484 | 0.0636 | 0.0764 | 0.0907 | 0.0940 | 0.0929 | 0.0957 | 0.0871 | 0.0763 | 0.0752 |
| 1.0 | 0.0293 | 0.0464 | 0.0601 | 0.0757 | 0.0845 | 0.0845 | 0.0908 | 0.0851 | 0.0743 | 0.0729 |
| 1.2 | 0.0173 | 0.0228 | 0.0319 | 0.0495 | 0.0632 | 0.0699 | 0.0796 | 0.0805 | 0.0716 | 0.0693 |
| 1.4 | 0.0098 | 0.0151 | 0.0202 | 0.0260 | 0.0341 | 0.0492 | 0.0652 | 0.0741 | 0.0692 | 0.0665 |
| 1.6 |  | 0.0093 | 0.0107 | 0.0177 | 0.0230 | 0.0316 | 0.0493 | 0.0649 | 0.0665 | 0.0641 |
| 1.8 | 0.0073 | 0.0081 | 0.0089 | 0.0104 | 0.0161 | 0.0225 | 0.0350 | 0.0544 | 0.0631 | 0.0615 |
| 2.0 |  |  | 0.0078 | 0.0087 | 0.0102 | 0.0157 | 0.0255 | 0.0444 | 0.0593 | 0.0593 |
| 2.2 | 0.0058 | 0.0063 | 0.0070 | 0.0076 | 0.0089 | 0.0105 | 0.0195 | 0.0341 | 0.0557 | 0.0569 |
| 2.4 |  |  | 0.0062 | 0.0068 | 0.0074 | 0.0087 | 0.0117 | 0.0270 | 0.0503 | 0.0541 |
| 2.6 | 0.0050 | 0.0052 | 0.0056 | 0.0061 | 0.0067 | 0.0075 | 0.0098 | 0.0204 | 0.0445 | 0.0512 |
| 3.6 | 0.0040 | 0.0042 | 0.0043 | 0.0046 | 0.0047 | 0.0048 | 0.0055 | 0.0071 | 0.0158 | 0.0283 |

TABLE A. $3(\mathrm{e}):$ TURBULENCE INTENSITY PROFILES FOR 40-GRIT CASE
Y $22 \mathrm{~mm} 24 \mathrm{~mm} \quad 26 \mathrm{~mm} \quad 28 \mathrm{~mm} \quad 30 \mathrm{~mm} \quad 36 \mathrm{~mm} \quad 40 \mathrm{~mm}$

| 0.3 | 0.0807 | 0.0888 | 0.0929 | 0.0943 | 0.0908 | 0.0909 | 0.0947 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 0.4 | 0.0733 | 0.0803 | 0.0853 | 0.0883 | 0.0847 | 0.0855 | 0.0887 |
| 0.5 | 0.0692 | 0.0755 | 0.0799 | 0.0829 | 0.0804 | 0.0816 | 0.0845 |
| 0.6 | 0.0669 | 0.0720 | 0.0753 | 0.0781 | 0.0763 | 0.0776 | 0.0807 |
| 0.7 | 0.0663 | 0.0697 | 0.0724 | 0.0755 | 0.0725 | 0.0752 | 0.0772 |
| 0.8 | 0.0659 | 0.0689 | 0.0705 | 0.0733 | 0.0696 | 0.0727 | 0.0749 |
| 0.9 | 0.0653 | 0.0677 | 0.0687 | 0.0719 | 0.0668 | 0.0707 | 0.0727 |
| 1.0 | 0.0636 | 0.0656 | 0.0665 | 0.0703 | 0.0641 | 0.0687 | 0.0709 |
| 1.2 | 0.0567 | 0.0600 | 0.0604 | 0.0633 | 0.0587 | 0.0644 | 0.0684 |
| 1.4 | 0.0476 | 0.0511 | 0.0528 | 0.0556 | 0.0521 | 0.0589 | 0.0635 |
| 1.6 | 0.0355 | 0.0397 | 0.0440 | 0.0477 | 0.0457 | 0.0533 | 0.0588 |
| 1.8 | 0.0242 | 0.0288 | 0.0343 | 0.0372 | 0.0361 | 0.0467 | 0.0539 |
| 2.0 | 0.0161 | 0.0195 | 0.0236 | 0.0269 | 0.0273 | 0.0382 | 0.0471 |
| 2.2 | 0.0113 | 0.0139 | 0.0168 | 0.0193 | 0.0204 | 0.0290 | 0.0386 |
| 2.4 | 0.0092 | 0.0106 | 0.0124 | 0.0147 | 0.0153 | 0.0220 | 0.0295 |
| 2.6 | 0.0075 | 0.0085 | 0.0099 | 0.0110 | 0.0119 | 0.0161 | 0.0225 |
| 3.6 | 0.0045 | 0.0045 | 0.0048 | 0.0052 | 0.0055 | 0.0065 | 0.0076 |
| 4.6 |  |  |  |  | 0.0039 | 0.0042 | 0.0046 |
| 5.6 |  |  |  |  | 0.0035 | 0.0036 | 0.0037 |

Y $\quad 46 \mathrm{~mm} \quad 50 \mathrm{~mm} \quad 60 \mathrm{~mm} \quad 70 \mathrm{~mm} \quad 90 \mathrm{~mm} \quad 110 \mathrm{~mm}$

```
0.3}00.09800.0951 0.1004 0.1024 0.1061 0.1077
0.4 0.0908 0.0889 0.0943 0.0960 0.1032 0.1048
0.5 0.0859 0.0843 0.0883 0.0900 0.0976 0.1001
0.6}00.0820 0.0804 0.0843 0.0849 0.0916 0.0948
0.7 0.0785 0.0772 0.0807 0.0813 0.0873 0.0895
0.8 0.0763 0.0749 0.0775 0.0784 0.0829 0.0852
0.9 0.0748 0.0724 0.0747 0.0761 0.0799 0.0820
1.0}00.07310.0711 0.0732 0.0743 0.0776 0.0792
1.2 0.0703 0.0685 0.0703 0.0712 0.0743 0.0757
1.4 0.0673 0.0653 0.0676 0.0691 0.0717 0.0732
1.6 0.0641 0.0621 0.0652 0.0672 0.0697 0.0711
1.8 0.0600 0.0585 0.0625 0.0647 0.0681 0.0696
2.0}00.0551 0.0544 0.0599 0.0629 0.0664 0.0679
2.2 0.0495 0.0504 0.0567 0.0608 0.0651 0.0665
2.4 0.0399}00.0425 0.0523 0.0569 0.0627 0.0645
2.6}00.0329 0.0356 0.0475 0.0528 0.0604 0.0627
3.6 0.0094 0.0106 0.0172 0.0260 0.0440 0.0513
4.6 0.0052
5.6 0.0039
```

TABLE A.3(f): TURBULENCE INTENSITY PROFILES FOR 4-GRIT CASE

| $Y$ | 22 mm | 24 mm | 26 mm | 28 mm | 31 mm | 40 mm |  |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 0.3 | 0.0951 | 0.1013 | 0.1036 | 0.1031 | 0.0983 | 0.0961 |  |
| 0.4 | 0.0917 | 0.0999 | 0.1011 | 0.1013 | 0.0953 | 0.0920 |  |
| 0.5 | 0.0879 | 0.0953 | 0.0964 | 0.0976 | 0.0915 | 0.0879 |  |
| 0.6 | 0.0845 | 0.0904 | 0.0923 | 0.0937 | 0.0884 | 0.0845 |  |
| 0.7 | 0.0829 | 0.0877 | 0.0893 | 0.0909 | 0.0863 | 0.0821 |  |
| 0.8 | 0.0812 | 0.0847 | 0.0876 | 0.0895 | 0.0851 | 0.0807 |  |
| 0.9 | 0.0809 | 0.0832 | 0.0860 | 0.0876 | 0.0845 | 0.0793 |  |
| 1.0 | 0.0815 | 0.0828 | 0.0856 | 0.0869 | 0.0835 | 0.0789 |  |
| 1.2 | 0.0861 | 0.0851 | 0.0864 | 0.0869 | 0.0833 | 0.0789 |  |
| 1.4 | 0.0953 | 0.0901 | 0.0897 | 0.0887 | 0.0845 | 0.0801 |  |
| 1.6 | 0.1056 | 0.0979 | 0.0947 | 0.0921 | 0.0861 | 0.0801 |  |
| 1.8 | 0.1153 | 0.1060 | 0.1011 | 0.0957 | 0.0885 | 0.0807 |  |
| 2.0 | 0.1245 | 0.1133 | 0.1068 | 0.1012 | 0.0915 | 0.0808 |  |
| 2.2 | 0.1313 | 0.1208 | 0.1132 | 0.1065 | 0.0960 | 0.0820 |  |
| 2.4 | 0.1328 | 0.1255 | 0.1183 | 0.1112 | 0.1003 | 0.0836 |  |
| 2.6 | 0.1273 | 0.1261 | 0.1205 | 0.1148 | 0.1032 | 0.0853 |  |
| 2.8 | 0.1156 | 0.1216 | 0.1196 | 0.1161 |  |  |  |
| 3.0 | 0.1015 | 0.1132 | 0.1149 | 0.1145 |  |  |  |
| 3.2 | 0.0807 | 0.0984 | 0.1044 | 0.1084 |  |  |  |
| 3.4 | 0.0616 | 0.0793 | 0.0895 | 0.0983 |  |  |  |
| 3.6 | 0.0459 | 0.0629 | 0.0729 | 0.0843 | 0.0868 | 0.0859 |  |
| 4.6 | 0.0147 | 0.0184 | 0.0221 | 0.0269 | 0.0324 | 0.0536 |  |
| 5.6 | 0.0069 | 0.0081 | 0.0093 | 0.0109 | 0.0124 | 0.0195 |  |
| 6.6 |  |  |  |  |  | 0.0064 | 0.0092 |

TABLE A.3(f): (cont'd)

| $Y$ | 50 mm | 60 mm | 70 mm | 90 mm | 110 mm |
| :--- | :--- | :--- | :--- | :--- | :--- |
| 0.3 | 0.0956 | 0.0960 | 0.1004 | 0.1045 | 0.1067 |
| 0.4 | 0.0908 | 0.0893 | 0.0945 | 0.1012 | 0.1048 |
| 0.5 | 0.0853 | 0.0835 | 0.0868 | 0.0943 | 0.0996 |
| 0.6 | 0.0805 | 0.0787 | 0.0813 | 0.0877 | 0.0925 |
| 0.7 | 0.0775 | 0.0749 | 0.0771 | 0.0832 | 0.0868 |
| 0.8 | 0.0751 | 0.0727 | 0.0732 | 0.0777 | 0.0821 |
| 0.9 | 0.0743 | 0.0707 | 0.0712 | 0.0744 | 0.0781 |
| 1.0 | 0.0736 | 0.0703 | 0.0691 | 0.0715 | 0.0751 |
| 1.2 | 0.0732 | 0.0683 | 0.0659 | 0.0667 | 0.0704 |
| 1.4 | 0.0741 | 0.0687 | 0.0640 | 0.0625 | 0.0661 |
| 1.6 | 0.0747 | 0.0691 | 0.0639 | 0.0603 | 0.0631 |
| 1.8 | 0.0755 | 0.0709 | 0.0649 | 0.0593 | 0.0597 |
| 2.0 | 0.0765 | 0.0719 | 0.0672 | 0.0591 | 0.0581 |
| 2.2 | 0.0765 | 0.0723 | 0.0673 | 0.0589 | 0.0575 |
| 2.4 | 0.0769 | 0.0721 | 0.0675 | 0.0587 | 0.0568 |
| 2.6 | 0.0771 | 0.0724 | 0.0683 | 0.0591 | 0.0556 |
| 3.6 | 0.0767 | 0.0708 | 0.0667 | 0.0587 | 0.0540 |
| 4.6 | 0.0615 | 0.0603 | 0.0584 | 0.0539 | 0.0509 |
| 5.6 | 0.0291 | 0.0356 | 0.0397 | 0.0428 | 0.0427 |
| 6.6 | 0.0124 | 0.0158 | 0.0199 | 0.0258 | 0.0295 |
| 7.6 | 0.0071 | 0.0083 | 0.0100 | 0.0139 | 0.0172 |
| 8.6 |  |  |  | 0.0078 | 0.0097 |

## TABLE A. 4 : FRICTION VELOCITY VERSUS NORMALIZED DISTANCE FROM REATTACHMENT

| $\begin{gathered} \text { Bare } \\ \text { (measured) } \end{gathered}$ |  | Bare |  | 100-Grit |  | 80-Grit |  | 60-Grit |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{x}_{\boldsymbol{E}}-\mathrm{l}_{\text {B }}$ | u. | $\mathrm{X}_{5}-1_{8}$ | U* | $\mathrm{X}_{\mathrm{s}}-1{ }_{1}$ | $u_{\text {. }}$ | $\mathrm{X}_{5}-\mathrm{l}_{8}$ | $\mathrm{u}^{*}$ | $\mathrm{X}_{\mathrm{E}^{-1}} \mathrm{l}_{\mathrm{B}}$ | U* |
| $\frac{18}{18}$ |  | $1_{8}$ |  | $1_{8}$ |  | $1{ }_{1}$ |  | $\frac{x_{8} 1_{B}}{l_{B}}$ |  |
| 0.0000 | 0.2650 | 0.0000 | 0.9426 | 0.0000 | 0.9484 | 0.0000 | 0.9367 | 0.0000 | 0.9367 |
| 0.0833 | 0.4600 | 0.0833 | 0.9190 | 0.0769 | 0.8895 | 0.1111 | 0.9190 | 0.1000 | 0.9308 |
| 0.1667 | 0.6000 | 0.1667 | 0.9013 | 0.1538 | 0.8895 | 0.2222 | 0.9249 | 0.2000 | 0.9308 |
| 0.2500 | 0.7000 | 0.2500 | 0.8895 | 0.2308 | 0.8601 | 0.3333 | 0.9072 | 0.3000 | 0.9367 |
| 0.3333 | 0.7550 | 0.3333 | 0.8778 | 0.4231 | 0.8424 | 0.4444 | 0.8778 | 0.4000 | 0.9367 |
| 0.5417 | 0.8050 | 0.5417 | 0.8778 | 0.6154 | 0.8365 | 0.5556 | 0.8719 | 0.5000 | 0.9367 |
| 0.7500 | 0.8300 | 0.7500 | 0.8483 | 1.0000 | 0.8306 | 0.6667 | 0.8719 | 0.6000 | 0.9308 |
| 1.1667 | 0.8200 | 1.1667 | 0.8424 | 1.7690 | 0.8012 | 0.7778 | 0.8719 | 0.8500 | 0.8954 |
| 1.5833 | 0.8150 |  |  | 2.5380 | 0.7835 | 1.0556 | 0.8306 | 1.1000 | 0.8836 |
| 2.0000 | 0.8050 |  |  |  |  | 1.3333 | 0.8247 | 1.6000 | 0.8542 |
| 2.4167 | 0.7950 |  |  |  |  | 1.8889 | 0.8130 | 2.6000 | 0.7953 |
| 2.8333 | 0.7850 |  |  |  |  | 3.0000 | 0.7835 | 3.6000 | 0.7776 |
| 3.2500 | 0.7750 |  |  |  |  | 4.1111 | 0.7717 |  |  |
| 3.6667 | 0.7650 |  |  |  |  |  |  |  |  |
| 4.0833 | 0.7600 |  |  |  |  |  |  |  |  |

TABLE A.5(a): NORMALIZED MEAN VELOCITY PROFILES FOR BARE CASE

| Y + | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 5.051 | 7.547 | 8.767 | 6.261 | 11.440 | 8.283 | 13.340 | 9.600 |
| 6.734 | 7.962 | 11.690 | 7.674 | 15.250 | 9.867 | 17.790 | 11.000 |
| 8.418 | 10.450 | 14.610 | 9.761 | 19.060 | 12.070 | 22.240 | 12.300 |
| 10.100 | 15.430 | 17.530 | 12.870 | 22.870 | 14.180 | 26.680 | 13.900 |
| 11.790 | 21.810 | 20.460 | 16.110 | 26.680 | 16.220 | 13.340 | 9.171 |
| 13.470 | 28.340 | 23.380 | 19.480 | 11.440 | 7.133 | 22.240 | 12.430 |
| 15.150 | 35.620 | 8.767 | 5.304 | 19.060 | 11.130 | 26.680 | 13.660 |
| 5.051 | 7.019 | 11.690 | 6.261 | 26.680 | 15.600 | 31.130 | 15.510 |
| 8.418 | 8.453 | 14.610 | 8.652 | 30.500 | 17.830 | 35.580 | 16.600 |
| 11.790 | 19.250 | 23.380 | 18.870 | 34.310 | 19.930 | 40.030 | 18.000 |
| 15.150 | 35.090 | 26.300 | 22.780 | 38.120 | 21.400 | 44.470 | 18.940 |
| 16.840 | 42.110 | 29.220 | 25.740 | 38.120 | 21.700 | 44.470 | 18.740 |
| 16.840 | 44.000 | 29.220 | 26.430 | 45.740 | 23.770 | 53.370 | 20.170 |
| 20.200 | 54.420 | 35.070 | 30.740 | 53.370 | 25.000 | 62.260 | 21.030 |
| 23.570 | 59.400 | 40.910 | 33.130 | 60.990 | 25.670 | 71.160 | 21.570 |
| 26.940 | 61.510 | 46.760 | 34.350 | 68.610 | 25.970 | 80.050 | 21.890 |
| 30.300 | 62.110 | 52.600 | 34.740 | 76.240 | 26.100 | 88.950 | 22.030 |
| 33.670 | 62.040 | 58.450 | 34.830 | 83.860 | 26.170 | 97.840 | 22.110 |
| 37.040 | 61.960 | 64.290 | 34.830 | 91.490 | 26.170 | 106.700 | 22.140 |
| 40.410 | 61.740 | 70.140 | 34.740 | 99.110 | 26.230 | 115.600 | 22.200 |
| 43.770 | 61.740 | 75.980 | 34.740 | 137.200 | 26.330 | 160.100 | 22.340 |
| 60.610 | 61.280 | 105.200 | 34.700 |  |  |  |  |

TABLE A.5(a): (cont'd)
$x=50 \mathrm{~mm} \quad x=55 \mathrm{~mm} \quad x=60 \mathrm{~mm} \quad x=70 \mathrm{~mm}$

| $Y$, | $U_{+}$ | $Y_{+}$ | $U_{+}$ | $Y_{+}$ | $U_{+}$ | $Y_{+}$ | $U_{+}$ |  |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 14.390 | 11.050 | 15.340 | 11.860 | 15.820 | 11.980 | 16.060 | 11.620 |  |
| 19.190 | 12.040 | 15.340 | 11.700 | 15.820 | 11.880 | 16.060 | 11.850 |  |
| 14.390 | 10.700 | 20.460 | 12.820 | 21.090 | 12.920 | 21.410 | 12.750 |  |
| 19.190 | 11.890 | 25.570 | 14.040 | 26.370 | 14.020 | 26.760 | 13.890 |  |
| 23.980 | 13.320 | 30.690 | 14.810 | 31.640 | 14.720 | 32.110 | 14.580 |  |
| 28.780 | 14.460 | 35.800 | 15.500 | 36.910 | 15.400 | 37.460 | 15.190 |  |
| 33.580 | 15.500 | 40.910 | 16.070 | 42.190 | 15.980 | 42.820 | 15.720 |  |
| 38.370 | 16.370 | 46.030 | 16.570 | 47.460 | 16.340 | 48.170 | 16.100 |  |
| 43.170 | 17.110 | 51.140 | 16.940 | 52.730 | 16.650 | 53.520 | 16.330 |  |
| 47.970 | 17.830 | 51.140 | 16.940 | 52.730 | 16.100 | 53.520 | 15.690 |  |
| 47.970 | 17.670 | 61.370 | 17.640 | 63.280 | 16.600 | 64.220 | 16.240 |  |
| 57.560 | 18.600 | 71.600 | 18.010 | 73.820 | 16.990 | 74.930 | 16.670 |  |
| 67.150 | 19.260 | 81.830 | 18.460 | 84.370 | 17.300 | 85.630 | 16.880 |  |
| 76.750 | 19.760 | 92.060 | 18.730 | 94.920 | 17.590 | 96.340 | 17.140 |  |
| 86.340 | 20.080 | 102.300 | 18.880 | 105.500 | 17.780 | 107.000 | 17.400 |  |
| 95.930 | 20.290 | 112.500 | 19.080 | 116.000 | 17.930 | 117.700 | 17.550 |  |
| 105.500 | 20.400 | 122.700 | 19.160 | 126.600 | 18.050 | 128.400 | 17.660 |  |
| 115.100 | 20.480 | 133.000 | 19.230 | 137.100 | 18.120 | 139.200 | 17.740 |  |
| 124.700 | 20.530 | 184.100 | 19.380 | 189.800 | 18.290 | 192.700 | 18.070 |  |
| 172.700 | 20.660 |  |  |  |  |  |  |  |

TABLE A.5(b): NORMALIZED MEAN VELOCITY PROFILES FOR 100-GRIT CASE

| $Y$. | $\mathrm{U}_{+}$ | Y | U | Y | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ | Y. | $\mathrm{U}_{+}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 5.051 | 9.358 | 8.767 | 9.522 | 11.440 | 10.470 | 12.870 | 11.140 | 14.870 | 10.540 |
| 6.734 | 13.660 | 11.690 | 11.960 | 15.250 | 12.330 | 17.150 | 12.530 | 19.820 | 11.900 |
| 8.418 | 19.470 | 14.610 | 14.650 | 19.060 | 14.300 | 21.440 | 13.780 | 24.780 | 13.000 |
| 10.100 | 25.660 | 17.530 | 17.300 | 22.870 | 16.100 | 25.730 | 15.020 | 29.730 | 13.970 |
| 11.790 | 32.830 | 20.460 | 19.870 | 26.680 | 17.670 | 30.020 | 16.150 | 34.690 | 14.720 |
| 13.470 | 38.940 | 23.380 | 22.430 | 30.500 | 19.130 | 34.310 | 17.100 | 39.640 | 15.360 |
| 15.150 | 44.830 | 26.300 | 24.830 | 34.310 | 20.430 | 38.600 | 17.960 | 44.600 | 15.870 |
| 16.840 | 50.420 | 29.220 | 26.910 | 38.120 | 21.500 | 42.880 | 18.670 | 49.560 | 16.360 |
| 20.200 | 56.830 | 35.070 | 29.960 | 45.740 | 23.330 | 51.460 | 19.910 | 59.470 | 17.130 |
| 23.570 | 60.150 | 40.910 | 31.830 | 53.370 | 24.400 | 60.040 | 20.800 | 69.380 | 17.720 |
| 26.940 | 61.510 | 46.760 | 32.870 | 60.990 | 25.130 | 68.610 | 21.420 | 79.290 | 18.230 |
| 30.300 | 61.740 | 52.600 | 33.350 | 68.610 | 25.570 | 77.190 | 21.960 | 89.200 | 18.640 |
| 33.670 | 61.890 | 58.450 | 33.570 | 76.240 | 25.730 | 85.770 | 22.220 | 99.110 | 19.000 |
| 37.040 | 61.890 | 75.980 | 33.780 | 83.860 | 25.830 | 94.350 | 22.400 | 109.000 | 19.210 |
| 40.410 | 61.960 | 105.200 | 33.960 | 91.490 | 25.870 | 102.900 | 22.460 | 118.900 | 19.360 |
| 43.770 | 62.040 |  |  | 99.110 | 25.930 | 111.500 | 22.520 | 128.800 | 19.490 |
| 60.610 | 62.040 |  |  | 137.200 | 26.070 | 154.400 | 22.640 | 178.400 | 19.620 |

$x=60 \mathrm{~mm} \quad \mathrm{x}=70 \mathrm{~mm} \quad \mathrm{x}=90 \mathrm{~mm} \quad \mathrm{x}=110 \mathrm{~mm}$

| Y+ | $\mathrm{U}_{+}$ | Y+ | $\mathrm{U}_{+}$ | $\mathrm{Y}_{+}$ | $\mathrm{U}_{+}$ | Y+ | $\mathrm{U}_{+}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 15.530 | 10.130 | 15.820 | 9.904 | 15.270 | 9.561 | 14.930 | 8.373 |
| 20.710 | 11.610 | 21.090 | 11.250 | 20.360 | 10.930 | 19.910 | 10.310 |
| 25.890 | 12.710 | 26.370 | 12.310 | 25.450 | 12.080 | 24.890 | 11.720 |
| 31.070 | 13.640 | 31.640 | 13.280 | 30.540 | 12.910 | 29.870 | 12.690 |
| 36.250 | 14.360 | 36.910 | 13.900 | 35.630 | 13.800 | 34.840 | 13.380 |
| 41.420 | 14.870 | 42.190 | 14.410 | 40.720 | 14.280 | 39.820 | 14.010 |
| 46.600 | 15.340 | 47.460 | 14.870 | 45.810 | 14.730 | 44.800 | 14.470 |
| 51.780 | 15.730 | 52.730 | 15.200 | 50.900 | 15.100 | 49.780 | 14.860 |
| 62.130 | 16.370 | 63.280 | 15.830 | 61.080 | 15.730 | 59.730 | 15.490 |
| 72.490 | 16.810 | 73.820 | 16.310 | 71.260 | 16.250 | 69.690 | 16.060 |
| 82.850 | 17.280 | 84.370 | 16.700 | 81.440 | 16.630 | 79.640 | 16.460 |
| 93.200 | 17.670 | 94.920 | 17.060 | 91.620 | 17.050 | 89.600 | 16.900 |
| 103.600 | 17.990 | 105.500 | 17.350 | 101.800 | 17.420 | 99.560 | 17.260 |
| 113.900 | 18.210 | 116.000 | 17.610 | 112.000 | 17.720 | 109.500 | 17.640 |
| 124.300 | 18.430 | 126.600 | 17.830 | 122.200 | 17.970 | 119.500 | 17.950 |
| 134.600 | 18.600 | 137.100 | 18.020 | 132.300 | 18.170 | 129.400 | 18.200 |
| 186.400 | 18.850 | 189.800 | 18.510 | 183.200 | 18.970 | 179.200 | 19.220 |

TABLE A.5(c): NORMALIZED MEAN VELOCITY PROFILES FOR 80-GRIT CASE


| $\mathrm{Y}_{+}$ | $\mathrm{U}_{+}$ | Y+ | $\mathrm{U}_{+}$ | $Y_{+}$ | U+ | Y | U, |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 15.370 | 8.864 | 15.560 | 9.732 | 15.750 | 9.711 | 15.760 | 9.680 |
| 20.490 | 10.520 | 20.750 | 11.390 | 21.000 | 11.370 | 21.010 | 11.120 |
| 25.620 | 11.850 | 25.930 | 12.490 | 26.250 | 12.460 | 26.270 | 12.110 |
| 30.740 | 12.890 | 31.120 | 13.290 | 31.500 | 13.330 | 31.520 | 12.810 |
| 35.860 | 13.780 | 36.310 | 13.950 | 36.750 | 14.020 | 36.780 | 13.360 |
| 40.990 | 14.590 | 41.490 | 14.830 | 42.000 | 14.700 | 42.030 | 13.970 |
| 46.110 | 15.360 | 46.680 | 15.440 | 47.250 | 15.300 | 47.280 | 14.420 |
| 51.230 | 16.020 | 51.870 | 16.000 | 52.500 | 15.810 | 52.540 | 14.900 |
| 61.480 | 17.040 | 62.240 | 16.850 | 63.000 | 16.630 | 63.040 | 15.650 |
| 71.730 | 17.780 | 72.610 | 17.510 | 73.500 | 17.300 | 73.550 | 16.210 |
| 81.970 | 18.270 | 82.990 | 18.000 | 84.000 | 17.780 | 84.060 | 16.690 |
| 92.220 | 18.670 | 93.360 | 18.410 | 94.500 | 18.220 | 94.570 | 17.140 |
| 102.500 | 18.940 | 103.700 | 18.710 | 105.000 | 18.510 | 105.100 | 17.510 |
| 112.700 | 19.110 | 114.100 | 18.880 | 115.500 | 18.750 | 115.600 | 17.770 |
| 123.000 | 19.210 | 124.500 | 19.000 | 126.000 | 18.820 | 126.100 | 17.960 |
| 133.200 | 19.260 | 134.900 | 19.070 | 136.500 | 18.920 | 136.600 | 18.080 |
| 184.400 | 19.330 | 186.700 | 19.170 | 189.000 | 19.040 | 189.100 | 18.280 |

TABLE A.5(c): (cont'd)

| $\mathrm{Y}_{4}$ | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ | Y. | $\mathrm{U}_{+}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 15.650 | 9.628 | 15.430 | 9.766 | 14.870 | 9.521 | 14.640 | 10.060 |
| 20.870 | 12.220 | 20.570 | 11.460 | 19.820 | 11.230 | 19.520 | 11.350 |
| 26.080 | 12.970 | 25.710 | 12.500 | 24.780 | 12.410 | 24.410 | 12.440 |
| 31.300 | 13.530 | 30.850 | 13.650 | 29.730 | 13.170 | 29.290 | 13.220 |
| 36.510 | 14.020 | 36.000 | 14.170 | 34.690 | 13.760 | 34.170 | 13.790 |
| 41.730 | 14.450 | 41.140 | 14.590 | 39.650 | 14.240 | 39.050 | 14.230 |
| 46.950 | 14.840 | 46.280 | 14.960 | 44.600 | . 14.630 | 43.930 | 14.590 |
| 52.160 | 15.180 | 51.420 | 15.250 | 49.560 | 14.980 | 48.810 | . 14.950 |
| 62.600 | 15.760 | 61.710 | 15.770 | 59.470 | 15.570 | 58.570 | 15.500 |
| 73.030 | 16.300 | 71.990 | 16.260 | 69.380 | 16.080 | 68.340 | 15.990 |
| 83.460 | 16.760 | 82.280 | 16.700 | 79.290 | 16.520 | 78.100 | 16.410 |
| 93.890 | 17.190 | 92.560 | 17.150 | 89.200 | 16.920 | 87.860 | 16.870 |
| 104.300 | 17.560 | 102.800 | 17.490 | 99.110 | 17.310 | 97.620 | 17.210 |
| 114.800 | 17.820 | 113.100 | 17.810 | 109.000 | 17.690 | 107.400 | 17.600 |
| 125.200 | 18.020 | 123.400 | 18.110 | 118.900 | 18.020 | 117.100 | 17.910 |
| 135.600 | 18.160 | 133.700 | 18.300 | 128.800 | 18.330 | 126.900 | 18.250 |
| 187.800 | 18.430 | 185.100 | 18.790 | 178.400 | 19.300 | 175.700 | 19.460 |

TABLE A.5(d): NORMALIZED MEAN VELOCITY PROFILES FOR 60-GRIT CASE


| $\mathrm{x}=46 \mathrm{~mm}$ |  | $\mathrm{x}=48 \mathrm{~mm}$ |  | $\mathrm{x}=50 \mathrm{~mm}$ |  | $\mathrm{x}=55 \mathrm{~mm}$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{Y}_{+}$ | $\mathrm{U}_{+}$ | $\mathrm{Y}_{+}$ | $\mathrm{U}_{+}$ | Y + | $\mathrm{U}_{+}$ | Y. | $\mathrm{U}_{+}$ |
| 14.530 | 8.361 | 15.000 | 7.775 | 15.190 | 7.802 | 15.840 | 7.929 |
| 19.380 | 10.300 | 20.000 | 9.750 | 20.250 | 9.975 | 21.130 | 9.751 |
| 24.220 | 12.260 | 25.000 | 11.450 | 25.310 | 11.600 | 26.410 | 10.980 |
| 29.060 | 14.140 | 30.000 | 13.250 | 30.380 | 13.160 | 31.690 | 12.240 |
| 33.910 | 15.900 | 35.000 | 14.850 | 35.440 | 14.490 | 36.970 | 13.250 |
| 38.750 | 17.470 | 40.000 | 16.200 | 40.500 | 15.850 | 42.250 | 14.340 |
| 43.590 | 18.660 | 45.000 | 17.470 | 45.560 | 17.010 | 47.530 | 15.290 |
| 48.440 | 19.460 | 50.000 | 18.400 | 50.630 | 17.830 | 52.810 | 16.050 |
| 58.130 | 20.260 | 60.000 | 19.450 | 60.750 | 18.860 | 63.380 | 17.070 |
| 67.810 | 20.490 | 70.000 | 19.820 | 70.880 | 19.460 | 73.940 | 17.680 |
| 77.500 | 20.570 | 80.000 | 19.950 | 81.000 | 19.600 | 84.500 | 17.940 |
| 87.190 | 20.570 | 90.000 | 19.970 | 91.130 | 19.650 | 95.060 | 18.060 |
| 96.880 | 20.570 | 100.000 | 20.000 | 101.300 | 19.680 | 105.600 | 18.110 |
| 106.600 | 20.570 | 110.000 | 20.020 | 111.400 | 19.680 | 116.200 | 18.130 |
| 116.300 | 20.570 | 120.000 | 20.020 | 121.500 | 19.680 | 126.800 | 18.130 |
| 125.900 | 20.570 | 130.000 | 20.020 | 131.600 | 19.680 | 137.300 | 18.150 |
| 174.400 | 20.570 | 180.000 | 20.050 | 182.300 | 19.700 | 190.100 | 18.150 |

TABLE A. 5(d): (cont'd)

| Y. | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ | Y, | $\mathrm{U}_{+}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 16.570 | 8.103 | 16.020 | 9.717 | 14.910 | 10.260 | 14.580 | 10.570 |
| 22.090 | 9.597 | 21.360 | 11.100 | 19.880 | 11.620 | 19.440 | 11.990 |
| 27.610 | 10.640 | 26.690 | 11.920 | 24.850 | 12.420 | 24.300 | 12.680 |
| 33.140 | 11.570 | 32.030 | 12.600 | 29.820 | 13.180 | 29.160 | 13.370 |
| 38.660 | 12.470 | 37.370 | 13.180 | 34.790 | 13.680 | 34.020 | 13.860 |
| 44.180 | 13.220 | 42.710 | 13.670 | 39.760 | 14.130 | 38.880 | 14.220 |
| 49.700 | 14.010 | 48.050 | 14.240 | 44.740 | 14.510 | 43.740 | 14.580 |
| 55.220 | 14.690 | 53.390 | 14.660 | 49.710 | 14.860 | 48.600 | 14.920 |
| 66.270 | 15.710 | 64.070 | 15.500 | 59.650 | 15.420 | 58.320 | 15.460 |
| 77.310 | 16.480 | 74.740 | 16.230 | 69.590 | 15.990 | 68.040 | 16.000 |
| 88.360 | 16.890 | 85.420 | 16.790 | 79.530 | 16.520 | 77.760 | 16.440 |
| 99.400 | 17.110 | 96.100 | 17.190 | 89.470 | 16.970 | 87.480 | 16.800 |
| 110.400 | 17.220 | 106.800 | 17.490 | 99.410 | 17.430 | 97.200 | 17.230 |
| 121.500 | 17.290 | 117.500 | 17.650 | 109.400 | 17.780 | 106.900 | 17.620 |
| 132.500 | 17.320 | 128.100 | 17.790 | 119.300 | 18.160 | 116.600 | 17.980 |
| 143.600 | 17.340 | 138.800 | 17.890 | 129.200 | 18.480 | 126.400 | 18.290 |
| 198.800 | 17.360 | 192.200 | 17.980 | 178.900 | 19.260 | 175.000 | 19.500 |

TABLE A.5(e): NORMALIZED MEAN VELOCITY PROFILES FOR 40-GRIT CASE

| Y | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ | y . | $\mathrm{U}_{+}$ | Y+ | $\mathrm{U}_{+}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 18.220 | 10.370 | 18.110 | 9.871 | 17.660 | 9.777 | 17.440 | 9.946 | 16.990 | 10.700 |
| 24.290 | 11.830 | 24.140 | 11.380 | 23.550 | 11.370 | 23.250 | 11.230 | 22.650 | 11.860 |
| 30.370 | 12.580 | 30.180 | 12.130 | 29.440 | 12.180 | 29.060 | 12.030 | 28.320 | 12.490 |
| 36.440 | 13.250 | 36.220 | 12.810 | 35.320 | 12.850 | 34.880 | 12.750 | 33.980 | 13.040 |
| 42.510 | 13.830 | 42.250 | 13.480 | 41.210 | 13.410 | 40.690 | 13.250 | 39.640 | 13.630 |
| 48.590 | 14.390 | 48.290 | 14.000 | 47.100 | 13.920 | 46.500 | 13.750 | 45.310 | 14.090 |
| 54.660 | 14.960 | 54.320 | 14.540 | 52.990 | 14.440 | 52.310 | 14.210 | 50.970 | 14.520 |
| 60.730 | 15.480 | 60.360 | 15.090 | 58.870 | 14.930 | 58.130 | 14.650 | 56.640 | 14.940 |
| 72.880 | 16.290 | 72.430 | 16.010 | 70.650 | 15.880 | 69.750 | 15.650 | 67.960 | 15.720 |
| 85.030 | 16.810 | 84.500 | 16.680 | 82.420 | 16.650 | 81.380 | 16.450 | 79.290 | 16.480 |
| 97.170 | 17.200 | 96.580 | 17.230 | 94.200 | 17.280 | 93.000 | 17.150 | 90.620 | 17.040 |
| 109.300 | 17.450 | 108.600 | 17.540 | 106.000 | 17.640 | 104.600 | 17.670 | 101.900 | 17.560 |
| 121.500 | 17.540 | 120.700 | 17.690 | 117.700 | 17.940 | 116.300 | 18.000 | 113.300 | 17.890 |
| 133.600 | 17.560 | 132.800 | 17.730 | 129.500 | 18.050 | 127.900 | 18.150 | 124.600 | 18.090 |
| 145.800 | 17.540 | 144.900 | 17.730 | 141.300 | 18.090 | 139.500 | 18.220 | 135.900 | 18.180 |
| 157.900 | 17.500 | 156.900 | 17.710 | 153.100 | 18.090 | 151.100 | 18.240 | 147.300 | 18.230 |
| 218.600 | 17.350 | 217.300 | 17.580 | 211.900 | 18.030 | 209.300 | 18.220 | 203.900 | 18.230 |
|  |  |  |  |  |  |  |  | 260.500 | 18.180 |
|  |  |  |  |  |  |  |  | 317.200 | 18.140 |

TABLE A.5(e): (cont'd)

| $\mathrm{x}=36 \mathrm{~mm}$ | $\mathrm{x}=40 \mathrm{~mm}$ | $\mathrm{x}=46 \mathrm{~mm}$ | $\mathrm{x}=50 \mathrm{~mm}$ |  |  |  |  |  |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| Y | $\mathrm{U}+$ | $\mathrm{Y}+$ | $\mathrm{U}_{+}$ | $\mathrm{Y}+$ | $\mathrm{U}_{+}$ | $\mathrm{Y}+$ | $\mathrm{U}_{+}$ |  |
|  |  |  |  |  |  |  |  |  |
| 16.430 | 10.850 | 15.980 | 10.450 | 15.650 | 10.500 | 15.200 | 10.630 |  |
| 21.910 | 11.890 | 21.310 | 11.780 | 20.870 | 11.910 | 20.270 | 11.860 |  |
| 27.390 | 12.470 | 26.640 | 12.420 | 26.080 | 12.540 | 25.340 | 12.630 |  |
| 32.870 | 13.050 | 31.970 | 12.990 | 31.300 | 13.100 | 30.410 | 13.130 |  |
| 38.340 | 13.490 | 37.300 | 13.530 | 36.510 | 13.600 | 35.470 | 13.600 |  |
| 43.820 | 13.950 | 42.630 | 13.910 | 41.730 | 13.990 | 40.540 | 13.980 |  |
| 49.300 | 14.340 | 47.950 | 14.340 | 46.950 | 14.360 | 45.610 | 14.380 |  |
| 54.780 | 14.730 | 53.280 | 14.700 | 52.160 | 14.700 | 50.680 | 14.700 |  |
| 65.730 | 15.500 | 63.940 | 15.380 | 62.600 | 15.330 | 60.810 | 15.300 |  |
| 76.690 | 16.190 | 74.600 | 16.120 | 73.030 | 15.980 | 70.950 | 15.930 |  |
| 87.640 | 16.810 | 85.250 | 16.710 | 83.460 | 16.540 | 81.080 | 16.500 |  |
| 98.600 | 17.370 | 95.910 | 17.310 | 93.890 | 17.150 | 91.220 | 17.050 |  |
| 109.600 | 17.810 | 106.600 | 17.850 | 104.300 | 17.650 | 101.400 | 17.570 |  |
| 120.500 | 18.180 | 117.200 | 18.300 | 114.800 | 18.120 | 111.500 | 18.000 |  |
| 131.500 | 18.380 | 127.900 | 18.640 | 125.200 | 18.580 | 121.600 | 18.470 |  |
| 142.400 | 18.520 | 138.500 | 18.830 | 135.600 | 18.890 | 131.800 | 18.800 |  |
| 197.200 | 18.610 | 191.800 | 19.090 | 187.800 | 19.430 | 182.400 | 19.470 |  |
| 252.000 | 18.590 | 245.100 | 19.060 | 240.000 | 19.400 |  |  |  |
| 306.700 | 18.570 | 298.400 | 19.040 | 292.100 | 19.380 |  |  |  |
|  |  |  |  |  |  |  |  |  |

TABLE A.5(e): (cont'd)

| Y. | $\mathrm{U}_{+}$ | $Y_{+}$ | $\mathrm{U}_{+}$ | Y | $\mathrm{U}_{+}$ | Y | U |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 14.760 | 10.130 | 14.420 | 10.610 | 13.970 | 9.859 | 13.750 | 9.826 |
| 19.670 | 11.730 | 19.230 | 12.050 | 18.630 | 11.540 | 18.330 | 11.570 |
| 24.590 | 12.600 | 24.030 | 12.840 | 23.290 | 12.440 | 22.920 | 12.500 |
| 29.510 | 13.120 | 28.840 | 13.370 | 27.950 | 13.150 | 27.500 | 13.140 |
| 34.430 | 13.580 | 33.650 | 13.790 | 32.600 | 13.610 | 32.080 | 13.690 |
| 39.350 | 14.020 | 38.450 | 14.210 | 37.260 | 14.100 | 36.670 | 14.160 |
| 44.270 | 14.380 | 43.260 | 14.550 | 41.920 | 14.420 | 41.250 | 14.490 |
| 49.180 | 14.690 | 48.060 | 14.840 | 46.580 | 14.720 | 45.830 | 14.790 |
| 59.020 | 15.300 | 57.680 | 15.420 | 55.890 | 15.240 | 55.000 | 15.350 |
| 68.860 | 15.870 | 67.290 | 15.920 | 65.210 | 15.780 | 64.160 | 15.790 |
| 78.690 | 16.380 | 76.900 | 16.400 | 74.520 | 16.210 | 73.330 | 16.230 |
| 88.530 | 16.900 | 86.520 | 16.870 | 83.840 | 16.650 | 82.500 | 16.620 |
| 98.370 | 17.360 | 96.130 | 17.320 | 93.160 | 17.060 | 91.660 | 17.000 |
| 108.200 | 17.770 | 105.700 | 17.710 | 102.500 | 17.460 | 100.800 | 17.360 |
| 118.000 | 18.240 | 115.400 | 18.210 | 111.800 | 17.840 | 110.000 | 17.750 |
| 127.900 | 18.650 | 125.000 | 18.610 | 121.100 | 18.220 | 119.200 | 18.110 |
| 177.100 | 19.830 | 173.000 | 20.080 | 167.700 | 19.910 | 165.000 | 19.740 |

TABLE A.5(f): NORMALIZED MEAN VELOCITY PROFILES FOR 4-GRIT CASE


TABLE A.5(f): (cont'd)


| $\mathrm{x}=70 \mathrm{~mm}$ |  | A.5(f) : (cont'd) |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\mathrm{x}=$ | 90 mm | $\mathrm{x}=$ | 10 mm |
| Y. | $\mathrm{U}_{\text {, }}$ | Y + | $\mathrm{U}_{+}$ | $\mathrm{Y}_{+}$ | $\mathrm{U}_{+}$ |
| 14.430 | 11.200 | 14.650 | 10.490 | 14.540 |  |
| 19.240 | 13.350 | 19.540 | 12.350 | 19.390 | 11.950 |
| 24.050 | 14.390 | 24.420 | 13.500 | 24.240 | 13.170 |
| 28.860 | 15.040 | 29.310 | 14.250 | 29.080 | 13.970 |
| 33.670 | 15.460 | 34.190 | 14.740 | 33.930 | 14.540 |
| 38.480 | 15.830 | 39.080 | 15.170 | 38.780 | 14.930 |
| 43.290 | 16.060 | 43.960 | 15.480 | 43.630 | 15.290 |
| 48.100 | 16.240 | 48.840 | 15.740 | 48.470 | 15.580 |
| 57.720 | 16.560 | 58.610 | 16.130 | 58.170 | 16.020 |
| 67.340 | 16.820 | 68.380 | 16.460 | 67.860 | 16.410 |
| 76.960 | 16.950 | 78.150 | 16.720 | 77.560 | 16.720 |
| 86.590 | 17.030 | 87.920 | 16.900 | 87.250 | 16.950 |
| 96.210 | 17.000 | 97.690 | 17.030 | 96.950 | 17.130 |
| 105.800 | 17.080 | 107.500 | 17.160 | 106.600 | 17.260 |
| 115.400 | 17.180 | 117.200 | 17.280 | 116.300 | 17.440 |
| 125.100 | 17.320 | 127.000 | 17.390 | 126.000 | 17.620 |
| 173.200 | 17.890 | 175.800 | 17.950 | 174.500 | 18.250 |
| 221.300 | 18.880 | 224.700 | 18.620 | 223.000 | 18.840 |
| 269.400 | 19.820 | 273.500 | 19.290 | 271.500 | 19.410 |
| 317.500 | 20.320 | 322.400 | 19.800 | 319.900 | 19.880 |
| 365.600 | 20.420 | 371.200 | 20.040 | 368.400 | 20.160 |
|  |  | 420.100 | 20.060 | 416.900 | 20.240 |

TABLE A.6: REDUCED PRESSURE DISTRIBUTION

| Bare |  | 100-Grit |  | 80-Grit |  | 60-Grit |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{x}_{5} / \mathrm{l}_{B}$ | - $\mathrm{C}_{\mathrm{P}}$ | $\mathrm{x}_{\mathrm{B}} / \mathrm{l}_{\mathrm{B}}$ | $-\mathrm{C}_{\mathrm{p}}$ | $\mathrm{x}_{8} / \mathrm{l}_{\mathrm{B}}$ | $-_{\text {P }}$ | $\mathrm{x}_{\mathrm{B}} / \mathrm{l}_{\mathrm{B}}$ | $-\mathrm{C}_{\mathrm{p}}$ |
| 0.0000 | -0.02699 | 0.0000 | -0.02932 | 0.0000 | 0.00000 | 0.0000 | 0.000000 |
| 0.0833 | -0.000692 | 0.0769 | 0.001260 | 0.1111 | 0.02197 | 0.1000 | 0.008362 |
| 0.1667 | 0.00692 | 0.1538 | 0.01001 | 0.2222 | 0.02449 | 0.2000 | 0.01679 |
| 0.2500 | 0.00000 | 0.2308 | 0.00000 | 0.3333 | 0.02755 | 0.3000 | 0.02426 |
| 0.3333 | 0.01799 | 0.3077 | 0.01421 | 0.4444 | 0.04674 | 0.4000 | 0.04469 |
| 0.4167 | 0.03460 | 0.3846 | 0.03289 | 0.5556 | 0.06266 | 0.5000 | 0.04469 0.06587 |
| 0.5000 | 0.06436 | 0.4615 | 0.05619 | 0.6667 | 0.09756 | 0.6000 | 0.1141 |
| 0.5833 | 0.07266 | 0.5385 | 0.06900 | 0.7778 | 0.12120 | 0.7000 | 0.1479 |
| 0.6667 | 0.08720 | 0.6154 | 0.07936 | 0.8889 | 0.1448 | 0.8000 | 0.1731 |
| 0.7500 | 0.09204 | 0.6923 | 0.09216 | 1.000 | 0.1755 | 0.9000 | 0.1949 |
| 0.8333 | 0.1073 | 0.7692 | 0.1118 | 1.111 | 0.2027 | 1.000 | 0.2161 |
| 0.9167 | 0.1308 | 0.8462 | 0.1358 | 1.222 | 0.2243 | 1.100 | 0.2270 |
| 1.000 | 0.1716 | 0.9231 | 0.1807 | 1.333 | 0.2351 | 1.200 | 0.2323 |
| 1.083 | 0.2131 | 1.000 | 0.2179 | 1.444 | 0.2407 | 1.300 | 0.2349 |
| 1.167 | 0.2401 | 1.077 | 0.2391 | 1.556 | 0.2448 | 1.400 | 0.2375 |
| 1.250 | 0.2524 | 1.154 | 0.2470 | 1.667 | 0.2484 | 1.500 | 0.2391 |
| 1.333 | 0.2566 | 1.231 | 0.2488 | 1.778 | 0.2507 | 1.600 | 0.2417 |
| 1.542 | 0.2625 | 1.423 | 0.2494 | 2.056 | 0.2559 | 1.850 | 0.2479 |
| 1.750 | 0.2609 | 1.615 | 0.2484 | 2.333 | 0.2599 | 2.100 | 0.2521 |
| 2.167 | 0.2606 | 2.000 | 0.2507 | 2.889 | 0.2662 | 2.600 | 0.2599 |
|  |  | 2.385 | 0.2548 | 3.444 | 0.2726 | 3.100 | 0.2651 |
|  |  | 2.769 | 0.2564 | 4.000 | 0.2767 | 3.600 | 0.2692 |
|  |  | 3.154 | 0.2591 | 4.556 | 0.2793 | 4.100 | 0.2729 |
|  |  | 3.538 | 0.2617 | 5.111 | 0.2813 | 4.600 | 0.2756 |
|  |  | 3.923 | 0.2636 |  |  |  |  |
|  |  | 4.308 | 0.2652 |  |  |  |  |
|  |  | 4.692 | 0.2667 |  |  |  |  |
|  |  | 5.077 | 0.2677 |  |  |  |  |
|  |  | 5.462 | 0.2688 |  |  |  |  |

TABLE A.7(a): DISPLACEMENT AND MOMENTUM THICKNESSES AND SHAPE FACTOR DISTRIBUTION FOR BARE CASE

| x | Displacement <br> Thickness <br> $\left(\mathrm{m} \mathrm{x} 10^{4}\right)$ | Momentum <br> Thickness <br> $\left(\mathrm{m} \times 10^{4}\right)$ | Shape <br> Factor |
| :---: | :---: | :---: | :---: |
|  |  |  |  |
| 22 | 3.22 | 0.58 | 5.514 |
| 24 | 4.64 | 0.72 | 6.416 |
| 26 | 5.83 | 0.78 | 7.481 |
| 28 | 5.91 | 0.86 | 6.842 |
| 30 | 6.63 | 0.94 | 7.089 |
| 32 | 7.43 | 1.02 | 7.292 |
| 34 | 8.03 | 1.06 | 7.546 |
| 36 | 7.95 | 1.28 | 6.209 |
| 38 | 8.53 | 1.25 | 6.849 |
| 40 | 8.64 | 1.37 | 6.284 |
| 42 | 8.13 | 1.64 | 4.954 |
| 44 | 7.15 | 2.26 | 3.161 |
| 46 | 6.02 | 2.57 | 2.338 |
| 48 | 5.24 | 2.63 | 1.988 |
| 50 | 4.80 | 2.60 | 1.848 |
| 55 | 4.37 | 2.50 | 1.753 |
| 60 | 4.04 | 2.38 | 1.696 |
| 70 | 4.21 | 2.57 | 1.638 |

TABLE A. $7(\mathrm{~b}):$ DISPLACEMENT AND MOMENTUM THICKNESSES AND SHAPE FACTOR DISTRIBUTION FOR 100-GRIT CASE

| x | Displacement <br> Thickness <br> $\left(\mathrm{m} \times 10^{4}\right)$ | Momentum <br> Thickness <br> $\left(\mathrm{m} \times 10^{4}\right)$ | Shape <br> Factor |
| :---: | :---: | :---: | :---: |
|  |  |  |  |
| 22 | 4.49 | 0.61 | 7.353 |
| 24 | 5.60 | 0.75 | 7.446 |
| 26 | 6.71 | 0.88 | 7.612 |
| 28 | 7.27 | 0.99 | 7.377 |
| 30 | 7.44 | 1.06 | 7.022 |
| 32 | 8.03 | 1.17 | 6.878 |
| 34 | 8.76 | 1.23 | 7.092 |
| 36 | 9.23 | 1.30 | 7.115 |
| 38 | 9.27 | 1.38 | 6.706 |
| 40 | 9.44 | 1.46 | 6.462 |
| 42 | 8.50 | 1.68 | 5.059 |
| 44 | 7.03 | 1.97 | 3.566 |
| 46 | 6.58 | 2.55 | 2.581 |
| 48 | 5.67 | 2.60 | 2.178 |
| 50 | 5.37 | 2.74 | 1.959 |
| 55 | 4.95 | 2.86 | 1.734 |
| 60 | 4.96 | 3.00 | 1.654 |
| 70 | 5.20 | 3.26 | 1.596 |
| 90 | 5.85 | 3.68 | 1.589 |
| 110 | 6.35 | 4.00 | 1.586 |
|  |  |  |  |

## TABLE A.7(c): DISPLACEMENT AND MOMENTUM THICKNESSES AND SHAPE FACTOR DISTRIBUTION FOR 80-GRIT CASE

| x | Displacement <br> Thickness <br> $(\mathrm{m} \mathrm{x} \mathrm{10}$ | Momentum <br> Thickness <br> $(\mathrm{m} \mathrm{x} \mathrm{10})$ | Shape <br> Factor |
| ---: | :---: | :---: | ---: |
| 22 | 4.16 | 0.35 | 11.985 |
| 24 | 5.53 | 0.72 | 7.640 |
| 26 | 6.02 | 0.81 | 7.458 |
| 28 | 6.92 | 0.97 | 7.149 |
| 30 | 6.93 | 1.10 | 6.308 |
| 32 | 6.78 | 1.41 | 4.814 |
| 34 | 6.37 | 1.72 | 3.711 |
| 36 | 6.31 | 1.87 | 3.370 |
| 38 | 6.01 | 2.33 | 2.585 |
| 40 | 5.36 | 2.50 | 2.143 |
| 42 | 5.14 | 2.72 | 1.890 |
| 44 | 4.76 | 2.63 | 1.811 |
| 46 | 4.79 | 2.74 | 1.749 |
| 48 | 4.76 | 2.77 | 1.722 |
| 50 | 4.78 | 2.81 | 1.701 |
| 55 | 4.90 | 3.00 | 1.631 |
| 60 | 4.98 | 3.06 | 1.627 |
| 70 | 5.43 | 3.39 | 1.604 |
| 90 | 6.34 | 3.99 | 1.590 |
| 110 | 6.66 | 4.20 | 1.587 |

## TABLE A.7(d): DISPLACEMENT AND MOMENTUM THICKNESSES AND SHAPE FACTOR DISTRIBUTION FOR 60-GRIT CASE

x
Displacement
Momentum Thickness Factor $\mathrm{mm}\left(\mathrm{m} \times 10^{4}\right) \quad\left(\mathrm{m} \times 10^{4}\right)$

| 22 | 3.04 | 0.54 | 5.639 |
| ---: | :--- | :--- | :--- |
| 24 | 3.75 | 0.75 | 5.037 |
| 26 | 4.60 | 0.77 | 5.968 |
| 28 | 4.94 | 0.89 | 5.579 |
| 30 | 5.35 | 0.89 | 6.020 |
| 32 | 5.55 | 0.97 | 5.724 |
| 34 | 5.15 | 1.13 | 4.540 |
| 36 | 5.11 | 1.36 | 3.749 |
| 38 | 5.16 | 1.33 | 3.889 |
| 40 | 4.80 | 1.59 | 3.015 |
| 42 | 4.44 | 1.75 | 2.541 |
| 44 | 4.09 | 1.85 | 2.208 |
| 46 | 3.95 | 1.87 | 2.114 |
| 48 | 4.09 | 2.05 | 1.990 |
| 50 | 4.08 | 2.10 | 1.947 |
| 55 | 3.86 | 2.14 | 1.808 |
| 60 | 3.82 | 2.26 | 1.694 |
| 70 | 4.57 | 2.77 | 1.647 |
| 90 | 6.25 | 3.91 | 1.600 |
| 110 | 6.69 | 4.21 | 1.589 |
|  |  |  |  |

TABLE A.7(e): DISPLACEMENT AND MOMENTUM THICKNESSES AND SHAPE FACTOR DISTRIBUTION FOR 40-GRIT CASE

| x | Displacement <br> Thickness <br> $(\mathrm{m} \mathrm{x} \mathrm{104)}$ | Momentum <br> Thickness <br> $\left(\mathrm{m} \times 10^{4}\right)$ | Shape <br> Factor |
| ---: | :---: | :---: | :---: |
|  |  |  |  |
| 22 | 3.24 | 1.83 | 1.771 |
| 24 | 3.57 | 2.07 | 1.718 |
| 26 | 4.06 | 2.42 | 1.678 |
| 28 | 4.36 | 2.63 | 1.657 |
| 30 | 4.23 | 2.50 | 1.694 |
| 36 | 4.94 | 3.00 | 1.644 |
| 40 | 5.53 | 3.38 | 1.633 |
| 46 | 6.08 | 3.76 | 1.616 |
| 50 | 6.36 | 3.96 | 1.608 |
| 60 | 7.00 | 4.37 | 1.601 |
| 70 | 7.29 | 4.55 | 1.602 |
| 90 | 7.43 | 4.65 | 1.599 |
| 110 | 7.28 | 4.56 | 1.598 |
|  |  |  |  |

TABLE A.7(f): DISPLACEMENT AND MOMENTUM THICKNESSES AND SHAPE FACTOR DISTRIBUTIONS FOR 4-GRIT CASE

| x | Displacement <br> Thickness <br> $\left(\mathrm{m} \times 10^{4}\right)$ | Momentum <br> Thickness <br> $\left(\mathrm{m} \times 10^{4}\right)$ | Shape <br> Factor |
| ---: | :---: | :---: | :---: |
|  |  |  |  |
| 22 | 12.74 | 6.46 | 1.971 |
| 24 | 13.16 | 6.98 | 1.887 |
| 26 | 13.01 | 7.26 | 1.791 |
| 28 | 12.99 | 7.50 | 1.732 |
| 31 | 12.72 | 7.56 | 1.682 |
| 40 | 11.96 | 7.69 | 1.554 |
| 50 | 10.91 | 7.32 | 1.490 |
| 60 | 10.11 | 6.93 | 1.459 |
| 70 | 9.93 | 6.77 | 1.468 |
| 90 | 9.88 | 6.68 | 1.480 |
| 110 | 10.16 | 6.79 | 1.496 |

TABLE A.8: MAXIMUM TURBULENCE INTENSITY DISTRIBUTION

All turbulence intensities given in $\mathrm{ms}^{-1}$.

| Bare |  | 100-Grit |  | 80-Grit |  | 60-Grit |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\frac{x_{s}}{1_{B}}$ | $\left(\overline{u^{2}}\right)^{1 / 2}$ | $\frac{x_{\mathrm{B}}}{1_{\mathrm{B}}}$ | $\left(\overline{u^{2}}\right)^{1 / 2}$ | $\frac{x_{B}}{l_{B}}$ | $\left(\overline{u^{2}}\right)^{1 / 2}$ | $\frac{x_{B}}{l_{B}}$ | $\left(\overline{u^{2}}\right)^{1 / 2}$ |
| 0.1667 | 0.4875 | 0.1539 | 0.5325 | 0.2222 | 0.4605 | 0.2000 | 0.5910 |
| 0.2500 | 0.6315 | 0.2308 | 0.5790 | 0.3333 | 0.4605 | 0.3000 | 0.7140 |
| 0.3333 | 0.7905 | 0.3077 | 0.7875 | 0.4444 | 0.5355 | 0.4000 | 0.8835 |
| 0.4167 | 0.9195 | 0.3846 | 0.8925 | 0.5556 | 0.8625 | 0.5000 | 0.8880 |
| 0.5000 | 1.0485 | 0.4615 | 1.0140 | 0.6667 | 1.2000 | 0.6000 | 0.7320 |
| 0.5833 | 1.2060 | 0.5385 | 1.1025 | 0.7778 | 1.3320 | 0.7000 | 0.8205 |
| 0.6667 | 1.4415 | 0.6154 | 1.1835 | 0.8889 | 1.7780 | 0.8000 | 0.9345 |
| 0.7500 | 1.6425 | 0.6923 | 1.3380 | 1.0000 | 2.0925 | 0.9000 | 1.2000 |
| 0.8333 | 2.1420 | 0.7692 | 1.5840 | 1.1111 | 2.0985 | 1.0000 | 1.3245 |
| 0.9167 | 2.8245 | 0.8462 | 1.8225 | 1.2222 | 1.9695 | 1.1000 | 1.4475 |
| 1.0000 | 2.9265 | 0.9231 | 2.0730 | 1.3333 | 1.9905 | 1.2000 | 1.5285 |
| 1.0833 | 2.9205 | 1.0000 | 2.1555 | 1.4444 | 1.9395 | 1.3000 | 1.5720 |
| 1.1667 | 2.7495 | 1.0769 | 2.1045 | 1.5556 | 1.7835 | 1.4000 | 1.6140 |
| 1.2500 | 2.5350 | 1.1538 | 2.1360 | 1.6667 | 1.6965 | 1.5000 | 1.6425 |
| 1.3333 | 2.3640 | 1.2308 | 1.8540 | 1.7778 | 1.6125 | 1.6000 | 1.6785 |
| 1.5417 | 1.9845 | 1.4231 | 1.6335 | 2.0556 | 1.5525 | 1.8500 | 1.7100 |
| 1.7500 | 1.7265 | 1.6154 | 1.5705 | 2.3333 | 1.4400 | 2.1000 | 1.7325 |
| 2.1667 | 1.6605 | 2.0000 | 1.5765 | 2.8889 | 1.4445 | 2.6000 | 1.6260 |
|  |  | 2.7692 | 1.6080 | 4.0000 | 1.4835 | 3.6000 | 1.5645 |
|  |  | 3.5385 | 1.6425 | 5.1111 | 1.6020 | 4.6000 | 1.5825 |

## TABLE A.9: TURBULENCE DATA NORMALIZED BY $\mathrm{U}_{\mathbf{r}}{ }^{2}$

| Bare |  | 100-Grit |  | 80-Grit |  | 60-Grit |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\frac{X_{B}}{l_{B}}$ | $\frac{\overline{u^{2}}}{U_{x}{ }^{2}}$ | $\frac{X_{B}}{l_{B}}$ | $\frac{\bar{u}{ }^{2}}{U_{x}{ }^{2}}$ | $\frac{x_{S}}{I_{B}}$ | $\frac{\overline{u^{2}}}{U_{x}{ }^{2}}$ | $\frac{X_{6}}{I_{B}}$ | $\frac{\bar{u}}{}{ }^{2}{ }^{U_{r}{ }^{2}}$ |
| 0.1667 | 0.001056 | 0.1538 | 0.001260 | 0.2222 | 0.000943 | 0.2000 | 0.001552 |
| 0.2500 | 0.001772 | 0.2308 | 0.001490 | 0.3333 | 0.001274 | 0.3000 | 0.002266 |
| 0.3333 | 0.002777 | 0.3077 | 0.002756 | 0.3333 | 0.002460 | 0.4000 | 0.003469 |
| 0.4167 | 0.003758 | 0.3846 | 0.003540 | 0.4444 | 0.003306 | 0.5000 | 0.003505 |
| 0.5000 | 0.004886 | 0.4615 | 0.004570 | 0.5556 | 0.006400 | 0.6000 | 0.002381 |
| 0.5833 | 0.006464 | 0.5385 | 0.005402 | 0.6667 | 0.007885 | 0.7000 | 0.002992 |
| 0.6667 | 0.009235 | 0.6154 | 0.006225 | 0.7778 | 0.014210 | 0.8000 | 0.003881 |
| 0.7500 | 0.011990 | 0.6923 | 0.007957 | 0.8889 | 0.019460 | 0.9000 | 0.006400 |
| 0.8333 | 0.020390 | 0.7692 | 0.011150 | 1.0000 | 0.019570 | 1.0000 | 0.007797 |
| 0.9167 | 0.035460 | 0.8462 | 0.014760 | 1.1111 | 0.017240 | 1.1000 | 0.009312 |
| 1.0000 | 0.038060 | 0.9231 | 0.019100 | 1.2222 | 0.017610 | 1.2000 | 0.010380 |
| 1.0833 | 0.037910 | 1.0000 | 0.020650 | 1.3333 | 0.016720 | 1.3000 | 0.010980 |
| 1.1667 | 0.033600 | 1.0769 | 0.019680 | 1.4444 | 0.014140 | 1.4000 | 0.011580 |
| 1.2500 | 0.028560 | 1.1538 | 0.020280 | 1.5556 | 0.012790 | 1.5000 | 0.011990 |
| 1.3333 | 0.024840 | 1.2308 | 0.015280 | 1.6667 | 0.011560 | 1.6000 | 0.012520 |
| 1.5417 | 0.017500 | 1.4231 | 0.011860 | 1.7778 | 0.010710 | 1.8500 | 0.013000 |
| 1.7500 | 0.013250 | 1.6154 | 0.010960 | 2.0556 | 0.009216 | 2.1000 | 0.013340 |
| 2.1667 | 0.012250 | 2.0000 | 0.011050 | 2.3333 | 0.009274 | 2.6000 | 0.011750 |
|  |  | 2.7692 | 0.011490 | 2.8889 | 0.009781 | 3.6000 | 0.010880 |
|  |  | 3.5385 | 0.011990 | 4.0000 | 0.010750 | 4.6000 | 0.011130 |
|  |  |  |  | 5.1111 | 0.011410 |  |  |


| Castro \& Haque |  |
| :---: | :---: |
|  |  |
| $\frac{\mathrm{x}_{\mathrm{s}}}{\mathrm{I}_{\mathrm{B}}}$ | $\frac{\overline{\mathrm{u}^{2}}}{\mathrm{U}_{\mathrm{r}}{ }^{2}}$ |
|  |  |
| 0.1290 | 0.053300 |
| 0.2310 | 0.058800 |
| 0.3530 | 0.088600 |
| 0.4550 | 0.093300 |
| 0.5730 | 0.093300 |
| 0.6750 | 0.091800 |
| 0.7880 | 0.085100 |
| 0.8980 | 0.072900 |
| 0.9610 | 0.063100 |
| 1.0120 | 0.056500 |
| 1.1250 | 0.045900 |
| 1.2390 | 0.040400 |
| 1.3490 | 0.040000 |

TABLE A. 10: SKIN FRICTION COEFFICIENT DISTRIBUTION
(Skin friction coefficient multiplied by 1000.)

| Present Study |  | Bradshaw \& Wong |  | Chandrsuda <br> \& Bradshaw |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| X* | $\mathrm{C}_{f}$ | X* | $C_{\text {f }}$ | X* | $\mathrm{C}_{\text {f }}$ |
| 0.000 | 0.624 | 0.119 | 0.500 | 0.017 | 0.083 |
| 0.083 | 1.881 | 0.167 | 0.735 | 0.525 | 1.976 |
| 0.167 | 3.200 | 0.238 | 1.132 | 1.024 | 2.321 |
| 0.250 | 4.356 | 0.286 | 1.515 | 1.532 | 2.345 |
| 0.333 | 5.067 | 0.310 | 1.662 |  |  |
| 0.542 | 5.760 | 0.381 | 1.838 |  |  |
| 0.750 | 6.124 | 0.429 | 1.941 |  |  |
| 1.167 | 5.977 | 0.476 | 2.059 |  |  |
| 1.583 | 5.904 | 0.548 | 2.265 |  |  |
| 2.000 | 5.760 | 0.595 | 2.353 |  |  |
| 2.417 | 5.618 | 1.059 | 2.882 |  |  |
| 2.833 | 5.478 | 1.157 | 2.882 |  |  |
| 3.250 | 5.339 | 1.206 | 2.926 |  |  |
| 3.667 | 5.202 | 1.255 | 2.971 |  |  |
| 4.083 | 5.134 | 1.304 | 3.000 |  |  |
|  |  | 1.451 | 3.000 |  |  |
|  |  | 1.549 | 3.015 |  |  |
|  |  | 1.598 | 3.000 |  |  |
|  |  | 1.696 | 3.044 |  |  |
|  |  | 2.579 | 3.015 |  |  |
|  |  | 3.093 | 3.044 |  |  |
|  |  | 3.583 | 3.000 |  |  |
|  |  | 4.098 | 3.000 |  |  |

